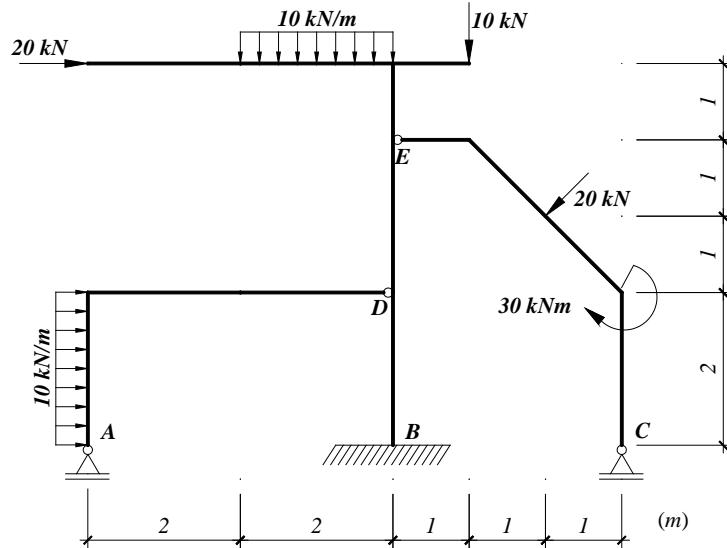


# VJEŽBA BR. 1

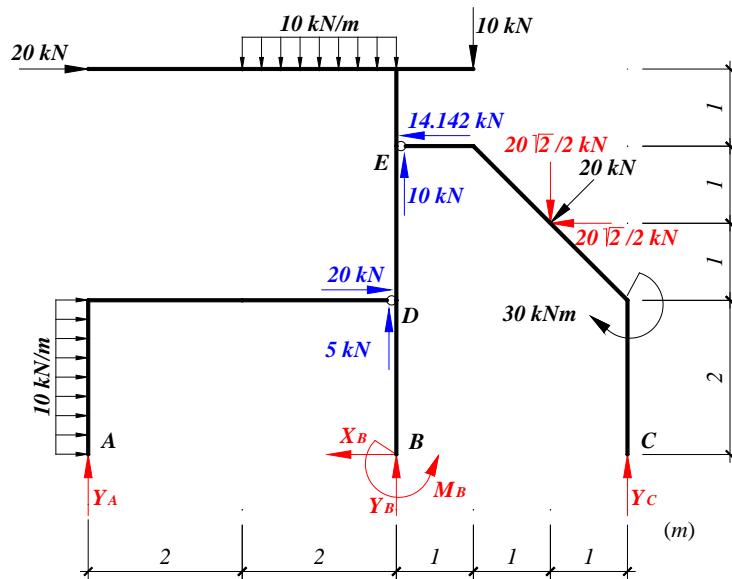
## PONAVLJANJE GRADIVA IZ GRAĐEVINSKE MEHANIKE 1

### RAVANSKI NOSAČI

- Za nosač na slici nacrtati dijagrame presječnih sila ( $M$ ,  $T$ ,  $N$ ).



### RJEŠENJE

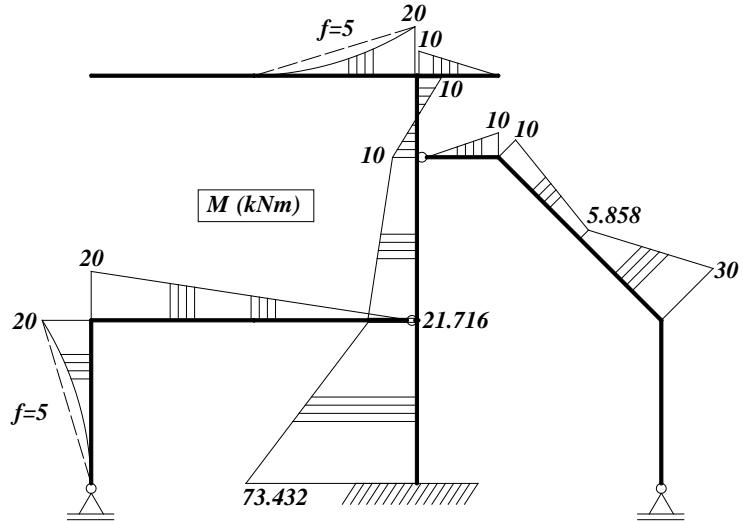
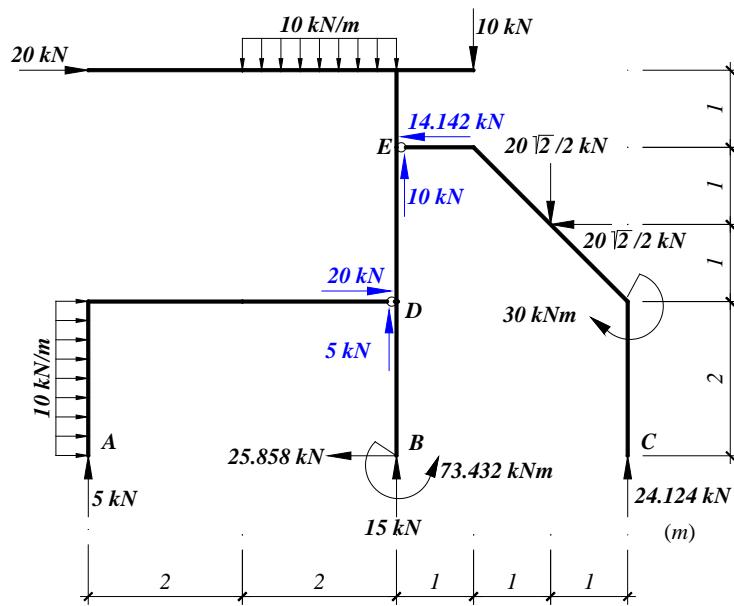


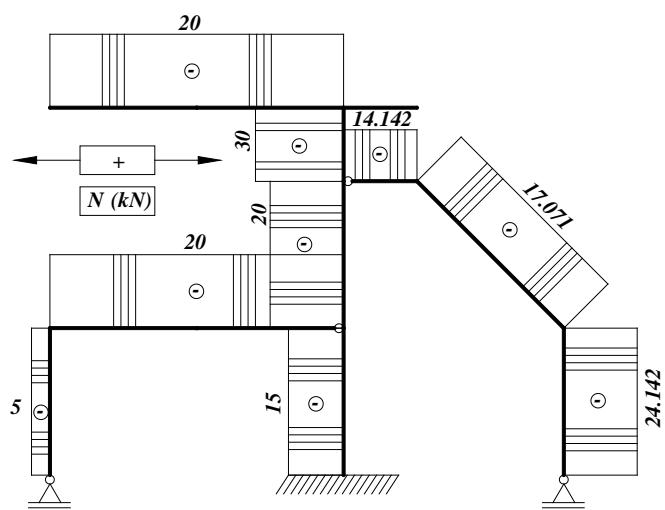
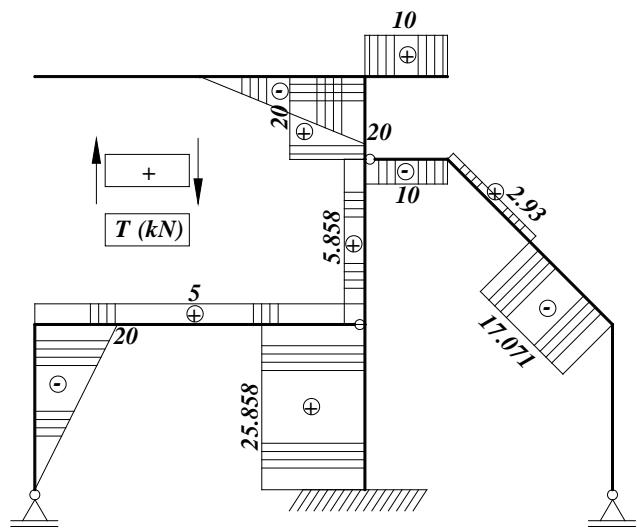
Uslovi ravnoteže:

- $\sum X = 0 \rightarrow X_B = 20 - \frac{20\sqrt{2}}{2} + 10 \cdot 2 \rightarrow X_B = 25.858 \text{ kN}$
- $\sum M_{D, \text{ljevo}} = 0 \rightarrow Y_A \cdot 4 - 10 \cdot 2 \cdot 1 = 0 \rightarrow Y_A = 5.0 \text{ kN}$
- $\sum M_{E, \text{desno}} = 0 \rightarrow Y_C \cdot 3 - 30 - \frac{20\sqrt{2}}{2} \cdot (2 + 1) = 0 \rightarrow Y_C = 24.142 \text{ kN}$
- $\sum Y = 0 \rightarrow Y_A + Y_B + Y_C - 10 \cdot 2 - 10 - \frac{20\sqrt{2}}{2} = 0 \rightarrow Y_B = 15.0 \text{ kN}$

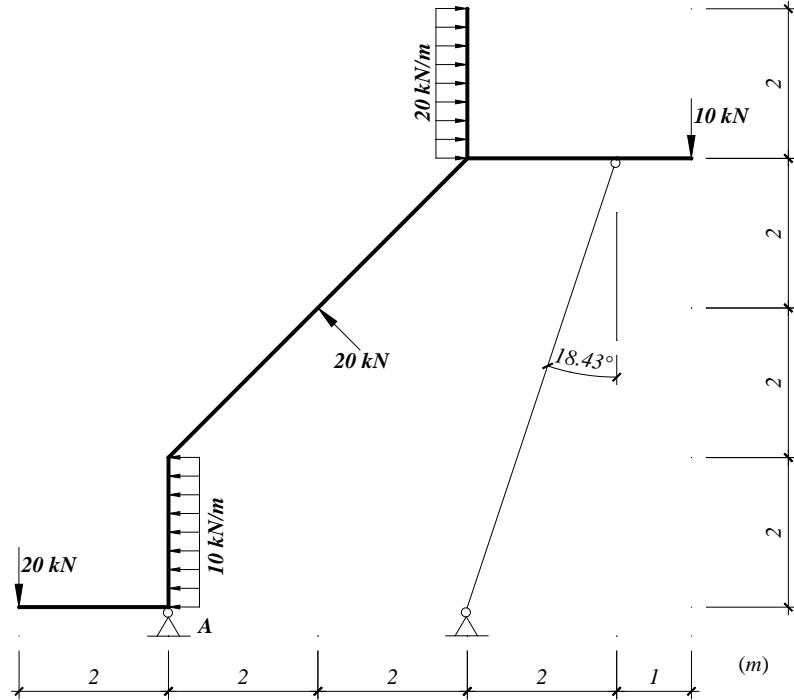
$$5. \quad \sum M_B = 0 \quad \rightarrow 20 \cdot 5 + 10 \cdot 1 - 10 \cdot 2 \cdot 1 + 20 \cdot 2 - 14.142 \cdot 4 = M_B \quad \rightarrow M_B = 73.432 \text{ kN}$$

Kontrola: npr.  $\sum M_A = 0$  – zadovoljeno

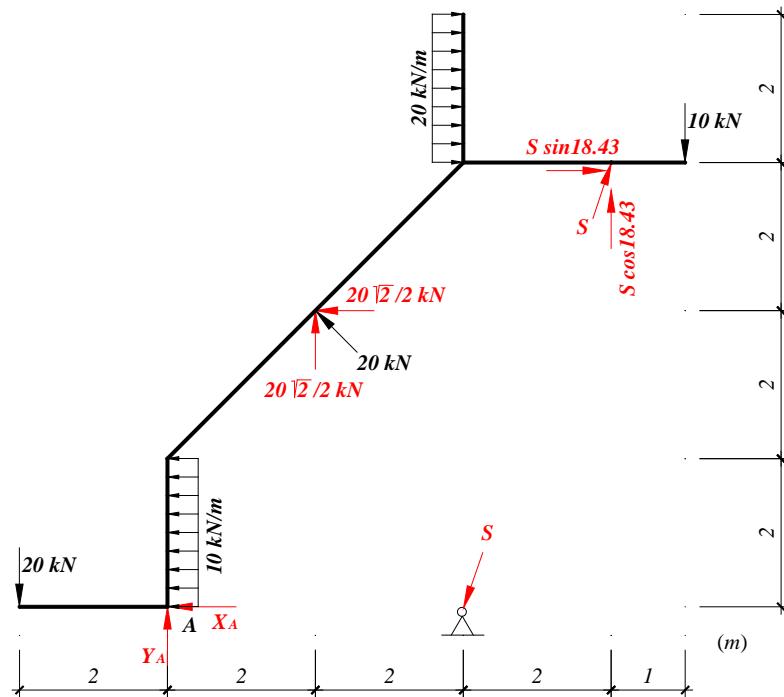




2. Za nosač na slici nacrtati dijagrame presječnih sila ( $M$ ,  $T$ ,  $N$ ).



### RJEŠENJE



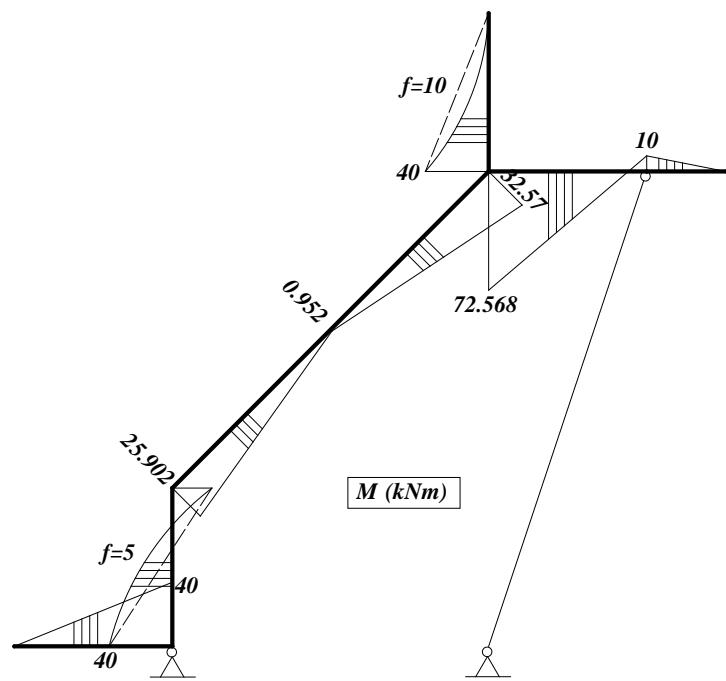
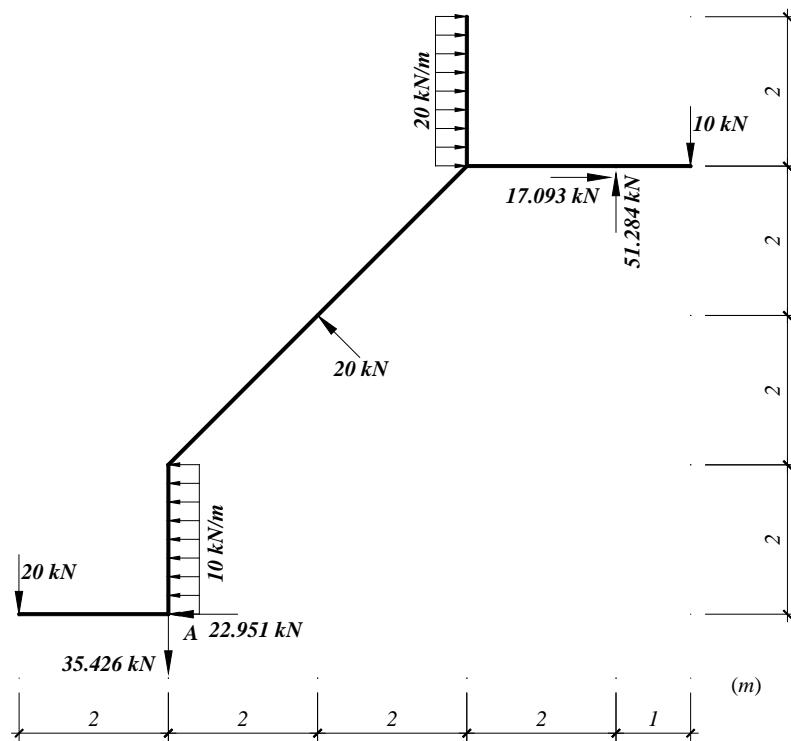
Uslovi ravnoteže:

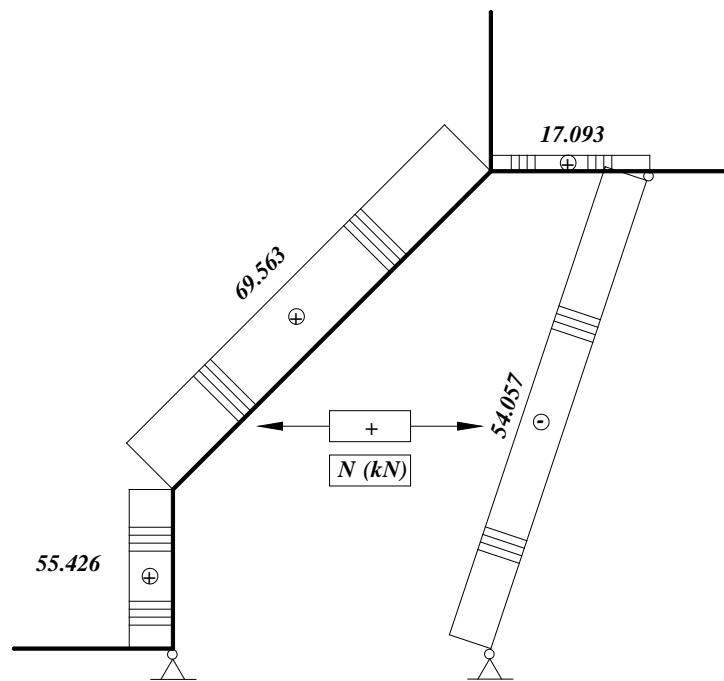
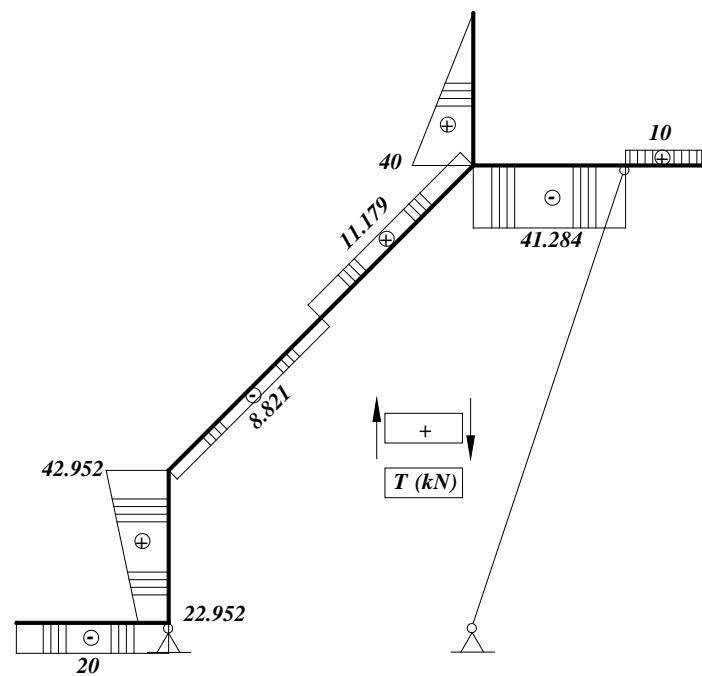
$$1. \sum M_A = 0 \rightarrow S \cos 18.43 \cdot 6 - S \sin 18.43 \cdot 6 - 20 \cdot 2 \cdot 7 + \frac{20\sqrt{2}}{2} \cdot (2+4) + 10 \cdot 2 \cdot 1 + 20 \cdot 2 - 10 \cdot 7 = 0 \rightarrow S = 54.057 \text{ kN}$$

$$2. \sum X = 0 \rightarrow X_A + 10 \cdot 2 + \frac{20\sqrt{2}}{2} - 20 \cdot 2 - S \sin 18.43 = 0 \rightarrow X_A = 22.95 \text{ kN}$$

$$3. \quad \sum Y = 0 \quad \rightarrow \quad Y_A - 20 + \frac{20\sqrt{2}}{2} - 10 + S \cos 18.43 = 0 \quad \rightarrow \quad Y_A = -35.426 \text{ kN}$$

Kontrola: npr.  $\sum M_{(bilo koja tačka ravni)} = 0$  – zadovoljeno



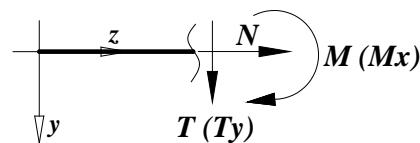


# VJEŽBA BR. 2

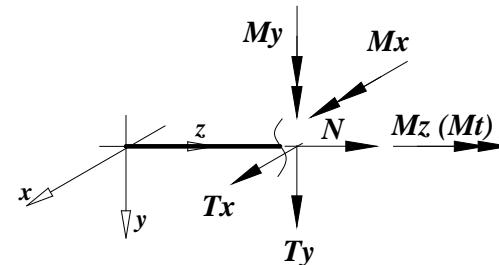
## PONAVLJANJE GRADIVA IZ GRAĐEVINSKE MEHANIKE 1

### PROSTORNI NOSAČI – IZLOMLJENE KONZOLE

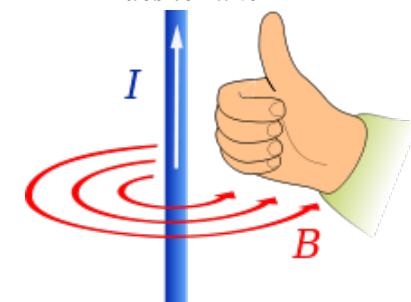
*3 presjecne sile  
nosaci u ravni*



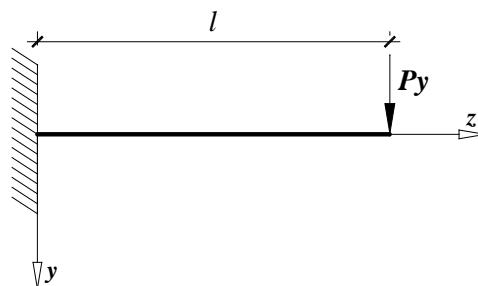
*6 presjecnih sila  
prostorni nosaci*



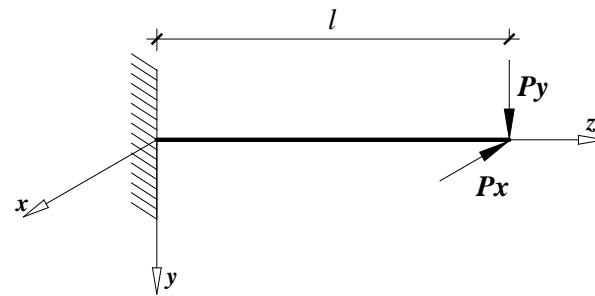
*Pravilo  
desne ruke*



*Nosac u ravni "yz"*



*Prostorni nosac*

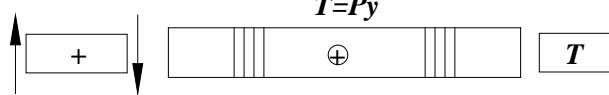


$$M = P_y \cdot l$$



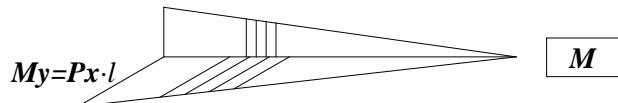
**M**

$$T = P_y$$



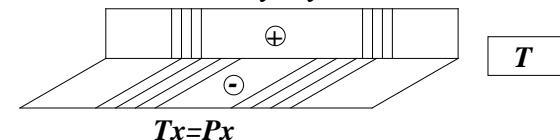
**T**

$$M_x = P_y \cdot l$$



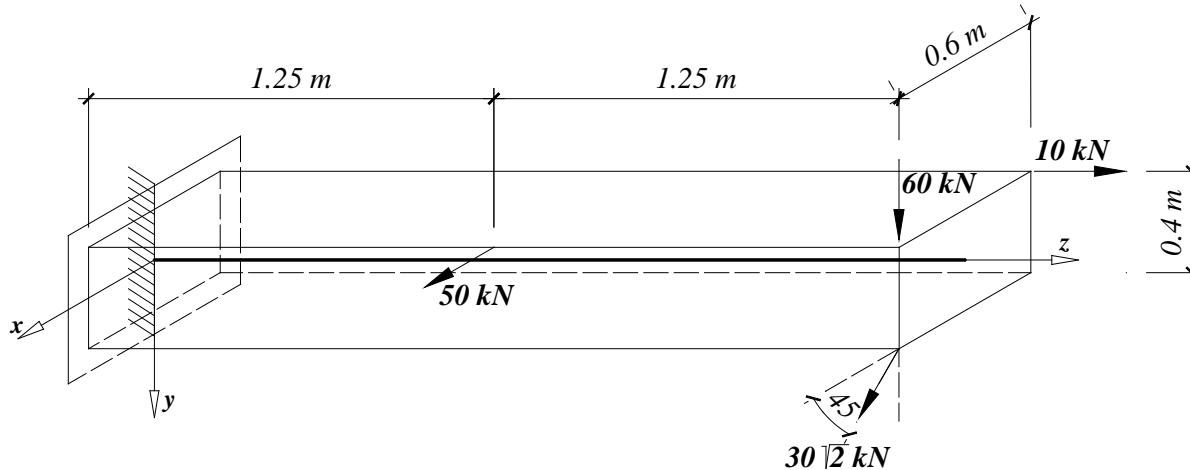
**M**

$$T_y = P_y$$

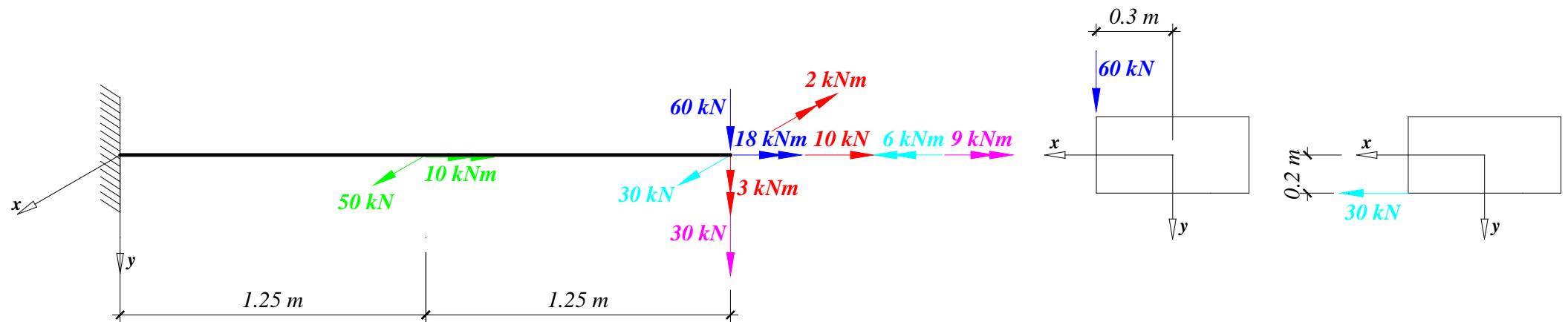


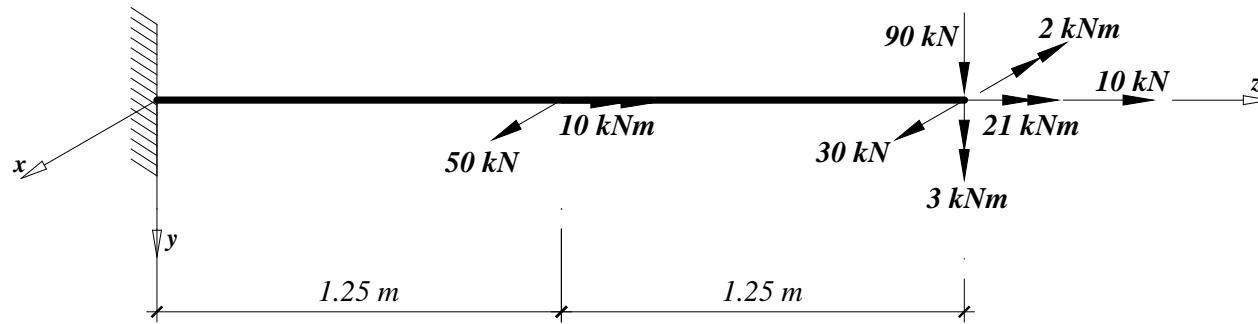
**T**

1. Za nosač na slici nacrtati dijagrame presječnih sila ( $M$ ,  $T$ ,  $N$ ).



### RJEŠENJE





$$Mx=227$$

$$My=140.5$$

$$M \text{ (kNm)}$$

$$Mz=3I$$

$$40.5$$

2

3

$$T \text{ (kN)}$$

$$Ty=90$$

$$Tx=80$$

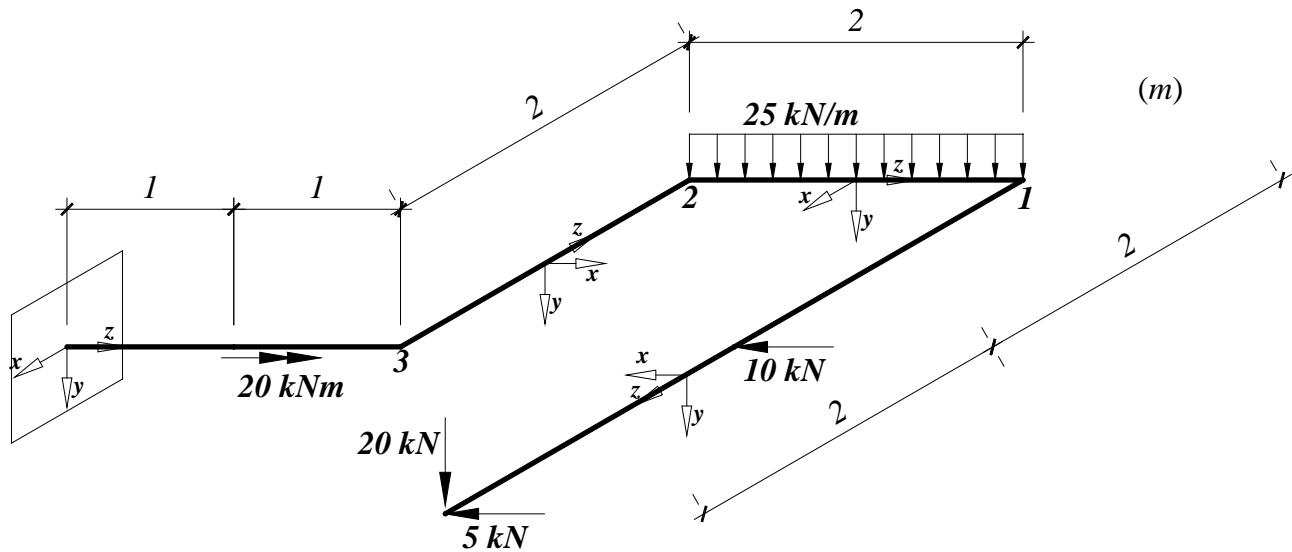
$$30$$

$$N \text{ (kN)}$$

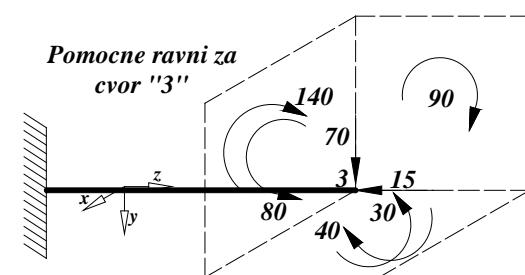
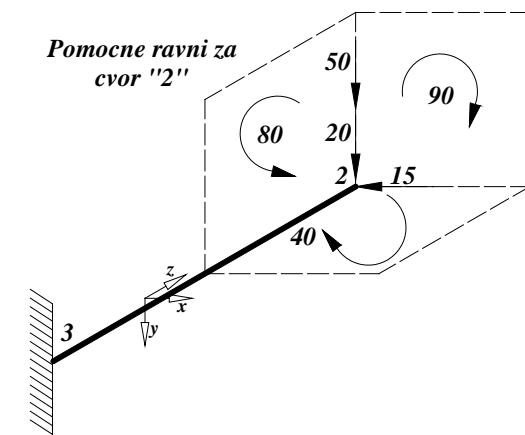
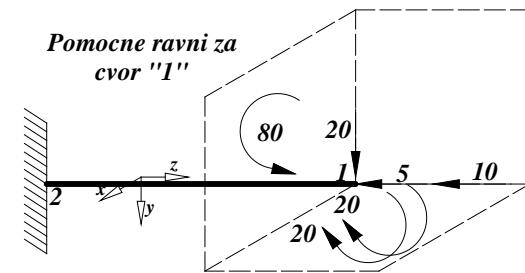
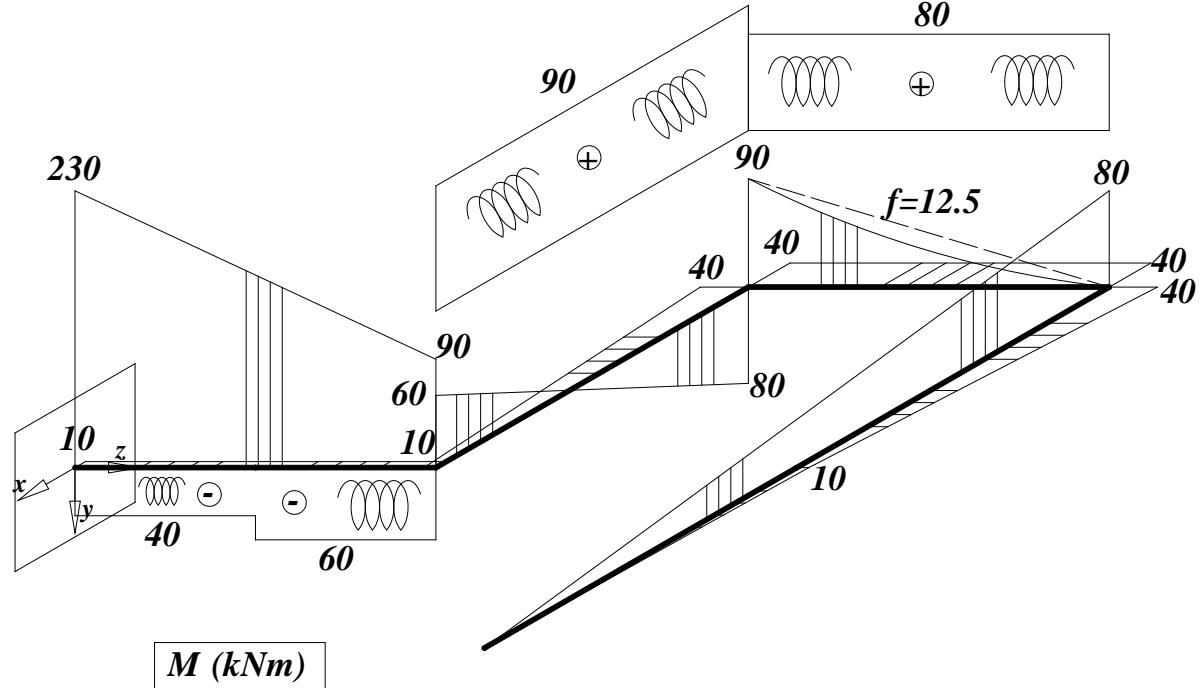
$$N=10$$

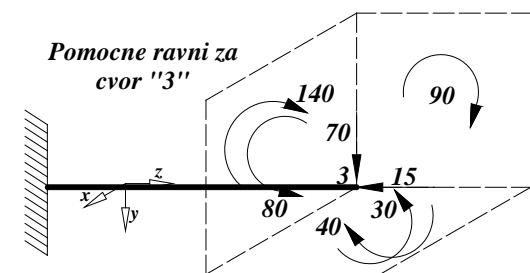
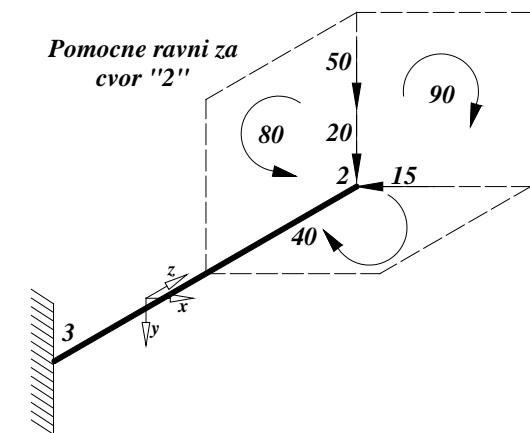
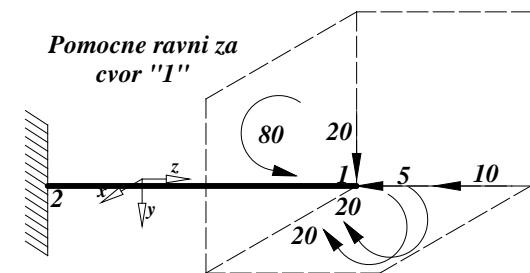
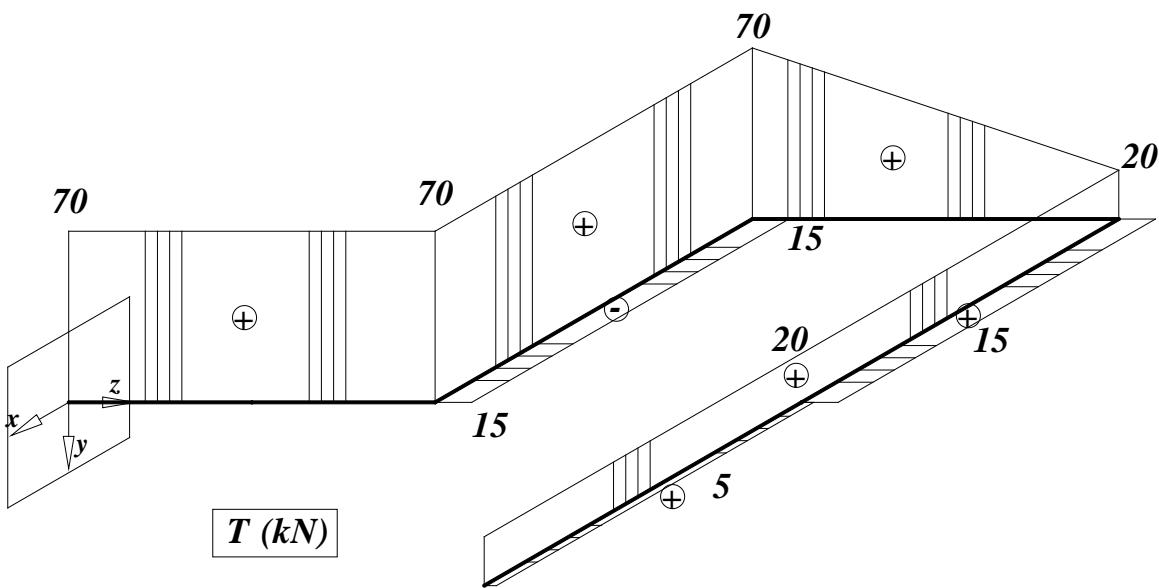


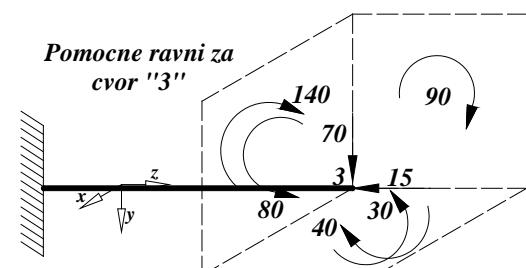
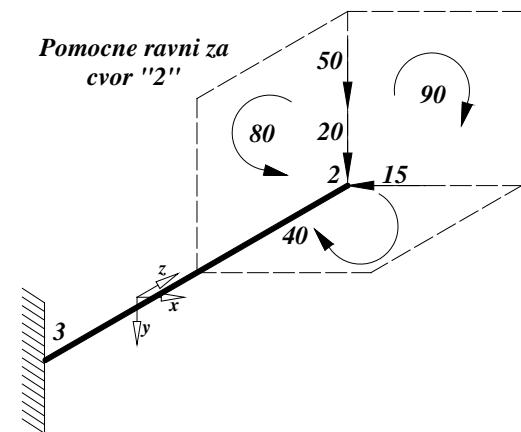
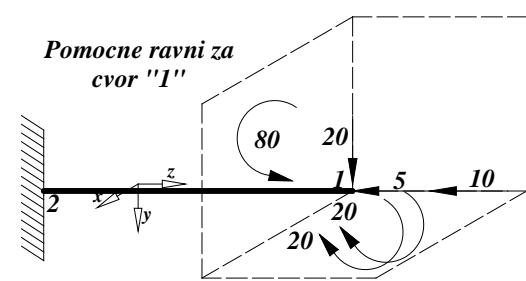
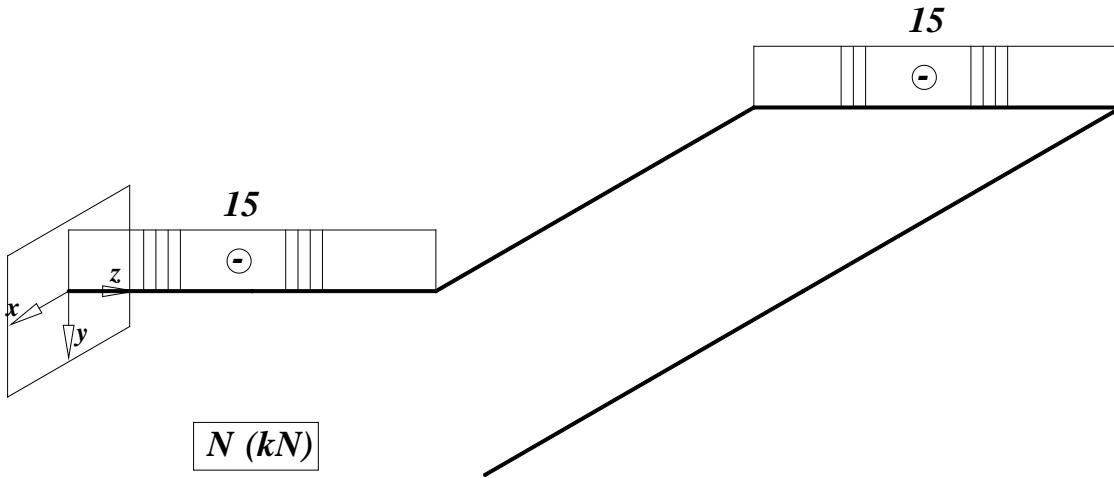
2. Za nosač na slici nacrtati dijagrame presječnih sila ( $M$ ,  $T$ ,  $N$ ).



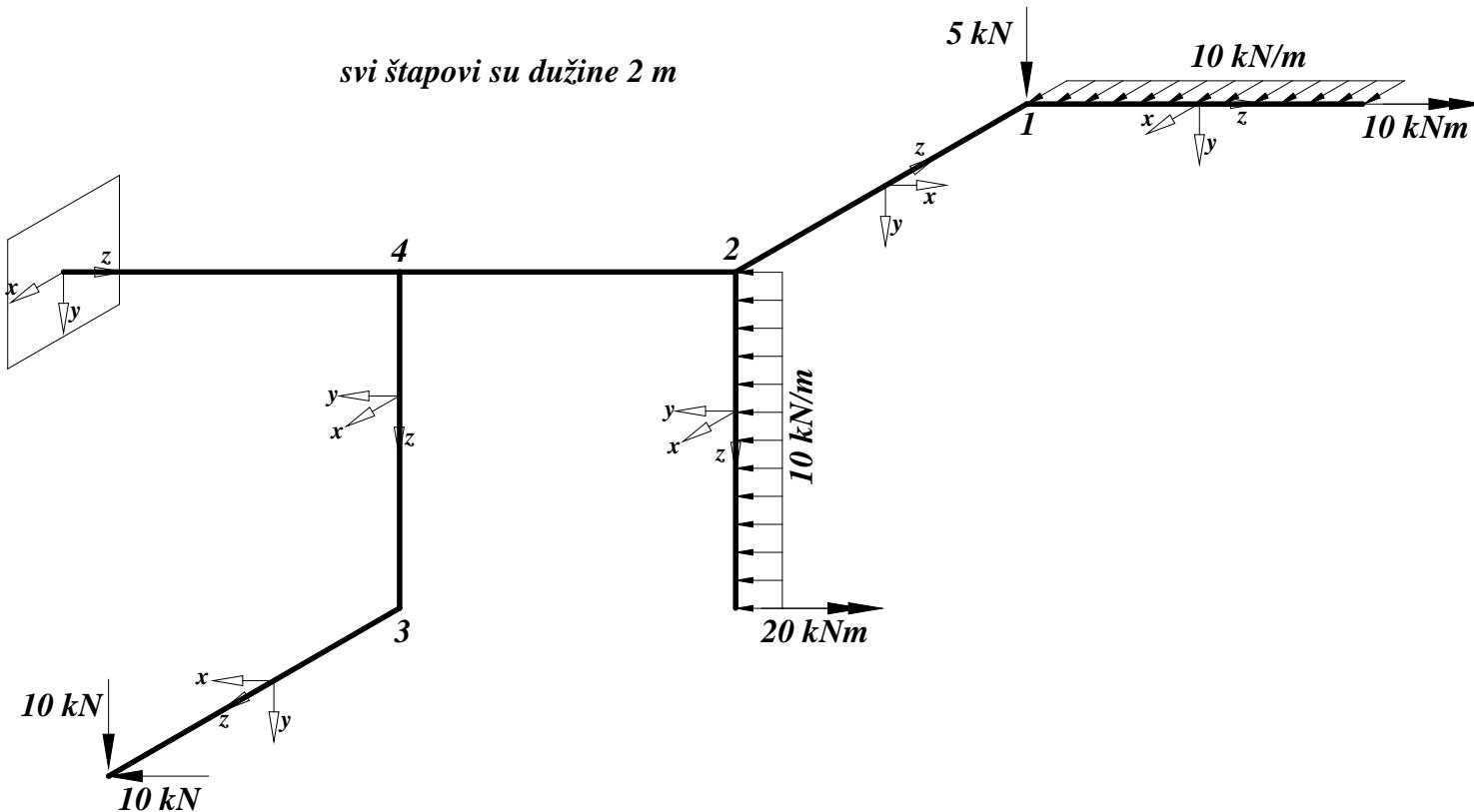
**RJEŠENJE**



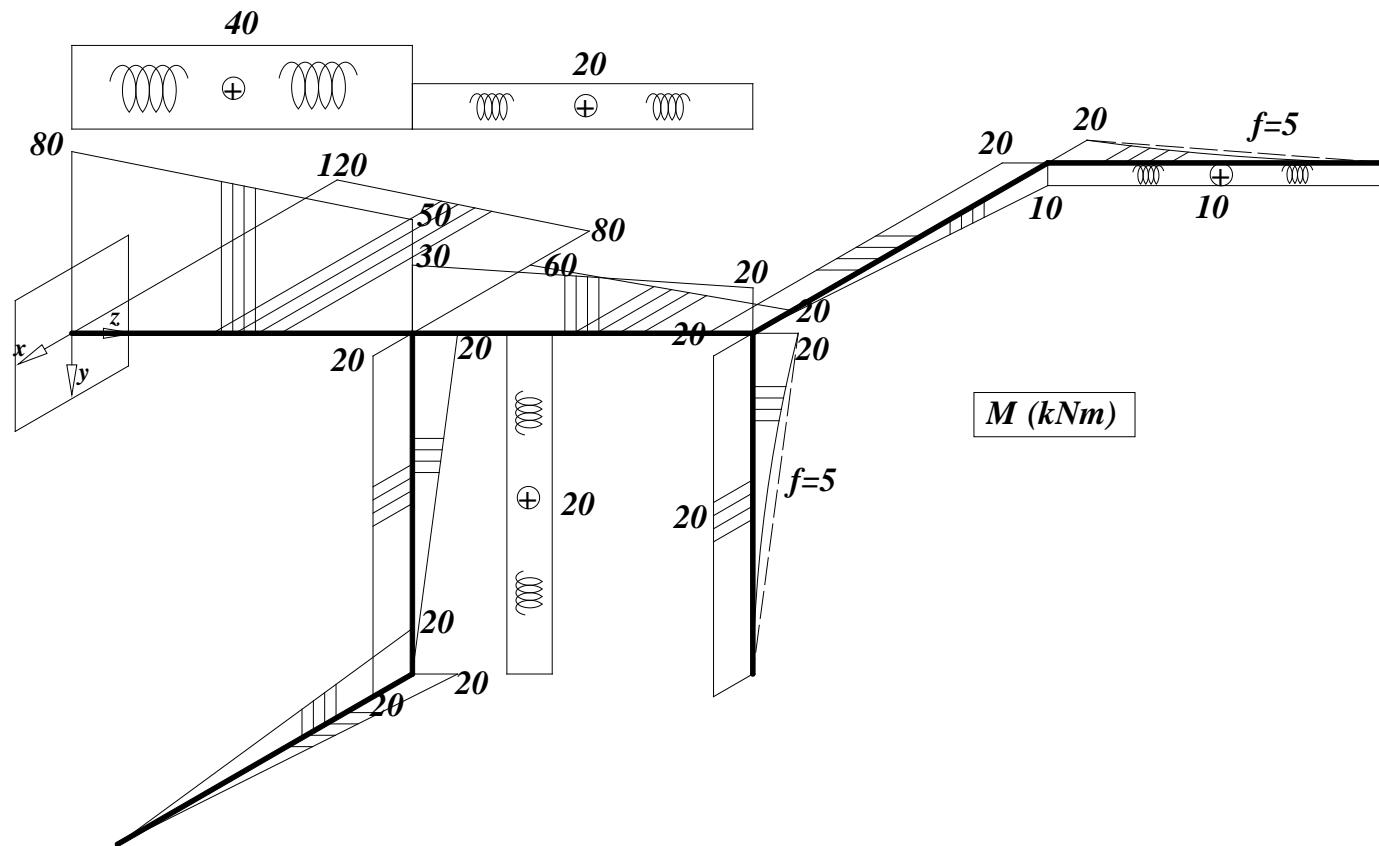




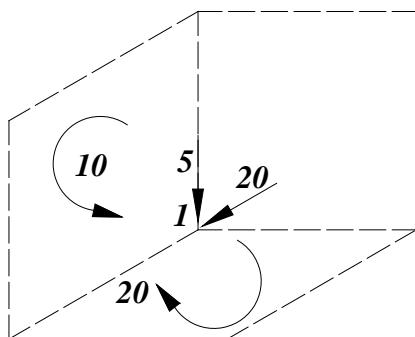
3. Za nosač na slici nacrtati dijagrame presječnih sila ( $M$ ,  $T$ ,  $N$ ).



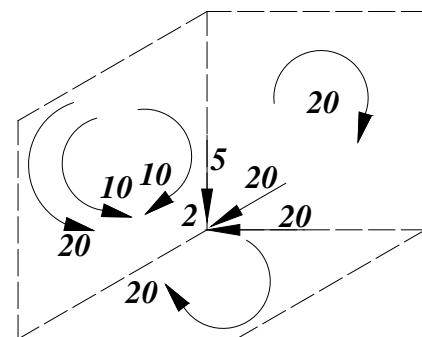
**RJEŠENJE**



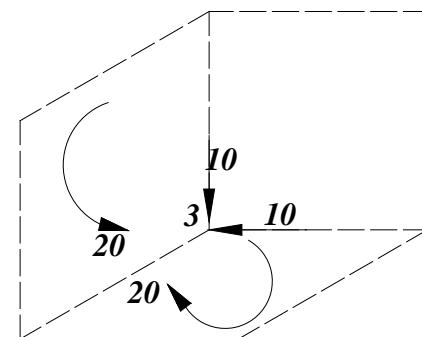
Pomocne ravni za  
cvor "1"



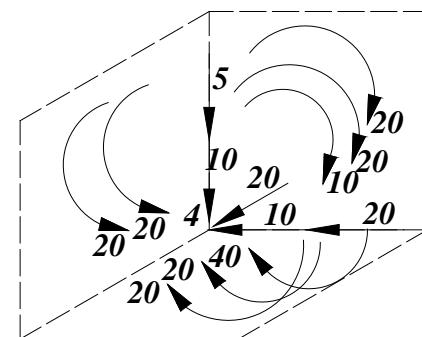
Pomocne ravni za  
cvor "2"

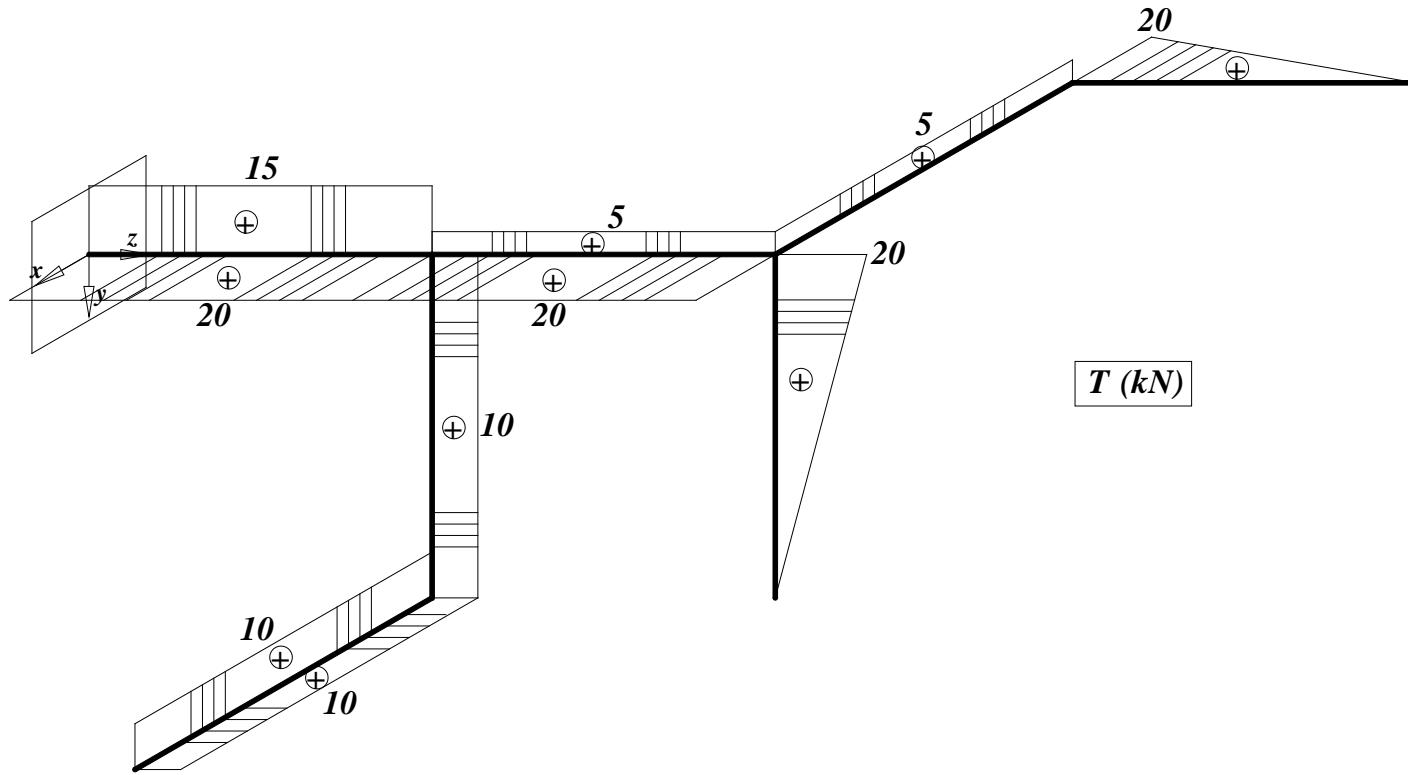


Pomocne ravni za  
cvor "3"

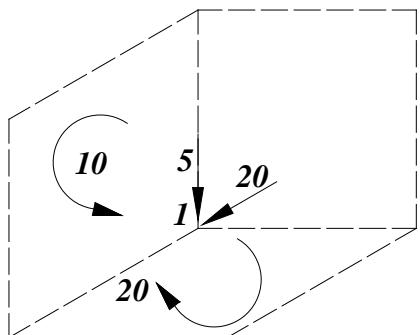


Pomocne ravni za  
cvor "4"

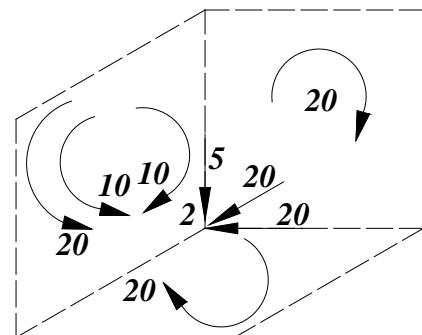




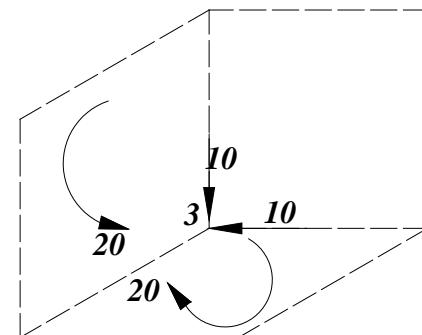
Pomocne ravni za  
cvor "1"



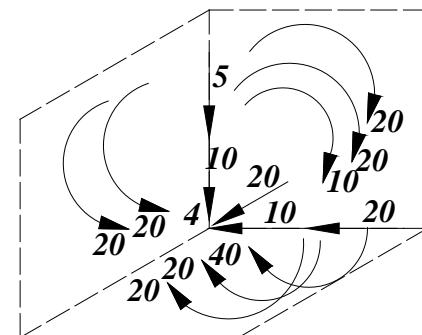
Pomocne ravni za  
cvor "2"

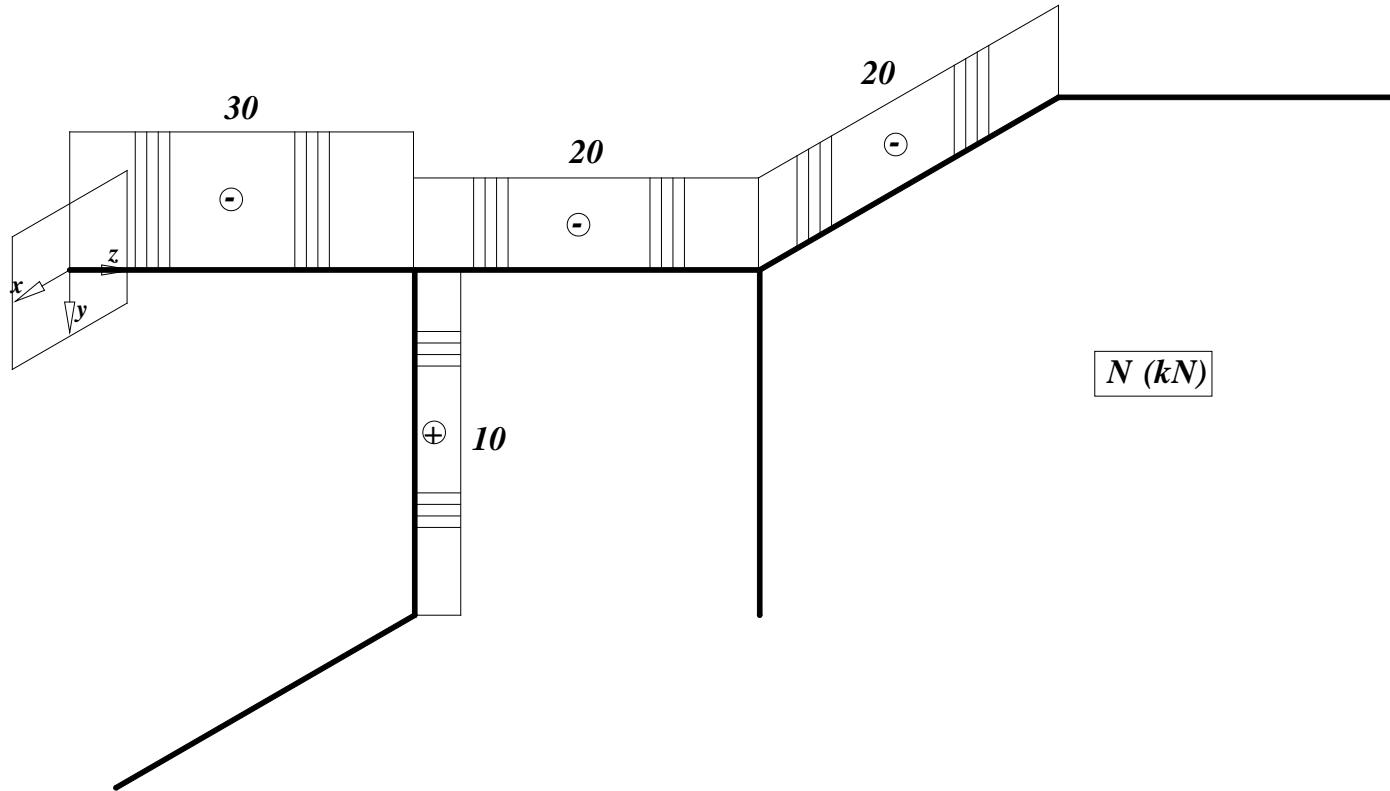


Pomocne ravni za  
cvor "3"

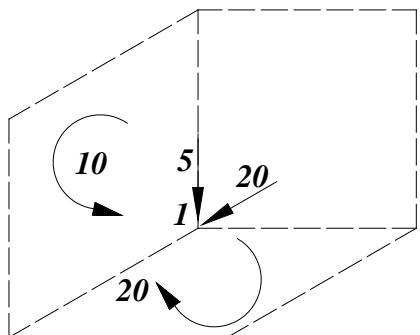


Pomocne ravni za  
cvor "4"

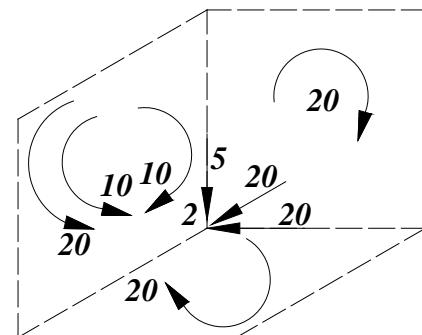




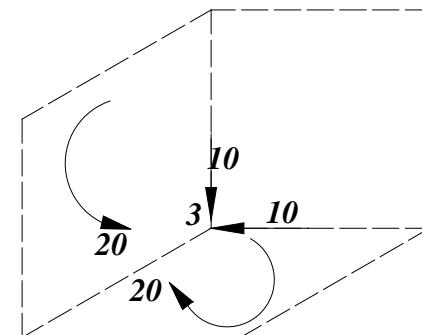
Pomocne ravni za  
cvor "1"



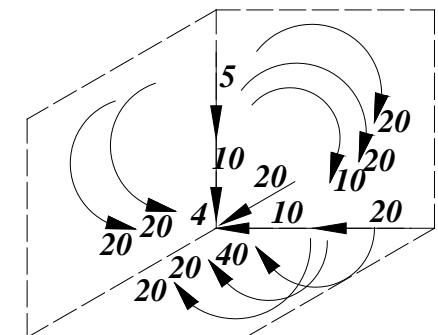
Pomocne ravni za  
cvor "2"



Pomocne ravni za  
cvor "3"



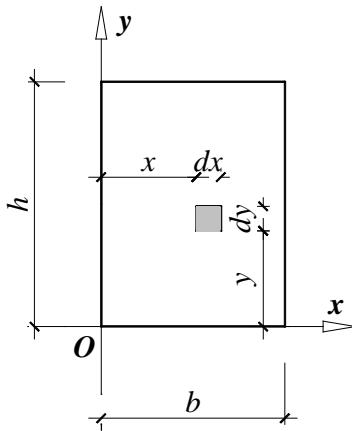
Pomocne ravni za  
cvor "4"



### VJEŽBA BR. 3

## GEOMETRIJSKE KARAKTERISTIKE RAVNIH POVRŠINA

1. Odrediti momente inercije u odnosu na  $x$  i  $y$  ose pravougaonika na slici.



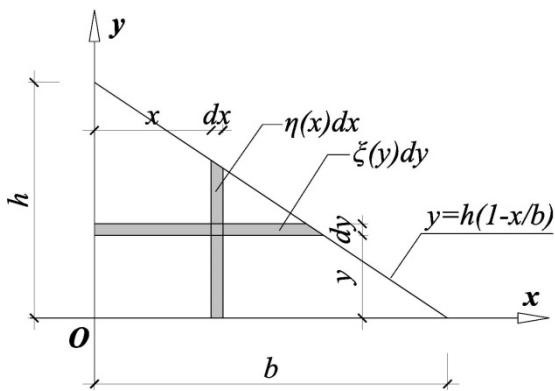
$$I_x = \int_A y^2 dA = \iint_A y^2 dx dy = \int_0^b dx \int_0^h y^2 dy = x \Big|_0^b \frac{y^3}{3} \Big|_0^h = \frac{bh^3}{3}$$

$$I_y = \int_A x^2 dA = \iint_A x^2 dx dy = \int_0^h dy \int_0^b x^2 dx = y \Big|_0^h \frac{x^3}{3} \Big|_0^b = \frac{hb^3}{3}$$

$$I_{xy} = \int_A xy dA = \iint_A xy dx dy = \int_0^b x dx \int_0^h y dy = \frac{x^2}{2} \Big|_0^b \frac{y^2}{2} \Big|_0^h = \frac{b^2 h^2}{4}$$

$$I_O = I_x + I_y = \frac{bh^3}{3} + \frac{b^3 h}{3} = \frac{bh}{3} (h^2 + b^2)$$

2. Odrediti momente inercije u odnosu na  $x$  i  $y$  ose pravouglog trougla na slici.



$$\xi(y) : b = (h - y) : h \Rightarrow \xi(y) = \frac{b}{h}(h - y)$$

$$\eta(x) : h = (b - x) : b \Rightarrow \eta(x) = \frac{h}{b}(b - x)$$

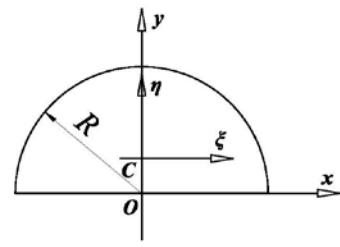
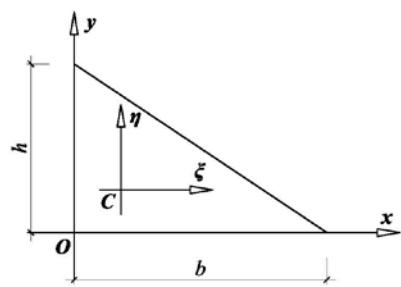
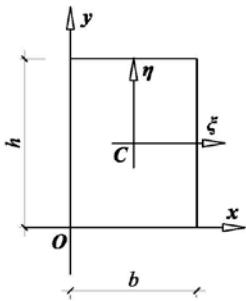
$$I_x = \int_A y^2 dA = \int_0^h y^2 \xi(y) dy = \int_0^h y^2 \frac{b}{h}(h - y) dy = \frac{b}{h} \int_0^h (y^2 h - y^3) dy = \frac{b}{h} \left( \frac{y^3}{3} h - \frac{y^4}{4} \right) \Big|_0^h = \frac{b}{h} \left( \frac{h^4}{3} - \frac{h^4}{4} \right) = \frac{bh^3}{12}$$

$$I_y = \int_A x^2 dA = \int_0^b x^2 \eta(x) dx = \int_0^b x^2 \frac{h}{b}(b - x) dx = \frac{h}{b} \int_0^b (x^2 b - x^3) dx = \frac{h}{b} \left( \frac{x^3}{3} b - \frac{x^4}{4} \right) \Big|_0^b = \frac{h}{b} \left( \frac{b^4}{3} - \frac{b^4}{4} \right) = \frac{hb^3}{12}$$

$$I_{xy} = \int_A xy dA = \int_0^b x dx \int_0^{h(1-x/b)} y dy = \int_0^b x \frac{1}{2} h^2 \left( 1 - \frac{x}{b} \right)^2 dx = \frac{h^2}{2} \int_0^b \left( x - 2 \frac{x^2}{b} + \frac{x^3}{b^2} \right) dx = \frac{h^2}{2} \left( \frac{x^2}{2} - 2 \frac{x^3}{3b} + \frac{x^4}{4b^2} \right) \Big|_0^b = \frac{h^2}{2} \left( \frac{b^2}{2} - \frac{2}{3} b^2 + \frac{1}{4} b^2 \right) = \frac{h^2 b^2}{2} \left( \frac{6-8+3}{12} \right) = \frac{b^2 h^2}{24}$$

$$I_O = I_x + I_y = \frac{bh^3}{12} + \frac{b^3 h}{12} = \frac{bh}{12} (h^2 + b^2)$$

3. Odrediti momente inercije figura na slici za sistem težišnih osa  $\xi\eta$ .



$$I_x = \frac{bh^3}{3} \quad I_y = \frac{b^3h}{3} \quad I_{xy} = \frac{b^2h^2}{4}$$

$$I_x = \frac{bh^3}{12} \quad I_y = \frac{b^3h}{12} \quad I_{xy} = \frac{b^2h^2}{24}$$

$$I_x = \frac{\pi R^4}{8} \quad I_y = \frac{\pi R^4}{8} \quad I_{xy} = 0$$

$$I_x = I_\xi + a^2A$$

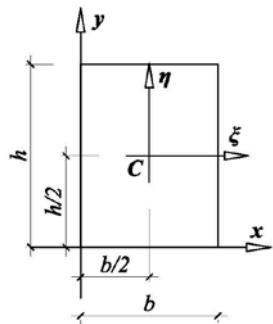
$$I_y = I_\eta + b^2A$$

$$I_{xy} = I_{\xi\eta} + abA$$

$$I_\xi = I_x - a^2A$$

$$I_\eta = I_y - b^2A$$

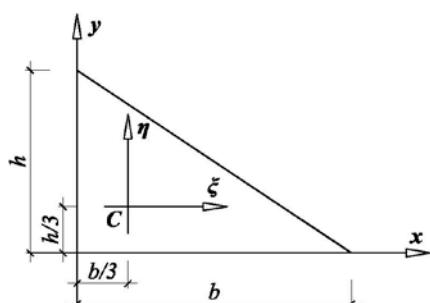
$$I_{\xi\eta} = I_{xy} - abA$$



$$I_\xi = \frac{bh^3}{3} - \left(\frac{h}{2}\right)^2 bh = \frac{bh^3}{12}$$

$$I_\eta = \frac{b^3h}{3} - \left(\frac{b}{2}\right)^2 bh = \frac{b^3h}{12}$$

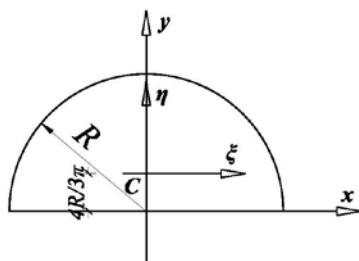
$$I_{\xi\eta} = \frac{b^2h^2}{4} - \frac{h}{2} \frac{b}{2} bh = 0$$



$$I_\xi = \frac{bh^3}{12} - \left(\frac{h}{3}\right)^2 \frac{1}{2} bh = \frac{bh^3}{36}$$

$$I_\eta = \frac{b^3h}{12} - \left(\frac{b}{3}\right)^2 \frac{1}{2} bh = \frac{b^3h}{36}$$

$$I_{\xi\eta} = \frac{b^2h^2}{24} - \frac{h}{3} \frac{b}{3} \frac{1}{2} bh = -\frac{b^2h^2}{72}$$

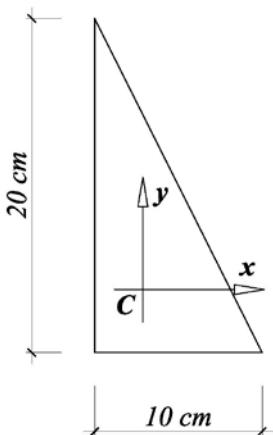


$$I_\xi = \frac{\pi R^4}{8} - \left(\frac{4R}{3\pi}\right)^2 \frac{1}{2} \pi R^2 = 0.10976 R^4$$

$$I_\eta = \frac{\pi R^4}{8} - 0 = \frac{\pi R^4}{8}$$

$$I_{\xi\eta} = 0 - 0 = 0$$

4. Za pravougli trougao na slici odrediti glavne centralne ose inercije i glavne centralne momente inercije.



Centralni momenti inercije:

$$I_x = \frac{bh^3}{36} = \frac{10 \cdot 20^3}{36} = 2222.22 \text{ cm}^4$$

$$I_y = \frac{b^3h}{36} = \frac{10^3 \cdot 20}{36} = 555.56 \text{ cm}^4$$

$$I_{xy} = -\frac{b^2h^2}{72} = -\frac{10^2 \cdot 20^2}{72} = -555.56 \text{ cm}^4$$

$$\begin{aligned} I_1 &= \frac{1}{2}(I_x + I_y) + \frac{1}{2}\sqrt{(I_x - I_y)^2 + 4I_{xy}^2} \\ I_2 &= \frac{1}{2}(I_x + I_y) - \frac{1}{2}\sqrt{(I_x - I_y)^2 + 4I_{xy}^2} \end{aligned} \Rightarrow I_{1/2} = \frac{1}{2}(I_x + I_y) \pm \frac{1}{2}\sqrt{(I_x - I_y)^2 + 4I_{xy}^2}$$

$$\begin{aligned} I_{1/2} &= \frac{1}{2}(I_x + I_y) \pm \frac{1}{2}\sqrt{(I_x - I_y)^2 + 4I_{xy}^2} = \\ &= \frac{1}{2}(2222.22 + 555.56) \pm \frac{1}{2}\sqrt{(2222.22 - 555.56)^2 + 4(-555.56)^2} = \\ &= 1388.89 \pm 1001.54 \end{aligned}$$

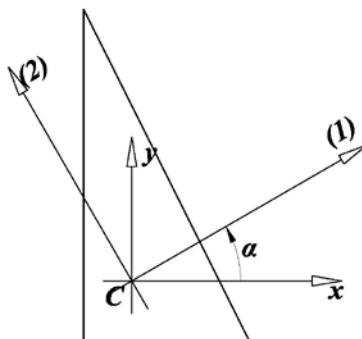
$$\begin{aligned} I_1 &= 2390.43 \text{ cm}^4 \\ I_2 &= 387.35 \text{ cm}^4 > 0 \end{aligned} \quad \text{Glavni centralni momenti inercije}$$

Kontrola  $I_x + I_y = I_1 + I_2$

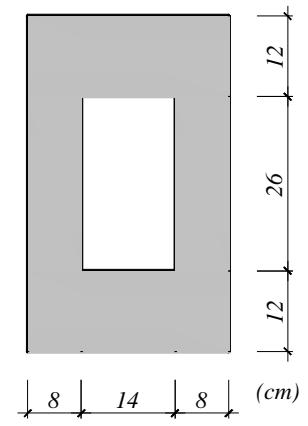
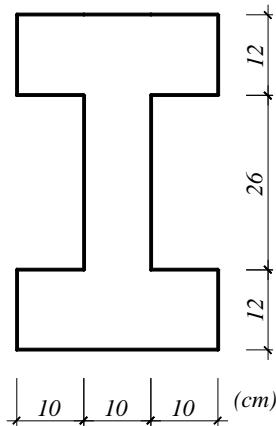
Položaj glavne centralne ose (1) u odnosu na horizontalnu osu (x) je definisan uglom ( $\alpha$ ):

$$\tan 2\alpha = \frac{-2I_{xy}}{I_x - I_y} = \frac{-2 \cdot (-555.56)}{2222.22 - 555.56} = 0.667$$

$$(I_x - I_y) > 0 \quad \alpha = \frac{1}{2} \arctan 0.667 = 16.845^\circ$$



5. Za presjeke na slici odrediti glavne centralne ose inercije i glavne centralne momente inercije.

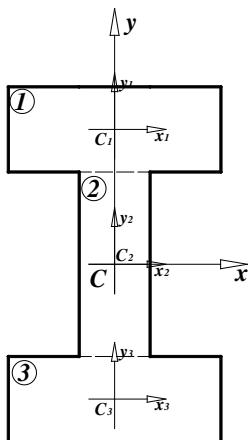


$$I_x = I_x^{[1]} + I_x^{[2]} + I_x^{[3]} = 283\ 206.67 \text{ cm}^4 = I_1$$

$$I_x^{[1]} = I_{x_1} + y_{C_1}^2 A_1 = \frac{30 \cdot 12^3}{12} + 19^2 \cdot 30 \cdot 12 = 134\ 280 \text{ cm}^4$$

$$I_x^{[2]} = I_{x_2} + y_{C_2}^2 A_2 = \frac{10 \cdot 26^3}{12} + 0 = 14\ 646.67 \text{ cm}^4$$

$$I_x^{[3]} = I_{x_3} + y_{C_3}^2 A_3 = \frac{30 \cdot 12^3}{12} + (-19)^2 \cdot 30 \cdot 12 = 134\ 280 \text{ cm}^4$$



$$I_y = I_y^{[1]} + I_y^{[2]} + I_y^{[3]} = 56\ 166.67 \text{ cm}^4 = I_2$$

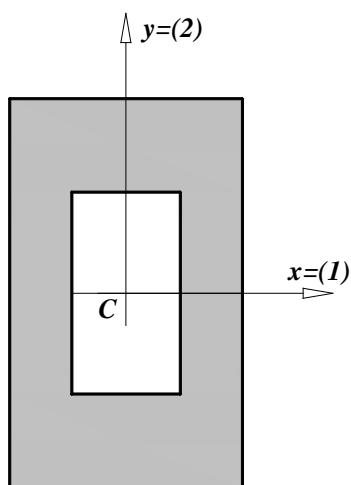
$$I_y^{[1]} = I_{y_1} + x_{C_1}^2 A_1 = \frac{30^3 \cdot 12}{12} + 0 = 27\ 000 \text{ cm}^4 = I_y^{[3]}$$

$$I_y^{[2]} = I_{y_2} + x_{C_2}^2 A_2 = \frac{26 \cdot 10^3}{12} + 0 = 2\ 166.67 \text{ cm}^4$$

$$I_{xy} = 0 \quad x \text{ i } y \text{ su ose simetrije}$$

$$I_1 \equiv I_x \quad x \equiv (1)$$

$$I_2 \equiv I_y \quad y \equiv (2)$$



$$I_x = 291\ 994.67 \text{ cm}^4 = I_1$$

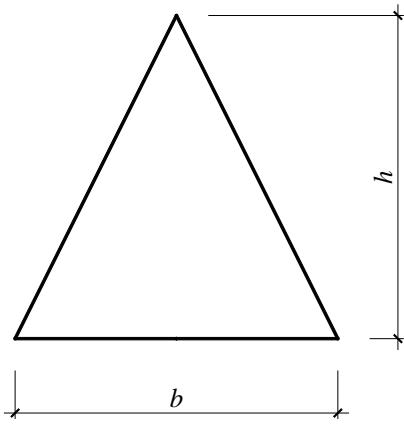
$$I_y = 106\ 554.67 \text{ cm}^4 = I_2$$

$$I_{xy} = 0 \quad x \text{ i } y \text{ su ose simetrije}$$

$$I_1 \equiv I_x \quad x \equiv (1)$$

$$I_2 \equiv I_y \quad y \equiv (2)$$

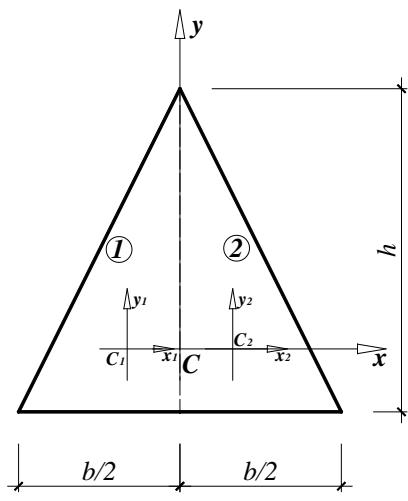
6. Za jednakokraki trougao na slici odrediti glavne centralne ose inercije i glavne centralne momente inercije.



$$I_x = I_x^{[1]} + I_x^{[2]} = \frac{\left(\frac{b}{2}\right)h^3}{36} + \frac{\left(\frac{b}{2}\right)h^3}{36} = \frac{bh^3}{36}$$

$$I_x^{[1]} = I_{x_1} + y_{C_1}^2 A_1 = \frac{\left(\frac{b}{2}\right)h^3}{36} + 0$$

$$I_x^{[2]} = I_{x_2} + y_{C_2}^2 A_2 = \frac{\left(\frac{b}{2}\right)h^3}{36} + 0$$



$$I_y = I_y^{[1]} + I_y^{[2]} = \frac{b^3 h}{96} + \frac{b^3 h}{96} = \frac{b^3 h}{48}$$

$$I_y^{[1]} = I_{y_1} + x_{C_1}^2 A_1 = \frac{\left(\frac{b}{2}\right)^3 h}{36} + \left(-\frac{b}{6}\right)^2 \frac{1}{2} \frac{b}{2} h = \frac{b^3 h}{96}$$

$$I_y^{[2]} = I_{y_2} + x_{C_2}^2 A_2 = \frac{\left(\frac{b}{2}\right)^3 h}{36} + \left(\frac{b}{6}\right)^2 \frac{1}{2} \frac{b}{2} h = \frac{b^3 h}{96}$$

$$I_{xy} = I_{xy}^{[1]} + I_{xy}^{[2]} = 0$$

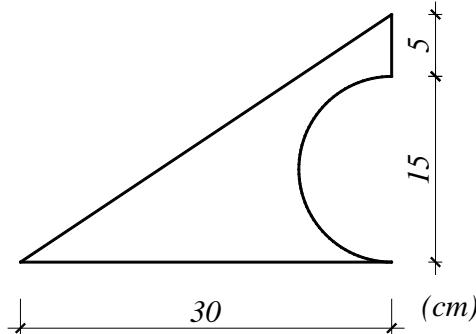
$$I_{xy}^{[1]} = I_{x_1 y_1} + x_{C_1} y_{C_1} A_1 = \frac{\left(\frac{b}{2}\right)^2 h^2}{72} + 0$$

$$I_{xy}^{[2]} = I_{x_2 y_2} + x_{C_2} y_{C_2} A_2 = -\frac{\left(\frac{b}{2}\right)^2 h^2}{72} + 0$$

## VJEŽBA BR. 4

### GEOMETRIJSKE KARAKTERISTIKE RAVNIH POVRŠINA

1. Za presjek na slici odrediti glavne centralne momente inercije i nacrtati glavne centralne ose inercije. Rezultat provjeriti Mohr-ovim krugom inercije.



**RJEŠENJE**

1. Težište

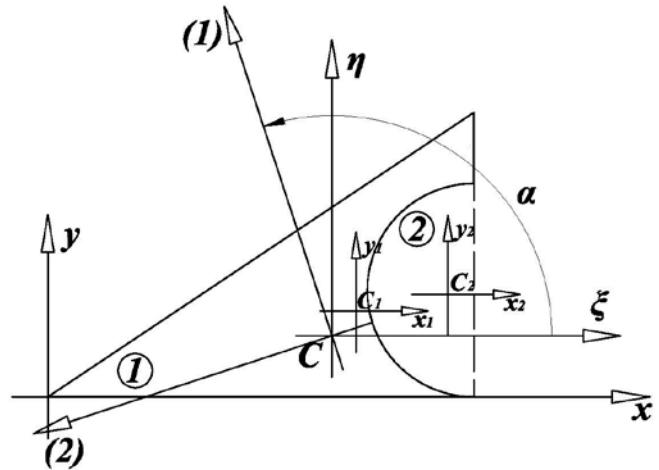
$$C_1(x_{C_1}, y_{C_1}) = (20, 6.667) \text{ cm} \quad A_1 = 300 \text{ cm}^2$$

$$C_2(x_{C_2}, y_{C_2}) = (26.817, 7.5) \text{ cm} \quad A_2 = 88.357 \text{ cm}^2$$

$$x_c = \frac{x_{C_1}A_1 - x_{C_2}A_2}{A_1 - A_2} = \frac{20 \cdot 300 - 26.817 \cdot 88.357}{300 - 88.357} = 17.154 \text{ cm}$$

$$y_c = \frac{y_{C_1}A_1 - y_{C_2}A_2}{A_1 - A_2} = \frac{6.667 \cdot 300 - 7.5 \cdot 88.357}{300 - 88.357} = 6.319 \text{ cm}$$

$$C(x_c, y_c) = (17.154, 6.319) \text{ cm}$$

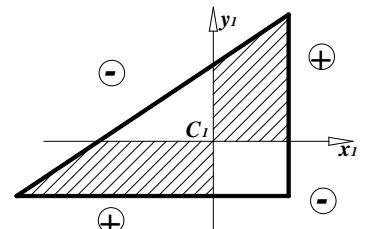


2. Centralni momenti inercije

$$I_\xi = I_\xi^{[1]} - I_\xi^{[2]} = 5\ 337.239 \text{ cm}^4$$

$$I_\xi^{[1]} = I_{x_1} + (y_{C_1} - y_c)^2 A_1 = \frac{30 \cdot 20^3}{36} + (6.667 - 6.319)^2 300 = 6\ 703 \text{ cm}^4$$

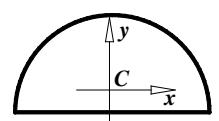
$$I_\xi^{[2]} = I_{x_2} + (y_{C_2} - y_c)^2 A_2 = \frac{7.5^4 \pi}{8} + (7.5 - 6.319)^2 88.357 = 1\ 365.761 \text{ cm}^4$$



$$I_\eta = I_\eta^{[1]} - I_\eta^{[2]} = 8\ 832.419 \text{ cm}^4$$

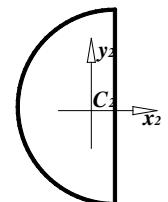
$$I_\eta^{[1]} = I_{y_1} + (x_{C_1} - x_c)^2 A_1 = \frac{30^3 \cdot 20}{36} + (20 - 17.154)^2 300 = 17\ 429.915 \text{ cm}^4$$

$$I_\eta^{[2]} = I_{y_2} + (x_{C_2} - x_c)^2 A_2 = 0.10976 R^4 + (26.817 - 17.154)^2 88.357 = 8\ 597.496 \text{ cm}^4$$



$$I_{\xi\eta} = I_{\xi\eta}^{[1]} - I_{\xi\eta}^{[2]} = 4\ 288.792 \text{ cm}^4$$

$$I_{\xi\eta}^{[1]} = I_{x_1 y_1} + (y_{C_1} - y_c)(x_{C_1} - x_c) A_1 = +\frac{30^2 \cdot 20^2}{72} + 0.348 \cdot 2.846 \cdot 300 = 5\ 297.122 \text{ cm}^4$$



$$I_{\xi\eta}^{[2]} = I_{x_2y_2} + (y_{c2} - y_c)(x_{c2} - x_c)A_l = 0 + 9.663 \cdot 1.181 \cdot 88.357 = 1008.330 \text{ cm}^4$$

### 3. Glavni centralni momenti inercije

$$\begin{aligned} I_{1/2} &= \frac{1}{2}(I_\xi + I_\eta) \pm \frac{1}{2}\sqrt{(I_\xi - I_\eta)^2 + 4I_{\xi\eta}^2} = \\ &= \frac{1}{2}(5337.239 + 8832.419) \pm \frac{1}{2}\sqrt{(5337.239 - 8832.419)^2 + 4(4288.792)^2} = \\ &= 7084.829 \pm 4631.178 \end{aligned}$$

$$I_1 = 11716.007 \text{ cm}^4$$

$$I_2 = 2453.651 \text{ cm}^4 > 0$$

Kontrola  $I_\xi + I_\eta = I_1 + I_2$

$$\tan 2\alpha = \frac{-2I_{\xi\eta}}{I_\xi - I_\eta} = \frac{-2 \cdot 4288.792}{5337.239 - 8832.419} = 2.454$$

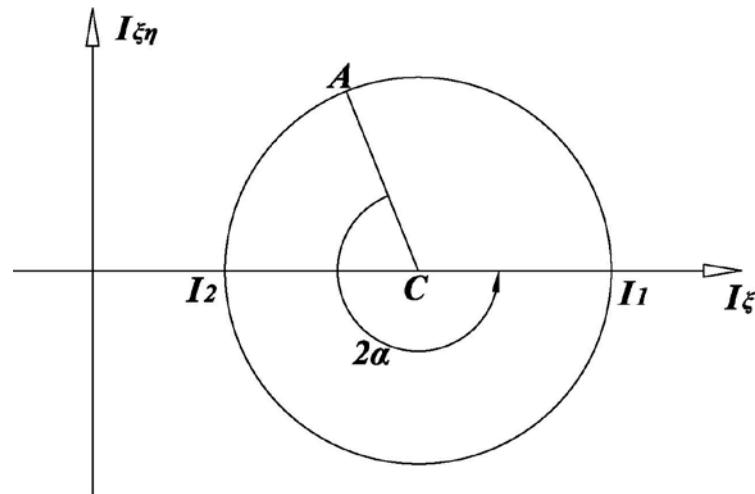
$$(I_\xi - I_\eta) < 0 \quad \alpha = \frac{1}{2} \arctan 2.454 + 90^\circ = 123.91^\circ$$

### 4. Mohr-ov krug inercije

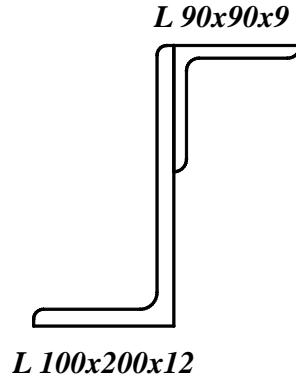
$$C\left[\frac{1}{2}(I_\xi + I_\eta); 0\right] = \left[\frac{1}{2}(5337.239 + 8832.419); 0\right] = [7084.829; 0] \text{ cm}^4$$

$$A[I_\xi; I_{\xi\eta}] = [5337.239; 4288.792] \text{ cm}^4$$

Usvaja se razmjera  $1000 \text{ cm}^4 = 1 \text{ cm}$

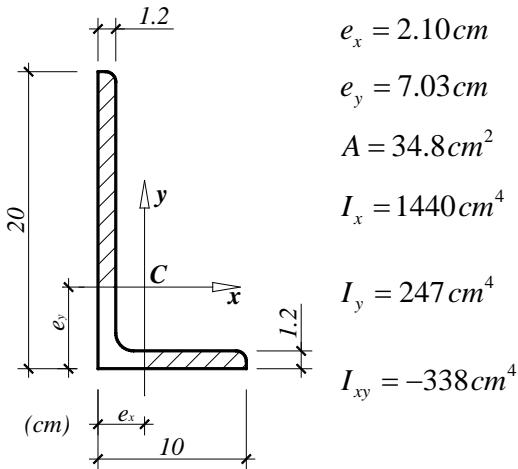


2. Za presjek na slici odrediti glavne centralne momente inercije i nacrtati glavne centralne ose inercije. Rezultat provjeriti Mohr-ovim krugom inercije.



**RJEŠENJE**

**L 100x200x12**



$$e_x = 2.10 \text{ cm}$$

$$e_y = 7.03 \text{ cm}$$

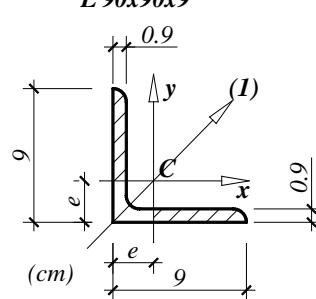
$$A = 34.8 \text{ cm}^2$$

$$I_x = 1440 \text{ cm}^4$$

$$I_y = 247 \text{ cm}^4$$

$$I_{xy} = -338 \text{ cm}^4$$

**L 90x90x9**



$$e_x = 2.54 \text{ cm}$$

$$A = 15.5 \text{ cm}^2$$

$$I_x = I_y = 116 \text{ cm}^4$$

$$I_y = 184 \text{ cm}^4$$

$$|I_{xy}| = I_1 - I_x = 184 - 116 = 68 \text{ cm}^4$$

$$I_{xy} = -68 \text{ cm}^4$$

## 1. Težiste

$$C_1(x_{C_1}, y_{C_1}) = (10 - 2.10, 7.03) = (7.9, 7.03) \text{ cm} \quad A_1 = 34.8 \text{ cm}^2$$

$$C_2(x_{C_2}, y_{C_2}) = (10 + 2.54, 20 - 2.54) = (12.54, 17.46) \text{ cm} \quad A_2 = 15.5$$

$$x_c = \frac{x_{c_1} A_1 + x_{c_2} A_2}{A_1 + A_2} = \frac{7.9 \cdot 34.8 + 12.54 \cdot 15.5}{34.8 + 15.5} = 9.33 \text{ cm}$$

$$y_c = \frac{y_{c_1} A_1 + y_{c_2} A_2}{A_1 + A_2} = \frac{7.03 \cdot 34.8 + 17.46 \cdot 15.5}{34.8 + 15.5} = 10.24 \text{ cm}$$

$$C(x_c, y_c) = (9.33, 10.24) \text{ cm}$$

## 2. Centralni momenti inercije

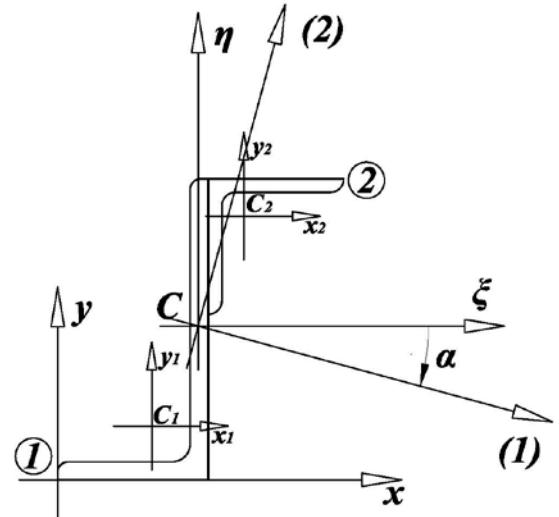
$$I_\xi = I_\xi^{[1]} + I_\xi^{[2]} = 2722.57 \text{ cm}^4$$

$$I_\xi^{[1]} = I_{x_1} + (y_{C_1} - y_c)^2 A_1 = 1440 + (7.03 - 10.24)^2 34.8 = 1798.58 \text{ cm}^4$$

$$I_\xi^{[2]} = I_{x_2} + (y_{C_2} - y_c)^2 A_2 = 116 + (17.46 - 10.24)^2 15.5 = 923.99 \text{ cm}^4$$

$$I_\eta = I_\eta^{[1]} + I_\eta^{[2]} = 593.87 \text{ cm}^4$$

$$I_\eta^{[1]} = I_{y_1} + (x_{C_1} - x_c)^2 A_1 = 247 + (7.9 - 9.33)^2 34.8 = 318.16 \text{ cm}^4$$



$$I_{\eta}^{[2]} = I_{y_2} + (x_{c2} - x_c)^2 A_2 = 116 + (12.54 - 9.33)^2 15.5 = 275.71 \text{ cm}^4$$

$$I_{\xi\eta} = I_{\xi\eta}^{[1]} + I_{\xi\eta}^{[2]} = 924.97 \text{ cm}^4$$

$$I_{\xi\eta}^{[1]} = I_{x_1 y_1} + (y_{c1} - y_c)(x_{c1} - x_c) A_1 = +338 + (-1.43) \cdot (-3.21) \cdot 34.8 = 497.74 \text{ cm}^4$$

$$I_{\xi\eta}^{[2]} = I_{x_2 y_2} + (y_{c2} - y_c)(x_{c2} - x_c) A_1 = 68 + 3.21 \cdot 7.22 \cdot 15.5 = 427.23 \text{ cm}^4$$

### 3. Glavni centralni momenti inercije

$$\begin{aligned} I_{1/2} &= \frac{1}{2}(I_{\xi} + I_{\eta}) \pm \frac{1}{2}\sqrt{(I_{\xi} - I_{\eta})^2 + 4I_{\xi\eta}^2} = \\ &= \frac{1}{2}(2722.57 + 593.87) \pm \frac{1}{2}\sqrt{(2722.57 - 593.87)^2 + 4(924.97)^2} = \\ &= 1658.22 \pm 1410.11 \end{aligned}$$

$$I_1 = 3068.33 \text{ cm}^4$$

$$I_2 = 248.11 \text{ cm}^4 > 0$$

$$\text{Kontrola } I_{\xi} + I_{\eta} = I_1 + I_2$$

$$\tan 2\alpha = \frac{-2I_{\xi\eta}}{I_{\xi} - I_{\eta}} = \frac{-2 \cdot 924.97}{2722.57 - 593.87} = -0.869$$

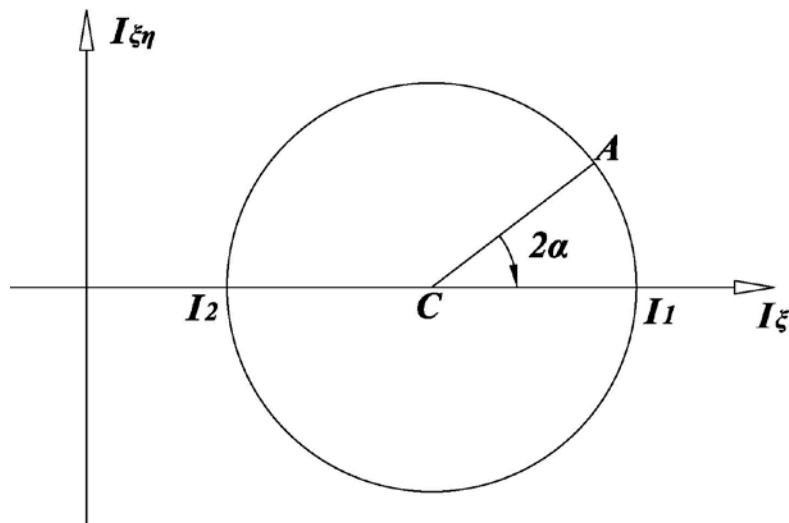
$$(I_{\xi} - I_{\eta}) > 0 \quad \alpha = \frac{1}{2} \arctan(-0.869) = -20.5^\circ$$

### 4. Mohr-ov krug inercije

$$C \left[ \frac{1}{2}(I_{\xi} + I_{\eta}); 0 \right] = \left[ \frac{1}{2}(2722.57 + 593.87); 0 \right] = [1658.22; 0] \text{ cm}^4$$

$$A \left[ I_{\xi}; I_{\xi\eta} \right] = [2722.57; 924.97] \text{ cm}^4$$

Usvaja se razmjera  $1000 \text{ cm}^4 = 1 \text{ cm}$



# VJEŽBA BR. 5

## ANALIZA NAPONA

1. Stanje napona u tački opterećenog tijela zadato je tenzorom napona  $[S]$ .

- Pokazati da je stanje napona linijsko;
- Odrediti pravac sa kojim su paralelni vektori svih totalnih napona;
- Napisati tenzor napona u sistemu glavnih osa.

$$[S] = \begin{bmatrix} 3 & 1 & 2 \\ 1 & 1/3 & 2/3 \\ 2 & 2/3 & 4/3 \end{bmatrix} \quad (\text{Pa})$$

**RJEŠENJE**

a. Linijsko stanje napona:  $\frac{\vec{p}_x}{\vec{p}_z} = \lambda \frac{\vec{p}_n}{\vec{p}_n}$  ili  $\det[S] = 0$

$$\left[ S \right] = \begin{bmatrix} 3 & 1 & 2 \\ 1 & 1/3 & 2/3 \\ 2 & 2/3 & 4/3 \end{bmatrix} \rightarrow \left. \begin{array}{l} \vec{p}_x = 3\vec{i} + \vec{j} + 2\vec{k} \\ \vec{p}_y = \vec{i} + \frac{1}{3}\vec{j} + \frac{2}{3}\vec{k} \\ \vec{p}_z = 2\vec{i} + \frac{2}{3}\vec{j} + \frac{4}{3}\vec{k} \end{array} \right\} \rightarrow \left. \begin{array}{l} \vec{p}_x = 3\vec{p}_y \\ \vec{p}_z = 2\vec{p}_y \end{array} \right\} \rightarrow \text{linijsko stanje napona, ili}$$

$$\det[S] = \begin{vmatrix} 3 & 1 & 2 \\ 1 & 1/3 & 2/3 \\ 2 & 2/3 & 4/3 \end{vmatrix} = 3 \begin{vmatrix} 1/3 & 2/3 \\ 2/3 & 4/3 \end{vmatrix} - 1 \begin{vmatrix} 1 & 2/3 \\ 2 & 4/3 \end{vmatrix} + 2 \begin{vmatrix} 1 & 1/3 \\ 2 & 2/3 \end{vmatrix} = \left. \begin{array}{l} = 3 \left( \frac{4}{9} - \frac{4}{9} \right) - 1 \left( \frac{4}{3} - \frac{4}{3} \right) + 2 \left( \frac{2}{3} - \frac{2}{3} \right) = 0 \end{array} \right\} \rightarrow \det[S] \rightarrow \text{linijsko stanje napona}$$

$$\left. \begin{array}{l} \begin{vmatrix} 1/3 & 2/3 \\ 2/3 & 4/3 \end{vmatrix} = 0 \quad \begin{vmatrix} 3 & 1 \\ 1 & 1/3 \end{vmatrix} = 0 \quad \begin{vmatrix} 3 & 1 \\ 2 & 2/3 \end{vmatrix} = 0 \\ \begin{vmatrix} 1 & 2/3 \\ 2 & 4/3 \end{vmatrix} = 0 \quad \begin{vmatrix} 3 & 2 \\ 1 & 2/3 \end{vmatrix} = 0 \quad \begin{vmatrix} 3 & 2 \\ 2 & 4/3 \end{vmatrix} = 0 \\ \begin{vmatrix} 1 & 1/3 \\ 2 & 2/3 \end{vmatrix} = 0 \quad \begin{vmatrix} 1 & 2 \\ 1/3 & 2/3 \end{vmatrix} = 0 \quad \begin{vmatrix} 1 & 2 \\ 2/3 & 4/3 \end{vmatrix} = 0 \end{array} \right\} \rightarrow \text{Minori} = 0 \right\} \rightarrow \text{linijsko stanje napona}$$

b. Linijsko stanje napona  $\rightarrow$  vektori totalnog napona za bilo koju presječnu ravan u okolini tačke imaju isti pravac. Neka je to pravac  $\vec{n} = \cos \alpha \vec{i} + \cos \beta \vec{j} + \cos \gamma \vec{k}$

Posmatrajmo npr. vektor totalnog napona za ravan sa normalom  $x$

$$\vec{p}_x = \sigma_x \vec{i} + \tau_{xy} \vec{j} + \tau_{xz} \vec{k} = 3\vec{i} + \vec{j} + 2\vec{k}$$

$$\cos \alpha = \frac{\sigma_x}{|\vec{p}_x|} = \frac{3}{\sqrt{3^2 + 1^2 + 2^2}} = \frac{3}{\sqrt{14}}$$

$$\cos \beta = \frac{\tau_{xy}}{\|\vec{p}_x\|} = \frac{1}{\sqrt{3^2 + 1^2 + 2^2}} = \frac{1}{\sqrt{14}}$$

$$\cos \gamma = \frac{\tau_{xz}}{\|\vec{p}_x\|} = \frac{2}{\sqrt{3^2 + 1^2 + 2^2}} = \frac{2}{\sqrt{14}}$$

$$\boxed{\vec{n} = \frac{3}{\sqrt{14}}\vec{i} + \frac{1}{\sqrt{14}}\vec{j} + \frac{2}{\sqrt{14}}\vec{k}} \quad - \text{ pravac sa kojim su paralelni vektori svih totalnih napona}$$

$$\vec{p}_n = p_{nx}\vec{i} + p_{ny}\vec{j} + p_{nz}\vec{k}$$

$$p_{nx} = \vec{p}_n \cdot \vec{i} = \vec{p}_x \cdot \vec{n} = (3\vec{i} + \vec{j} + 2\vec{k}) \cdot \left( \frac{3}{\sqrt{14}}\vec{i} + \frac{1}{\sqrt{14}}\vec{j} + \frac{2}{\sqrt{14}}\vec{k} \right) = 3 \frac{3}{\sqrt{14}} + \frac{1}{\sqrt{14}} + 2 \frac{2}{\sqrt{14}} = 3.741$$

$$p_{ny} = \vec{p}_n \cdot \vec{j} = \vec{p}_y \cdot \vec{n} = (\vec{i} + \frac{1}{3}\vec{j} + \frac{2}{3}\vec{k}) \cdot \left( \frac{3}{\sqrt{14}}\vec{i} + \frac{1}{\sqrt{14}}\vec{j} + \frac{2}{\sqrt{14}}\vec{k} \right) = \frac{3}{\sqrt{14}} + \frac{1}{3} \frac{1}{\sqrt{14}} + \frac{2}{3} \frac{2}{\sqrt{14}} = 1.247$$

$$p_{nz} = \vec{p}_n \cdot \vec{k} = \vec{p}_z \cdot \vec{n} = (2\vec{i} + \frac{2}{3}\vec{j} + \frac{4}{3}\vec{k}) \cdot \left( \frac{3}{\sqrt{14}}\vec{i} + \frac{1}{\sqrt{14}}\vec{j} + \frac{2}{\sqrt{14}}\vec{k} \right) = 2 \frac{3}{\sqrt{14}} + \frac{2}{3} \frac{1}{\sqrt{14}} + \frac{4}{3} \frac{2}{\sqrt{14}} = 1.247$$

$$\vec{p}_n = 3.741\vec{i} + 1.247\vec{j} + 2.494\vec{k}$$

$$|\vec{p}_n| = \sqrt{3.741^2 + 1.247^2 + 2.494^2} = 4.666 \text{ Pa}$$

$$\sigma_n = \vec{p}_n \cdot \vec{n} = (3.741\vec{i} + 1.247\vec{j} + 2.494\vec{k}) \cdot \left( \frac{3}{\sqrt{14}}\vec{i} + \frac{1}{\sqrt{14}}\vec{j} + \frac{2}{\sqrt{14}}\vec{k} \right) = 3.741 \frac{3}{\sqrt{14}} + 1.247 \frac{1}{\sqrt{14}} + 2.494 \frac{2}{\sqrt{14}} = 4.666 \text{ Pa}$$

c. Tenzor napona u sistemu glavnih osa

$$[S] = \begin{bmatrix} 4.666 & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & 0 \end{bmatrix} (\text{Pa})$$

2. Stanje napona u tački opterećenog tijela zadato je tenzorom napona  $[S]$ .

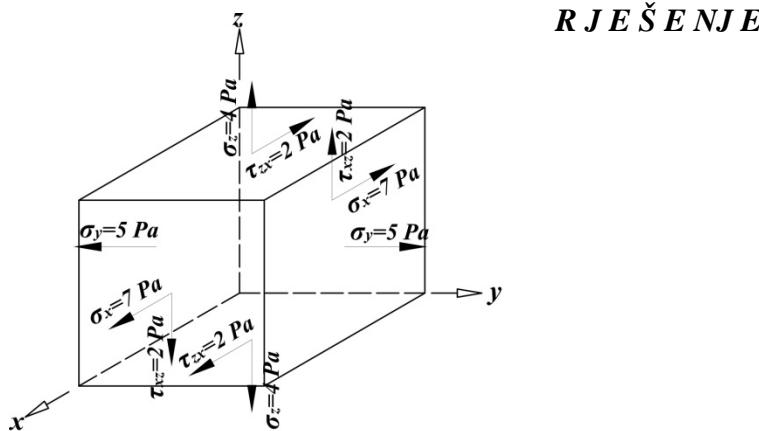
a. Prikazati okolinu tačke;

b. Odrediti totalni napon za ravan sa normalom  $\vec{n} = \left\{ \frac{2}{3}; -\frac{2}{3}; \frac{1}{3} \right\}$ ;

c. Za totalni napon iz tačke c. odrediti normalni napon i totalni smičući napon.

$$[S] = \begin{bmatrix} 7 & 0 & -2 \\ 0 & 5 & 0 \\ -2 & 0 & 4 \end{bmatrix} (\text{Pa})$$

a. Okolina tačke



**RJEŠENJE**

b. Totalni napon za ravan sa normalom  $\vec{n}$

$$[S] = \begin{bmatrix} 7 & 0 & -2 \\ 0 & 5 & 0 \\ -2 & 0 & 4 \end{bmatrix} \rightarrow \left. \begin{array}{l} \vec{p}_x = 7\vec{i} - 2\vec{k} \\ \vec{p}_y = 5\vec{j} \\ \vec{p}_z = -2\vec{i} + 4\vec{k} \end{array} \right\} \text{ - Vektori totalnog napona za 3 međusobno upravne ravni}$$

$$\vec{n} = \left\{ \frac{2}{3}; -\frac{2}{3}; \frac{1}{3} \right\} = \frac{2}{3}\vec{i} - \frac{2}{3}\vec{j} + \frac{1}{3}\vec{k}$$

$$|\vec{n}| = \sqrt{\left(\frac{2}{3}\right)^2 + \left(-\frac{2}{3}\right)^2 + \left(\frac{1}{3}\right)^2} = 1 \text{ u slučaju da nijesmo dobili 1} \rightarrow \vec{n} = \cos \alpha \vec{i} + \cos \beta \vec{j} + \cos \gamma \vec{k}$$

$$\cos \alpha = \frac{n_x}{|\vec{n}|} = \frac{2/3}{1}; \cos \beta = \frac{n_y}{|\vec{n}|} = \frac{-2/3}{1}; \cos \gamma = \frac{n_z}{|\vec{n}|} = \frac{1/3}{1}.$$

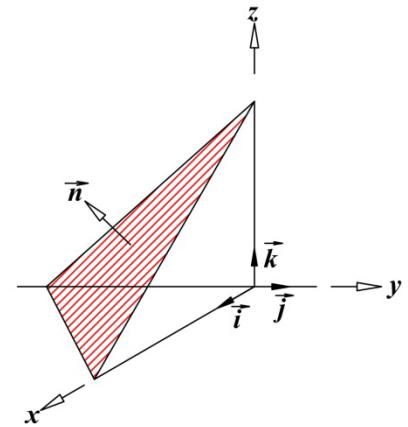
$$\vec{p}_n = p_{nx}\vec{i} + p_{ny}\vec{j} + p_{nz}\vec{k}$$

$$p_{nx} = \vec{p}_n \cdot \vec{i} = \vec{p}_x \cdot \vec{n} = (7\vec{i} - 2\vec{k}) \cdot \left( \frac{2}{3}\vec{i} - \frac{2}{3}\vec{j} + \frac{1}{3}\vec{k} \right) = 7 \frac{2}{3} - 2 \frac{1}{3} = 4$$

$$p_{ny} = \vec{p}_n \cdot \vec{j} = \vec{p}_y \cdot \vec{n} = (5\vec{j}) \cdot \left( \frac{2}{3}\vec{i} - \frac{2}{3}\vec{j} + \frac{1}{3}\vec{k} \right) = 5 \left( -\frac{2}{3} \right) = -10/3$$

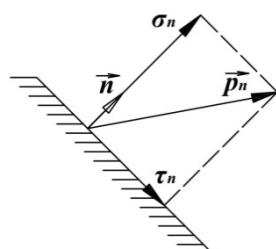
$$p_{nz} = \vec{p}_n \cdot \vec{k} = \vec{p}_z \cdot \vec{n} = (-2\vec{i} + 4\vec{k}) \cdot \left( \frac{2}{3}\vec{i} - \frac{2}{3}\vec{j} + \frac{1}{3}\vec{k} \right) = -2 \frac{2}{3} + 4 \frac{1}{3} = 0$$

$$\boxed{\vec{p}_n = 4\vec{i} - \frac{10}{3}\vec{j}} \quad \text{- Vektor totalnog napona}$$



$$|\vec{p}_n| = \sqrt{p_{nx}^2 + p_{ny}^2 + p_{nz}^2} = \sqrt{4^2 + \left(-\frac{10}{3}\right)^2} = 5.207 \text{ Pa} \quad \text{- Intenzitet totalnog napona}$$

$$\left. \begin{array}{l} \cos \alpha_{\vec{p}_n} = \frac{p_{nx}}{|\vec{p}_n|} = \frac{4}{5.207} = 0.768 \\ \cos \beta_{\vec{p}_n} = \frac{p_{ny}}{|\vec{p}_n|} = \frac{-10/3}{5.207} = -0.640 \\ \cos \gamma_{\vec{p}_n} = \frac{p_{nz}}{|\vec{p}_n|} = \frac{0}{5.207} = 0 \end{array} \right\} \text{ - Pravac totalnog napona } \vec{p}_n$$



c. Normalni napon i totalni smičući napon

$$\sigma_n = \vec{p}_n \cdot \vec{n} = \left( 4\vec{i} - \frac{10}{3}\vec{j} \right) \cdot \left( \frac{2}{3}\vec{i} - \frac{2}{3}\vec{j} + \frac{1}{3}\vec{k} \right) = 4 \frac{2}{3} - \frac{10}{3} \left( -\frac{2}{3} \right) = 4.889 \text{ Pa} \quad \text{- Normalni napon}$$

$$\tau_n = \sqrt{|\vec{p}_n|^2 - \sigma_n^2} = \sqrt{5.207^2 - 4.889^2} = 1.792 \text{ Pa} \quad \text{- Totalni smičući napon}$$

3. Stanje napona u tački opterećenog tijela zadato je tenzorom napona  $[S]$ .

- Pokazati da je stanje napona ravno;
- Prikazati okolinu tačke;
- Sračunati glavne napone i pravce glavnih napona i prikazati okolinu tačke;
- Sračunati maksimalni smičući napon i odgovarajući normalni napon. Prikazati okolinu tačke;
- Odrediti komponentalne napone za ravni čije normale leže u ravni  $yz$  i sa osom  $y$  grede uglove  $\varphi_1=30^\circ$  odnosno  $\varphi_2=-45^\circ$ ;
- Nacrtati Mohr-ov krug napona i provjeriti rješenja iz prethodnih tačaka zadatka.

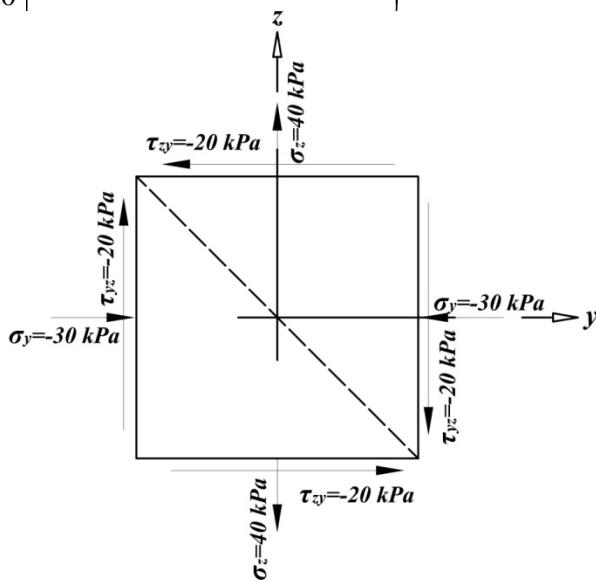
$$[S] = \begin{bmatrix} 0 & 0 & 0 \\ 0 & -30 & -20 \\ 0 & -20 & 40 \end{bmatrix} \text{ (kPa)}$$

### RJEŠENJE

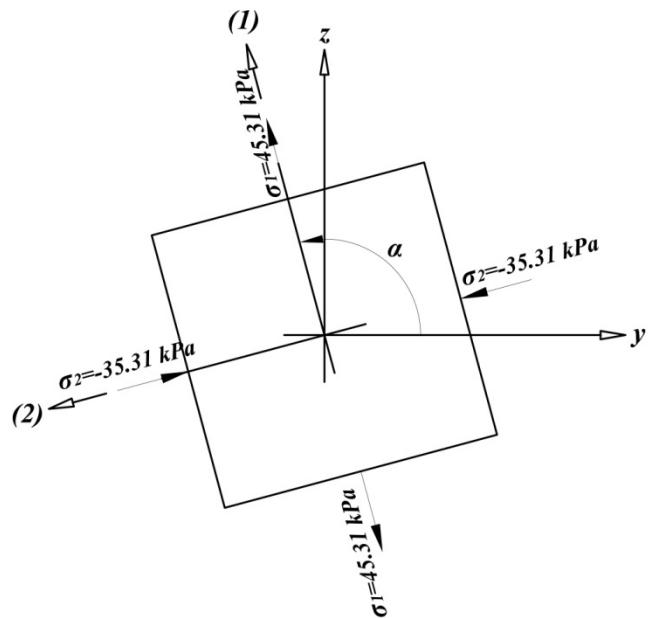
a. Ravno stanje napona:  $\det[S] = 0$

Makar 1 minor  $\neq 0$

$$\left. \begin{array}{l} \det[S] = \begin{vmatrix} 0 & 0 & 0 \\ 0 & -30 & -20 \\ 0 & -20 & 40 \end{vmatrix} = 0 \\ \det[S_1] = \begin{vmatrix} -30 & -20 \\ -20 & 40 \end{vmatrix} = -1200 - 400 = -1600 \neq 0 \end{array} \right\} \rightarrow \text{Stanje napona je ravno}$$



b. Okolina tačke



c. Glavni normalni naponi

$$\begin{aligned} \sigma_{1/2} &= \frac{1}{2}(\sigma_y + \sigma_z) \pm \frac{1}{2}\sqrt{(\sigma_y - \sigma_z)^2 + 4\tau_{yz}^2} = \\ &= \frac{1}{2}(-30 + 40) \pm \frac{1}{2}\sqrt{(-30 - 40)^2 + 4(-20)^2} = \\ &= 5 \pm 40.31 \end{aligned}$$

$$\left. \begin{array}{l} \sigma_1 = 45.31 \text{ kPa} \\ \sigma_2 = -35.31 \text{ kPa} \end{array} \right\} \text{ - Glavni normalni naponi}$$

Kontrola  $\sigma_1 + \sigma_2 = \sigma_y + \sigma_z$

$$\operatorname{tg} 2\alpha = \frac{2\tau_{yz}}{\sigma_y - \sigma_z} = \frac{2 \cdot (-20)}{-30 - 40} = \frac{4}{7}$$

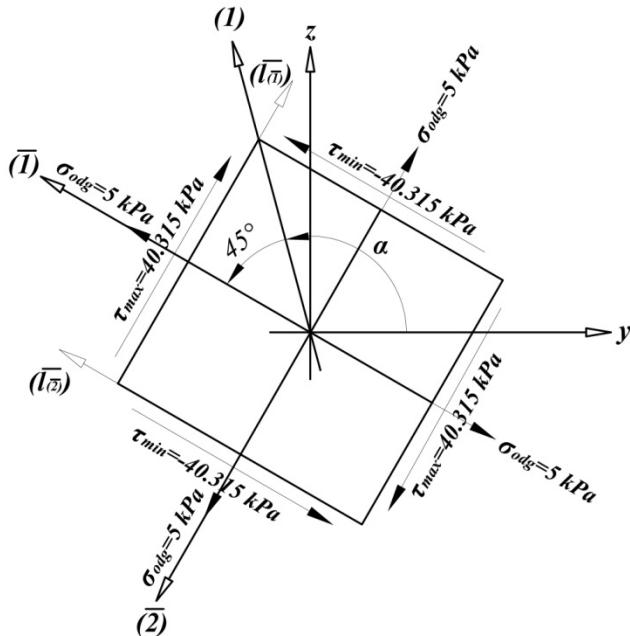
$$(\sigma_y - \sigma_z) = -70 < 0 \rightarrow \alpha = \frac{1}{2} \operatorname{arctg} \left( \frac{4}{7} \right) + 90^\circ = 104.87^\circ \text{ - Ugao koji definiše glavni pravac (1)}$$

d. Maksimalni smičući napon i odgovarajući normalni napon

$$\tau_{max} = \frac{1}{2}(\sigma_1 - \sigma_2) = \frac{1}{2}(45.31 - (-35.31)) = 40.31 \text{ kPa} \text{ - Maksimalni smičući napon}$$

$$\sigma_s = \sigma_{odg} = \frac{1}{2}(\sigma_1 + \sigma_2) = \frac{1}{2}(45.31 + (-35.31)) = 5 \text{ kPa} \text{ - Odgovarajući normalni napon}$$

$\bar{\alpha} = \alpha + 45^\circ$  - Ugao koji definiše normalu za ravan u kojoj djeluje maksimalni smičući napon

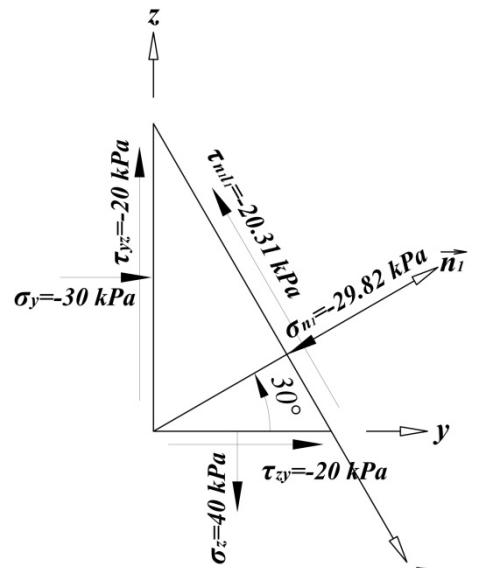


e. Komponentalni naponi za ravni  $\varphi_1=30^\circ$  odnosno  $\varphi_2=-45^\circ$

$$\vec{n}_1 \leftrightarrow \varphi_1 = 30^\circ$$

$$\begin{aligned} \sigma_{n_1} &= \frac{1}{2}(\sigma_y + \sigma_z) + \frac{1}{2}(\sigma_y - \sigma_z)\cos 2\varphi_1 + \tau_{yz} \sin 2\varphi_1 = \\ &= \frac{1}{2}(-30 + 40) + \frac{1}{2}(-30 - 40)\cos(2 \cdot 30^\circ) + (-20)\sin(2 \cdot 30^\circ) = \\ &= -29.82 \text{ kPa} \end{aligned}$$

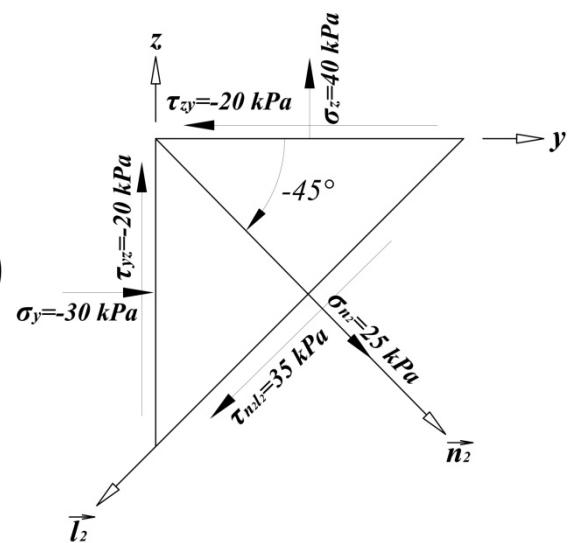
$$\begin{aligned} \tau_{n_1 l_1} &= \frac{1}{2}(\sigma_y - \sigma_z)\sin 2\varphi_1 - \tau_{yz} \cos 2\varphi_1 = \\ &= \frac{1}{2}(-30 - 40)\sin(2 \cdot 30^\circ) - (-20)\cos(2 \cdot 30^\circ) = -20.31 \text{ kPa} \end{aligned}$$



$$\vec{n}_2 \leftrightarrow \varphi_2 = -45^\circ$$

$$\begin{aligned} \sigma_{n_2} &= \frac{1}{2}(\sigma_y + \sigma_z) + \frac{1}{2}(\sigma_y - \sigma_z)\cos 2\varphi_2 + \tau_{yz} \sin 2\varphi_2 = \\ &= \frac{1}{2}(-30 + 40) + \frac{1}{2}(-30 - 40)\cos(2 \cdot (-45^\circ)) + (-20)\sin(2 \cdot (-45^\circ)) \\ &= 25 \text{ kPa} \end{aligned}$$

$$\begin{aligned} \tau_{n_2 l_2} &= \frac{1}{2}(\sigma_y - \sigma_z)\sin 2\varphi_2 - \tau_{yz} \cos 2\varphi_2 = \\ &= \frac{1}{2}(-30 - 40)\sin(2 \cdot (-45^\circ)) - (-20)\cos(2 \cdot (-45^\circ)) = 35 \text{ kPa} \end{aligned}$$

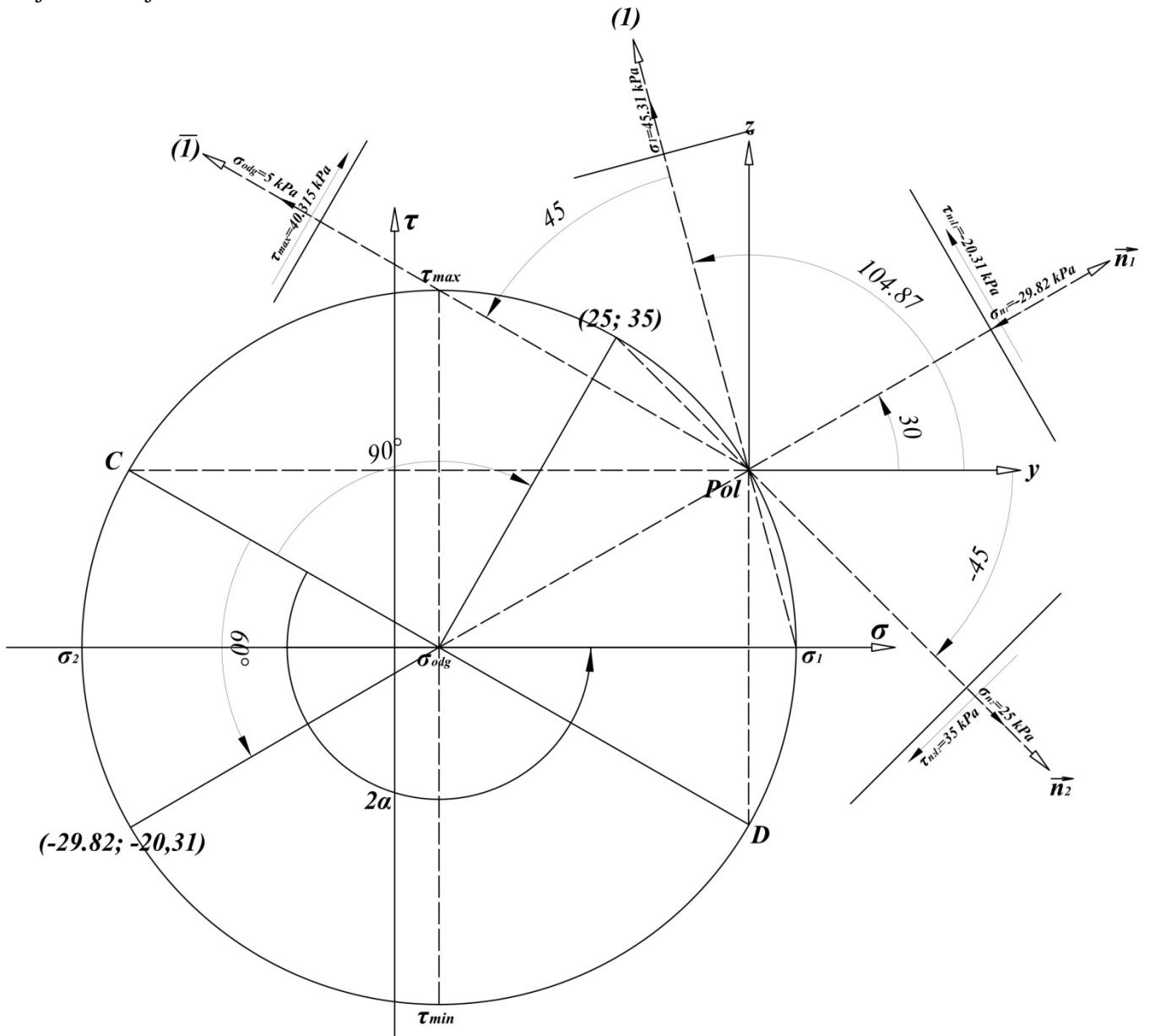


f. Mohr-ov krug napona

$$C[\sigma_y; -\tau_{yz}] = [-30; 20] \text{ kPa}$$

$$D[\sigma_z; \tau_{yz}] = [40; -20] \text{ kPa}$$

Usvaja se razmjer  $10 \text{ kPa} = 1 \text{ cm}$



# VJEŽBA BR. 6

## ANALIZA NAPONA

1. Stanje napona u tački opterećenog tijela zadato je tenzorom napona  $[S]$ . Odrediti:

a. Vektor totalnog napona za ravan sa normalom  $\vec{n} = \left\{ \frac{1}{3}; \frac{2}{3}; n_z < 0 \right\}$ ;

$$[S] = \begin{bmatrix} -20 & 0 & -6 \\ 0 & 6 & 0 \\ -6 & 0 & 6 \end{bmatrix} \text{ (MPa)}$$

- b. Normalni i smičući napon u toj ravni;  
 c. Glavne napone i glavne pravce;  
 d. Maksimalni smičući napon i odgovarajući normalni napon;  
 e. Sferni i devijatorski dio tenzora napona.

***RJEŠENJE***

- a. Vektor totalnog napona za ravan sa normalom  $\vec{n}$

$$|\vec{n}| = \sqrt{\left(\frac{1}{3}\right)^2 + \left(\frac{2}{3}\right)^2 + (n_z)^2} = 1 \rightarrow \left(\frac{1}{3}\right)^2 + \left(\frac{2}{3}\right)^2 + (n_z)^2 = 1 \rightarrow n_z^2 = \frac{4}{9} \rightarrow n_z = \pm \frac{2}{3} \quad \boxed{n_z = -\frac{2}{3}}$$

$$\vec{n} = \left\{ \frac{1}{3}; \frac{2}{3}; -\frac{2}{3} \right\} = \frac{1}{3}\vec{i} + \frac{2}{3}\vec{j} - \frac{2}{3}\vec{k}$$

$$[S] = \begin{bmatrix} -20 & 0 & -6 \\ 0 & 6 & 0 \\ -6 & 0 & 6 \end{bmatrix} \text{ (MPa)} \rightarrow \begin{aligned} \vec{p}_x &= -20\vec{i} - 6\vec{k} \\ \vec{p}_y &= 6\vec{j} \\ \vec{p}_z &= -6\vec{i} + 6\vec{k} \end{aligned}$$

$$\vec{p}_n = p_{nx}\vec{i} + p_{ny}\vec{j} + p_{nz}\vec{k}$$

$$p_{nx} = \vec{p}_n \cdot \vec{i} = \vec{p}_x \cdot \vec{n} = (-20\vec{i} - 6\vec{k}) \cdot \left( \frac{1}{3}\vec{i} + \frac{2}{3}\vec{j} - \frac{2}{3}\vec{k} \right) = -20 \frac{1}{3} - 6 \left( -\frac{2}{3} \right) = -\frac{8}{3}$$

$$p_{ny} = \vec{p}_n \cdot \vec{j} = \vec{p}_y \cdot \vec{n} = (6\vec{j}) \cdot \left( \frac{1}{3}\vec{i} + \frac{2}{3}\vec{j} - \frac{2}{3}\vec{k} \right) = 6 \frac{2}{3} = 4$$

$$p_{nz} = \vec{p}_n \cdot \vec{k} = \vec{p}_z \cdot \vec{n} = (-6\vec{i} + 6\vec{k}) \cdot \left( \frac{1}{3}\vec{i} + \frac{2}{3}\vec{j} - \frac{2}{3}\vec{k} \right) = -6 \frac{1}{3} + 6 \left( -\frac{2}{3} \right) = -6$$

$$\vec{p}_n = -\frac{8}{3}\vec{i} + 4\vec{j} - 6\vec{k}$$

- b. Normalni i smičući napon

$$|\vec{p}_n| = \sqrt{\left(\frac{8}{3}\right)^2 + 4^2 + (-6)^2} = 7.688 \text{ MPa}$$

$$\sigma_n = \vec{p}_n \cdot \vec{n} = \left( -\frac{8}{3}\vec{i} + 4\vec{j} - 6\vec{k} \right) \cdot \left( \frac{1}{3}\vec{i} + \frac{2}{3}\vec{j} - \frac{2}{3}\vec{k} \right) = -\frac{8}{3} \frac{1}{3} + 4 \frac{2}{3} - 6 \left( -\frac{2}{3} \right) = 5.77 \text{ MPa}$$

$$\tau_n = \sqrt{|\vec{p}_n|^2 - \sigma_n^2} = \sqrt{7.688^2 - 5.77^2} = 5.08 \text{ MPa}$$

c. Glavni naponi i pravci glavnih napona (**I način**)

Intenzitet glavnih normalnih napona

$$\sigma_i^3 - I_\sigma \sigma_i^2 + II_\sigma \sigma_i - III_\sigma = 0$$

$$I_\sigma = \sigma_x + \sigma_y + \sigma_z = -20 + 6 + 6 = -8$$

$$II_\sigma = \begin{vmatrix} \sigma_y & \tau_{yz} \\ \tau_{zy} & \sigma_z \end{vmatrix} + \begin{vmatrix} \sigma_x & \tau_{xz} \\ \tau_{zx} & \sigma_z \end{vmatrix} + \begin{vmatrix} \sigma_x & \tau_{xy} \\ \tau_{yx} & \sigma_y \end{vmatrix} =$$

$$= \sigma_x \sigma_y + \sigma_y \sigma_z + \sigma_z \sigma_x - \tau_{xy}^2 - \tau_{yz}^2 - \tau_{zx}^2 = (-20) \cdot 6 + 6 \cdot 6 + 6 \cdot (-20) - (-6)^2 = -240$$

$$III_\sigma = \begin{vmatrix} \sigma_x & \tau_{xy} & \tau_{xz} \\ \tau_{yx} & \sigma_y & \tau_{yz} \\ \tau_{zx} & \tau_{zy} & \sigma_z \end{vmatrix} = \begin{vmatrix} -20 & 0 & -6 \\ 0 & 6 & 0 \\ -6 & 0 & 6 \end{vmatrix} = -20 \cdot (6 \cdot 6) - 6 \cdot (36) = -936$$

$$\sigma_i^3 + 8\sigma_i^2 - 240\sigma_i + 936 = 0 \quad \boxed{\sigma_a = 6 \text{ MPa}} \text{ jedno od 3 rješenja kubne jednačine}$$

$$(\sigma_i^3 + 8\sigma_i^2 - 240\sigma_i + 936) : (\sigma_i - 6) = \sigma_i^2 + 14\sigma_i - 156$$

$$\underline{\sigma_i^3 - 6\sigma_i^2}$$

$$\underline{14\sigma_i^2 - 240\sigma_i}$$

$$\underline{14\sigma_i^2 - 84\sigma_i}$$

$$\underline{-156\sigma_i + 936}$$

$$\underline{-156\sigma_i + 936}$$

$$0.0$$

$$\sigma_i^2 + 14\sigma_i - 156 = 0$$

$$\sigma_{b/c} = \frac{-14 \pm \sqrt{196 + 4 \cdot 156}}{2} - 7 \pm 14.32$$

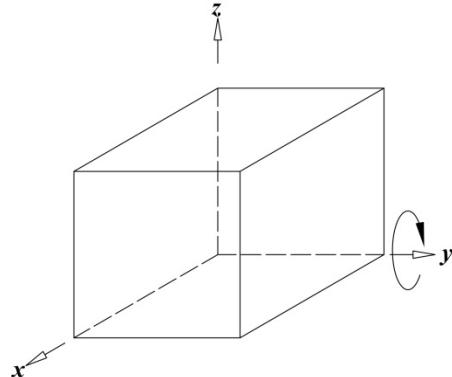
$$\sigma_b = 7.32 \text{ MPa}$$

$$\sigma_c = -21.32 \text{ Pa}$$

$$\sigma_1 = \sigma_b = 7.32 \text{ MPa}$$

$$\sigma_2 = \sigma_a = 6 \text{ MPa}$$

$$\sigma_3 = \sigma_c = -21.32 \text{ MPa}$$



Pravci glavnih normalnih napona  $\vec{n}_i = \cos \alpha_i \vec{i} + \cos \beta_i \vec{j} + \cos \gamma_i \vec{k} \quad i = 1, 2, 3$

$$\cos \alpha_i = \pm \frac{A_i}{D_i}; \quad \cos \beta_i = \pm \frac{B_i}{D_i}; \quad \cos \gamma_i = \pm \frac{C_i}{D_i}$$

$$A_i = \begin{vmatrix} (\sigma_y - \sigma_i) & \tau_{yz} \\ \tau_{zy} & (\sigma_z - \sigma_i) \end{vmatrix}; \quad B_i = -\begin{vmatrix} \tau_{yx} & \tau_{yz} \\ \tau_{zx} & (\sigma_z - \sigma_i) \end{vmatrix}; \quad C_i = \begin{vmatrix} \tau_{yx} & (\sigma_y - \sigma_i) \\ \tau_{zx} & \tau_{zy} \end{vmatrix}; \quad D_i = \sqrt{A_i^2 + B_i^2 + C_i^2}$$

Pravac (1)

$$A_1 = \begin{vmatrix} (\sigma_y - \sigma_1) & \tau_{yz} \\ \tau_{zy} & (\sigma_z - \sigma_1) \end{vmatrix} = \begin{vmatrix} (6 - 7.32) & 0 \\ 0 & (6 - 7.32) \end{vmatrix} = 1.7424$$

$$B_1 = -\begin{vmatrix} \tau_{yx} & \tau_{yz} \\ \tau_{zx} & (\sigma_z - \sigma_1) \end{vmatrix} = -\begin{vmatrix} 0 & 0 \\ -6 & (6 - 7.32) \end{vmatrix} = 0$$

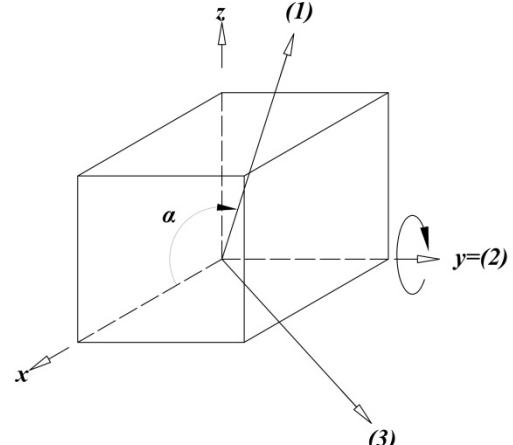
$$C_1 = \begin{vmatrix} \tau_{yx} & (\sigma_y - \sigma_1) \\ \tau_{zx} & \tau_{zy} \end{vmatrix} = \begin{vmatrix} 0 & (6 - 7.32) \\ -6 & 0 \end{vmatrix} = -7.92$$

$$D_1 = \sqrt{A_1^2 + B_1^2 + C_1^2} = \sqrt{1.7424^2 + 0^2 + (-7.92)^2} = 8.11$$

$$\left. \begin{array}{l} \cos \alpha_1 = \pm \frac{A_1}{D_1} = \pm \frac{1.7421}{8.11} = \pm 0.215 \\ \cos \beta_1 = \pm \frac{B_1}{D_1} = \pm \frac{0}{8.11} = 0 \\ \cos \gamma_1 = \pm \frac{C_1}{D_1} = \pm \frac{-7.92}{8.11} = \mp 0.977 \end{array} \right\} \rightarrow \boxed{\vec{n}_1 = \pm 0.215\vec{i} \mp 0.977\vec{k}}$$

Pravac (2)

$$\sigma_2 = \sigma_y = 6 \text{ MPa} \quad y \text{ osa je glavna osa} \rightarrow \boxed{\vec{n}_2 = \pm \vec{j}}$$



Pravac (3)

$$A_3 = \begin{vmatrix} (\sigma_y - \sigma_3) & \tau_{yz} \\ \tau_{zy} & (\sigma_z - \sigma_3) \end{vmatrix} = \begin{vmatrix} (6 + 21.32) & 0 \\ 0 & (6 + 21.32) \end{vmatrix} = 746.38$$

$$B_3 = -\begin{vmatrix} \tau_{yx} & \tau_{yz} \\ \tau_{zx} & (\sigma_z - \sigma_3) \end{vmatrix} = -\begin{vmatrix} 0 & 0 \\ -6 & (6 + 21.32) \end{vmatrix} = 0$$

$$C_3 = \begin{vmatrix} \tau_{yx} & (\sigma_y - \sigma_3) \\ \tau_{zx} & \tau_{zy} \end{vmatrix} = \begin{vmatrix} 0 & (6 + 21.32) \\ -6 & 0 \end{vmatrix} = 163.92$$

$$D_3 = \sqrt{A_3^2 + B_3^2 + C_3^2} = \sqrt{746.38^2 + 0^2 + 163.92^2} = 764.17$$

$$\left. \begin{array}{l} \cos \alpha_3 = \pm \frac{A_3}{D_3} = \pm \frac{746.38}{764.17} = \pm 0.977 \\ \cos \beta_3 = \pm \frac{B_3}{D_3} = \pm \frac{0}{764.17} = 0 \\ \cos \gamma_3 = \pm \frac{C_3}{D_3} = \pm \frac{163.92}{764.17} = \pm 0.215 \end{array} \right\} \rightarrow \boxed{\vec{n}_3 = \pm 0.977\vec{i} \pm 0.215\vec{k}}$$

c. Glavni naponi i pravci glavnih napona (**II način**)

$$[S] = \begin{bmatrix} -20 & 0 & -6 \\ 0 & 6 & 0 \\ -6 & 0 & 6 \end{bmatrix} \rightarrow \sigma_y = 6 \text{ MPa} \quad \text{je jedan od tri glavna napona, a } y \text{ osa je jedna od tri glavne osa}$$

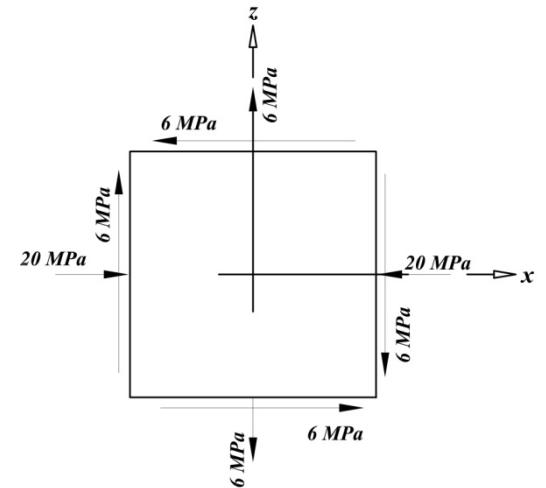
Preostale dvije glavne ose, čemo dobiti rotacijom elementarnog kvadra (beskonačno mala okolina tačke) oko ose  $y$  → Možemo iskoristiti formule za proračun glavnih normalnih naponi i glavnih osa za ravno stanje napona.

$$[S_{xz}] = \begin{bmatrix} \sigma_x & \tau_{xz} \\ \tau_{zx} & \sigma_z \end{bmatrix} = \begin{bmatrix} -20 & -6 \\ -6 & 6 \end{bmatrix}$$

$$\begin{aligned} \sigma_{a/b} &= \frac{1}{2}(\sigma_x + \sigma_z) \pm \frac{1}{2}\sqrt{(\sigma_x - \sigma_z)^2 + 4\tau_{xz}^2} = \\ &= \frac{1}{2}(-20 + 6) \pm \frac{1}{2}\sqrt{(-20 - 6)^2 + 4(-6)^2} = \\ &= -7 \pm 14.31 \end{aligned}$$

$$\left. \begin{array}{l} \sigma_a = 7.32 \text{ MPa} \\ \sigma_b = -21.32 \text{ MPa} \end{array} \right\} \text{ - Preostala dva glavna normalna naponi}$$

$$\text{Kontrola } \sigma_a + \sigma_b = \sigma_x + \sigma_z$$



$$\operatorname{tg} 2\alpha = \frac{2\tau_{xz}}{\sigma_x - \sigma_z} = \frac{2 \cdot (-6)}{-20 - 6} = 0.4615$$

$$(\sigma_x - \sigma_z) < 0 \rightarrow \alpha = \frac{1}{2} \operatorname{arctg}(0.4615) + 90^\circ = 102.38^\circ \text{ - Ugao koji definiše glavni pravac (a)}$$

$\sigma_1 = \sigma_a = 7.32 \text{ MPa}$
$\sigma_2 = \sigma_c = 6 \text{ MPa}$
$\sigma_3 = \sigma_b = -21.32 \text{ MPa}$

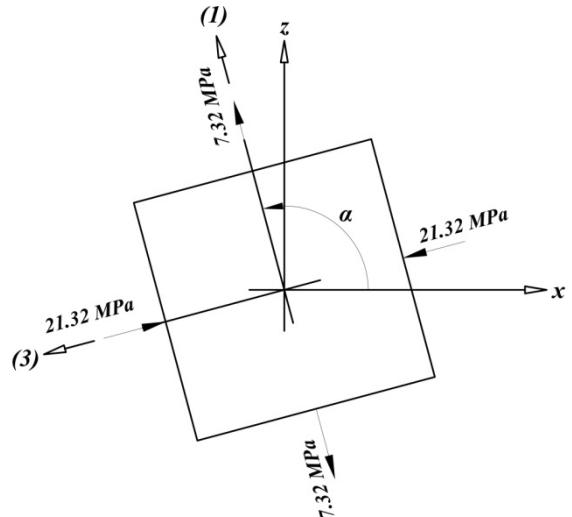
Konačno

Kontrola:

$$\text{Pravac (1)} \begin{cases} \cos \alpha_1 = \cos 102.38 = -0.215 \\ \cos \beta_1 = \cos 90 = 0 \\ \cos \gamma_1 = \cos(102.38 - 90) = 0.977 \end{cases}$$

$$\text{Pravac (2)} \begin{cases} \cos \alpha_2 = \cos 90 = 0 \\ \cos \beta_2 = \cos 0 = 1 \\ \cos \gamma_2 = \cos 90 = 0 \end{cases}$$

$$\text{Pravac (3)} \begin{cases} \cos \alpha_3 = \cos(102.38 + 90) = -0.977 \\ \cos \beta_3 = \cos 90 = 0 \\ \cos \gamma_3 = \cos 102.38 = -0.215 \end{cases}$$

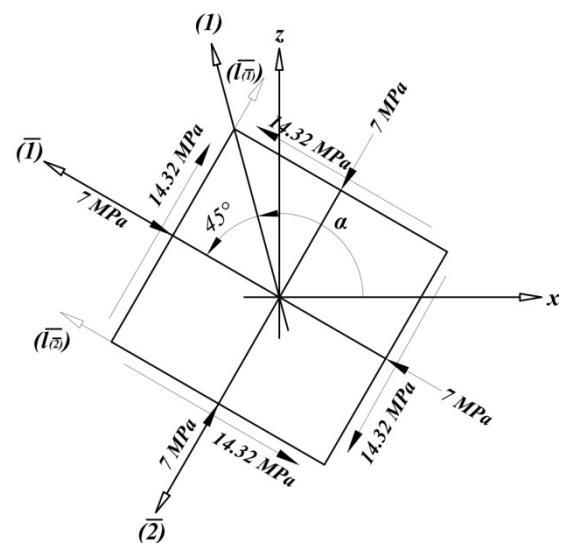


Očigledno je da je II način značajno jednostavniji za primjenu. Naravno, ograničenje je da mora biti poznat jedan od tri glavna pravca i jedan od tri glavna normalna naponi.

d. Maksimalni smičući napon i odgovarajući normalni napon

$$\tau_{max} = \frac{1}{2}(\sigma_1 - \sigma_3) = \frac{1}{2}(7.32 - (-21.32)) = 14.32 \text{ MPa}$$

$$\sigma_s = \sigma_{odg} = \frac{1}{2}(\sigma_1 + \sigma_3) = \frac{1}{2}(7.32 + (-21.32)) = -7 \text{ MPa}$$



e. Sferni i devijatorski dio tenzora napona

$$\sigma_s = \frac{1}{3}(\sigma_x + \sigma_y + \sigma_z) = \frac{1}{3}(-20 + 6 + 6) = -\frac{8}{3} \text{ MPa}$$

$$S = \begin{bmatrix} \sigma_x & \tau_{xy} & \tau_{xz} \\ \tau_{yx} & \sigma_y & \tau_{yz} \\ \tau_{zx} & \tau_{zy} & \sigma_z \end{bmatrix} = \begin{bmatrix} (\sigma_x - \sigma_s) & \tau_{xy} & \tau_{xz} \\ \tau_{yx} & (\sigma_y - \sigma_s) & \tau_{yz} \\ \tau_{zx} & \tau_{zy} & (\sigma_z - \sigma_s) \end{bmatrix} + \begin{bmatrix} \sigma_s & 0 & 0 \\ 0 & \sigma_s & 0 \\ 0 & 0 & \sigma_s \end{bmatrix} = S'' + S'$$

$$[S'] = \begin{bmatrix} \sigma_s & 0 & 0 \\ 0 & \sigma_s & 0 \\ 0 & 0 & \sigma_s \end{bmatrix} = \begin{bmatrix} -\frac{8}{3} & 0 & 0 \\ 0 & -\frac{8}{3} & 0 \\ 0 & 0 & -\frac{8}{3} \end{bmatrix} \text{ (MPa)}$$

$$[S''] = \begin{bmatrix} (\sigma_x - \sigma_s) & \tau_{xy} & \tau_{xz} \\ \tau_{yx} & (\sigma_y - \sigma_s) & \tau_{yz} \\ \tau_{zx} & \tau_{zy} & (\sigma_z - \sigma_s) \end{bmatrix} = \begin{bmatrix} -20 + \frac{8}{3} & 0 & -6 \\ 0 & 6 + \frac{8}{3} & 0 \\ -6 & 0 & 6 + \frac{8}{3} \end{bmatrix} = \begin{bmatrix} -17.33 & 0 & -6 \\ 0 & 8.66 & 0 \\ -6 & 0 & 8.66 \end{bmatrix} \text{ (MPa)}$$

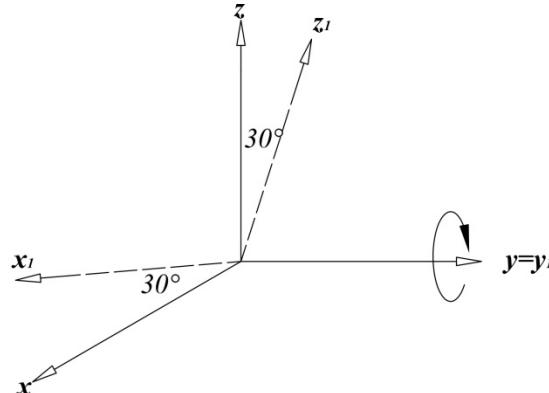
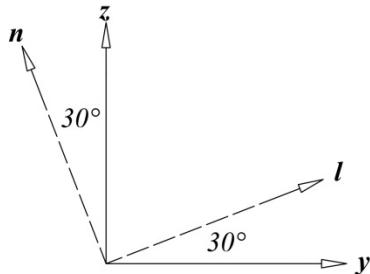
[S'] - Sferni dio tenzora napona (hidrostatski pritisak) - utiče na promjenu zapremine

[S''] Devijatorski dio tenzora napona (čisto smicanje) - utiče na promjenu oblika tijela

2. U tački u kojoj vlada ravno stanje napona poznati su sljedeći podaci:

Odrediti:

- Tenzor napona i izdvojiti element na koji djeluju;
- Glavne napone i glavne pravce i izdvojiti element na koji djeluju;
- Maksimalni smičući napon i odgovarajući normalni napon i izdvojiti element na koji djeluju;
- Komponentalne napone za ravni sa normalama  $\vec{n}_1 = \frac{1}{2}\vec{j} - \frac{\sqrt{3}}{2}\vec{k}$  i  $\vec{n}_2 = \frac{\sqrt{2}}{2}\vec{j} + \frac{\sqrt{2}}{2}\vec{k}$  i izdvojiti element na koji djeluju;
- Tenzor napona u novom transformisanom koordinatnom sistemu  $(x_1, y_1, z_1)$  koji se dobija rotacijom koordinatnog sistema  $(x_1, y_1, z_1)$  oko y ose za ugao  $30^\circ$ ;
- Mohr*-ov krug napona.



**RJEŠENJE**

- a. Tenzor napona

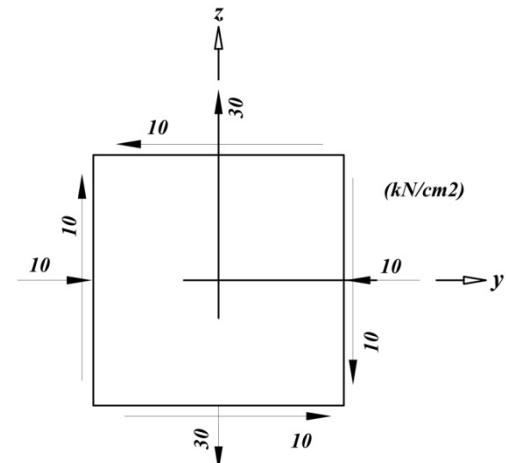
$$[S] = \begin{bmatrix} \sigma_y & \tau_{yz} \\ \tau_{zy} & \sigma_z \end{bmatrix} = \begin{bmatrix} -10 & \tau_{yz} \\ \tau_{zy} & 30 \end{bmatrix} \quad \tau_{yz} = ?$$

$$\tau_{nl} = \frac{1}{2}(\sigma_y - \sigma_z) \sin 2\varphi - \tau_{yz} \cos 2\varphi$$

$$12.32 = \frac{1}{2}(-10 - 30) \sin(2 \cdot 120^\circ) - (\tau_{yz}) \cos(2 \cdot 120^\circ)$$

$$\tau_{yz} = -10 \frac{kN}{cm^2}$$

$$[S] = \begin{bmatrix} -10 & -10 \\ -10 & 30 \end{bmatrix} \frac{kN}{cm^2}$$

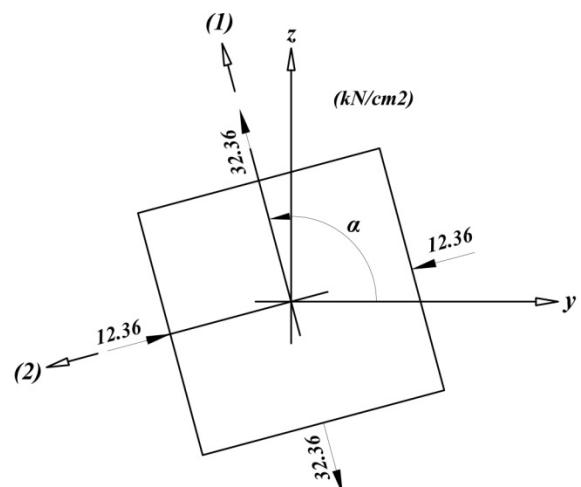


- b. Glavni naponi i glavni pravci

$$\begin{aligned} \sigma_{1/2} &= \frac{1}{2}(\sigma_y + \sigma_z) \pm \frac{1}{2}\sqrt{(\sigma_y - \sigma_z)^2 + 4\tau_{yz}^2} = \\ &= \frac{1}{2}(-10 + 30) \pm \frac{1}{2}\sqrt{(-10 - 30)^2 + 4(-10)^2} = \\ &= 10 \pm 22.36 \end{aligned}$$

$$\left. \begin{array}{l} \sigma_1 = 32.36 \frac{kN}{cm^2} \\ \sigma_2 = -12.36 \frac{kN}{cm^2} \end{array} \right\} \text{ - Glavni normalni naponi}$$

$$\text{Kontrola } \sigma_1 + \sigma_2 = \sigma_y + \sigma_z$$



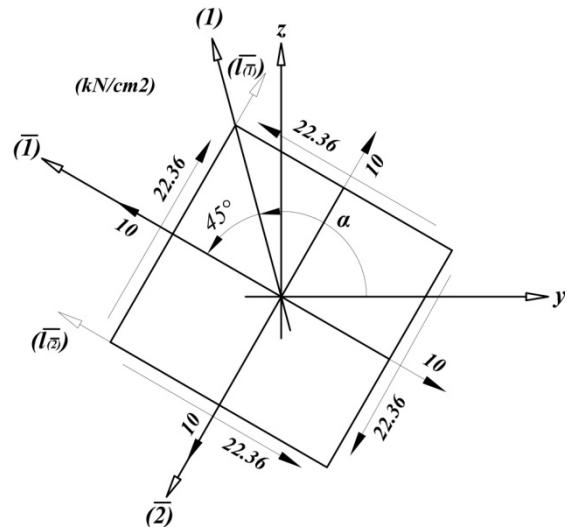
$$\operatorname{tg} 2\alpha = \frac{2\tau_{yz}}{\sigma_y - \sigma_z} = \frac{2 \cdot (-10)}{-10 - 30} = \frac{1}{2}$$

$$(\sigma_y - \sigma_z) = -40 < 0 \rightarrow \alpha = \frac{1}{2} \operatorname{arctg} \left( \frac{1}{2} \right) + 90^\circ = 103.28^\circ - \text{Ugao koji definiše glavni pravac (1)}$$

c. Maksimalni smičući napon i odgovarajući normalni napon

$$\tau_{max} = \frac{1}{2} (\sigma_1 - \sigma_2) = \frac{1}{2} (32.36 - (-12.36)) = 22.36 \frac{kN}{cm^2}$$

$$\sigma_s = \sigma_{odg} = \frac{1}{2} (\sigma_1 + \sigma_2) = \frac{1}{2} (32.36 + (-12.36)) = 10 \frac{kN}{cm^2}$$



d. Komponentalni naponi za ravni sa normalama  $\vec{n}_1$  i  $\vec{n}_2$

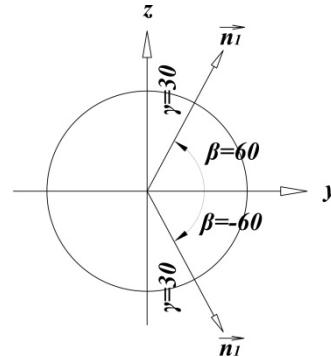
$$\vec{n}_1 = \frac{1}{2} \vec{j} - \frac{\sqrt{3}}{2} \vec{k} = \cos \beta \vec{j} + \cos \gamma \vec{k}$$

$\beta$  – ugao koji  $\vec{n}_1$  gradi sa "y" osom

$\gamma$  – ugao koji  $\vec{n}_1$  gradi sa "z" osom

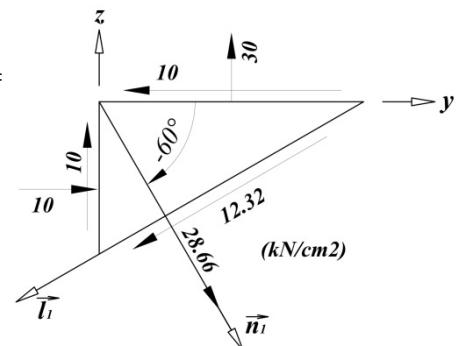
$$\left. \begin{array}{l} \cos \beta = \frac{1}{2} \\ \cos \gamma = -\frac{\sqrt{3}}{2} \end{array} \right\} \rightarrow \varphi_1 = -60^\circ$$

$$\boxed{\vec{n}_1 \leftrightarrow \varphi_1 = -60^\circ}$$



$$\begin{aligned} \sigma_{n_1} &= \frac{1}{2} (\sigma_y + \sigma_z) + \frac{1}{2} (\sigma_y - \sigma_z) \cos 2\varphi_1 + \tau_{yz} \sin 2\varphi_1 = \\ &= \frac{1}{2} (-10 + 30) + \frac{1}{2} (-10 - 30) \cos (2 \cdot (-60^\circ)) + (-10) \sin (2 \cdot (-60^\circ)) = \\ &= 28.66 \frac{kN}{cm^2} \end{aligned}$$

$$\begin{aligned} \tau_{n_1 l_1} &= \frac{1}{2} (\sigma_y - \sigma_z) \sin 2\varphi_1 - \tau_{yz} \cos 2\varphi_1 = \\ &= \frac{1}{2} (-10 - 30) \sin (2 \cdot (-60^\circ)) - (-10) \cos (2 \cdot (-60^\circ)) = 12.32 \frac{kN}{cm^2} \end{aligned}$$

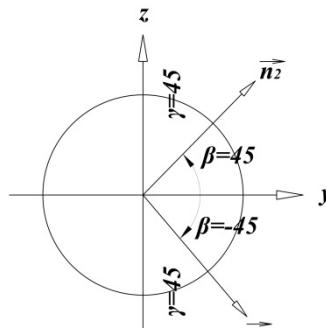


$$\vec{n}_2 = \frac{\sqrt{2}}{2} \vec{j} + \frac{\sqrt{2}}{2} \vec{k} = \cos \beta \vec{j} + \cos \gamma \vec{k}$$

$\beta$  – ugao koji  $\vec{n}_2$  gradi sa "y" osom

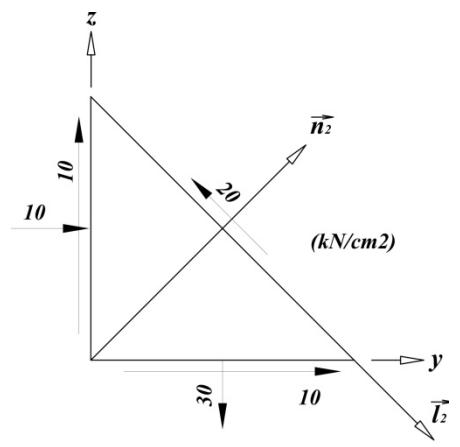
$\gamma$  – ugao koji  $\vec{n}_2$  gradi sa "z" osom

$$\left. \begin{array}{l} \cos \beta = \frac{\sqrt{2}}{2} \\ \cos \gamma = \frac{\sqrt{2}}{2} \end{array} \right\} \rightarrow \varphi_2 = 45^\circ$$



$$\overrightarrow{n_2} \leftrightarrow \varphi_2 = 45^\circ$$

$$\begin{aligned}\sigma_{n_2} &= \frac{1}{2}(\sigma_y + \sigma_z) + \frac{1}{2}(\sigma_y - \sigma_z)\cos 2\varphi_2 + \tau_{yz} \sin 2\varphi_2 = \\ &= \frac{1}{2}(-10 + 30) + \frac{1}{2}(-10 - 30)\cos(2 \cdot 45^\circ) + (-10)\sin(2 \cdot 45^\circ) = \\ &= 0 \frac{kN}{cm^2} \\ \tau_{n_2 l_2} &= \frac{1}{2}(\sigma_y - \sigma_z)\sin 2\varphi_2 - \tau_{yz} \cos 2\varphi_2 = \\ &= \frac{1}{2}(-10 - 30)\sin(2 \cdot 45^\circ) - (-10)\cos(2 \cdot 45^\circ) = -20 \frac{kN}{cm^2}\end{aligned}$$



e. Tenzor napona u novom transformisanom koordinatnom sistemu ( $x_1, y_1, z_1$ )

$$[S'] = [A][S][A]^T$$

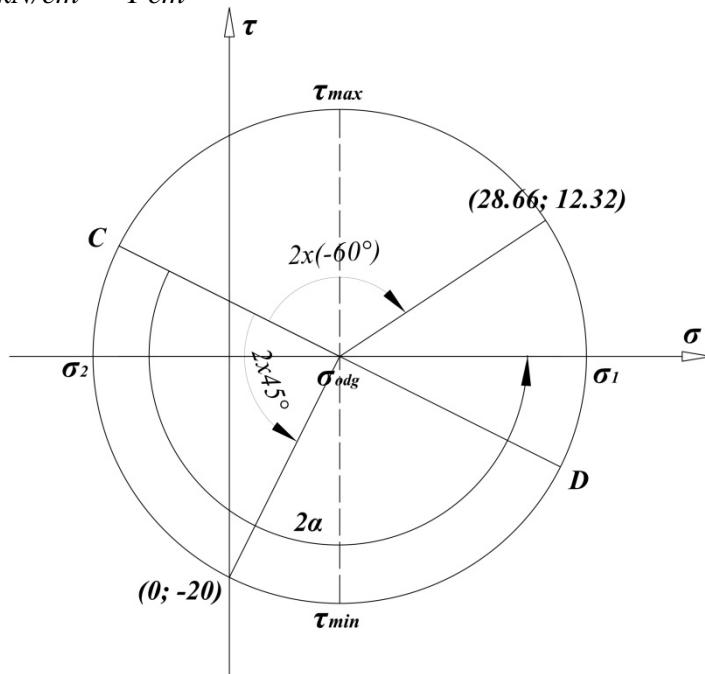
$$\begin{aligned}[A] &= \begin{bmatrix} \cos \alpha_{x_1} & \cos \beta_{x_1} & \cos \gamma_{x_1} \\ \cos \alpha_{y_1} & \cos \beta_{y_1} & \cos \gamma_{y_1} \\ \cos \alpha_{z_1} & \cos \beta_{z_1} & \cos \gamma_{z_1} \end{bmatrix} = \begin{bmatrix} \frac{\sqrt{3}}{2} & 0 & \frac{1}{2} \\ 0 & 1 & 0 \\ -\frac{1}{2} & 0 & \frac{\sqrt{3}}{2} \end{bmatrix} \\ [S'] &= [A][S][A]^T = \begin{bmatrix} \frac{\sqrt{3}}{2} & 0 & \frac{1}{2} \\ 0 & 1 & 0 \\ -\frac{1}{2} & 0 & \frac{\sqrt{3}}{2} \end{bmatrix} \begin{bmatrix} 0 & 0 & 0 \\ 0 & -10 & -10 \\ 0 & -10 & 30 \end{bmatrix} \begin{bmatrix} \frac{\sqrt{3}}{2} & 0 & -\frac{1}{2} \\ 0 & 1 & 0 \\ \frac{1}{2} & 0 & \frac{\sqrt{3}}{2} \end{bmatrix} = \begin{bmatrix} & & \\ & & \\ & & \end{bmatrix}\end{aligned}$$

f. Mohr-ov krug napona

$$C[\sigma_y; -\tau_{yz}] = [-10; 10] \frac{kN}{cm^2}$$

$$D[\sigma_z; \tau_{yz}] = [30; -10] \frac{kN}{cm^2}$$

Usvaja se razmjer  $10 \text{ kN/cm}^2 = 1 \text{ cm}$



# VJEŽBA BR. 7

## ANALIZA DEFORMACIJE

1. Stanje deformacije u tački opterećenog tijela zadato je tenzorom deformacije  $[D]$ .

Odrediti:

- Dilataciju pravca  $\vec{n}$  koji sa koordinatnim osama gradi jednake uglove;
- Klizanje između pravca  $\vec{n}$  i njemu upravnog pravca  $\vec{l}$  koji leži u ravni  $(x, y)$ ;
- Tenzor deformacije u novom transformisanom koordinatnom sistemu  $\vec{n}, \vec{l}, \vec{m}$ , pri čemu  $\vec{m}$  gradi oštar ugao sa osom  $x$ .

$$[D] = \begin{bmatrix} 5 & -1 & -1 \\ -1 & 4 & 0 \\ -1 & 0 & 4 \end{bmatrix} (\cdot 10^{-4})$$

**RJEŠENJE**

a.

$$[D] = \begin{bmatrix} 5 & -1 & -1 \\ -1 & 4 & 0 \\ -1 & 0 & 4 \end{bmatrix} (\cdot 10^{-4}) \rightarrow \begin{aligned} \vec{d}_x &= 5\vec{i} - \vec{j} - \vec{k} \\ \vec{d}_y &= -\vec{i} + 4\vec{j} \quad (\cdot 10^{-4}) \\ \vec{d}_z &= -\vec{i} + 4\vec{k} \end{aligned}$$

$$\vec{n} = \cos \alpha \vec{i} + \cos \beta \vec{j} + \cos \gamma \vec{k},$$

$$\cos \alpha = \cos \beta = \cos \gamma$$

$$|\vec{n}| = \sqrt{\cos^2 \alpha + \cos^2 \beta + \cos^2 \gamma} = 1$$

$$\cos \alpha = \pm \frac{1}{\sqrt{3}}$$

$$\vec{n} = \frac{1}{\sqrt{3}} \vec{i} + \frac{1}{\sqrt{3}} \vec{j} + \frac{1}{\sqrt{3}} \vec{k}$$

$$\vec{d}_n = d_{nx} \vec{i} + d_{ny} \vec{j} + d_{nz} \vec{k}$$

$$d_{nx} = \vec{d}_x \cdot \vec{n} = (5\vec{i} - \vec{j} - \vec{k}) \cdot \left( \frac{1}{\sqrt{3}} \vec{i} + \frac{1}{\sqrt{3}} \vec{j} + \frac{1}{\sqrt{3}} \vec{k} \right) = \frac{5}{\sqrt{3}} - \frac{1}{\sqrt{3}} - \frac{1}{\sqrt{3}} = \frac{3}{\sqrt{3}} = \sqrt{3} (\cdot 10^{-4})$$

$$d_{ny} = \vec{d}_y \cdot \vec{n} = (-\vec{i} + 4\vec{j}) \cdot \left( \frac{1}{\sqrt{3}} \vec{i} + \frac{1}{\sqrt{3}} \vec{j} + \frac{1}{\sqrt{3}} \vec{k} \right) = -\frac{1}{\sqrt{3}} + \frac{4}{\sqrt{3}} = \frac{3}{\sqrt{3}} = \sqrt{3} (\cdot 10^{-4})$$

$$d_{nz} = \vec{d}_z \cdot \vec{n} = (-\vec{i} + 4\vec{k}) \cdot \left( \frac{1}{\sqrt{3}} \vec{i} + \frac{1}{\sqrt{3}} \vec{j} + \frac{1}{\sqrt{3}} \vec{k} \right) = -\frac{1}{\sqrt{3}} + \frac{4}{\sqrt{3}} = \frac{3}{\sqrt{3}} = \sqrt{3} (\cdot 10^{-4})$$

$$\vec{d}_n = \sqrt{3}\vec{i} + \sqrt{3}\vec{j} + \sqrt{3}\vec{k} (\cdot 10^{-4}) \text{ - vektor deformacije za ravan sa normalom } \vec{n}$$

$$\varepsilon_n = \vec{d}_n \cdot \vec{n} = (\sqrt{3}\vec{i} + \sqrt{3}\vec{j} + \sqrt{3}\vec{k}) \cdot \left( \frac{1}{\sqrt{3}} \vec{i} + \frac{1}{\sqrt{3}} \vec{j} + \frac{1}{\sqrt{3}} \vec{k} \right) = 3 \cdot 10^{-4}$$

b.

- $\vec{l} = l_x \vec{i} + l_y \vec{j} + l_z \vec{k}, \quad (l_z = 0)$  - vektor  $\vec{l}$  leži u ravni  $(x, y)$ ;

- $\vec{n} \cdot \vec{l} = 0$  - uslov ortogonalnosti dva pravca;

$$\vec{n} \cdot \vec{l} = \left( \frac{1}{\sqrt{3}} \vec{i} + \frac{1}{\sqrt{3}} \vec{j} + \frac{1}{\sqrt{3}} \vec{k} \right) \cdot (l_x \vec{i} + l_y \vec{j}) = \frac{l_x}{\sqrt{3}} + \frac{l_y}{\sqrt{3}} = 0 \rightarrow l_x = -l_y$$

$$3. \quad |\vec{l}| = \sqrt{l_x^2 + l_y^2 + l_z^2} = 1 \rightarrow l_x^2 + l_y^2 = 1 \rightarrow l_x = \pm \frac{1}{\sqrt{2}}$$

$$\vec{l} = \frac{1}{\sqrt{2}}\vec{i} - \frac{1}{\sqrt{2}}\vec{j} \quad \text{ili} \quad \vec{l} = -\frac{1}{\sqrt{2}}\vec{i} + \frac{1}{\sqrt{2}}\vec{j}$$

$$\frac{1}{2}\gamma_{nl} = \vec{d}_n \cdot \vec{l} = (\sqrt{3}\vec{i} + \sqrt{3}\vec{j} + \sqrt{3}\vec{k}) \cdot \left( \frac{1}{\sqrt{2}}\vec{i} - \frac{1}{\sqrt{2}}\vec{j} \right) = 0$$

$$c. \quad \vec{m} = m_x \vec{i} + m_y \vec{j} + m_z \vec{k}$$

$$1. \quad \vec{m} \cdot \vec{l} = 0 \rightarrow (m_x \vec{i} + m_y \vec{j} + m_z \vec{k}) \cdot \left( \frac{1}{\sqrt{2}}\vec{i} - \frac{1}{\sqrt{2}}\vec{j} \right) = m_x \frac{1}{\sqrt{2}} - m_y \frac{1}{\sqrt{2}} = 0 \rightarrow m_x = m_y$$

$$2. \quad \begin{aligned} \vec{m} \cdot \vec{n} &= 0 \rightarrow (m_x \vec{i} + m_y \vec{j} + m_z \vec{k}) \cdot \left( \frac{1}{\sqrt{3}}\vec{i} + \frac{1}{\sqrt{3}}\vec{j} + \frac{1}{\sqrt{3}}\vec{k} \right) = \\ &= m_x \frac{1}{\sqrt{3}} + m_y \frac{1}{\sqrt{3}} + m_z \frac{1}{\sqrt{3}} = 2m_x \frac{1}{\sqrt{3}} + m_z \frac{1}{\sqrt{3}} 0 \rightarrow 2m_x + m_z = 0 \rightarrow m_z = -2m_x \end{aligned}$$

$$3. \quad |\vec{m}| = \sqrt{m_x^2 + m_y^2 + m_z^2} = 1 \rightarrow 2m_x^2 + m_z^2 = 2m_x^2 + (-2m_x)^2 = 1 \rightarrow m_x = \pm \frac{1}{\sqrt{6}}$$

$$\vec{m} \text{ gradi oštar ugao sa osom } x \rightarrow m_x = \frac{1}{\sqrt{6}} \rightarrow m_y = \frac{1}{\sqrt{6}} \quad m_z = -\frac{2}{\sqrt{6}}$$

$$\vec{m} = \frac{1}{\sqrt{6}}\vec{i} + \frac{1}{\sqrt{6}}\vec{j} - \frac{2}{\sqrt{6}}\vec{k}$$

$$[D'] = [A][D][A]^T$$

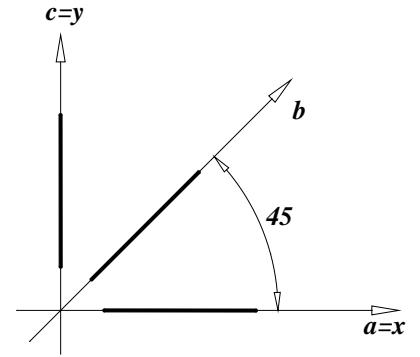
$$[A] = \begin{bmatrix} n_x & n_y & n_z \\ l_x & l_y & l_z \\ m_x & m_y & m_z \end{bmatrix} = \begin{bmatrix} \frac{1}{\sqrt{3}} & \frac{1}{\sqrt{3}} & \frac{1}{\sqrt{3}} \\ \frac{1}{\sqrt{2}} & -\frac{1}{\sqrt{2}} & 0 \\ \frac{1}{\sqrt{6}} & \frac{1}{\sqrt{6}} & -\frac{2}{\sqrt{6}} \end{bmatrix}$$

$$[D'] = [A][D][A]^T = \begin{bmatrix} \frac{1}{\sqrt{3}} & \frac{1}{\sqrt{3}} & \frac{1}{\sqrt{3}} \\ \frac{1}{\sqrt{2}} & -\frac{1}{\sqrt{2}} & 0 \\ \frac{1}{\sqrt{6}} & \frac{1}{\sqrt{6}} & -\frac{2}{\sqrt{6}} \end{bmatrix} \begin{bmatrix} 5 & -1 & -1 \\ -1 & 4 & 0 \\ -1 & 0 & 4 \end{bmatrix} (\cdot 10^{-4}) \begin{bmatrix} \frac{1}{\sqrt{3}} & \frac{1}{\sqrt{2}} & \frac{1}{\sqrt{6}} \\ \frac{1}{\sqrt{3}} & -\frac{1}{\sqrt{2}} & \frac{1}{\sqrt{6}} \\ \frac{1}{\sqrt{3}} & 0 & -\frac{2}{\sqrt{6}} \end{bmatrix} = \begin{bmatrix} 3 & 0 & 0 \\ 0 & 5.5 & \frac{\sqrt{3}}{2} \\ 0 & \frac{\sqrt{3}}{2} & 4.5 \end{bmatrix} (\cdot 10^{-4})$$

2. U tački napregnutog tijela u kojoj vlada ravno stanje deformacije pomoću mernih traka su izmjerene sljedeće dilatacije:  $\varepsilon_a = -5 \cdot 10^{-4}$ ;  $\varepsilon_b = 2 \cdot 10^{-4}$ ;  $\varepsilon_c = 3 \cdot 10^{-4}$ .

Odrediti:

- Tenzor deformacije u toj tački i prikazati deformisani okolinu tačke;
- Dilataciju u pravcu koji zaklapa ugao  $60^0$  sa osom  $x$ ;
- Klizanje između tog i njemu upravnog pravca;
- Glavne dilatacije, pravce glavnih dilatacija i prikazati deformisani okolinu tačke u sistemu glavnih osa;
- Maksimalnu smičuću deformaciju (klizanje) i uglove između kojih se događa;
- Mohr*-ov krug deformacije.



### RJEŠENJE

a.

$$\varepsilon_n = \frac{\varepsilon_x + \varepsilon_y}{2} + \frac{\varepsilon_x - \varepsilon_y}{2} \cos 2\varphi + \frac{1}{2} \gamma_{xy} \sin 2\varphi$$

$$\varepsilon_b = 2 \cdot 10^{-4} \quad (\varphi = 45^0)$$

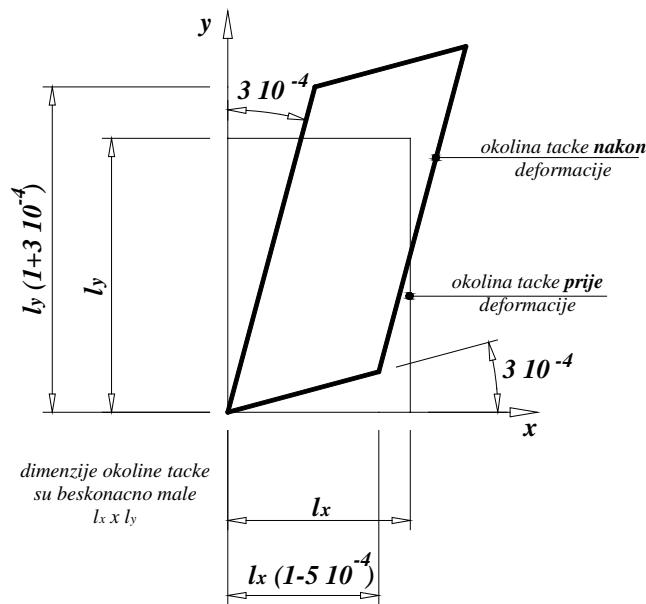
$$\varepsilon_b = \frac{1}{2}(-5+3)10^{-4} + \frac{1}{2}(-5-3)10^{-4} \cos(2 \cdot 45^0) + \frac{1}{2} \gamma_{xy} \cdot 10^{-4} \sin(2 \cdot 45^0) = 2 \cdot 10^{-4} \quad | /10^{-4} \rightarrow \frac{1}{2} \gamma_{xy} = 3 \cdot 10^{-4}$$

$$[D] = \begin{bmatrix} \varepsilon_x & \frac{1}{2} \gamma_{xy} \\ \frac{1}{2} \gamma_{yx} & \varepsilon_y \end{bmatrix} = \begin{bmatrix} -5 & 3 \\ 3 & 3 \end{bmatrix} \cdot 10^{-4}$$

$l_x$  - elementarna dužina u pravcu x prije deformacije

$$\varepsilon_x = \frac{\Delta l_x}{l_x} - \text{dilatacija u pravcu } x$$

$$l'_x = l_x + \Delta l_x = l_x + \varepsilon_x l_x = l_x (1 + \varepsilon_x) - \text{elementarna dužina } \underline{\text{nakon}} \text{ deformacije u pravcu } x$$



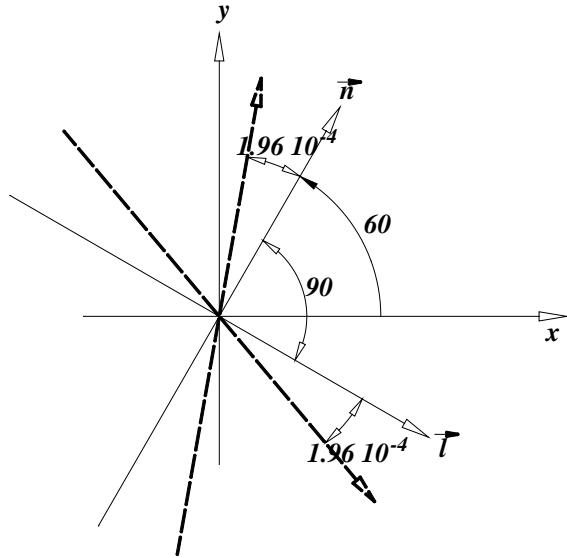
b.

$$\vec{n} \leftrightarrow \varphi = 60^\circ$$

$$\begin{aligned}\varepsilon_n &= \frac{\varepsilon_x + \varepsilon_y}{2} + \frac{\varepsilon_x - \varepsilon_y}{2} \cos 2\varphi + \frac{1}{2} \gamma_{xy} \sin 2\varphi = \\ &= \frac{1}{2}(-5+3)10^{-4} + \frac{1}{2}(-5-3)10^{-4} \cos(2 \cdot 60^\circ) + 3 \cdot 10^{-4} \sin(2 \cdot 60^\circ) = 3.6 \cdot 10^{-4} \text{ izduženje}\end{aligned}$$

c.

$$\begin{aligned}\frac{1}{2} \gamma_{nl} &= \frac{\varepsilon_x - \varepsilon_y}{2} \sin 2\varphi - \frac{1}{2} \gamma_{xy} \cos 2\varphi = \\ &= \frac{1}{2}(-5-3) \cdot 10^{-4} \sin(2 \cdot 60^\circ) - 3 \cdot 10^{-4} \cos(2 \cdot 60^\circ) = -1.96 \cdot 10^{-4} \text{ pravi ugao se povećao}\end{aligned}$$



$$\left. \begin{aligned}\frac{1}{2} \gamma_{ln}(\varphi = 150^\circ) &= \frac{\varepsilon_x - \varepsilon_y}{2} \sin 2\varphi - \frac{1}{2} \gamma_{xy} \cos 2\varphi = \\ &= \frac{1}{2}(-5-3) \cdot 10^{-4} \sin(2 \cdot 150^\circ) - 3 \cdot 10^{-4} \cos(2 \cdot 150^\circ) = +1.96 \cdot 10^{-4} \text{ pravi ugao se smanjio}\end{aligned} \right\}$$

d.

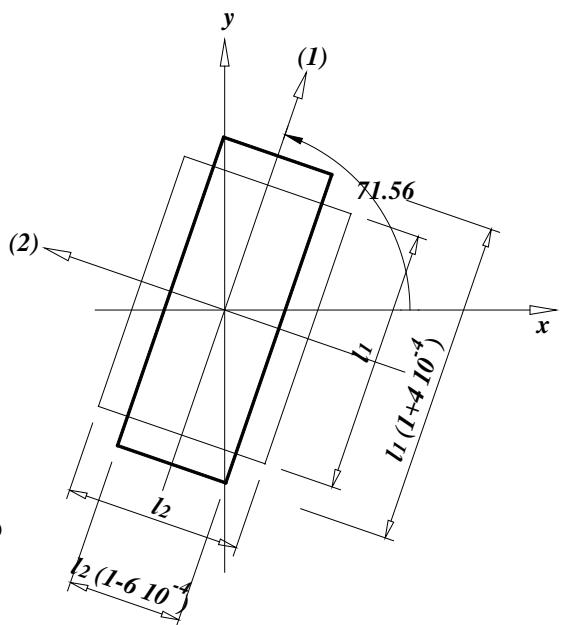
$$\begin{aligned}\varepsilon_{1/2} &= \frac{1}{2}(\varepsilon_x + \varepsilon_y) \pm \frac{1}{2} \sqrt{(\varepsilon_x - \varepsilon_y)^2 + \gamma_{xy}^2} = \\ &= \frac{1}{2}(-5+3) \cdot 10^{-4} \pm \frac{1}{2} \cdot 10^{-4} \sqrt{(-5-3)^2 + 6^2} = (-1 \pm 5) \cdot 10^{-4}\end{aligned}$$

$$\varepsilon_1 = 4 \cdot 10^{-4}$$

$$\varepsilon_2 = -6 \cdot 10^{-4}$$

$$\tan 2\alpha = \frac{\gamma_{xy}}{\varepsilon_x - \varepsilon_y} = \frac{6 \cdot 10^{-4}}{(-5-3) \cdot 10^{-4}} = \frac{6}{-8}$$

$$(\varepsilon_x - \varepsilon_y) = (-5-3) \cdot 10^{-4} < 0 \rightarrow \alpha = \frac{1}{2} \operatorname{arctg} \left( \frac{6}{-8} \right) + 90^\circ = 71.56^\circ$$

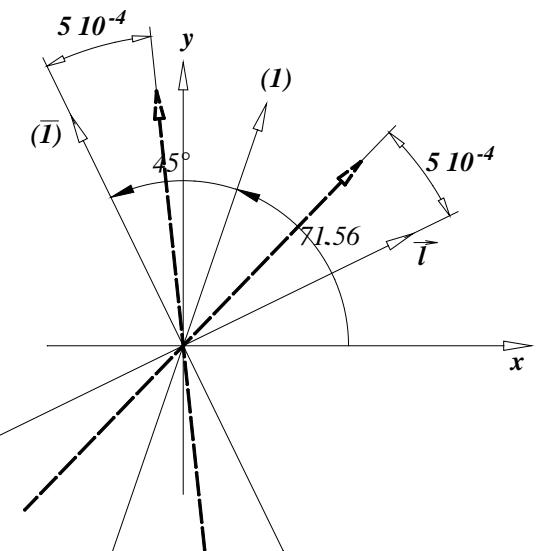


e.

$$\frac{1}{2}\gamma_{max} = \frac{1}{2}(\varepsilon_1 - \varepsilon_2) = \frac{1}{2}(4 - (-6)) \cdot 10^{-4} = 5 \cdot 10^{-4}$$

$$\varepsilon_{odg} = \varepsilon_s = \frac{1}{2}(\varepsilon_1 + \varepsilon_2) = \frac{1}{2}(4 + (-6)) \cdot 10^{-4} = -1 \cdot 10^{-4}$$

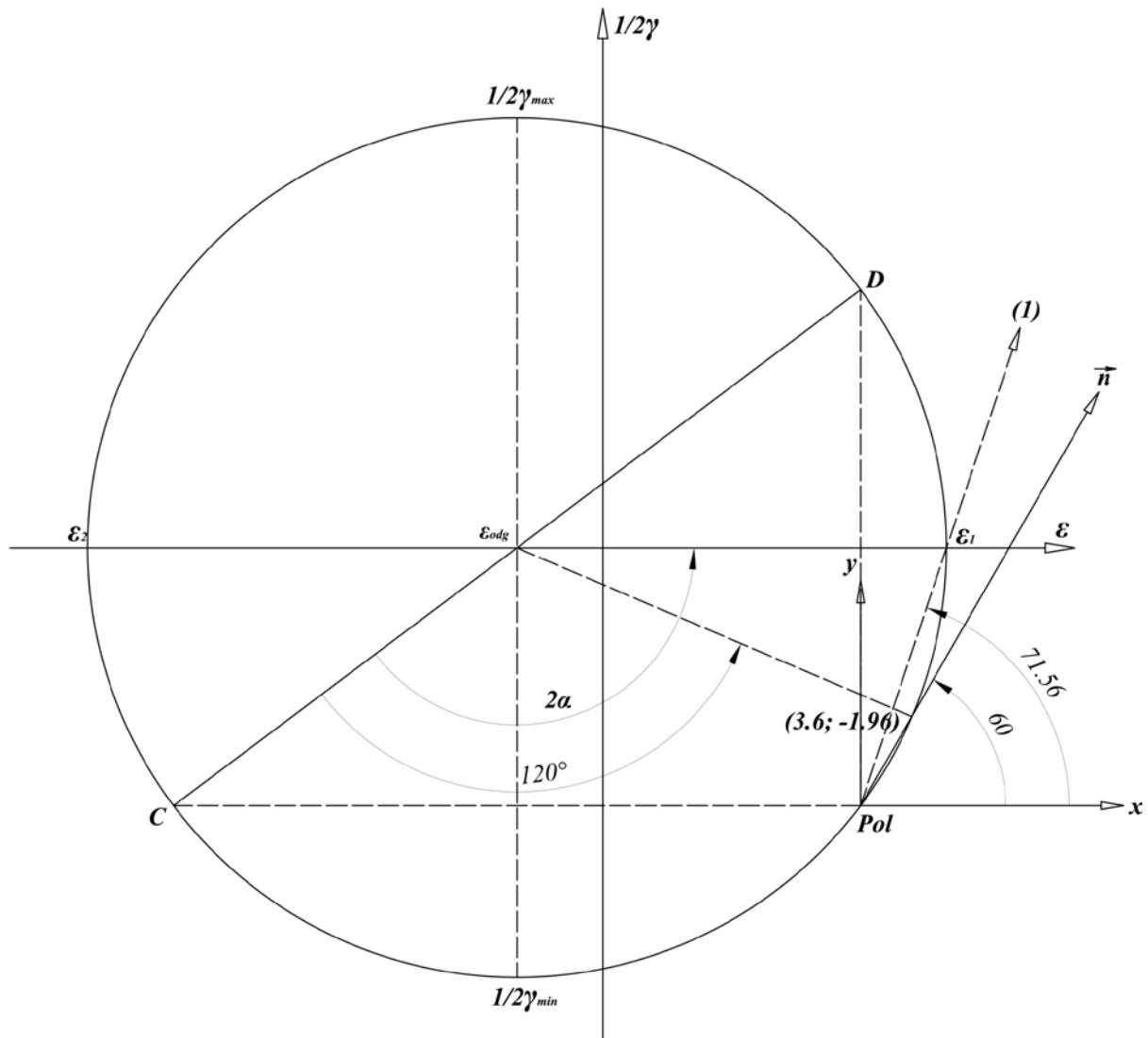
$$\overline{\alpha} = \alpha + 45^\circ$$



f.  $C \left[ \varepsilon_x; -\frac{1}{2}\gamma_{xy} \right] = [-5; -3] \cdot 10^{-4}$

$$D \left[ \varepsilon_y; \frac{1}{2}\gamma_{xy} \right] = [3; 3] \cdot 10^{-4}$$

Usvaja se razmjer  $10^{-4} = 1 \text{ cm}$



# VJEŽBA BR. 8

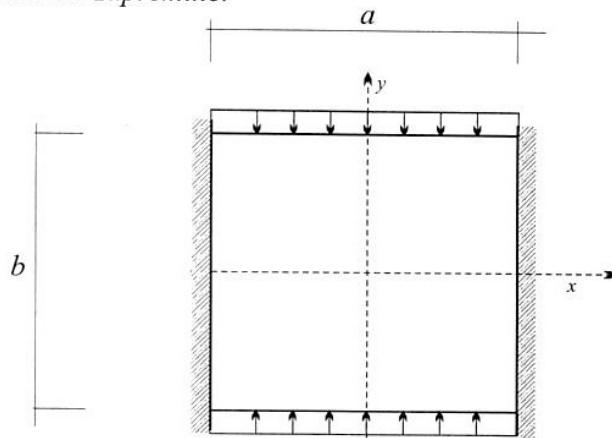
## VEZE IZMEĐU NAPONA I DEFORMACIJA

1.

### ZADATAK:

Pravougaona ploča  $a \times b = 100 \times 100 \text{ cm}$  debeline  $t = 10 \text{ cm}$  koja je uklještena između dva idealno kruta i glatka zida opterećena je pritiskom od  $10 \text{ MPa}$ . Odrediti napone i deformacije u ploči ako se ona zagreje za  $\Delta t = 50^\circ\text{C}$  kao i specifičnu i absolutnu promenu zapremine.

$$E = 210 \text{ GPa}, \\ v = 0.3 \\ \alpha = 1.25 \cdot 10^{-5} \text{ } ^\circ\text{C}^l$$



**RJEŠENJE**

Početni (granični) uslovi:

$\sigma_z = 0$	$\varepsilon_z \neq 0$	$\tau_{xy} = 0$	$\gamma_{xy} = 0$
$\sigma_y = -10 \text{ MPa}$	$\varepsilon_y \neq 0$	$\tau_{xz} = 0$	$\gamma_{xz} = 0$
$\sigma_x \neq 0$	$\varepsilon_x = 0$	$\tau_{yz} = 0$	$\gamma_{yz} = 0$

$$\varepsilon_x = \frac{1}{E} [\sigma_x - v(\sigma_y + \sigma_z)] + \alpha_t \Delta t = \frac{1}{E} [\sigma_x - v\sigma_y] + \alpha_t \Delta t = 0$$

$$\sigma_x = v\sigma_y - E\alpha_t \Delta t = 0.3(-10) - 210 \cdot 10^3 \cdot 1.25 \cdot 10^{-5} \cdot 50 = -134.58 \text{ MPa}$$

$$[S] = \begin{bmatrix} -134.58 & 0 & 0 \\ 0 & -10 & 0 \\ 0 & 0 & 0 \end{bmatrix} (\text{MPa}) \text{ - tenzor napona}$$

$$\varepsilon_y = \frac{1}{E} [\sigma_y - v(\sigma_z + \sigma_x)] + \alpha_t \Delta t = \frac{1}{210 \cdot 10^3} [-10 - 0.3(0 - 134.58)] + 1.25 \cdot 10^{-5} \cdot 50 = 0.77 \cdot 10^{-3}$$

$$\varepsilon_z = \frac{1}{E} [\sigma_z - v(\sigma_x + \sigma_y)] + \alpha_t \Delta t = \frac{1}{210 \cdot 10^3} [0 - 0.3(-10 - 134.58)] + 1.25 \cdot 10^{-5} \cdot 50 = 0.83 \cdot 10^{-3}$$

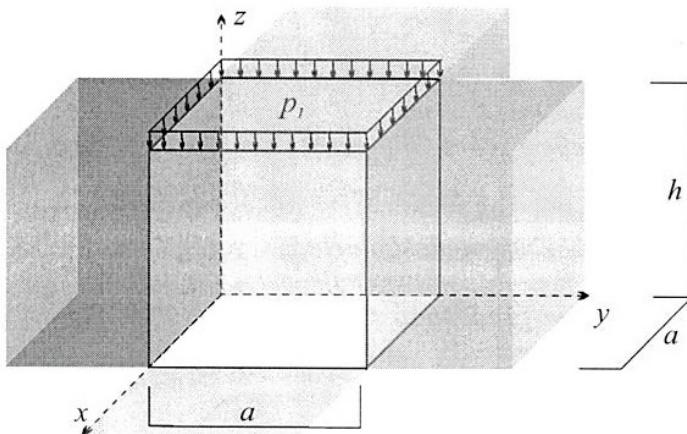
$$[D] = \begin{bmatrix} 0 & 0 & 0 \\ 0 & 0.77 & 0 \\ 0 & 0 & 0.83 \end{bmatrix} \cdot 10^{-3} \text{ - tenzor deformacije}$$

$$e = \varepsilon_x + \varepsilon_y + \varepsilon_z = (0 + 0.77 + 0.83) \cdot 10^{-3} = 1.6 \cdot 10^{-3} \text{ - specifična promjena zapremine}$$

$$\Delta V = e \cdot V = 1.6 \cdot 10^{-3} \cdot 1 \cdot 1 \cdot 0.1 = 0.16 \cdot 10^{-3} \text{ m}^3 \text{ - absolutna promjena zapremine}$$

**ZADATAK:**

Blok materijala na slici sabija se pritiskom  $p = 50 \text{ MPa}$  u bunaru čiji su zidovi kruti i istovremeno se zagreje za  $\Delta T = 50^\circ \text{C}$ . Zanemarujući trenje između materijala i zidova bunara, odrediti:



- a/ komponentalne napone i deformacije,
- b/ komponentalna pomeranja,
- c/ promenu visine bloka,
- d/ promenu zapremljenosti bloka,
- e/ vrednost kubne dilatacije

Poznato je:

$$E = 200 \text{ GPa}, \nu = 1/3, \alpha = 1.25 \cdot 10^{-5} \text{ }^\circ\text{C}^{-1}$$

$$a = 0.1 \text{ m}, h = 1 \text{ m}$$

**RJEŠENJE**

## a. komponente napona i deformacije

$\sigma_z = -50 \text{ MPa}$	$\varepsilon_z \neq 0$	$\tau_{xy} = 0$	$\gamma_{xy} = 0$
$\sigma_y \neq 0$	$\varepsilon_y = 0$	$\tau_{xz} = 0$	$\rightarrow \gamma_{xz} = 0$
$\sigma_x \neq 0$	$\varepsilon_x = 0$	$\tau_{yz} = 0$	$\gamma_{yz} = 0$

$$\left. \begin{array}{l} \varepsilon_x = \frac{1}{E} [\sigma_x - \nu(\sigma_y + \sigma_z)] + \alpha_t \Delta t = 0 \\ \varepsilon_y = \frac{1}{E} [\sigma_y - \nu(\sigma_z + \sigma_x)] + \alpha_t \Delta t = 0 \end{array} \right\} \rightarrow \begin{array}{l} \sigma_x = -E\alpha_t \Delta t + \nu(\sigma_y + \sigma_z) \\ \sigma_y = -E\alpha_t \Delta t + \nu(\sigma_x + \sigma_z) \end{array}$$

$$\sigma_y = -E\alpha_t \Delta t + \nu(-E\alpha_t \Delta t + \nu(\sigma_y + \sigma_z) + \sigma_z)$$

$$\sigma_y - \nu^2 \sigma_y = -E\alpha_t \Delta t - \nu E\alpha_t \Delta t + \nu^2 \sigma_z + \nu \sigma_z$$

$$\sigma_y = \frac{1}{1-\nu^2} [-E\alpha_t \Delta t - \nu E\alpha_t \Delta t + \nu^2 \sigma_z + \nu \sigma_z] =$$

$$= \frac{1}{1-\frac{1}{9}} \left[ -200 \cdot 10^3 \cdot 1.25 \cdot 10^{-5} \cdot 50 - \frac{1}{3} 200 \cdot 10^3 \cdot 1.25 \cdot 10^{-5} \cdot 50 + \frac{1}{9} (-50) + \frac{1}{3} (-50) \right] = -212.5 \text{ MPa}$$

$$\sigma_x = -E\alpha_t \Delta t + \nu(\sigma_y + \sigma_z) = -200 \cdot 10^3 \cdot 1.25 \cdot 10^{-5} \cdot 50 + \frac{1}{3} ((-50) + (-212.5)) = -212.5 \text{ MPa}$$

$$[S] = \begin{bmatrix} -212.5 & 0 & 0 \\ 0 & -212.5 & 0 \\ 0 & 0 & -50 \end{bmatrix} (\text{MPa}) \text{ - tenzor napona}$$

$$\varepsilon_z = \frac{1}{E} [\sigma_z - \nu(\sigma_x + \sigma_y)] + \alpha_t \Delta t = \frac{1}{200 \cdot 10^3} \left[ -50 - \frac{1}{3} (-212.5 - 212.5) \right] + 1.25 \cdot 10^{-5} \cdot 50 = 1.083 \cdot 10^{-3} \text{ - izduženje}$$

$$[D] = \begin{bmatrix} 0 & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & 1.083 \end{bmatrix} \cdot 10^{-3} \text{ - tenzor deformacije}$$

b. komponentalna pomjeranja

$$\varepsilon_z = \frac{\partial w}{\partial z} = 1.083 \cdot 10^{-3} \rightarrow \partial w = 1.083 \cdot 10^{-3} \partial z \quad | \int \rightarrow w = 1.083 \cdot 10^{-3} \cdot z + C$$

Granični uslov: za  $z=0 w=0 \rightarrow 0 = 1.083 \cdot 10^{-3} \cdot 0 + C \rightarrow C = 0$

Konačno:  $u = 0, v = 0, w = 1.083 \cdot 10^{-3} \cdot z$

c. promjena visine bloka ( $\Delta h$ )

$$\varepsilon_z = \frac{\Delta h}{h} \rightarrow \Delta h = \varepsilon_z \cdot h = 1.083 \cdot 10^{-3} \cdot 100 \approx 0.11 \text{ cm}$$

d. vrijednost kubne dilatacije (specifična promjena zapremina)

$$e = \varepsilon_x + \varepsilon_y + \varepsilon_z = (0+0+1.083) \cdot 10^{-3} = 1.083 \cdot 10^{-3} \approx 11 \cdot 10^{-4}$$

e. apsolutna promjena zapremine bloka

$$\Delta V = e \cdot V = 11 \cdot 10^{-4} \cdot 1 \cdot 0.1 \cdot 0.1 = 11 \cdot 10^{-6} \text{ m}^3 = 11 \text{ cm}^3$$

### 3.

#### ZADATAK:

U tački A napregnutog tela u kojoj vlada ravno stanje napona, merenjem dobijeni su sledeći podaci:  
 $\sigma_z = 30 \text{ MPa}$ ,

$$\sigma_a = -0.98 \text{ MPa} \text{ i } \tau_{ab} = 23.66 \text{ MPa}.$$

Odrediti:

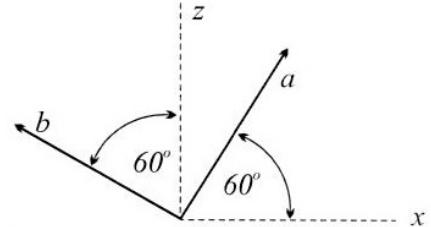
a/ tenzor napona, glavne napone, glavne pravce i izdvojiti element na koji deluju,

b/ pravce za koje je normalni napon jednak nuli,

c/Mohr-ov krug napona (proveriti rezultate pod a/ i b/),

d/ vrednost Poisson-ovog koeficijenta  $v$  ( $\varepsilon_y = -5 \cdot 10^{-5}$ ,  $E = 200 \text{ GPa}$ ),

e/ tenzor deformacije, glavne dilatacije i glavne pravce.



**RJEŠENJE**

a. tenzor napona i glavni naponi

$$\begin{aligned} \varphi_a = 60^\circ \rightarrow \sigma_a &= \frac{1}{2}(\sigma_x + \sigma_z) + \frac{1}{2}(\sigma_x - \sigma_z)\cos 2\varphi_a + \tau_{xz} \sin 2\varphi_a = \\ &= \frac{1}{2}(\sigma_x + 30) + \frac{1}{2}(\sigma_x - 30)\cos(2 \cdot 60^\circ) + \tau_{xz} \sin(2 \cdot 60^\circ) = -0.98 \text{ MPa} \end{aligned}$$

$$\begin{aligned} \varphi_b = 150^\circ \rightarrow \tau_{ba} &= \frac{1}{2}(\sigma_x - \sigma_z)\sin 2\varphi_b - \tau_{xz} \cos 2\varphi_b = \\ &= \frac{1}{2}(\sigma_x - 30)\sin(2 \cdot 150^\circ) - \tau_{xz} \cos(2 \cdot 150^\circ) = 23.66 \text{ MPa} \end{aligned}$$

Iz sistema jednačina se dobija:  $\sigma_x = 10 \text{ MPa}$     $\tau_{xz} = -30 \text{ MPa}$     $\rightarrow [S] = \begin{bmatrix} 10 & -30 \\ -30 & 30 \end{bmatrix} \text{ MPa}$  - **ravno stanje napona u tački**

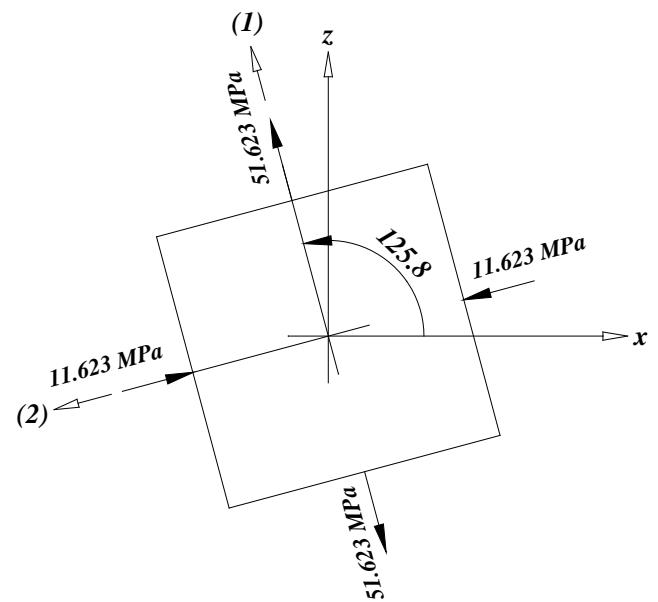
$$\sigma_{1/2} = \frac{1}{2}(10+30) \pm \frac{1}{2}\sqrt{(10-30)^2 + 4(-30)^2} = 20 \pm 31.623$$

$$\left. \begin{array}{l} \sigma_1 = 51.623 \text{ MPa} \\ \sigma_2 = -11.623 \text{ MPa} \end{array} \right\} \text{ - Glavni normalni naponi}$$

Kontrola  $\sigma_1 + \sigma_2 = \sigma_y + \sigma_z$

$$\tan 2\alpha = \frac{2 \cdot (-30)}{10 - 30} = 3$$

$$(\sigma_x - \sigma_z) < 0 \rightarrow \alpha = \frac{1}{2} \operatorname{arctg}(3) + 90^\circ = 125.8^\circ$$



- b. pravci za koje je normalni napon jednak nula (ako postoje)

$$\sigma_n = \frac{1}{2}(\sigma_x + \sigma_z) + \frac{1}{2}(\sigma_x - \sigma_z)\cos 2\varphi + \tau_{xz}\sin 2\varphi = 0$$

$$\sigma_n = \frac{1}{2}(\sigma_1 + \sigma_2) + \frac{1}{2}(\sigma_1 - \sigma_2)\cos 2\varphi + \tau_{12}\sin 2\varphi = 0 \quad (\text{u sistemu glavnih osa } \tau_{12} = 0)$$

$$\sigma_n = \frac{1}{2}(51.623 - 11.623) + \frac{1}{2}(51.623 + 11.623)\cos 2\varphi = 0$$

$$20 + 31.623\cos 2\varphi = 0$$

$$\cos 2\varphi = -0.632 \rightarrow \text{postoji rješenje}$$

$$2\varphi = \operatorname{arccos}(-0.632) = \pm 129.2^\circ$$

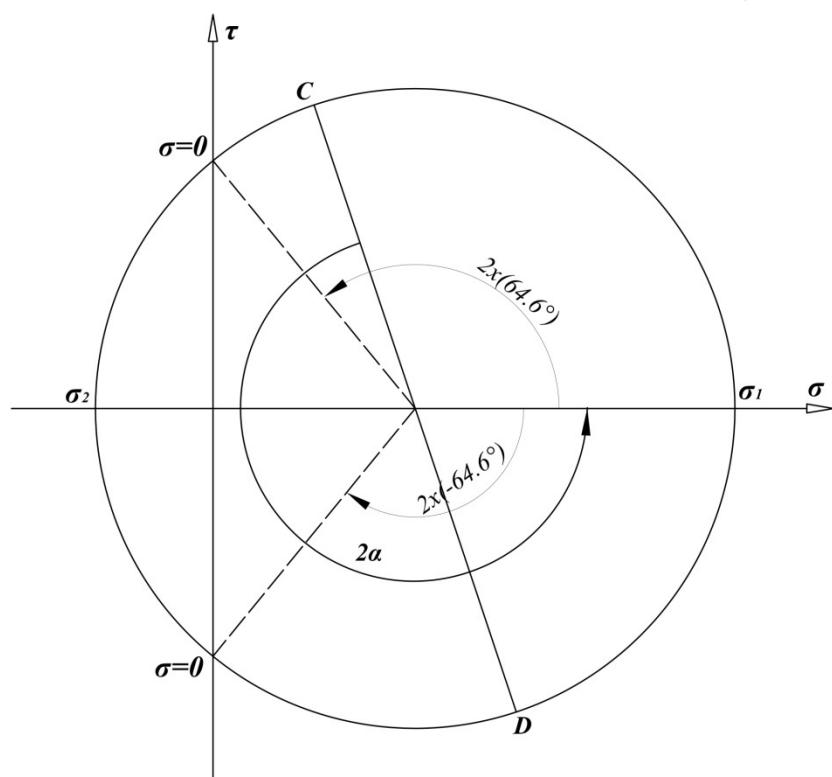
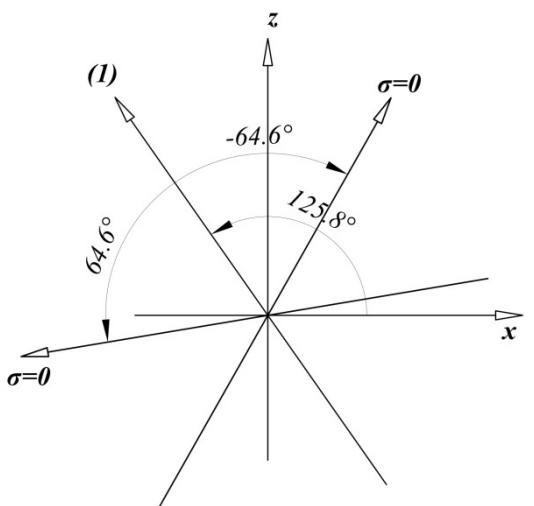
$$\varphi = \pm 64.6^\circ \text{ - u odnosu na osu (1)}$$

- c. Mohr-ov krug napona

$$C[\sigma_x; -\tau_{xz}] = [10; 30] \text{ MPa}$$

$$D[\sigma_z; \tau_{xz}] = [30; -30] \text{ MPa}$$

Usvaja se razmjer  $10 \text{ MPa} = 1 \text{ cm}$



d. Poisson-ov koeficijent

$$\varepsilon_y = \frac{1}{E} [\sigma_y - \nu(\sigma_z + \sigma_x)] = -5 \cdot 10^{-5}$$

$$-\nu(10 + 30) = -5 \cdot 10^{-5} \cdot 200 \cdot 10^3$$

$$\nu = \frac{1}{4} = 0.25$$

e. Tenzor deformacije i glavne dilatacije

$$\varepsilon_x = \frac{1}{E} [\sigma_x - \nu(\sigma_y + \sigma_z)] = \frac{1}{200 \cdot 10^3} [10 - 0.25(0 + 30)] = 1.25 \cdot 10^{-5}$$

$$\varepsilon_z = \frac{1}{E} [\sigma_z - \nu(\sigma_x + \sigma_y)] = \frac{1}{200 \cdot 10^3} [30 - 0.25(10 + 0)] = 13.75 \cdot 10^{-5}$$

$$G = \frac{E}{2(1+\nu)} = \frac{200}{2(1+0.25)} = 80 \text{ GPa}$$

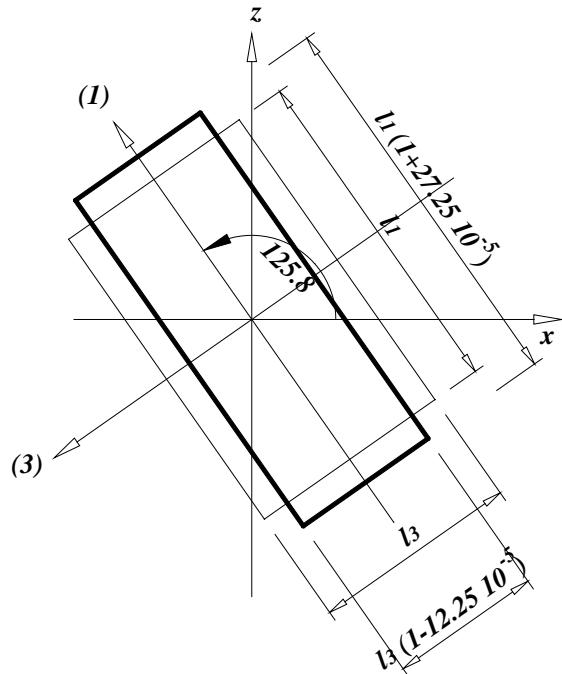
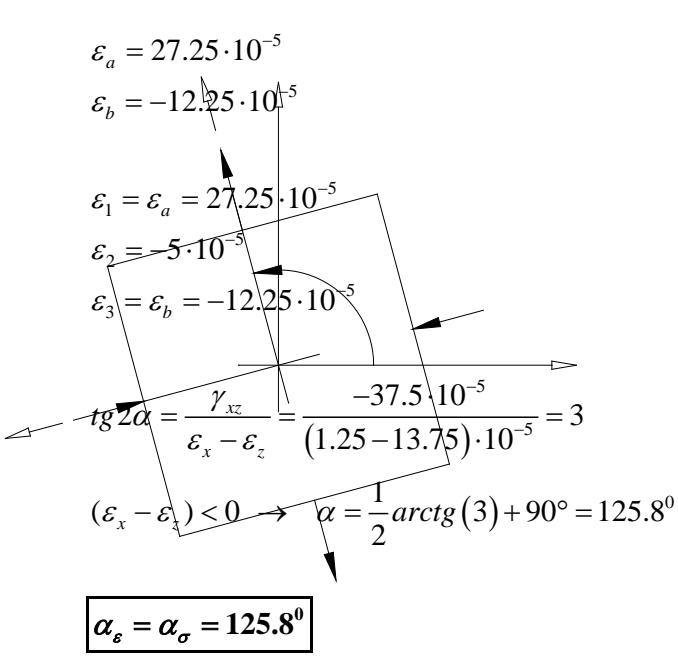
$$\gamma_{xy} = \frac{\tau_{xy}}{G} = 0$$

$$\gamma_{xz} = \frac{\tau_{xz}}{G} = \frac{-30}{80 \cdot 10^3} = -37.5 \cdot 10^{-5}$$

$$\gamma_{yz} = \frac{\tau_{yz}}{G} = 0$$

$$[D] = \begin{bmatrix} 1.25 & 0 & -18.75 \\ 0 & -5 & 0 \\ -18.75 & 0 & 13.75 \end{bmatrix} \cdot 10^{-5} - \text{prostorno stanje deformacije u tački}$$

$$\begin{aligned} \varepsilon_{a/b} &= \frac{1}{2} (\varepsilon_x + \varepsilon_z) \pm \frac{1}{2} \sqrt{(\varepsilon_x - \varepsilon_z)^2 + \gamma_{xz}^2} = \\ &= \frac{1}{2} (1.25 + 13.75) \cdot 10^{-5} \pm \frac{1}{2} \cdot 10^{-4} \sqrt{(1.25 - 13.75)^2 + 37.5^2} = (7.5 \pm 19.75) \cdot 10^{-4} \end{aligned}$$



**ZADATAK:**

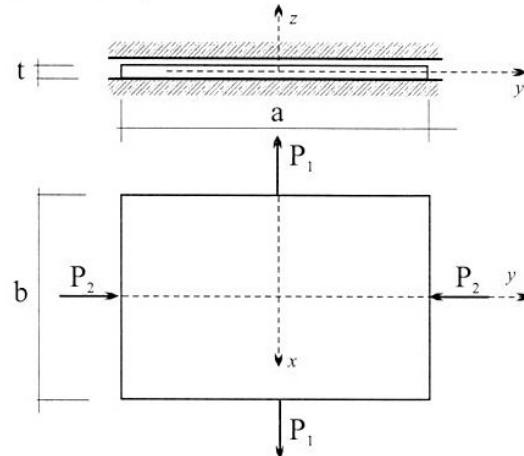
Pravougaona ploča a x b = 100 x 80 cm debljine t = 3 cm leži između dve paralelne nepokretne ravni na međusobnom rastojanju 3.001 cm. Ako se ploča zagreje za  $\Delta t = 30^\circ\text{C}$  i optereti silama zatezanja  $P_1 = 30 \text{ kN}$  i silama pritiska  $P_2 = 90 \text{ kN}$  kao na slici, odrediti:

- a/ pritisak koji ploča vrši na nepokretne ravni,
- b/ tenzor napona u karakterističnoj tački,
- c/ tenzor deformacija u sistemu glavnih osa,
- d/ specifičnu i absolutnu promenu zapremine.

$$E = 2 \cdot 10^5 \text{ MPa},$$

$$\nu = 0.3$$

$$\alpha = 1.25 \cdot 10^{-5}$$



**RJEŠENJE**

- a. pritisak koji ploča vrši na nepokretne ravni

$\sigma_x = \frac{P_1}{a \cdot t} = \frac{30}{100 \cdot 3} = 0.1 \frac{\text{kN}}{\text{cm}^2}$	$\varepsilon_x \neq 0$
$\sigma_y = -\frac{P_2}{b \cdot t} = -\frac{90}{80 \cdot 3} = -0.375 \frac{\text{kN}}{\text{cm}^2}$	$\varepsilon_y \neq 0$
$\sigma_z \neq 0$	$\varepsilon_z = \frac{\Delta l_z}{l_z}$

$$\varepsilon_z = \frac{1}{E} [\sigma_z - \nu(\sigma_x + \sigma_y)] + \alpha_t \Delta t = \frac{\Delta l_z}{l_z}$$

$$\frac{1}{2 \cdot 10^4} [\sigma_z - 0.3(0.1 - 0.375)] + 1.25 \cdot 10^{-5} \cdot 30 = \frac{0.001}{3} \rightarrow \sigma_z = -0.916 \frac{\text{kN}}{\text{cm}^2}$$

$$N = \sigma_z \cdot a \cdot b = -0.916 \cdot 100 \cdot 80 = 7328 \text{ kN}$$

- b. tenzor napona

$$[S] = \begin{bmatrix} 0.1 & 0 & 0 \\ 0 & -0.375 & 0 \\ 0 & 0 & -0.916 \end{bmatrix} \left( \frac{\text{kN}}{\text{cm}^2} \right)$$

- c. tenzor deformacije

$$\varepsilon_x = \frac{1}{2 \cdot 10^4} [0.1 - 0.3(-0.375 - 0.916)] + 1.25 \cdot 10^{-5} \cdot 30 = 3.994 \cdot 10^{-4}$$

$$\varepsilon_y = \frac{1}{2 \cdot 10^4} [-0.375 - 0.3(0.1 - 0.916)] + 1.25 \cdot 10^{-5} \cdot 30 = 3.685 \cdot 10^{-4}$$

$$[D] = \begin{bmatrix} 3.994 & 0 & 0 \\ 0 & 3.685 & 0 \\ 0 & 0 & 3.333 \end{bmatrix} \cdot 10^{-4}$$

d. specifična i absolutna promjena zapremine

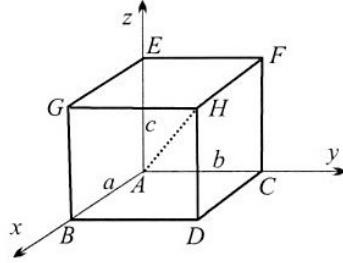
$$e = \varepsilon_x + \varepsilon_y + \varepsilon_z = (3.994 + 3.685 + 3.333) \cdot 10^{-4} = 11.012 \cdot 10^{-4}$$

$$\Delta V = e \cdot V = 11.012 \cdot 10^{-4} \cdot 100 \cdot 80 \cdot 3 = 26.429 \text{ cm}^3$$

5.

### ZADATAK

Dat je kvadar čije su ivice  $a=4 \text{ mm}$ ,  $b=3 \text{ mm}$  i  $c=5 \text{ mm}$ . Posle deformacije dužine ivica su  $a_i=3.997 \text{ mm}$ ,  $b_i=3.002 \text{ mm}$  i  $c_i=4.998 \text{ mm}$ , dok ugao  $\angle CAE$  prelazi u  $90^\circ 00' 35''$ .



Odrediti:

- a/ tenzor deformacije, glavne dilatacije, glavne pravce i prikazati deformaciju elementa u ravni  $yz$ ,
- b/ vrednost dilatacije u pravcu dijagonale  $AH$ , kao i promenu njene dužine,
- c/ tenzor napona ( $E = 200 \text{ Pa}$ ,  $\nu = 1/3$ ), glavne napone i glavne pravce i izdvojiti element na koji deluju,
- d/ komponentalne napone za ravan koja leži u  $yz$  ravni i čija normala sa  $x$  osom gradi ugao  $\varphi = -30^\circ$ ,
- e/ Mohr-ov krug napona.

### RJEŠENJE

a. tenzor deformacije

$$\varepsilon_x = \frac{3.997 - 4}{4} = -7.5 \cdot 10^{-4}$$

$$\varepsilon_y = \frac{3.002 - 3}{3} = 6.667 \cdot 10^{-4}$$

$$\varepsilon_z = \frac{4.998 - 5}{5} = -4 \cdot 10^{-4}$$

$$\gamma_{yz} = -35 \frac{1}{3600} \frac{\pi}{180} = -1.697 \cdot 10^{-4}$$

$$[D] = \begin{bmatrix} -7.5 & 0 & 0 \\ 0 & 6.667 & -\frac{1}{2}1.697 \\ 0 & -\frac{1}{2}1.697 & -4 \end{bmatrix} \cdot 10^{-4}$$

$$\varepsilon_1 = 6.7341 \cdot 10^{-4}$$

$$\varepsilon_2 = -4.0671 \cdot 10^{-5}$$

$$\varepsilon_3 = -7.5 \cdot 10^{-5}$$

$$\alpha = -4.52^0$$

b. dilatacija u pravcu  $AH$

$$\varepsilon_{AH} = \frac{\Delta AH}{AH} = \frac{AH' - AH}{AH}$$

$AH' \neq \sqrt{a_1^2 + b_1^2 + c_1^2}$  uglovi nijesu pravi

$$\cos \alpha_{AH} = \frac{4}{\sqrt{50}}; \quad \cos \beta_{AH} = \frac{3}{\sqrt{50}}; \quad \cos \gamma_{AH} = \frac{5}{\sqrt{50}}$$

$$\varepsilon_n = \varepsilon_x \cos^2 \alpha + \varepsilon_y \cos^2 \beta + \varepsilon_z \cos^2 \gamma + \gamma_{xy} \cos \alpha \cos \beta + \gamma_{yz} \cos \beta \cos \gamma + \gamma_{xz} \cos \alpha \cos \gamma$$

$$\varepsilon_{AH} = -7.5 \cdot 10^{-4} \cdot \left( \frac{4}{\sqrt{50}} \right)^2 + 6.667 \cdot 10^{-4} \cdot \left( \frac{3}{\sqrt{50}} \right)^2 - 4 \cdot 10^{-4} \cdot \left( \frac{5}{\sqrt{50}} \right)^2 - 1.697 \cdot 10^{-4} \cdot \frac{3}{\sqrt{50}} \cdot \frac{5}{\sqrt{50}} = -3.709 \cdot 10^{-4}$$

$$\Delta AH = -3.709 \cdot 10^{-4} \cdot AH = -3.709 \cdot 10^{-4} \cdot \sqrt{50} = -2.622 \cdot 10^{-3} \text{ mm}$$

c. tenzor napona

$$\mu = G = \frac{E}{2(1+\nu)} = \frac{200 \cdot 10^3}{2\left(1+\frac{1}{3}\right)} = 75 \cdot 10^3 \text{ MPa}$$

$$\lambda = \frac{\nu E}{(1+\nu)(1-2\nu)} = \frac{\frac{1}{3}200 \cdot 10^3}{\left(1+\frac{1}{3}\right)\left(1-2\frac{1}{3}\right)} = 150 \cdot 10^3 \text{ MPa}$$

$$e = \varepsilon_x + \varepsilon_y + \varepsilon_z = (-7.5 + 6.667 - 4) \cdot 10^{-4} = -4.833 \cdot 10^{-4}$$

$$\sigma_x = 2\mu\varepsilon_x + \lambda e = 2 \cdot 75 \cdot 10^3 \cdot (-7.5 \cdot 10^{-4}) + 150 \cdot 10^3 \cdot (-4.833 \cdot 10^{-4}) = -185 \text{ MPa}$$

$$\sigma_y = 2\mu\varepsilon_y + \lambda e = 2 \cdot 75 \cdot 10^3 \cdot 6.667 \cdot 10^{-4} + 150 \cdot 10^3 \cdot (-4.833 \cdot 10^{-4}) = 27.51 \text{ MPa}$$

$$\sigma_z = 2\mu\varepsilon_z + \lambda e = 2 \cdot 75 \cdot 10^3 \cdot (-4 \cdot 10^{-4}) + 150 \cdot 10^3 \cdot (-4.833 \cdot 10^{-4}) = -132.5 \text{ MPa}$$

$$\tau_{xy} = \mu\gamma_{xy} = 0$$

$$\tau_{yz} = \mu\gamma_{yz} = 75 \cdot 10^3 \cdot (-1.697 \cdot 10^{-4}) = -12.73 \text{ MPa}$$

$$\tau_{zx} = \mu\gamma_{zx} = 0$$

$$[S] = \begin{bmatrix} -185 & 0 & 0 \\ 0 & 27.5 & -12.73 \\ 0 & -12.73 & -132.5 \end{bmatrix} (\text{MPa})$$

d. naponi u ravni ( $\phi=-30^\circ$ )

$$\sigma_n = -1.469 \text{ MPa}$$

$$\tau_{nl} = -75.65 \text{ MPa}$$

## 6.

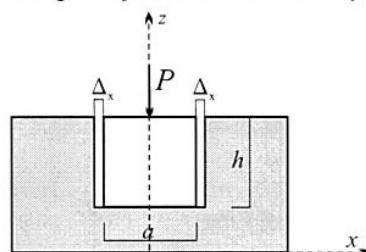
### ZADATAK

Kvadrat  $a \times b \times h = 30 \times 40 \times 20 \text{ cm}$  stavljen je u otvor u masivnoj ploči koju možemo smatrati nedeformabilnom. Na gornju površinu kvadra deluje sila pritiska  $P$  i kvadar se zagreva za  $\Delta t = 30^\circ C$ . Pre nanošenja opterećenja, između stranice kvadra i ivica otvora postojali su zazori u  $x$  i  $y$  pravcu  $\Delta_x = 0.021 \text{ mm}$  i  $\Delta_y = 0.054 \text{ mm}$ .

a/ Odrediti vrednost sile  $P$  iz uslova da se u  $x$  pravcu pojavi napon pritiska od  $5 \text{ MPa}$ .

b/ Napisati tenzor napona i deformacije u sistemu glavnih osa.

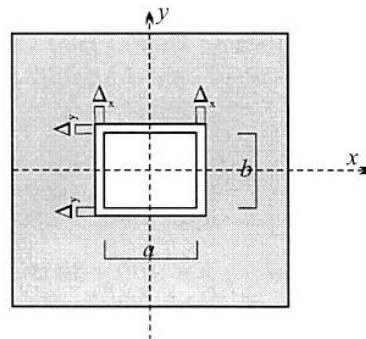
c/ Izračunati promenu dužina ivica kvadra kao i specifičnu i apsolutnu promenu zapremine.



$$E = 20 \text{ GPa},$$

$$\nu = 0.3$$

$$\alpha = 1.1 \cdot 10^{-5}$$



a. sila  $P$

$\sigma_x = -0.5 \frac{kN}{cm^2}$	$\varepsilon_x = \frac{2\Delta x}{a} = \frac{2 \cdot 0.00021}{30} = 1.4 \cdot 10^{-4}$
$\sigma_y \neq 0$	$\varepsilon_y = \frac{2\Delta y}{b} = \frac{2 \cdot 0.00054}{40} = 2.7 \cdot 10^{-4}$
$\sigma_z = -\frac{P}{a \cdot b}$	$\varepsilon_z \neq 0$

$$\left. \begin{array}{l} \varepsilon_x = \frac{1}{20 \cdot 10^2} [-0.5 - 0.3(\sigma_y + \sigma_z)] + 1.1 \cdot 10^{-5} \cdot 30 = 1.4 \cdot 10^{-4} \\ \varepsilon_y = \frac{1}{20 \cdot 10^2} [\sigma_y - 0.3(-0.5 + \sigma_z)] + 1.1 \cdot 10^{-5} \cdot 30 = 2.7 \cdot 10^{-4} \end{array} \right\} \rightarrow \begin{array}{l} \sigma_y = -0.3 \frac{kN}{cm^2} \\ \sigma_z = -0.1 \frac{kN}{cm^2} \end{array}$$

$$\sigma_z = -\frac{P}{a \cdot b} = -\frac{P}{40 \cdot 30} = -0.1 \rightarrow P = 120 \text{ kN}$$

b. naponi i deformacije u sistemu glavnih osa

$$[S] = \begin{bmatrix} -0.5 & 0 & 0 \\ 0 & -0.3 & 0 \\ 0 & 0 & -0.1 \end{bmatrix} \left( \frac{kN}{cm^2} \right)$$

$$\varepsilon_z = \frac{1}{20 \cdot 10^2} [-0.1 - 0.3(-0.5 - 0.3)] + 1.1 \cdot 10^{-5} \cdot 30 = 4 \cdot 10^{-4}$$

$$[D] = \begin{bmatrix} 1.4 & 0 & 0 \\ 0 & 2.7 & 0 \\ 0 & 0 & 4 \end{bmatrix} \cdot 10^{-4}$$

c. promjena dužina stranica kvadra, specifična i absolutna promjena zapremine

$$\Delta a = a \cdot \varepsilon_x = 30 \cdot 1.4 \cdot 10^{-4} = 4.2 \cdot 10^{-3} \text{ cm}$$

$$\Delta b = b \cdot \varepsilon_y = 40 \cdot 2.7 \cdot 10^{-4} = 10.8 \cdot 10^{-3} \text{ cm}$$

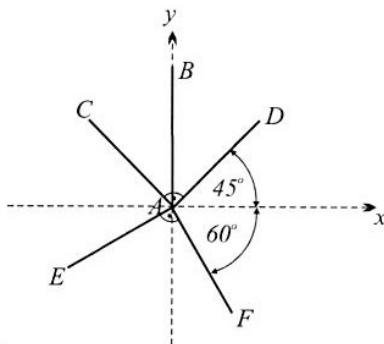
$$\Delta h = h \cdot \varepsilon_z = 20 \cdot 4 \cdot 10^{-4} = 8 \cdot 10^{-3} \text{ cm}$$

$$e = \varepsilon_x + \varepsilon_y + \varepsilon_z = (1.4 + 2.7 + 4) \cdot 10^{-4} = 8.1 \cdot 10^{-4}$$

$$\Delta V = e \cdot V = 8.1 \cdot 10^{-4} \cdot 20 \cdot 30 \cdot 40 = 19.44 \text{ cm}^3$$

**ZADATAK:**

Na slici prikazana je duž  $AB$  od 5 cm, kao i pravi uglovi definisani oznakama  $\angle CAD$  i  $\angle EAF$ . Posle deformacije rastojanje između tačaka  $A$  i  $B$  postalo je 5.0001 cm, dok su početni pravi uglovi  $\angle CAD$  i  $\angle EAF$  prešli u  $90^\circ 00' 14''$ , odnosno  $89^\circ 59' 52''$ .



Odrediti:

- tenzor ravnog stanja deformacije, glavne dilatacije, glavne pravce i prikazati deformaciju elementa u ravni  $xy$ ,
- vrednost modula elastičnosti  $E$ , ako je  $\sigma_x = 24.667 \text{ MPa}$  i  $\nu = 0.3$ ,
- tenzor napona, glavne napone i glavne pravce i izdvojiti element na koji deluju.

**RJEŠENJE**

a. tenzor deformacije...

$$\varphi = -60^\circ \quad \gamma_{nl} = \frac{8}{3600} \frac{\pi}{180} = 3.8785 \cdot 10^{-5}$$

$$\varphi = 135^\circ \quad \gamma_{nl} = -\frac{14}{3600} \frac{\pi}{180} = -6.787 \cdot 10^{-5}$$

$$\varepsilon_y = \frac{5.0001 - 5}{5} = 2 \cdot 10^{-5}$$

$$\frac{1}{2} \gamma_{nl} = \frac{1}{2} (\varepsilon_x - \varepsilon_y) \sin 2\varphi - \frac{1}{2} \gamma_{xy} \cos 2\varphi$$

$$\left. \begin{aligned} \frac{1}{2} 3.8785 \cdot 10^{-5} &= \frac{1}{2} (\varepsilon_x - 2) \cdot 10^{-5} \sin(2 \cdot (-60^\circ)) - \frac{1}{2} \gamma_{xy} \cdot 10^{-5} \cos(2 \cdot (-60^\circ)) \\ \frac{1}{2} (-6.787 \cdot 10^{-5}) &= \frac{1}{2} (\varepsilon_x - 2) \cdot 10^{-5} \sin(2 \cdot 135^\circ) - \frac{1}{2} \gamma_{xy} \cdot 10^{-5} \cos(2 \cdot 135^\circ) \end{aligned} \right\} \rightarrow \begin{aligned} \varepsilon_x &= 8.787 \cdot 10^{-5} \\ \frac{1}{2} \gamma_{xy} &= 9.7562 \cdot 10^{-5} \end{aligned}$$

$$[D] = \begin{bmatrix} \varepsilon_x & \frac{1}{2} \gamma_{xy} \\ \frac{1}{2} \gamma_{yx} & \varepsilon_y \end{bmatrix} = \begin{bmatrix} 8.787 & 9.7562 \\ 9.7562 & 2 \end{bmatrix} \cdot 10^{-5} \text{ tenzor ravnog stanja deformacije u tački}$$

$$\varepsilon_1 = 15.723 \cdot 10^{-5}$$

$$\varepsilon_2 = -4.936 \cdot 10^{-5}$$

$$\alpha = 35.41^\circ$$

b. modul elastičnosti  $E$

$$\sigma_x = 2\mu\varepsilon_x + \lambda e$$

$$\mu = G = \frac{E}{2(1+\nu)} = \frac{E}{2(1+0.3)}$$

$$\lambda = \frac{\nu E}{(1+\nu)(1-2\nu)} = \frac{0.3 \cdot E}{(1+0.3)(1-2 \cdot 0.3)}$$

$$e = \varepsilon_x + \varepsilon_y + \varepsilon_z = (8.787 + 2) \cdot 10^{-5} = 10.787 \cdot 10^{-5}$$

$$\sigma_x = 24.667 = 2 \cdot \frac{E \cdot 10^3}{2(1+0.3)} \cdot 8.787 \cdot 10^{-5} + \frac{0.3 \cdot E \cdot 10^3}{(1+0.3)(1-2 \cdot 0.3)} \cdot 10.787 \cdot 10^{-5} \rightarrow E = 190 \text{ GPa}$$

c. tenzor napona...

$$\mu = \frac{190}{2(1+0.3)} = 73.077 \text{ GPa}$$

$$\lambda = \frac{0.3 \cdot 190}{(1+0.3)(1-2 \cdot 0.3)} = 109.615 \text{ GPa}$$

$$e = \varepsilon_x + \varepsilon_y + \varepsilon_z = (8.787 + 2) \cdot 10^{-5} = 10.787 \cdot 10^{-5}$$

$$\sigma_x = 24.667 \text{ MPa}$$

$$\sigma_y = 2\mu\varepsilon_y + \lambda e = 2 \cdot 73.077 \cdot 10^3 \cdot 2 \cdot 10^{-5} + 109.615 \cdot 10^3 \cdot 10.787 \cdot 10^{-5} = 14.747 \text{ MPa}$$

$$\sigma_z = 2\mu\varepsilon_z + \lambda e = 2 \cdot 73.077 \cdot 10^3 \cdot 0 + 109.615 \cdot 10^3 \cdot 10.787 \cdot 10^{-5} = 11.824 \text{ MPa}$$

$$\tau_{xy} = \mu\gamma_{xy} = 73.077 \cdot 10^3 \cdot (2 \cdot 9.7562 \cdot 10^{-5}) = 14.259 \text{ MPa}$$

$$\tau_{yz} = \mu\gamma_{yz} = 0$$

$$\tau_{zx} = \mu\gamma_{zx} = 0$$

$$[S] = \begin{bmatrix} 24.667 & 14.259 & 0 \\ 14.259 & 14.747 & 0 \\ 0 & 0 & 11.824 \end{bmatrix} (\text{MPa}) \text{ tenzor } \underline{\text{prostornog stanja napona}} \text{ u tački}$$

$$\sigma_1 = 34.804 \text{ MPa}$$

$$\sigma_2 = 11.824 \text{ MPa}$$

$$\sigma_3 = 4.610 \text{ MPa}$$

$$\alpha = 35.41^\circ$$

$$\boxed{\alpha_\varepsilon = \alpha_\sigma = 35.41^\circ}$$

**ZADATAK:**

Telo na slici se zagreva za  $\Delta t$  i opterećeno je koncentrisanom silom  $P=350 \text{ kN}$ .

a/ Odrediti vrednost temperature  $\Delta t$  ako je najveća dozvoljena pozitivna dilatacija  $\varepsilon = +2.5 \cdot 10^{-4}$ , pod uslovom:  
1/ da se telo može slobodno deformisati,

2/ da je telo upasovano u otvor sa krutim glatkim ivicama na rastojanju  $\Delta x = 2 \cdot 10^{-4} \text{ cm}$ .

Zatim za oba slučaja:

b/ napisati tenzore napona i deformacija u sistemu glavnih osa, 1/

c/ odrediti promene dimenzija bloka  $l_x, l_y$  i  $l_z$ , kao i promenu površine poprečnog presjeka  $A$ ,

c/ izračunati specifične i absolutne promene zapremine.

$$E = 2 \cdot 10^4 \text{ kN/cm}^2,$$

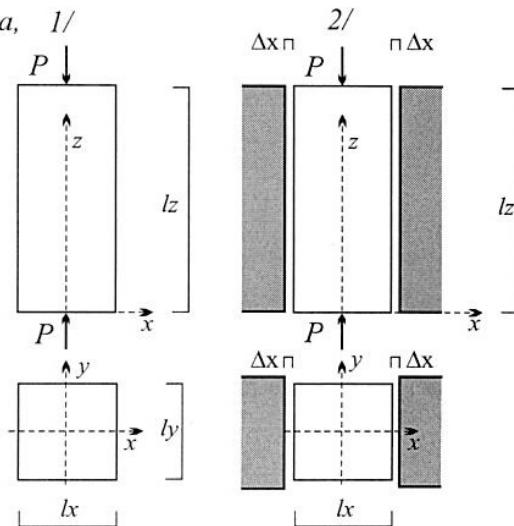
$$\nu = 0.2$$

$$\alpha = 1.2 \cdot 10^{-5}$$

$$l_z = 100 \text{ cm}$$

$$l_x = 5 \text{ cm}$$

$$l_y = 7 \text{ cm}$$



**RJEŠENJE**

**Slučaj 1/**

a. temperaturna promjena  $\Delta t$

$\sigma_x = 0$	$\varepsilon_x = 2.5 \cdot 10^{-4}$
$\sigma_y = 0$	$\varepsilon_y = 2.5 \cdot 10^{-4}$
$\sigma_z = -\frac{P}{l_x \cdot l_y} = -\frac{350}{5 \cdot 7} = -10 \frac{\text{kN}}{\text{cm}^2}$	$\varepsilon_z \neq 0$

$$\varepsilon_y = \varepsilon_y = -\nu \frac{\sigma_z}{E} + \alpha \cdot \Delta t = -0.2 \frac{-10}{2 \cdot 10^4} + 1.2 \cdot 10^{-5} \cdot \Delta t = 2.5 \cdot 10^{-4} \rightarrow \Delta t = 12.5 \text{ } ^\circ\text{C}$$

b. tenzori napona i deformacija

$$[S] = \begin{bmatrix} 0 & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & -10 \end{bmatrix} \left( \frac{\text{kN}}{\text{cm}^2} \right)$$

$$\varepsilon_z = \frac{\sigma_z}{E} + \alpha \cdot \Delta t = \frac{-10}{2 \cdot 10^4} + 1.2 \cdot 10^{-5} \cdot 12.5 = -3.5 \cdot 10^{-4}$$

$$[D] = \begin{bmatrix} 2.5 & 0 & 0 \\ 0 & 2.5 & 0 \\ 0 & 0 & -3.5 \end{bmatrix} \cdot 10^{-4}$$

c. promjena dimenzija bloka i promjena površine poprečnog presjeka bloka

$$\Delta l_x = l_x \cdot \varepsilon_x = 5 \cdot 2.5 \cdot 10^{-4} = 12.5 \cdot 10^{-4} \text{ cm}$$

$$\Delta l_y = l_y \cdot \varepsilon_y = 7 \cdot 2.5 \cdot 10^{-4} = 17.5 \cdot 10^{-4} \text{ cm}$$

$$\Delta l_z = l_z \cdot \varepsilon_z = 100 \cdot (-3.5) \cdot 10^{-4} = -0.035 \text{ cm}$$

$$\Delta A = (\varepsilon_x + \varepsilon_y) \cdot A = (2.5 + 2.5) \cdot 10^{-4} \cdot 5 \cdot 7 = 0.0175 \text{ cm}^2$$

d. specifična i apsolutna promjena zapremine bloka

$$e = \varepsilon_x + \varepsilon_y + \varepsilon_z = (2.5 + 2.5 - 3.5) \cdot 10^{-4} = 1.5 \cdot 10^{-4}$$

$$\Delta V = e \cdot V = 1.5 \cdot 10^{-4} \cdot 5 \cdot 7 \cdot 100 = 0.525 \text{ cm}^3$$

### Slučaj 2/

a. temperaturna promjena  $\Delta t$

$\varepsilon_x = \frac{2\Delta x}{l_x} = \frac{2 \cdot 2 \cdot 10^{-4}}{5} = 0.8 \cdot 10^{-4} < 2.5 \cdot 10^{-4} \rightarrow \sigma_x \neq 0$	
$\varepsilon_y = 2.5 \cdot 10^{-4}$	$\sigma_y = 0$
$\varepsilon_z \neq 0$	$\sigma_z = -\frac{P}{l_x \cdot l_y} = -\frac{350}{5 \cdot 7} = -10 \frac{kN}{cm^2}$

$$\left. \begin{aligned} \varepsilon_x &= \frac{1}{E} [\sigma_x - \nu(\sigma_y + \sigma_z)] + \alpha_t \Delta t = \frac{1}{2 \cdot 10^4} [\sigma_x - 0.2(0 - 10)] + 1.2 \cdot 10^{-5} \cdot \Delta t = 0.8 \cdot 10^{-4} \\ \varepsilon_y &= \frac{1}{E} [\sigma_y - \nu(\sigma_z + \sigma_x)] + \alpha_t \Delta t = \frac{1}{2 \cdot 10^4} [0 - 0.2(\sigma_x - 10)] + 1.2 \cdot 10^{-5} \cdot \Delta t = 2.5 \cdot 10^{-4} \end{aligned} \right\} \rightarrow \begin{aligned} \sigma_x &= -2.8333 \frac{kN}{cm^2} \\ \Delta t &= 10.138^0 \end{aligned}$$

b. tenzori napona i deformacija

$$[S] = \begin{bmatrix} -2.833 & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & -10 \end{bmatrix} \left( \frac{kN}{cm^2} \right)$$

$$\varepsilon_z = \frac{1}{E} (\sigma_z - \nu \cdot \sigma_x) + \alpha_t \Delta t = \frac{1}{2 \cdot 10^4} [-10 - 0.2 \cdot (-2.8333)] + 1.2 \cdot 10^{-5} \cdot 10.138 = -3.5 \cdot 10^{-4}$$

$$[D] = \begin{bmatrix} 0.8 & 0 & 0 \\ 0 & 2.5 & 0 \\ 0 & 0 & -3.5 \end{bmatrix} \cdot 10^{-4}$$

c. promjena dimenzija bloka i promjena površine poprečnog presjeka bloka

$$\Delta l_x = l_x \cdot \varepsilon_x = 5 \cdot 0.8 \cdot 10^{-4} = 4 \cdot 10^{-4} \text{ cm}$$

$$\Delta l_y = l_y \cdot \varepsilon_y = 7 \cdot 2.5 \cdot 10^{-4} = 17.5 \cdot 10^{-4} \text{ cm}$$

$$\Delta l_z = l_z \cdot \varepsilon_z = 100 \cdot (-3.5) \cdot 10^{-4} = -0.035 \text{ cm}$$

$$\Delta A = (\varepsilon_x + \varepsilon_y) \cdot A = (0.8 + 2.5) \cdot 10^{-4} \cdot 5 \cdot 7 = 0.01155 \text{ cm}^2$$

d. specifična i apsolutna promjena zapremine bloka

$$e = \varepsilon_x + \varepsilon_y + \varepsilon_z = (0.8 + 2.5 - 3.5) \cdot 10^{-4} = -0.2 \cdot 10^{-4}$$

$$\Delta V = e \cdot V = (-0.2) \cdot 10^{-4} \cdot 5 \cdot 7 \cdot 100 = -0.07 \text{ cm}^3$$

**ZADATAK:**

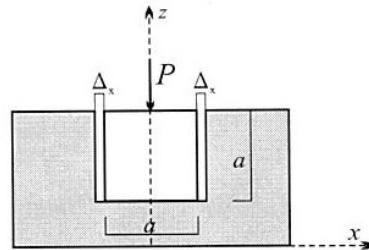
Kocka stranice  $a = 25 \text{ cm}$  stavljen je u otvor u masivnoj ploči koju možemo smatrati nedeformabilnom.

Na gornju površinu kocke deluje sila pritiska  $P$  i kocka se zagreva za  $\Delta t = 15^\circ\text{C}$ . Pre nanošenja opterećenja, između stranice kocke i ivica otvora postojao je u  $x$  pravcu zazor  $\Delta_x = 1.5 \cdot 10^{-3} \text{ cm}$ .

- a/ Odrediti vrednost sile  $P$  iz uslova da se u  $x$  pravcu pojavi napon pritiska od  $1 \text{ kN/cm}^2$ .

- b/ Napisati tenzor napona i deformacije u sistemu glavnih osa.

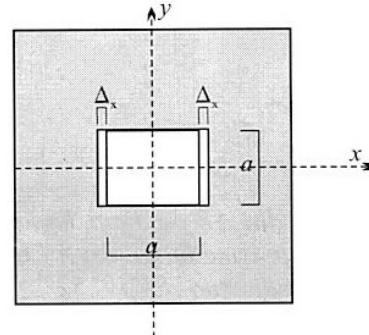
- c/ Izračunati specifičnu i apsolutnu promenu zapremine.



$$E = 10^4 \text{ kN/cm}^2,$$

$$\nu = 0.2$$

$$\alpha = 1.1 \cdot 10^{-5}$$



**RJEŠENJE**

- a. sila  $P$

$\sigma_x = -1 \frac{\text{kN}}{\text{cm}^2}$	$\varepsilon_x = \frac{2\Delta x}{a} = \frac{2 \cdot 1.5 \cdot 10^{-3}}{25} = 1.2 \cdot 10^{-4}$
$\sigma_y \neq 0$	$\varepsilon_y = 0$
$\sigma_z = -\frac{P}{a^2}$	$\varepsilon_z \neq 0$

$$\left. \begin{aligned} \varepsilon_x &= \frac{1}{10^4} \left[ -1 - 0.2(\sigma_y + \sigma_z) \right] + 1.1 \cdot 10^{-5} \cdot 15 = 1.2 \cdot 10^{-4} \\ \varepsilon_y &= \frac{1}{10^4} \left[ \sigma_y - 0.2(-1 + \sigma_z) \right] + 1.1 \cdot 10^{-5} \cdot 15 = 0 \end{aligned} \right\} \rightarrow \begin{aligned} \sigma_y &= -2 \frac{\text{kN}}{\text{cm}^2} \\ \sigma_z &= -0.75 \frac{\text{kN}}{\text{cm}^2} \end{aligned}$$

$$\sigma_z = -\frac{P}{a \cdot a} = -\frac{P}{25 \cdot 25} = -0.75 \rightarrow P = 468.75 \text{ kN}$$

- b. naponi i deformacije u sistemu glavnih osa

$$[S] = \begin{bmatrix} -1 & 0 & 0 \\ 0 & -2 & 0 \\ 0 & 0 & -0.75 \end{bmatrix} \left( \frac{\text{kN}}{\text{cm}^2} \right) \text{ - prostorno stanje napona}$$

$$\varepsilon_z = \frac{1}{10^4} \left[ -0.75 - 0.2(-1 - 2) \right] + 1.1 \cdot 10^{-5} \cdot 15 = 1.5 \cdot 10^{-4}$$

$$[D] = \begin{bmatrix} 1.2 & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & 1.5 \end{bmatrix} \cdot 10^{-4} \text{ - ravno stanje deformacije}$$

c. specifična i absolutna promjena zapremine

$$\varepsilon = \varepsilon_x + \varepsilon_y + \varepsilon_z = (1.2 + 1.5) \cdot 10^{-4} = 2.7 \cdot 10^{-4}$$

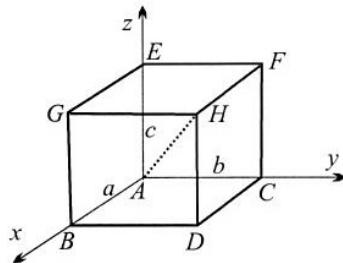
$$\Delta V = e \cdot V = 2.7 \cdot 10^{-4} \cdot 25^3 = 4.22 \text{ cm}^3$$

**10.**

### ZADATAK

a/ Dat je kvadar čije su ivice  $a=4 \text{ mm}$ ,  $b=2 \text{ mm}$  i  $c=3 \text{ mm}$ . Posle deformacije dužine ivica su  $a_i=4.004 \text{ mm}$ ,  $b_i=2.0006 \text{ mm}$  i  $c_i=2.9985 \text{ mm}$ . Uglovi ostaju nepromjenjeni.

b/ Dat je kvadar čije dužine ivica stoje u odnosu  $a : b : c = 2 : 3 : 5$ . Kvadar se tako deformeša da dužine ivica ostaju nepromjenjene, a pravi uglovi prelaze u:



$$\angle BAC = 90^\circ 00'03''$$

$$\angle CAE = 89^\circ 59'54''$$

$$\angle EAB = 89^\circ 59'56''$$

Za dva naznačena slučaja:

1/ Napisati tenzore napona i deformacije,

2/ Odrediti vrednost normalnog napona o dilatacije u pravcu dijagonale AH.

**RJEŠENJE**

### Slučaj 1/

a. tenzori napona i deformacija

$$\varepsilon_x = \frac{4.004 - 4}{4} = 1 \cdot 10^{-3}$$

$$\varepsilon_y = \frac{2.0006 - 2}{2} = 0.3 \cdot 10^{-3} \quad \rightarrow \quad [D] = \begin{bmatrix} 1 & 0 & 0 \\ 0 & 0.3 & 0 \\ 0 & 0 & -0.5 \end{bmatrix} \cdot 10^{-3}$$

$$\varepsilon_z = \frac{2.9985 - 3}{3} = -0.5 \cdot 10^{-3}$$

$$\mu = G = \frac{E}{2(1+\nu)} = \frac{200 \cdot 10^3}{2\left(1+\frac{1}{3}\right)} = 75 \cdot 10^3 \text{ MPa}$$

$$\lambda = \frac{\nu E}{(1+\nu)(1-2\nu)} = \frac{\frac{1}{3}200 \cdot 10^3}{\left(1+\frac{1}{3}\right)\left(1-2\frac{1}{3}\right)} = 150 \cdot 10^3 \text{ MPa}$$

$$e = \varepsilon_x + \varepsilon_y + \varepsilon_z = (1 + 0.3 - 0.5) \cdot 10^{-3} = 0.8 \cdot 10^{-3}$$

$$\sigma_x = 2\mu\varepsilon_x + \lambda e = 2 \cdot 75 \cdot 10^3 \cdot 1 \cdot 10^{-3} + 150 \cdot 10^3 \cdot 0.8 \cdot 10^{-3} = 270 \text{ MPa}$$

$$\sigma_y = 2\mu\varepsilon_y + \lambda e = 2 \cdot 75 \cdot 10^3 \cdot 0.3 \cdot 10^{-3} + 150 \cdot 10^3 \cdot 0.8 \cdot 10^{-3} = 165 \text{ MPa}$$

$$\sigma_z = 2\mu\varepsilon_z + \lambda e = 2 \cdot 75 \cdot 10^3 \cdot (-0.5 \cdot 10^{-3}) + 150 \cdot 10^3 \cdot 0.8 \cdot 10^{-3} = 45 \text{ MPa}$$

$$\tau_{xy} = \mu\gamma_{xy} = 0$$

$$\tau_{yz} = \mu\gamma_{yz} = 0$$

$$\tau_{zx} = \mu\gamma_{zx} = 0$$

$$[S] = \begin{bmatrix} 270 & 0 & 0 \\ 0 & 165 & 0 \\ 0 & 0 & 45 \end{bmatrix} (MPa)$$

b. normalni napon i dilatacija u pravcu AH

$$\varepsilon_{AH} = \frac{\Delta AH}{AH} = \frac{AH' - AH}{AH}$$

$$AH = \sqrt{4^2 + 2^2 + 3^2} = \sqrt{29}$$

$$AH' = \sqrt{4.004^2 + 2.0006^2 + 2.9985^2} = \sqrt{29.0254186}$$

$$\varepsilon_{AH} = \frac{\sqrt{29.0254186} - \sqrt{29}}{\sqrt{29}} = 0.438 \cdot 10^{-3}$$

$$\cos \alpha_{AH} = \frac{4}{\sqrt{29}}; \quad \cos \beta_{AH} = \frac{2}{\sqrt{29}}; \quad \cos \gamma_{AH} = \frac{3}{\sqrt{29}}$$

$$\varepsilon_n = \varepsilon_x \cos^2 \alpha + \varepsilon_y \cos^2 \beta + \varepsilon_z \cos^2 \gamma + \gamma_{xy} \cos \alpha \cos \beta + \gamma_{yz} \cos \beta \cos \gamma + \gamma_{xz} \cos \alpha \cos \gamma$$

$$\varepsilon_{AH} = 1 \cdot 10^{-3} \cdot \left( \frac{4}{\sqrt{29}} \right)^2 + 0.3 \cdot 10^{-3} \cdot \left( \frac{2}{\sqrt{29}} \right)^2 - 0.5 \cdot 10^{-3} \cdot \left( \frac{3}{\sqrt{29}} \right)^2 = 0.438 \cdot 10^{-3}$$

$$\sigma_n = \sigma_x \cos^2 \alpha + \sigma_y \cos^2 \beta + \sigma_z \cos^2 \gamma + 2\tau_{xy} \cos \alpha \cos \beta + 2\tau_{yz} \cos \beta \cos \gamma + 2\tau_{xz} \cos \alpha \cos \gamma$$

$$\sigma_{AH} = 270 \cdot \left( \frac{4}{\sqrt{29}} \right)^2 + 165 \cdot \left( \frac{2}{\sqrt{29}} \right)^2 + 45 \cdot \left( \frac{3}{\sqrt{29}} \right)^2 = 214.14 MPa$$

## Slučaj 2/

a. tenzori napona i deformacija

$$\gamma_{xy} = -3'' = -\frac{3}{3600} \frac{\pi}{180} rad = -0.145 \cdot 10^{-4} \quad (\mu = 75 \cdot 10^3 MPa)$$

$$\gamma_{yz} = 6'' = \frac{6}{3600} \frac{\pi}{180} rad = 0.294 \cdot 10^{-4} \quad \rightarrow \quad \tau_{xy} = \mu\gamma_{xy} = -1.0875 MPa$$

$$\gamma_{xz} = 4'' = \frac{4}{3600} \frac{\pi}{180} rad = 0.194 \cdot 10^{-4} \quad \tau_{yz} = \mu\gamma_{yz} = 2.1828 MPa$$

$$\tau_{zx} = \mu\gamma_{zx} = 1.455 MPa$$

$$[D] = \begin{bmatrix} 0 & -0.725 & 0.97 \\ -0.725 & 0 & 1.455 \\ 0.97 & 1.455 & 0 \end{bmatrix} \cdot 10^{-5}$$

$$[S] = \begin{bmatrix} 0 & -1.0875 & 1.455 \\ -1.0875 & 0 & 2.1825 \\ 1.455 & 2.1825 & 0 \end{bmatrix} (MPa)$$

b. normalni napon i dilatacija u pravcu  $AH$

$$a = 2x \quad b = 3x \quad c = 5x$$

$$AH = \sqrt{(2x)^2 + (3x)^2 + (5x)^2} = x\sqrt{38}$$

$$\cos \alpha_{AH} = \frac{2x}{x\sqrt{38}} = \frac{2}{\sqrt{38}}; \quad \cos \beta_{AH} = \frac{3}{\sqrt{38}}; \quad \cos \gamma_{AH} = \frac{5}{\sqrt{38}}$$

$$\varepsilon_n = \varepsilon_x \cos^2 \alpha + \varepsilon_y \cos^2 \beta + \varepsilon_z \cos^2 \gamma + \gamma_{xy} \cos \alpha \cos \beta + \gamma_{yz} \cos \beta \cos \gamma + \gamma_{xz} \cos \alpha \cos \gamma$$

$$\varepsilon_{AH} = (-0.145) \cdot 10^{-4} \cdot \frac{2}{\sqrt{38}} \frac{3}{\sqrt{38}} + 0.291 \cdot 10^{-4} \cdot \frac{3}{\sqrt{38}} \frac{5}{\sqrt{38}} + 0.194 \cdot 10^{-4} \cdot \frac{2}{\sqrt{38}} \frac{5}{\sqrt{38}} = 0.143 \cdot 10^{-3}$$

$$\sigma_n = \sigma_x \cos^2 \alpha + \sigma_y \cos^2 \beta + \sigma_z \cos^2 \gamma + 2\tau_{xy} \cos \alpha \cos \beta + 2\tau_{yz} \cos \beta \cos \gamma + 2\tau_{xz} \cos \alpha \cos \gamma$$

$$\sigma_{AH} = 2 \cdot (-1.0875) \cdot \frac{2}{\sqrt{38}} \frac{3}{\sqrt{38}} + 2 \cdot 2.1825 \cdot \frac{3}{\sqrt{38}} \frac{5}{\sqrt{38}} + 2 \cdot 1.455 \cdot \frac{2}{\sqrt{38}} \frac{5}{\sqrt{38}} = 2.145 \text{ MPa}$$

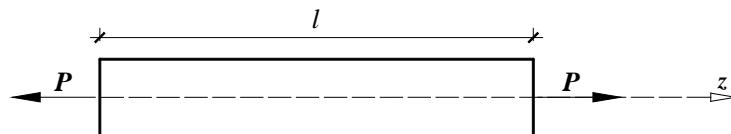
# VJEŽBA BR. 9

## AKSIJALNO NAPREZANJE I

- I. Štep dužina  $l=1m$ , pravougaonog poprečnog presjeka dimenzija  $4 \times 5cm$  opterećen je silom zatezanja  $P=250 kN$ . ( $E=20 MN/cm^2$ ;  $\nu=0.3$ )

Odrediti:

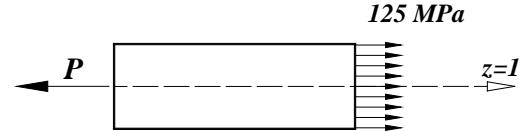
- Napone i deformaciju štapa i ukupno izduženje štapa;
- Promjenu zapremine i površine poprečnog presjeka štapa;
- Komponentalne napone u tačkama koje se nalaze u presječnoj ravni štapa koja zaklapa ugao  $60^0$  sa osom  $z$ ;
- Maksimalni smičući napon i ravan u kojoj djeluje;
- Mohr*-ov krug napona.



**RJEŠENJE**

- a. Naponi:

$$\sigma_z = \frac{P}{A} = \frac{250}{4 \cdot 5} = 12.5 \frac{kN}{cm^2} \quad [S] = \begin{bmatrix} 0 & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & 12.5 \end{bmatrix} \left[ \frac{kN}{cm^2} \right] - \text{Tenzor napona}$$



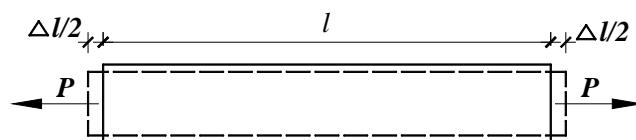
Deformacije:

$$\varepsilon_z = \frac{\sigma_z}{E} = \frac{12.5}{20 \cdot 10^3} = 6.25 \cdot 10^{-4} \quad [D] = \begin{bmatrix} -1.875 & 0 & 0 \\ 0 & -1.875 & 0 \\ 0 & 0 & 6.25 \end{bmatrix} \cdot 10^{-4} - \text{Tenzor deformacija}$$

$$\varepsilon_x = \varepsilon_y = -\nu \cdot \varepsilon_z = -0.3 \cdot 6.25 \cdot 10^{-4} = -1.875 \cdot 10^{-4}$$

Ukupno izduženje štapa:

$$\Delta l = \frac{P \cdot l}{E \cdot A} = \frac{250 \cdot 100}{20 \cdot 10^3 \cdot 20} = 0.0625 \text{ cm} \quad \text{ili} \quad \Delta l = \varepsilon_z \cdot l = 6.25 \cdot 10^{-4} \cdot 100 = 0.0625 \text{ cm}$$



- b. Promjena zapremine:

$$\Delta V = e \cdot V = (\varepsilon_x + \varepsilon_y + \varepsilon_z) \cdot V = (-1.875 - 1.875 + 6.25) \cdot 10^{-4} \cdot 4 \cdot 5 \cdot 100 = 0.5 \text{ cm}^3 - \text{povećala se zapremina}$$

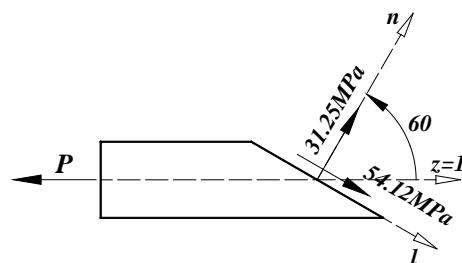
Promjena površine poprečnog presjeka:

$$\Delta A = (\varepsilon_x + \varepsilon_y) \cdot A = (-1.875 - 1.875) \cdot 10^{-4} \cdot 4 \cdot 5 = -0.0075 \text{ cm}^2 - \text{smanjila se površina pp}$$

c.

$$\sigma_n = \frac{1}{2} \sigma_z (1 + \cos 2\varphi) = \frac{1}{2} 12.5 (1 + \cos 2 \cdot 60^0) = 3.125 \frac{kN}{cm^2}$$

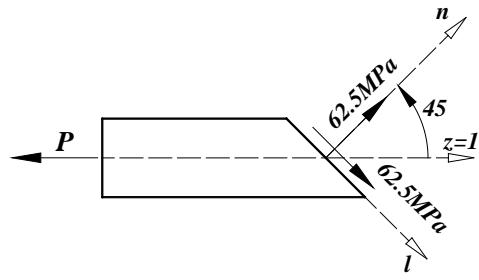
$$\tau_{nl} = \frac{1}{2} \sigma_z \sin 2\varphi = \frac{1}{2} 12.5 \sin 2 \cdot 60^0 = 5.412 \frac{kN}{cm^2}$$



d.

$$\tau_{\max} = \frac{1}{2}\sigma_z = \frac{1}{2}12.5 = 6.25 \frac{kN}{cm^2}$$

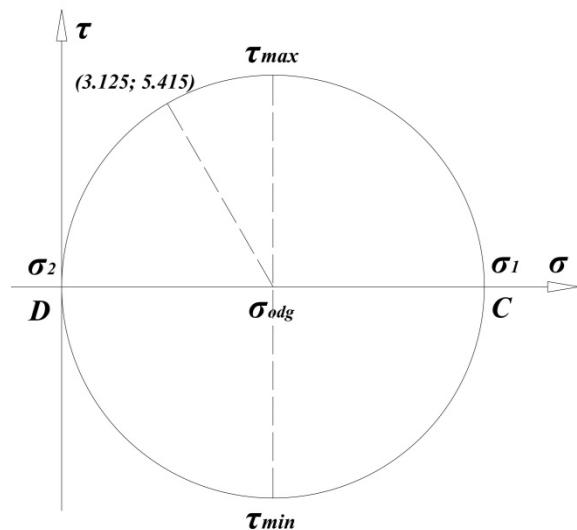
$$\sigma_{odg} = \frac{1}{2}\sigma_z = \frac{1}{2}12.5 = 6.25 \frac{kN}{cm^2}$$



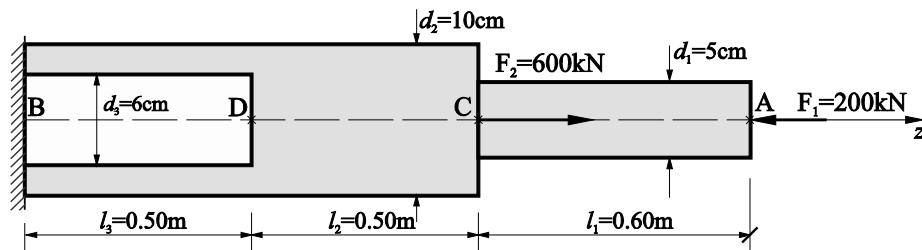
e.

$$C[\sigma_z; -\tau_{yz}] = [12.5; 0] \frac{kN}{cm^2}$$

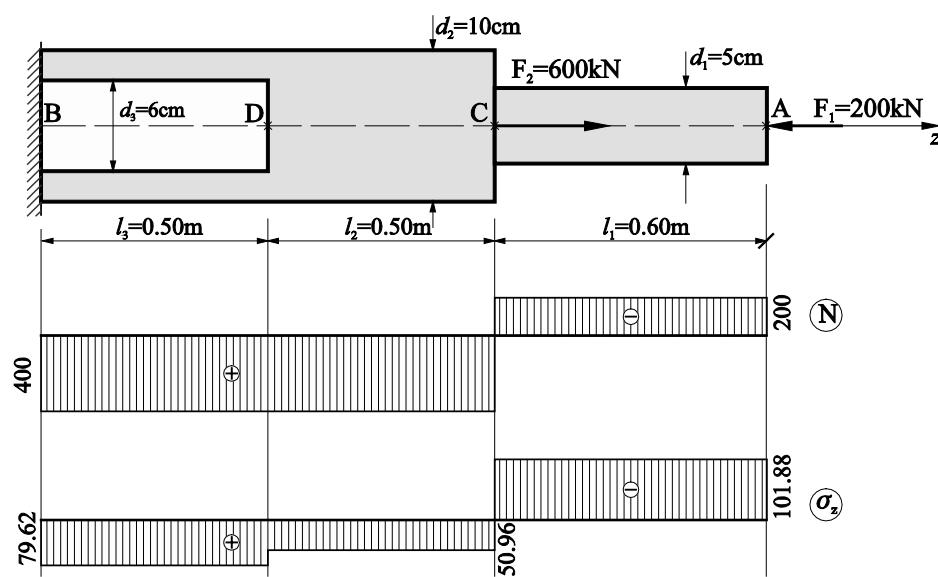
$$D[\sigma_y; \tau_{yz}] = [0; 0] \frac{kN}{cm^2}$$



2. Za nosač na slici nacrtati dijagram normalnih napona duž štapa, sračunati ukupnu promjenu dužine štapa i nacrtati dijagram normalnih napona u presjecima **B** i **Cdesno**. Štap je izrađen od čelika sa modulom elastičnosti  $E = 2.1 \cdot 10^5 \text{ MPa}$ . Štap je skokovitog kružnog i prstenastog poprečnog presjeka sa prečnicima presjeka datim na slici.



**RJEŠENJE**



Dijagram normalnih sila - presječne normalne sile u svim karakterističnim presjecima štapa:

$$\textbf{Dio } AC: \quad N_{AC} = -F_1 = -200 \text{ kN}$$

$$\textbf{Dio } CD: \quad N_{CD} = -F_1 + F_2 = 400 \text{ kN}$$

$$\textbf{Dio } DB: \quad N_{DB} = N_{CD} = 400 \text{ kN}$$

Površine poprečnih presjeka:

$$\textbf{Dio } AC: \quad A_{AC} = \frac{d_1^2 \pi}{4} = \frac{5^2 \pi}{4} = 19.63 \text{ cm}^2$$

$$\textbf{Dio } CD: \quad A_{CD} = \frac{d_2^2 \pi}{4} = \frac{10^2 \pi}{4} = 78.50 \text{ cm}^2$$

$$\textbf{Dio } DB: \quad A_{DB} = \frac{d_2^2 \pi}{4} - \frac{d_3^2 \pi}{4} = \frac{10^2 \pi}{4} - \frac{6^2 \pi}{4} = 50.24 \text{ cm}^2$$

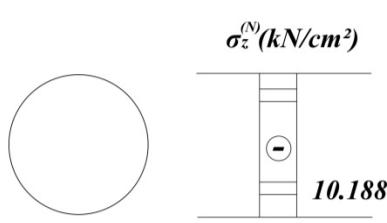
Naponi u karakterističnim poprečnim presjecima štapa:

$$\text{Dio } AC \text{ (Presjeci } A \text{ i } C_{desno}): \quad \sigma_{AC} = \frac{N_{AC}}{A_{AC}} = -\frac{200}{19.63} = -10.188 \frac{kN}{cm^2}$$

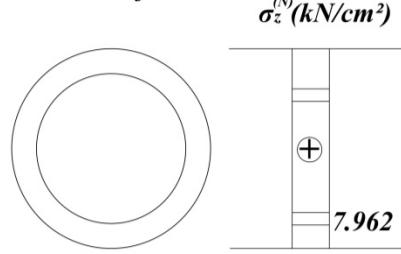
$$\text{Dio } CD \text{ (Presjeci } C_{lijevo} \text{ i } D_{desno}): \quad \sigma_{CD} = \frac{N_{CD}}{A_{CD}} = \frac{400}{78.50} = 5.096 \frac{kN}{cm^2}$$

$$\text{Dio } DB \text{ (Presjeci } B \text{ i } D_{lijevo}): \quad \sigma_{DB} = \frac{N_{DB}}{A_{DB}} = \frac{400}{50.24} = 7.962 \frac{kN}{cm^2}$$

Dijagram normalnih napona  
Presjek  $C_{desno}$



Dijagram normalnih napona  
Presjek  $B$



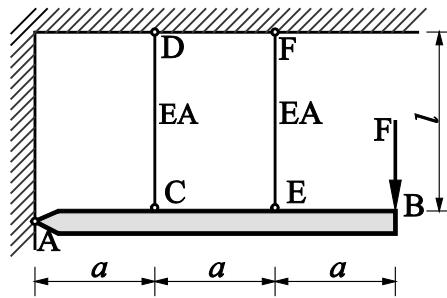
Ukupnu promjenu dužine štapa:

$$\Delta l = \sum_{i=1}^n \frac{N_i l_i}{EA_i}$$

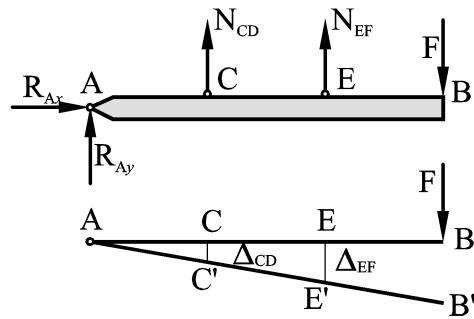
$$\Delta l_{AB} = \Delta l_{AC} + \Delta l_{CD} + \Delta l_{DB}$$

$$\begin{aligned} \Delta l_{AB} &= \frac{N_{AC} l_{AC}}{EA_{AC}} + \frac{N_{CD} l_{CD}}{EA_{CD}} + \frac{N_{DB} l_{DB}}{EA_{DB}} = \\ &= -\frac{200 \cdot 0.60 \cdot 10^2}{2.1 \cdot 10^4 \cdot 19.63} + \frac{400 \cdot 0.50 \cdot 10^2}{2.1 \cdot 10^4 \cdot 78.50} + \frac{400 \cdot 0.50 \cdot 10^2}{2.1 \cdot 10^4 \cdot 50.24} = -0.0292 + 0.0121 + 0.0190 = 0.0019 \text{ cm} \end{aligned}$$

3. Apsolutno kruta greda  $AB$  u tački  $A$  je vezana zglobno za vertikalni zid i sa dva čelična štapa istog poprečnog presjeka za plafon. Ako na gredu  $AB$  u tački  $B$  djeluje vertikalna sila  $F$  odrediti reakcije veza.



**RJEŠENJE**



Uslovi ravnoteže:

$$\sum M^A = 0 \quad N_{CD} \cdot a + N_{EF} \cdot 2a - F \cdot 3a = 0$$

$$\sum X = 0 \quad R_{Ax} = 0$$

$$\sum Y = 0 \quad R_{Ay} + N_{CD} + N_{EF} - F = 0$$

Deformacijski uslov:

$$\Delta_{CD} : a = \Delta_{EF} : 2a \quad \rightarrow \quad \Delta_{EF} = 2\Delta_{CD} \quad \rightarrow \quad \frac{N_{EF} l}{EA} = \frac{2N_{CD} l}{EA} \quad \rightarrow \quad N_{EF} = 2N_{CD}$$

Konačno, iz uslova ravnoteže dobijamo:

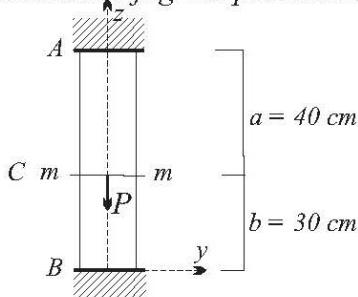
$$N_{CD} + 2N_{CD} \cdot 2 - F \cdot 3 = 0 \quad \rightarrow \quad N_{CD} = \frac{3}{5}F$$

$$N_{EF} = \frac{6}{5}F$$

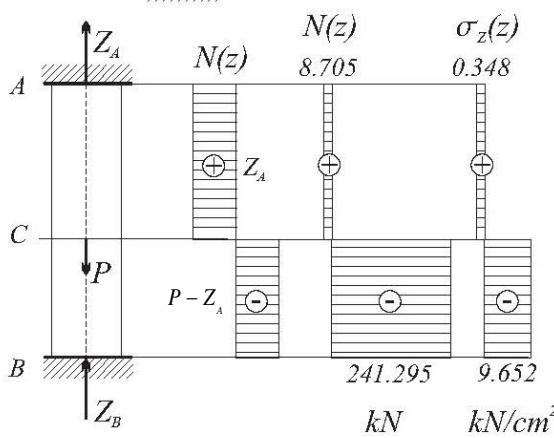
$$R_{Ay} = -N_{CD} - N_{EF} + F = -\frac{3}{5}F - \frac{6}{5}F + F = -\frac{4}{5}F$$

4.

Obostrano uklješten čelični štap opterećen je aksijalnom silom  $P$  u preseku  $m-m$  i zagrejan za  $15^\circ C$ . Nacrtati dijagram promene normalne sile i napona duž ose štapa i odrediti položaj preseka  $m-m$ .



$$\begin{aligned}E &= 210 \text{ GPa} \\ \alpha &= 12.5 \cdot 10^{-6} \text{ }^{\circ}\text{C}^{-1} \\ A &= 25 \text{ cm}^2 \\ P &= 250 \text{ kN}\end{aligned}$$



$Z_A = 8.705 \text{ kN} \text{ (zatezanje)}$

$Z_B = 241.295 \text{ kN} \text{ (pritisak)}$

$\sigma_A = Z_A/A = 8.705/25 = 0.348 \text{ kN/cm}^2$

$\sigma_B = Z_B/A = -241.295/25 = -9.652 \text{ kN/cm}^2$

Rešenje:

$\text{uslov ravnoteže}$

$\Sigma Z = 0$

$Z_A + Z_B = P$

$\Delta l = 0$

Deformacijski uslov napisan u razvijenom obliku

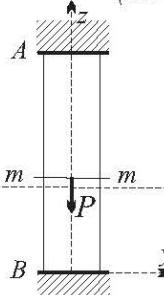
$\Delta l = \Delta l_{ac} + \Delta l_{cb} =$

$$\left( \frac{Z_A a}{EA} + \alpha \Delta t a \right) + \left( \frac{(Z_A - P)b}{EA} + \alpha \Delta t b \right) = 0$$

$$\Delta l_{ac} \qquad \qquad \qquad \Delta l_{cb}$$

$Z_A = \frac{EA}{(a+b)} \left[ \frac{Pb}{EA} - \alpha \Delta t (a+b) \right] =$

$$\frac{210 \cdot 10^2 \cdot 25}{(40+30)} \left[ \frac{250 \cdot 30}{210 \cdot 10^2 \cdot 25} - 12.5 \cdot 10^{-6} \cdot 15 \cdot 70 \right] = 8.705 \text{ kN}$$



Deo AC doživljava izduženje

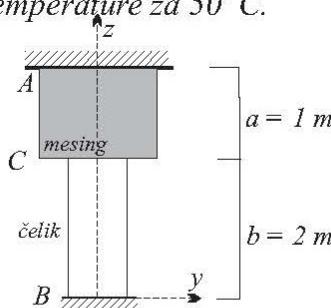
$\Delta l_{ac} = \frac{Z_A a}{EA} + \alpha \Delta t a =$

$$= \frac{8.705 \cdot 40}{210 \cdot 10^2 \cdot 25} + 12.5 \cdot 10^{-6} \cdot 15 \cdot 40 = 0.00816 \text{ cm}$$

za istu vrednost deo CB doživljava skraćenje

5.

Dat je štap delimično od čelika, a delimično od mesinga. Odrediti normalni napon pri smanjenju temperature za  $50^\circ C$ .



$$\begin{aligned}E_c &= 210 \text{ GPa} & E_m &= 100 \text{ GPa} \\ \alpha_c &= 12.5 \cdot 10^{-6} \text{ }^{\circ}\text{C}^{-1} & \alpha_m &= 17 \cdot 10^{-6} \text{ }^{\circ}\text{C}^{-1} \\ A_c &= 20 \text{ cm}^2 & A_m &= 30 \text{ cm}^2\end{aligned}$$

Rešenje:

$\text{uslov ravnoteže}$

$\Sigma Z = 0$

$Z_A - Z_B = 0$

$\Delta l = 0$

Deformacijski uslov napisan u razvijenom obliku

$\Delta l = \Delta l_{ac} + \Delta l_{cb} =$

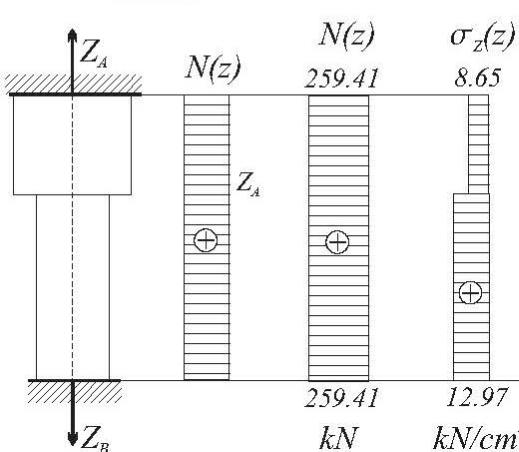
$$\left( \frac{Z_A a}{E_m A_m} - \alpha_m \Delta t a \right) + \left( \frac{Z_B b}{E_c A_c} - \alpha_c \Delta t b \right) = 0$$

$$\Delta l_{ac} \qquad \qquad \qquad \Delta l_{cb}$$

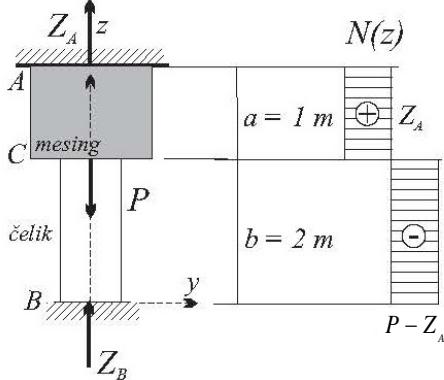
$Z_A = Z_B = 259.41 \text{ kN}$

$\sigma_A = Z_A/A_m = 259.41/30 = 8.65 \text{ kN/cm}^2$

$\sigma_B = Z_B/A_c = 259.41/20 = 12.97 \text{ kN/cm}^2$



U slučaju da je u prethodnom zadatku bila zadata i sila:



$$uslov ravnoteže$$

$$deform. uslov$$

$$\Sigma Z = 0$$

$$Z_A + Z_B = P$$

$$\Delta l = 0$$

Deformacijski uslov napisan u razvijenom obliku

$$\Delta l = \Delta l_{ac} + \Delta l_{cb} =$$

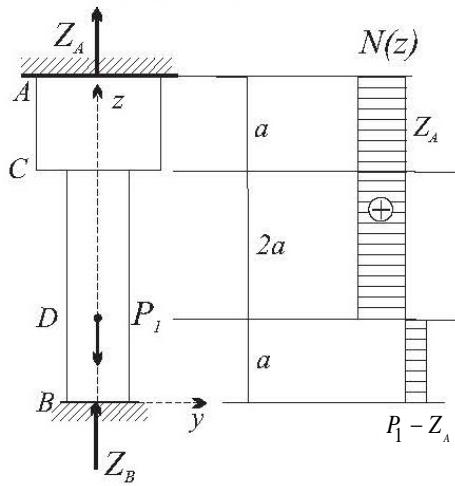
$$\left( \frac{Z_A a}{E_m A_m} - \alpha_m \Delta t a \right) + \left( \frac{(Z_A - P)b}{E_c A_c} - \alpha_c \Delta t b \right) = 0$$

znak minus je zbog smanjenja temperature

6.

Obostrano uklješten čelični štap opterećen je aksijalnom silom  $P = 50 \text{ kN}$ , kao na slici.

Nacrtati dijagram promene normalne sile i napona duž ose štapa i odrediti položaj preseka C i D.



$$E = 210 \text{ GPa} \quad a = 1 \text{ m}$$

$$A_{AC} = 15 \text{ cm}^2 \quad A_{CB} = 10 \text{ cm}^2$$

$$uslov ravnoteže$$

$$deform. uslov$$

$$\Sigma Z = 0 \quad Z_A + Z_B - P_1 = 0$$

$$\Delta l = 0$$

Deformacijski uslov napisan u razvijenom obliku

$$\Delta l = \Delta l_{ac} + \Delta l_{cd} + \Delta l_{db} =$$

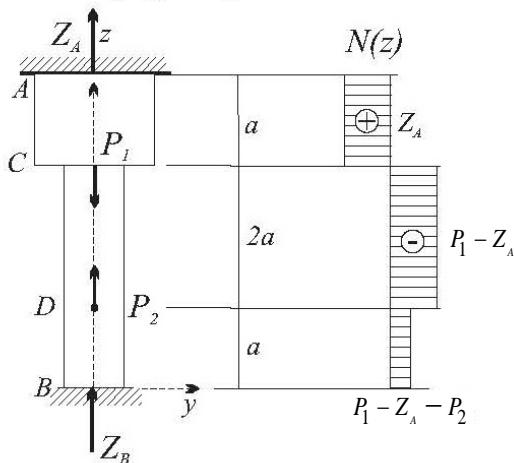
$$\left( \frac{Z_A a}{EA_{AC}} \right) + \left( \frac{Z_A 2a}{EA_{CB}} \right) + \left( \frac{(Z_A - P_1)a}{EA_{CB}} \right) = 0$$

$$\Delta l_{ac} \quad \Delta l_{cd} \quad \Delta l_{db}$$

7.

Obostrano uklješten čelični štap opterećen je aksijalnim silama  $P_1 = 20 \text{ kN}$  i  $P_2 = 5 \text{ kN}$ , kao na slici.

Nacrtati dijagram promene normalne sile i napona duž ose štapa i odrediti položaj preseka C i D.



$$E = 210 \text{ GPa} \quad a = 1 \text{ m}$$

$$A_{AC} = 15 \text{ cm}^2 \quad A_{CB} = 10 \text{ cm}^2$$

$$uslov ravnoteže$$

$$deform. uslov$$

$$\Sigma Z = 0 \quad Z_A + Z_B - P_1 + P_2 = 0$$

$$\Delta l = 0$$

Deformacijski uslov napisan u razvijenom obliku

$$\Delta l = \Delta l_{ac} + \Delta l_{cd} + \Delta l_{db} =$$

$$\left( \frac{Z_A a}{EA_{AC}} \right) + \left( \frac{(Z_A - P_1)2a}{EA_{CB}} \right) + \left( \frac{(Z_A - P_1 + P_2)a}{EA_{CB}} \right) = 0$$

$$\Delta l_{ac} \quad \Delta l_{cd} \quad \Delta l_{db}$$

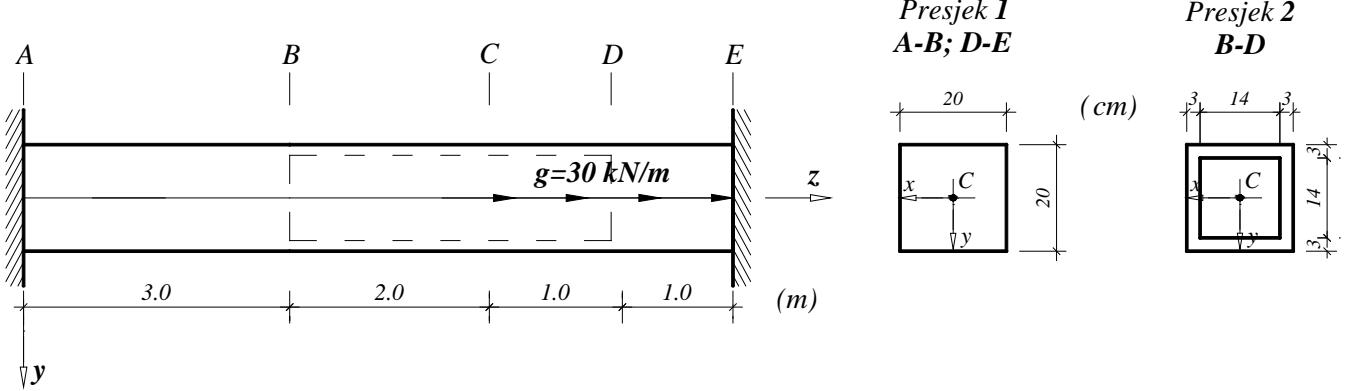
U slučaju da je u prethodnom zadatku postojalo i zagrevanje dela CB deformacijski uslov bi glasio:

$$\Delta l = \Delta l_{ac} + \Delta l_{cd} + \Delta l_{db} =$$

$$\left( \frac{Z_A a}{EA_{AC}} \right) + \left[ \left( \frac{(Z_A - P_1)2a}{EA_{CB}} + \alpha \Delta t 2a \right) + \left[ \left( \frac{(Z_A - P_1 + P_2)a}{EA_{CB}} + \alpha \Delta t a \right) \right] \right] = 0$$

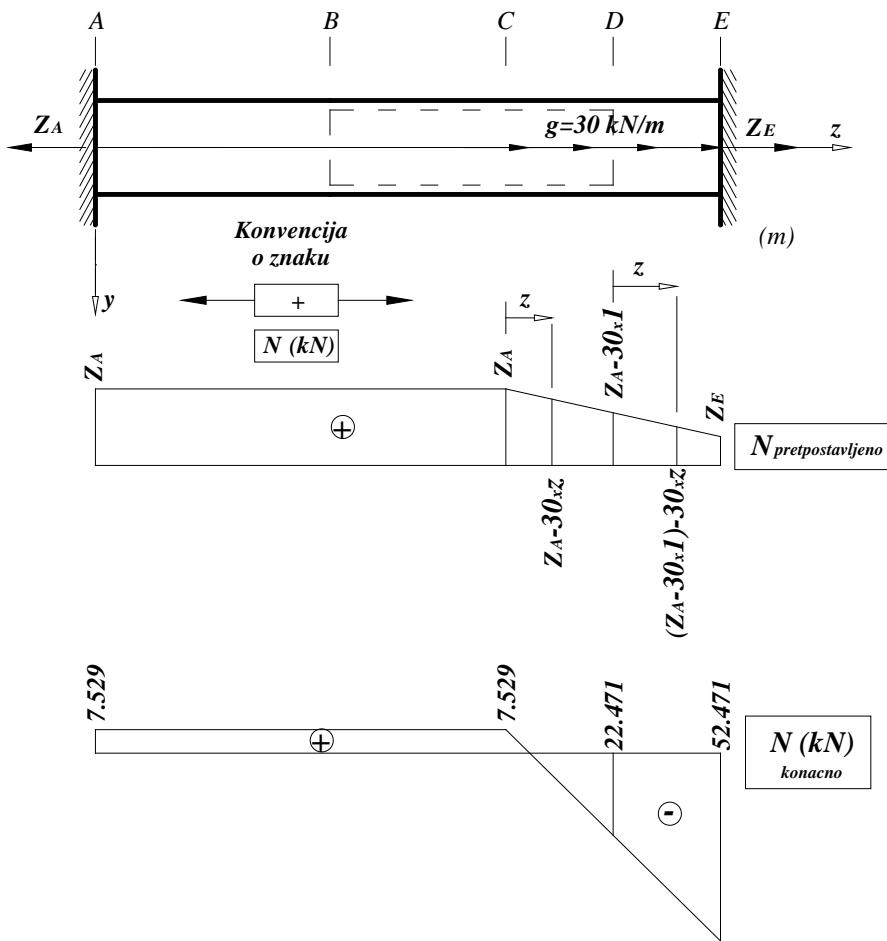
8. Za nosač ( $E=21 \text{ GPa}$ ) i opterećenje kao na slici odrediti:

- Reakcije oslonaca i nacrati dijagram raspodjele normalnih sila u nosaču;
- Dijagrame normalnih napona u presjecima  $B_{lijevo}$  i  $D_{lijevo}$ ;
- Pomjeranje tačaka u presjecima  $B$ ,  $C$  i  $D$ .



**RJEŠENJE**

a.



$$\Delta l = \int_0^l \frac{N(z)}{EA(z)} dz \quad \text{za } N = \text{const} \text{ i } A = \text{const} \rightarrow \Delta l = \frac{N}{EA} \int_0^l dz = \frac{Nl}{EA}$$

Površine poprečnih presjeka:

$$A_1 = 20^2 = 400 \text{ cm}^2$$

$$A_2 = 20^2 - 14^2 = 204 \text{ cm}^2$$

Uslov ravnoteže:

$$\Sigma Z = 0 \rightarrow g \cdot 2 = Z_A - Z_E \rightarrow Z_E = Z_A - 30 \cdot 2$$

Deformacijski uslov:

$$\Delta l = 0 \rightarrow \Delta l_{AB} + \Delta l_{BC} + \Delta l_{CD} + \Delta l_{DE} = 0$$

$$\frac{Z_A \cdot 3}{EA_1} + \frac{Z_A \cdot 2}{EA_2} + \int_0^1 \frac{(Z_A - g \cdot z)}{EA_2} dz + \int_0^1 \frac{[(Z_A - g \cdot 1) - g \cdot z]}{EA_1} dz = 0 \quad | \cdot EA_1$$

$$3Z_A + \frac{A_1}{A_2} 2Z_A + \frac{A_1}{A_2} \left[ Z_A \cdot z - g \frac{z^2}{2} \right]_0^1 + \left[ (Z_A - g \cdot 1) \cdot z - g \frac{z^2}{2} \right]_0^1 = 0$$

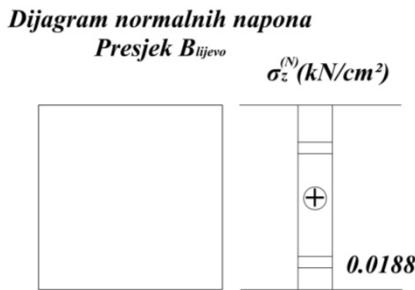
$$3Z_A + 1.961 \cdot 2Z_A + 1.961 \left[ Z_A \cdot 1 - 30 \frac{1^2}{2} \right] + \left[ (Z_A - 30 \cdot 1) \cdot 1 - 30 \frac{1^2}{2} \right] = 0 \rightarrow Z_A = 7.529 \text{ kN}$$

$$Z_E = -52.47 \text{ kN}$$

b.

$$\sigma_z^{B_{\text{lijevo}}} = \frac{7.529}{400} = 0.0188 \frac{\text{kN}}{\text{cm}^2}$$

$$\sigma_z^{D_{\text{lijevo}}} = \frac{-22.471}{204} = -0.11 \frac{\text{kN}}{\text{cm}^2}$$



c.

Pomjeranje presjeka **B**

$$\Delta l_{AB} = \frac{N_{AB} \cdot l_{AB}}{E \cdot A_{AB}} = \frac{7.529 \cdot 300}{21 \cdot 10^2 \cdot 400} = 0.00269 \text{ cm}$$

$$\Delta l_{AC} = \Delta l_{AB} + \Delta l_{BC} = \Delta l_{AB} + \frac{N_{BC} \cdot l_{BC}}{E \cdot A_{BC}} = 0.00269 + \frac{7.529 \cdot 200}{21 \cdot 10^2 \cdot 204} = 0.00269 + 0.00351 = 0.0062 \text{ cm}$$

$$\begin{aligned} \Delta l_{AD} &= \Delta l_{AC} + \Delta l_{CD} = \Delta l_{AC} + \int_0^1 \frac{(7.529 - 30 \cdot z)}{EA_2} dz = \\ &= 0.0062 + \frac{7.529 \cdot 100 - 30 \cdot 10^{-2} \cdot \frac{100^2}{2}}{21 \cdot 10^2 \cdot 204} = 0.0062 - 0.00174 = 0.00446 \text{ cm} \end{aligned}$$

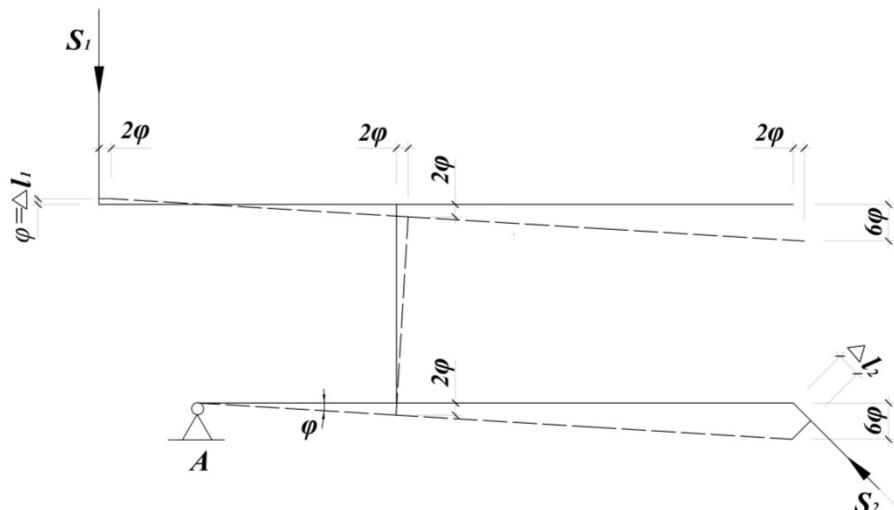
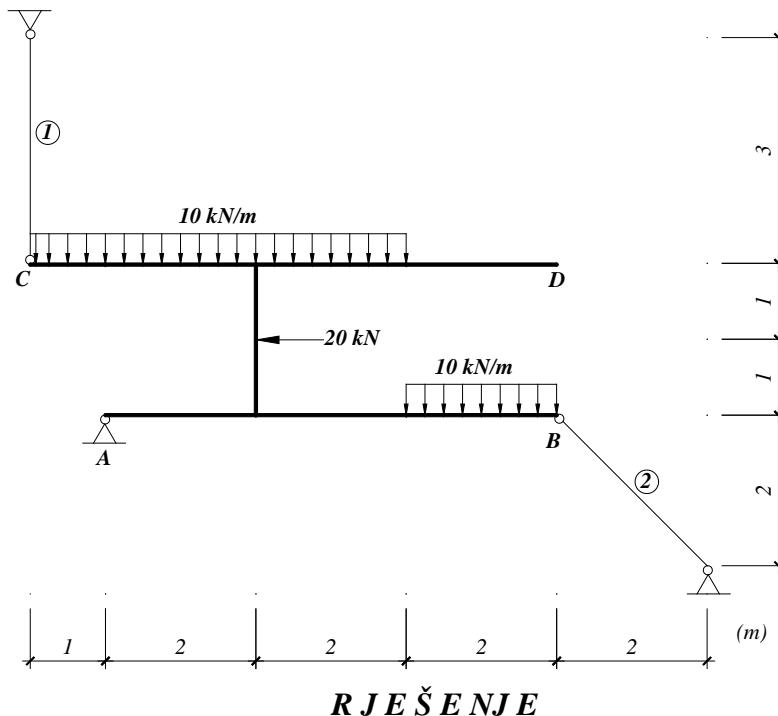
Kontrola  $\Delta l_{AE} = 0$

$$\begin{aligned} \Delta l_{AE} &= \Delta l_{AD} + \Delta l_{DE} = \Delta l_{AD} + \int_0^1 \frac{(-22.471 - 30 \cdot z)}{EA_1} dz = \\ &= 0.00446 + \frac{-22.471 \cdot 100 - 30 \cdot 10^{-2} \cdot \frac{100^2}{2}}{21 \cdot 10^2 \cdot 400} = 0.00446 - 0.00446 = 0 \text{ cm} \end{aligned}$$

# VJEŽBA BR. 10

## AKSIJALNO NAPREZANJE II

- I. Apsolutno krutu gredu **ABCD** u ravnoteži održavaju dva deformabilna štapa. Sračunati sile u štapovima.  
 $(A_1 = 0.7A_2; E_1 = E_2)$



Uslov ravnoteže:

$$\Sigma M_A = 0 \rightarrow S_1 \cdot 1 + S_2 \frac{\sqrt{2}}{2} \cdot 6 - 10 \cdot 5 \cdot 1.5 - 10 \cdot 2 \cdot 5 + 20 \cdot 1 = 0$$

$$S_1 + S_2 \cdot 3\sqrt{2} = 155 \quad (1)$$

Deformacijski uslov:

$$\left. \begin{array}{l} \Delta l_1 = \varphi \\ \frac{\Delta l_2}{\cos 45^\circ} = 6\varphi \end{array} \right\} \rightarrow \varphi = \frac{\Delta l_1}{\Delta l_2} = \frac{\Delta l_1}{6\cos 45^\circ} \rightarrow \Delta l_1 = \frac{\Delta l_2}{6\cos 45^\circ} \rightarrow \frac{S_1 \cdot 3}{E \cdot 0.7A_2} = \frac{S_2 \cdot 2\sqrt{2}}{6\sqrt{2}/2 \cdot E \cdot A_2} \rightarrow S_1 = 0.156S_2 \quad (2)$$

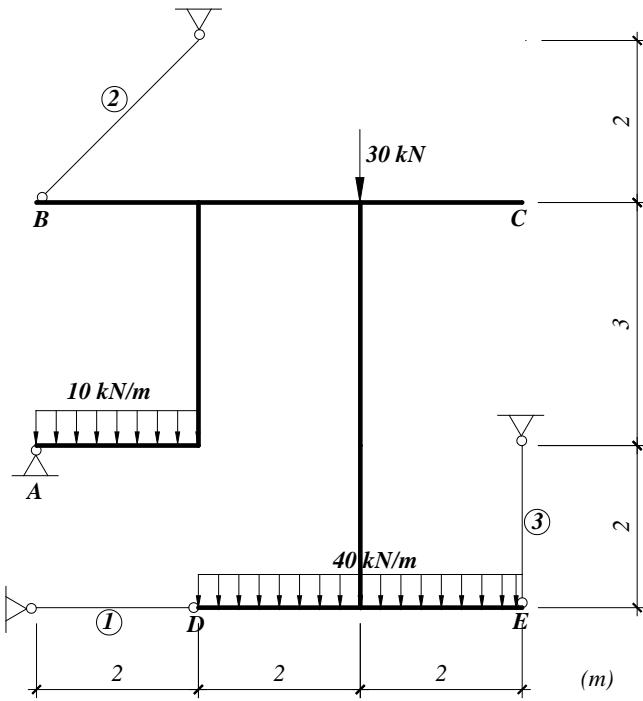
$$(2) \rightarrow (1): 0.156S_2 + S_2 \cdot 3\sqrt{2} = 155$$

$$S_2 = 35.24 \text{ kN}$$

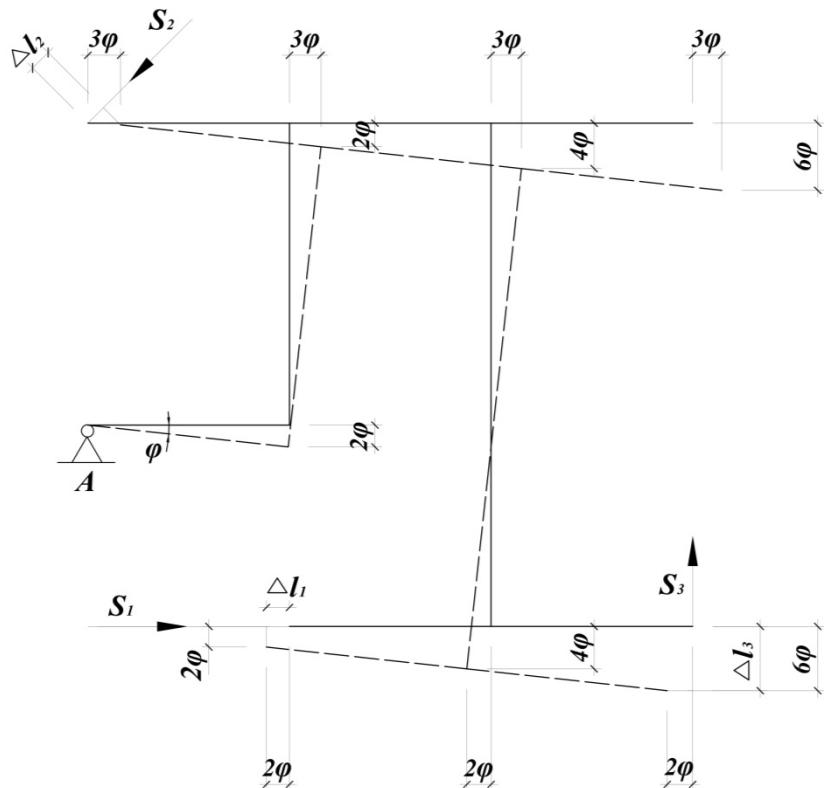
$$S_1 = 5.5 \text{ kN}$$

2. Apsolutno krutu gredu **ABCDE** u ravnoteži održavaju tri deformabilna štapa. Dimenzionisati štapove ako je:

$$A_1 = 0.8A_2 = 0.4A_3; E_1 = E_2 = E_3 = E; \sigma_{dop} = 10 \frac{kN}{cm^2}$$



**RJEŠENJE**



Uslov ravnoteže:

$$\Sigma M_A = 0 \rightarrow S_1 \cdot 2 + S_2 \frac{\sqrt{2}}{2} \cdot 3 + S_3 \cdot 6 = 10 \cdot 2 \cdot 1 + 30 \cdot 4 + 40 \cdot 4 \cdot 4$$

$$2S_1 + \frac{3}{2}\sqrt{2}S_2 + 6S_3 = 780 \quad (1)$$

Deformacijski uslovi:

$$\left. \begin{array}{l} \Delta l_1 = 2\varphi \rightarrow \varphi = \frac{\Delta l_1}{2} \\ \frac{\Delta l_2}{\cos 45^\circ} = 3\varphi \rightarrow \varphi = \frac{\Delta l_2}{3\cos 45^\circ} \\ \Delta l_3 = 6\varphi \rightarrow \varphi = \frac{\Delta l_3}{6} \end{array} \right\} \rightarrow \begin{array}{l} \frac{\Delta l_1}{2} = \frac{\Delta l_2}{3\cos 45^\circ} \rightarrow \frac{S_1 \cdot 2}{2 \cdot E \cdot 0.8A_2} = \frac{S_2 \cdot 2\sqrt{2}}{3\sqrt{2} / 2 \cdot E \cdot A_2} \rightarrow S_2 = 0.9375S_1 \quad (2) \\ \frac{\Delta l_1}{2} = \frac{\Delta l_3}{6} \rightarrow \frac{S_1 \cdot 2}{2 \cdot E \cdot 0.4A_3} = \frac{S_3 \cdot 2}{6 \cdot E \cdot A_3} \rightarrow S_3 = 7.5S_1 \end{array} \quad (3)$$

$$(2) (3) \rightarrow (1): 2S_1 + \frac{3}{2}\sqrt{2} \cdot 0.9375S_1 + 6 \cdot 7.5S_1 = 780$$

$$S_1 = 15.922 \text{ kN}$$

$$S_2 = 14.927 \text{ kN}$$

$$S_3 = 119.415 \text{ kN}$$

Dimenzionisanje:

$$\sigma_z = \frac{S}{A} \leq \sigma_{dop} \rightarrow \text{mjerodavan je štap najmanjeg poprečnog presjeka izložen najvećoj aksijalnoj sili}$$

Mjerodavan je štap 3

$$A_{3(potr)} \geq \frac{119.415}{10} = 11.94 \text{ cm}^2 \quad \text{Usvaja se } A_3 = 12 \text{ cm}^2 \rightarrow A_1 = 0.4 \cdot 12 = 4.8 \text{ cm}^2 \quad i \quad A_2 = \frac{0.4}{0.8} \cdot 12 = 6 \text{ cm}^2$$

Kontrola napona:

$$\sigma_{z,1} = \frac{S_1}{A_1} = \frac{15.922}{4.8} = 3.32 \frac{\text{kN}}{\text{cm}^2} \leq 10 \frac{\text{kN}}{\text{cm}^2} = \sigma_{dop}$$

$$\sigma_{z,2} = \frac{S_2}{A_2} = \frac{14.927}{6} = 2.49 \frac{\text{kN}}{\text{cm}^2} \leq 10 \frac{\text{kN}}{\text{cm}^2} = \sigma_{dop}$$

$$\sigma_{z,3} = \frac{S_3}{A_3} = \frac{119.415}{12} = 9.95 \frac{\text{kN}}{\text{cm}^2} \leq 10 \frac{\text{kN}}{\text{cm}^2} = \sigma_{dop}$$

3. Apsolutno krutu gredu **ABCD** u ravnoteži održavaju tri deformabilna štapa. Odrediti:

- Sile u štapovima;
- Pomjeranje tačke A.

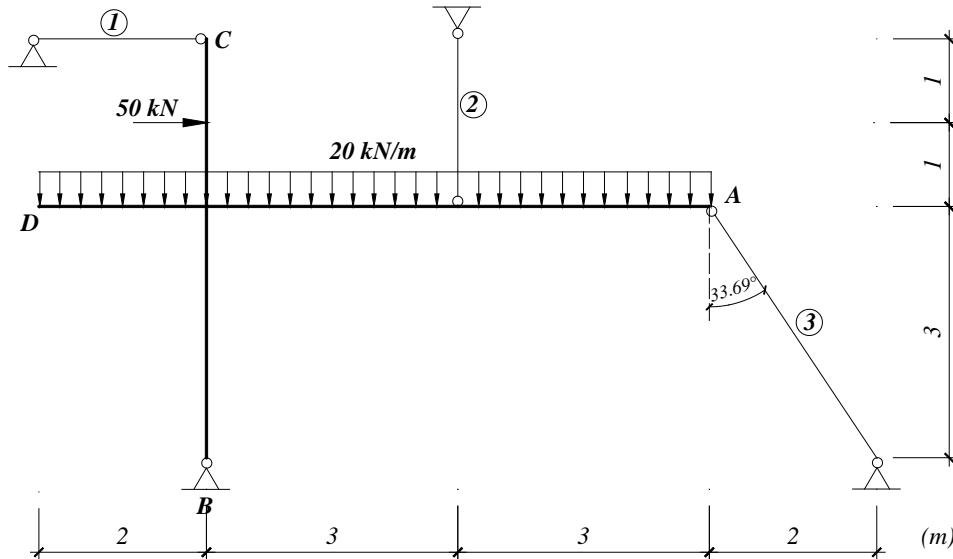
$$A_1 = A$$

$$A_2 = A$$

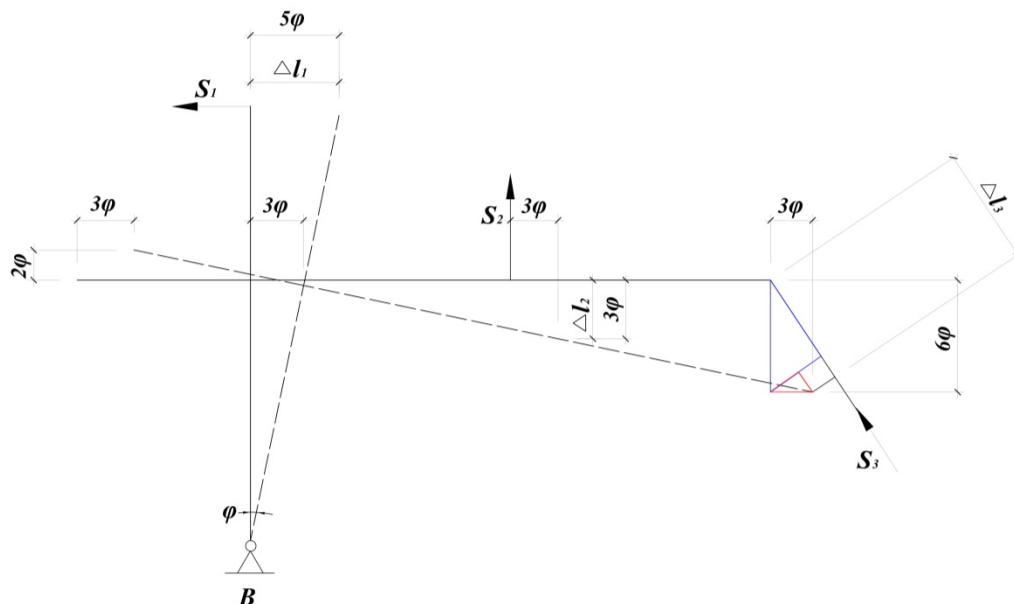
$$A_3 = 3A$$

$$A = 5 \text{ cm}^2$$

$$E_1 = E_2 = E_3 = E = 2.1 \cdot 10^4 \frac{\text{kN}}{\text{cm}^2}$$



**RJEŠENJE**



- Sile u štap

$$\tan \alpha = \frac{2}{3} \quad \alpha = \arctan \frac{2}{3} = 33.69^\circ \rightarrow \begin{aligned} \sin \alpha &= 0.5547 \\ \cos \alpha &= 0.832 \end{aligned}$$

Uslov ravnoteže:

$$\sum M_B = 0 \rightarrow S_1 \cdot 5 + S_2 \cdot 3 + S_3 \cos \alpha \cdot 6 + S_3 \sin \alpha \cdot 3 = 20 \cdot 8 \cdot 2 + 50 \cdot 4$$

$$5S_1 + 3S_2 + 6.6561S_3 = 520 \quad (1)$$

Deformacijski uslovi:

$$\left. \begin{array}{l} \Delta l_1 = 5\varphi \rightarrow \varphi = \frac{\Delta l_1}{5} \\ \Delta l_2 = 3\varphi \rightarrow \varphi = \frac{\Delta l_2}{3} \\ \Delta l_3 = 6.6561\varphi \rightarrow \varphi = \frac{\Delta l_3}{6.6561} \\ \Delta l_3 = 6\varphi \cos \alpha + 3\varphi \sin \alpha = 6.6561\varphi \end{array} \right\} \rightarrow \begin{array}{l} \frac{\Delta l_1}{5} = \frac{\Delta l_2}{3} \rightarrow \frac{S_1 \cdot 2}{5 \cdot E \cdot A} = \frac{S_2 \cdot 2}{3 \cdot E \cdot A} \rightarrow S_2 = \frac{3}{5} S_1 \\ \frac{\Delta l_1}{5} = \frac{\Delta l_3}{6.6561} \rightarrow \frac{S_1 \cdot 2}{5 \cdot E \cdot A} = \frac{S_3 \cdot \sqrt{13}}{6.6561 \cdot E \cdot 3A} \rightarrow S_3 = 2.2152 S_1 \end{array} \quad (2)$$

$$(2) (3) \rightarrow (1): 5S_1 + 3 \cdot \frac{3}{5} S_1 + 6.6561 \cdot 2.2152 S_1 = 520$$

$$S_1 = 24.14 \text{ kN}$$

$$S_2 = 14.48 \text{ kN}$$

$$S_3 = 53.47 \text{ kN}$$

### b. Pomjeranje tačke A

$$\left. \begin{array}{l} \Delta l_1 = \frac{S_1 \cdot l_1}{E \cdot A_1} = \frac{24.14 \cdot 200}{2.1 \cdot 10^4 \cdot 5} = 0.04597 \text{ cm} \\ \Delta l_1 = 5\varphi \end{array} \right\} \rightarrow \varphi = \frac{\Delta l_1}{5} = \frac{0.04597}{5 \cdot 10^2} = 0.009194 \cdot 10^{-2} \text{ rad}$$

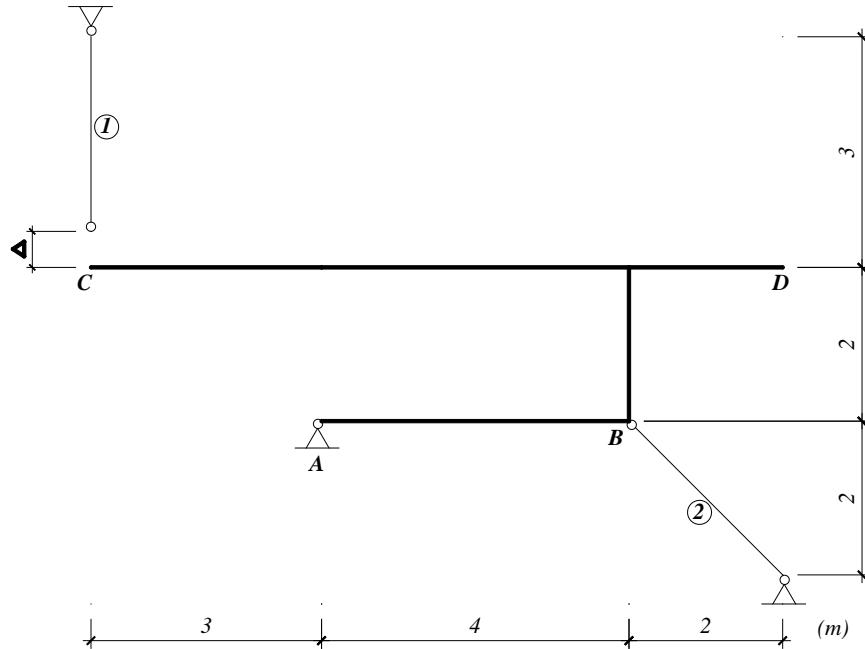
$$\delta_{HOR}^A = 3\varphi = 3 \cdot 10^2 \cdot 0.009194 \cdot 10^{-2} = 0.02758 \text{ cm}$$

$$\delta_{VER}^A = 6\varphi = 6 \cdot 10^2 \cdot 0.009194 \cdot 10^{-2} = 0.05517 \text{ cm}$$

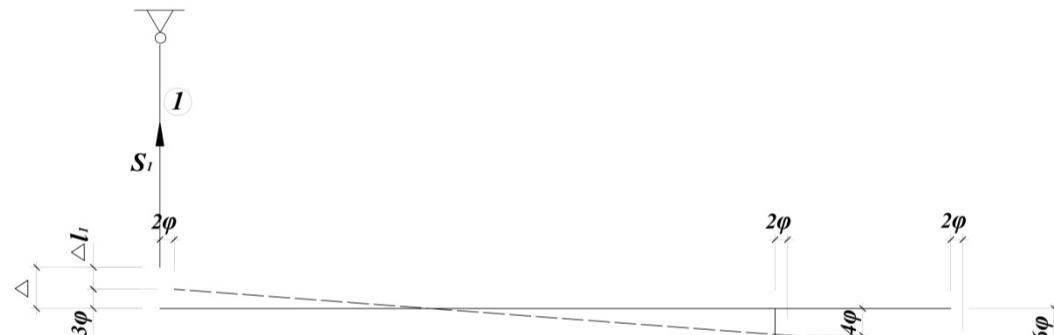
$$\delta^A = \sqrt{(\delta_{VER}^A)^2 + (\delta_{HOR}^A)^2} = \sqrt{0.02758^2 + 0.05517^2} = 0.06168 \text{ cm}$$

$$\Delta l_3 = \frac{S_3 \cdot l_3}{E \cdot A_3} = \frac{53.47 \cdot \sqrt{13} \cdot 100}{2.1 \cdot 10^4 \cdot 3 \cdot 5} = 0.0612 \text{ cm}$$

4. Izvršena je prisilna montaža apsolutno krute grede **ABCD** pošto je štap broj **I** izrađen kraći za  $\Delta = 2\text{mm}$ . Odrediti sile u štapovima ako je poznato:  $E_1 = E_2 = E = 210 \text{ GPa}$ ;  $A_1 = 3\text{cm}^2$ ;  $A_2 = 5\text{cm}^2$

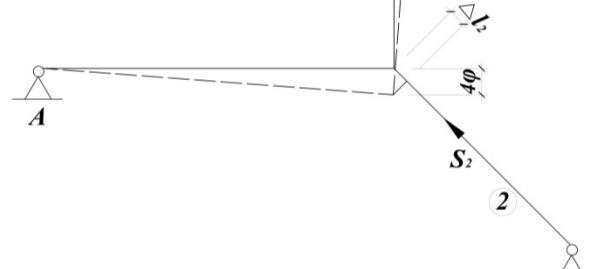


**RJEŠENJE**



Uslov ravnoteže:

$$\Sigma M_A = 0 \rightarrow S_1 \cdot$$



Deformacijski uslov:

$$\left. \begin{array}{l} \Delta - \Delta l_1 = 3\varphi \\ \frac{\Delta l_2}{\cos 45^\circ} = 4\varphi \end{array} \right\} \rightarrow \varphi = \frac{\Delta - \Delta l_1}{3} = \frac{\Delta l_2}{4 \cdot \sqrt{2}} \rightarrow \Delta - \frac{S_1 \cdot 300}{210 \cdot 10^2 \cdot 3} = \frac{3}{4} \frac{S_2 \cdot 200\sqrt{2}}{\sqrt{2} \cdot 210 \cdot 10^2 \cdot 5}$$

$$0.2 - 0.004762S_1 = 0.0028571S_2 \quad (2)$$

$$(1) \rightarrow (2): \quad 0.2 - 0.004762 \cdot \frac{2}{3} \sqrt{2} S_1 = 0.0028571 S_2$$

$$S_1 = 25.66 \text{ kN}$$

$$S_2 = 27.22 \text{ kN}$$

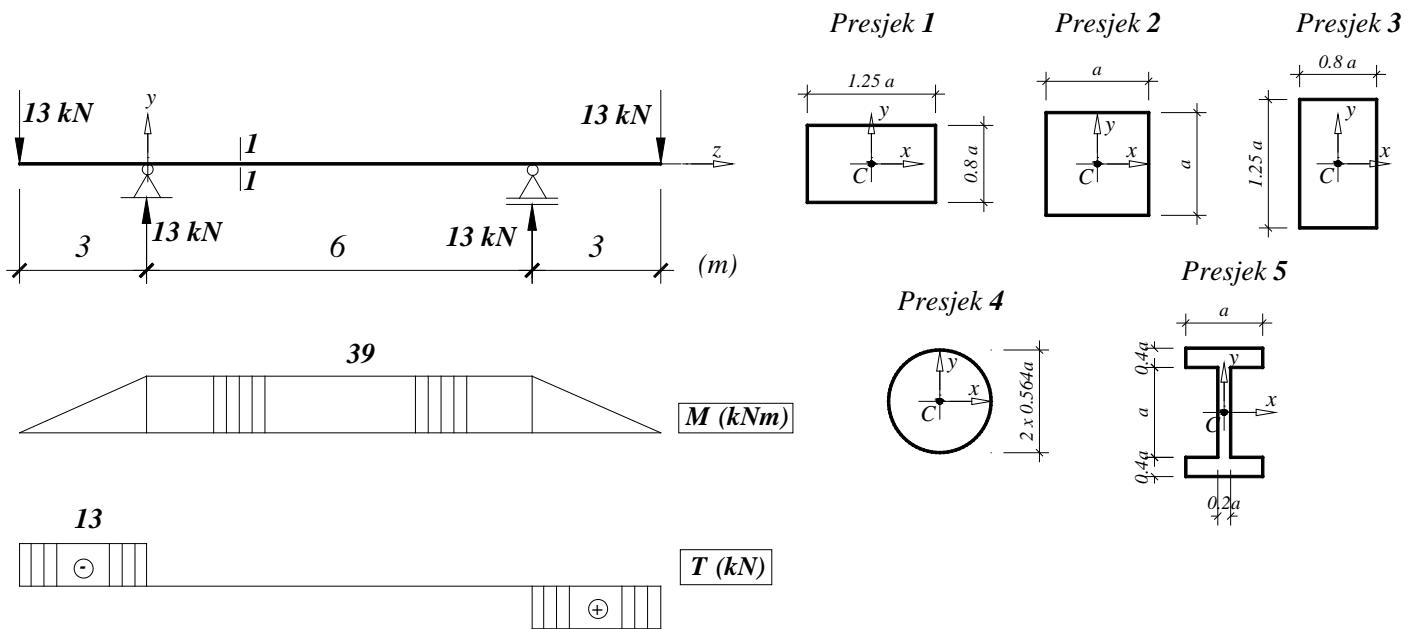
# VJEŽBA BR. 11

## ČISTO PRAVO SAVIJANJE

I.

Za nosač na slici:

- Od ponuđenih oblika poprečnih presjeka (istih površina) izračunati najpovoljniji i dimenzionisati ga za presjek I-I ako je  $\sigma_{doz} = 12 \text{ MPa}$ ;
- Za odabrani i dimenzionisani poprečni presjek nacrtati dijagram normalnih napona u presjeku I-I.



**RJEŠENJE**

- Odabir najpovoljnijeg presjeka i dimenzionisanje:

$\sigma_{z,\max} \leq \sigma_{doz}$  - uslov za dimenzionisanje

$$\sigma_{z,\max} = \frac{M}{I_x} y_{\max} = \frac{M}{W_x} \quad \left( W_x = \frac{I_x}{y_{\max}} \right) \text{ - presjek sa najvećim otpornim momentom je najpovoljniji}$$

$$\text{Pravougaoni poprečni presjek dimenzija } b \times h \text{ - } W_x = \frac{I_x}{y_{\max}} = \frac{\frac{b \cdot h^3}{12}}{\frac{h}{2}} = \frac{b \cdot h^2}{6}$$

$$\text{Presjek 1: } W_x = \frac{b \cdot h^2}{6} = \frac{1.25a \cdot (0.8a)^2}{6} = 0.1333a^3$$

$$\text{Presjek 2: } W_x = \frac{b \cdot h^2}{6} = \frac{a \cdot (a)^2}{6} = 0.1666a^3$$

$$\text{Presjek 3: } W_x = \frac{b \cdot h^2}{6} = \frac{0.8a \cdot (1.25a)^2}{6} = 0.2083a^3$$

$$\text{Presjek 4: } W_x = \frac{I_x}{y_{\max}} = \frac{\frac{\pi \cdot R^4}{4}}{R} = \frac{\pi \cdot R^3}{4} = \frac{\pi \cdot (0.564a)^3}{4} = 0.141a^3$$

$$\text{Presjek 5: } I_x = 2 \cdot \frac{a \cdot (0.4a)^3}{12} + 2 \cdot 0.4a \cdot a \cdot (0.7a)^2 + \frac{0.2a \cdot a^3}{12} = 0.4193a^4$$

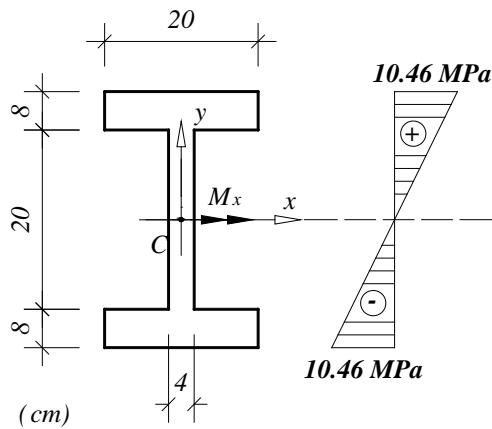
$$W_x = \frac{I_x}{y_{\max}} = \frac{0.4193a^4}{0.9a} = 0.466a^3 - \underline{\text{Presjek 5 je najpovoljniji}}$$

$$\sigma_{z,\max} = \frac{M}{W_x} \leq \sigma_{doz} \quad \rightarrow \quad W_x \geq \frac{M}{\sigma_{doz}} \quad \rightarrow \quad 0.466a^3 \geq \frac{39 \cdot 10^2}{1.2} \quad \rightarrow \quad a \geq 19.1 \text{ cm}$$

Usvaja se:  $a = 20 \text{ cm}$   $\rightarrow W_x = 0.466 \cdot 20^3 = 3728 \text{ cm}^3$

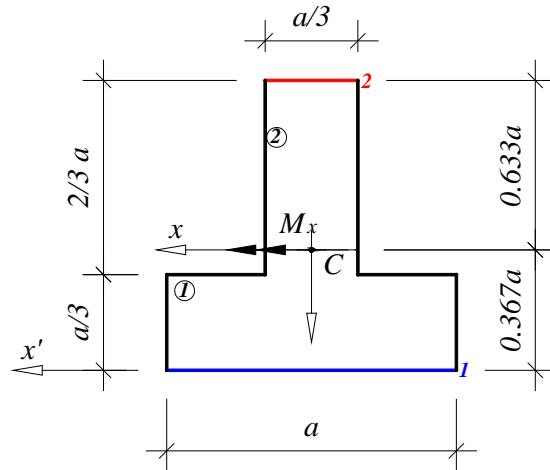
b. Kontrola, odnosno ekstremni normalni naponi:

$$\sigma_{z,\max} = \pm \frac{M}{W_x} = \pm \frac{39 \cdot 10^2}{3728} = \pm 1.046 \frac{kN}{cm^2} \leq \sigma_{doz}$$



2.

Greda poprečnog presjeka kao na slici opterećena je na savijanje spregovima  $M_x=15 \text{ kNm}$ . Ako je dozvoljeni napon na zatezanje  $\sigma_{doz}^+ = 20 \text{ MPa}$ , a na pritisak  $\sigma_{doz}^- = 80 \text{ MPa}$ , izvršiti dimenzionisanje poprečnog presjeka grede i nacrtati dijagram normalnih napona.



RJEŠENJE

Geometrijske karakteristike poprečnog presjeka:

$$C_1\left(0; -\frac{a}{6}\right) \quad A_l = \frac{a^2}{3}$$

$$C_2\left(0; -\frac{2}{3}a\right) \quad A_2 = \frac{2}{9}a^2$$

$$y_C = -\frac{\frac{a}{6} \cdot \frac{a^2}{3} + \frac{2}{3}a \cdot \frac{2}{9}a^2}{\frac{a^2}{3} + \frac{2}{9}a^2} = -0.367a$$

$$I_x = \frac{a \cdot \left(\frac{a}{3}\right)^3}{12} + \frac{a^2}{3} \cdot (0.2a)^2 + \frac{\frac{a}{3} \cdot \left(\frac{2}{3}a\right)^3}{12} + \frac{2}{9}a^2 \cdot (-0.3a)^2 = 0.04465a^4$$

### Dimenzionisanje:

**1 – Zategnuto vlakno max**

$$\sigma_{z,\max} = \frac{M}{W_{x,1}} \leq \sigma_{doz} \quad \rightarrow \quad W_{x,1} \geq \frac{M}{\sigma_{doz}}$$

$$W_{x,1} = \frac{I_x}{y_1} = \frac{0.04465a^4}{0.367a} = 0.12177a^3 \geq \frac{15 \cdot 10^2}{2} = 750 \text{ cm}^3 \quad \rightarrow \quad a \geq 18.33 \text{ cm}$$

## 2 – Pritisnuto vlakno *min*

$$\left| \sigma_{z,\min} \right| = \frac{M}{W_{x,2}} \leq \sigma_{doz} \quad \rightarrow \quad W_{x,2} \geq \frac{M}{\sigma_{doz}}$$

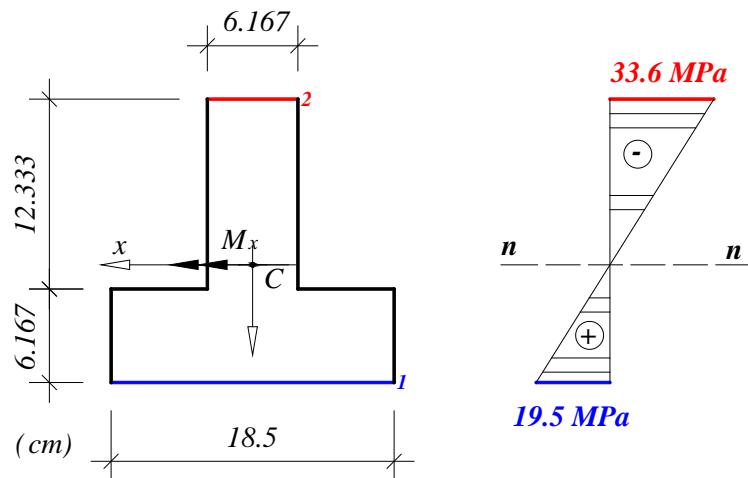
$$W_{x,2} = \frac{I_x}{y_2} = \frac{0.04465a^4}{0.633a} = 0.0705a^3 \geq \frac{15 \cdot 10^2}{8} = 187.5 \text{ cm}^3 \quad \rightarrow \quad a \geq 13.85 \text{ cm}$$

Usvaja se:  $a = 18.5 \text{ cm}$   $\rightarrow$   $W_{x,1} = 0.12177 \cdot 18.5^3 = 771 \text{ cm}^3$   
 $W_{x,2} = 0.0705 \cdot 18.5^3 = 446.4 \text{ cm}^3$

Kontrola, odnosno ekstremni normalni naponi:

$$\sigma_{z,\max} = \frac{15 \cdot 10^2}{771} = 1.95 \frac{\text{kN}}{\text{cm}^2} \leq \sigma_{doz} \square = 2 \frac{\text{kN}}{\text{cm}^2}$$

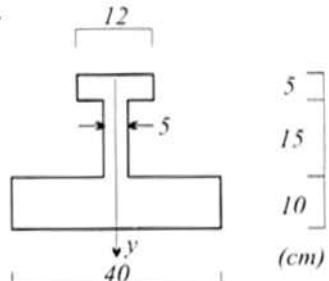
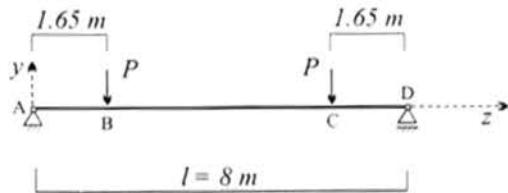
$$|\sigma_{z,\min}| = \frac{15 \cdot 10^2}{446.4} = 3.36 \frac{\text{kN}}{\text{cm}^2} \leq \sigma_{doz} \square = 8 \frac{\text{kN}}{\text{cm}^2}$$



Data je prosta greda opterećena prema slici. Opterećenje leži u ravni yz.

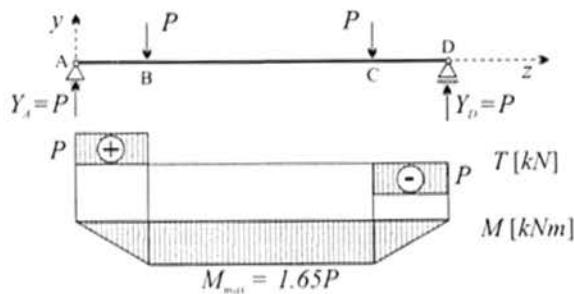
a/ Odrediti maksimalni intenzitet sile P koje može da nosi greda zadatog poprečnog preseka, ako je  $\sigma_z = 120 \text{ MPa}$ .

b/ Za usvojene vrednosti izvršiti proveru napona  $\sigma_z$  i nacrtati dijagram.



### RJEŠENJE

a/ Sa dijagrama presečnih sila vidi se da je na delu BC nosač izložen čistom pravom savijanju oko ose x.

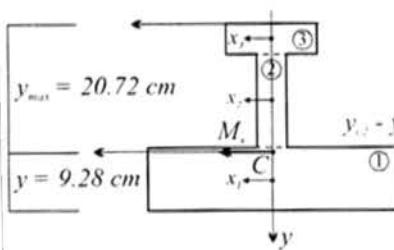


Određivanje težišta preseka

Zbog simetrije preseka težište se nalazi na osi simetrije (osi y), koja je ujedno i glavna centralna osa inercije. Znači, treba odrediti samo položaj ose x.

$$y_c = \frac{y_{c1}A_1 + y_{c2}A_2 + y_{c3}A_3}{A_1 + A_2 + A_3} = \frac{25 \cdot 400 + 12.5 \cdot 75 + 2.5 \cdot 60}{400 + 75 + 60} = 20.72 \text{ cm}$$

Geometrijske karakteristike preseka



$$I_y = \frac{40 \cdot 10^3}{12} + 400 \cdot 4.28^2 + \frac{5 \cdot 15^3}{12} + 75 \cdot (-8.22)^2 + \frac{12.5^3}{12} + 60 \cdot (-18.22)^2 = 37177.67 \text{ cm}^4$$

$$W_y = \frac{I_y}{y_{max}} = \frac{37177.67}{20.72} = 1794.29 \text{ cm}^3$$

Iz uslova za dimenzionisanje određujemo moment nosivosti.

$$\sigma_{zmax} = \frac{M_y}{I_y} y_{max} = \frac{M_y}{W_y} \leq \sigma_{doz} \Rightarrow M_y = W_y \cdot \sigma_{doz} = 1794.29 \cdot 12 = 21531.5 \text{ kNm} = 215.315 \text{ kNm}$$

Maksimalna vrednost sile P

$$M_{max} = 1.65 \cdot P \Rightarrow P = \frac{M_{max}}{1.65} = \frac{215.315}{1.65} = 130.49 \text{ kN}$$

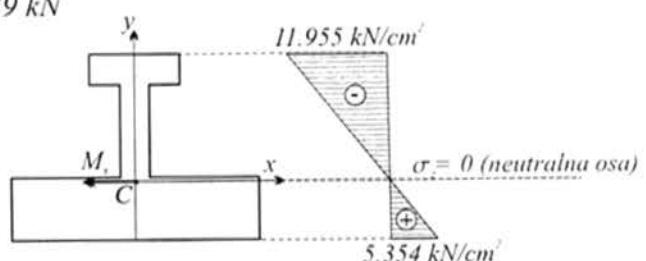
usvojeno:  $P = 130 \text{ kN}$

b/ Stvarni uticaji u nosaču su:

$$M_{max} = 1.65 \cdot P_{nev} = 1.65 \cdot 130 = 214.5 \text{ kNm}$$

$$\sigma_{zmax} = \frac{M_y}{I_y} y_{max} = \frac{M_y}{W_y} = \frac{214.5 \cdot 10^3}{1794.29} (-20.72) = -11.955 \text{ kN/cm}^2 = -119.55 \text{ MPa}$$

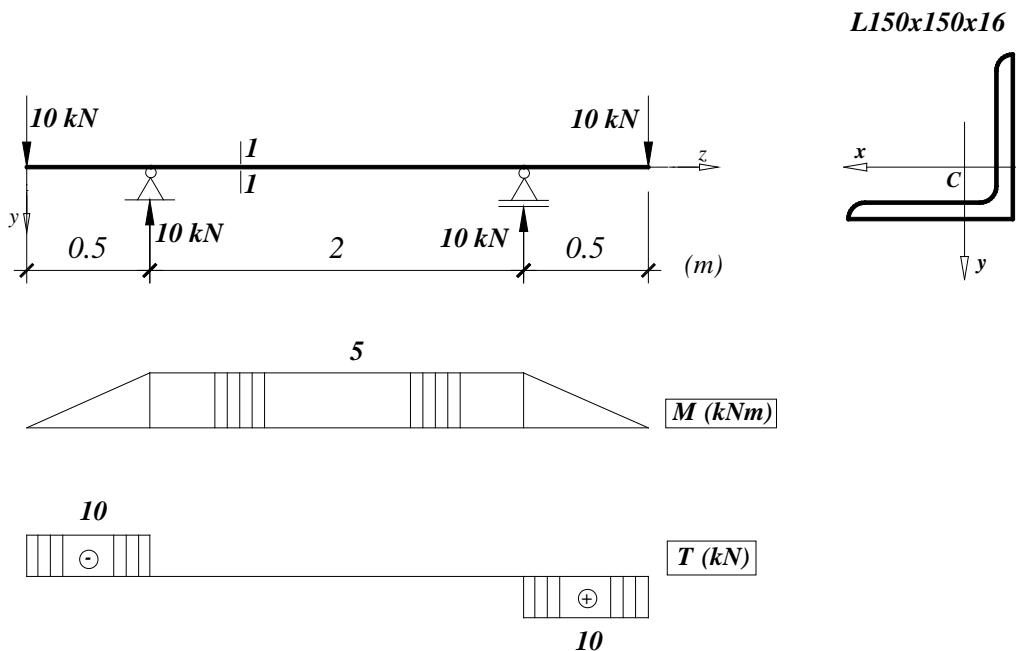
$$\sigma_{z(y=9.28)} = \frac{M_y}{I_y} y = \frac{214.5 \cdot 10^3}{37177.67} 9.28 = 5.354 \text{ kN/cm}^2 = 53.54 \text{ MPa}$$



# VJEŽBA BR. 12

## ČISTO KOSO SAVIJANJE

- I. Poprečni presjek grede sa prepustima je ugaonik **L150x150x16**. Nacrtati dijagram ekstremnih normalnih napona u presjeku **I-I** grede.



**RJEŠENJE**

Geometrijske karakteristike poprečnog presjeka:

**L150x150x16**

$$e = 4.29 \text{ cm}$$

$$A = 45.7 \text{ cm}^2$$

$$I_1 = 1510 \text{ cm}^4$$

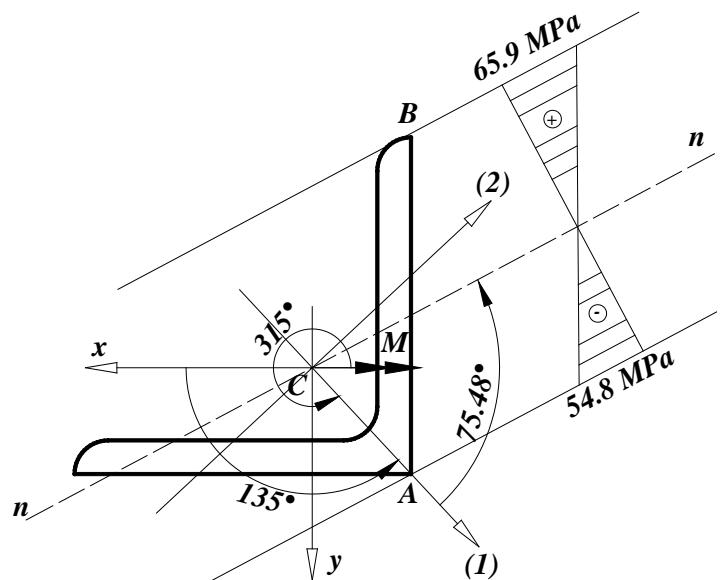
$$I_2 = 391 \text{ cm}^4$$

Ekstremni normalni naponi:

$$\beta = 315^\circ$$

$$\tan \varphi = -\frac{I_1}{I_2} \tan \beta = -\frac{1510}{391} \tan 315^\circ = 3.862$$

$$\varphi = \arctan 3.862 = 75.48^\circ$$



Tačke u kojima se javljaju ekstremne vrijednosti normalnih napona u sistemu osa **xy**:

$$A(x_A; y_A) = (-4.29; 4.29) \text{ cm}$$

$$B(x_B; y_B) = (-4.29; -10.71) \text{ cm}$$

Tačke u kojima se javljaju ekstremne vrijednosti normalnih napona u sistemu glavnih osa **(1)(2)**:

$$(1)_A = x_A \cos \alpha + y_A \sin \alpha = (-4.29) \cos 135^\circ + 4.29 \sin 135^\circ = 6.067 \text{ cm}$$

$$(2)_A = -x_A \sin \alpha + y_A \cos \alpha = 4.29 \sin 135^\circ + 4.29 \cos 135^\circ = 0 \text{ cm}$$

$$(1)_B = x_B \cos \alpha + y_B \sin \alpha = (-4.29) \cos 135^\circ - 10.71 \sin 135^\circ = -4.54 \text{ cm}$$

$$(2)_B = -x_B \sin \alpha + y_B \cos \alpha = 4.29 \sin 135^\circ - 10.71 \cos 135^\circ = 10.607 \text{ cm}$$

$$A((1)_A; (2)_A) = (6.067; 0) \text{ cm}$$

$$B((1)_B; (2)_B) = (-4.54; 10.607) \text{ cm}$$

$$\sigma_z^{A,B} = M \left( \frac{\cos \beta}{I_1} \cdot (2)_{A,B} + \frac{\sin \beta}{I_2} \cdot (1)_{A,B} \right)$$

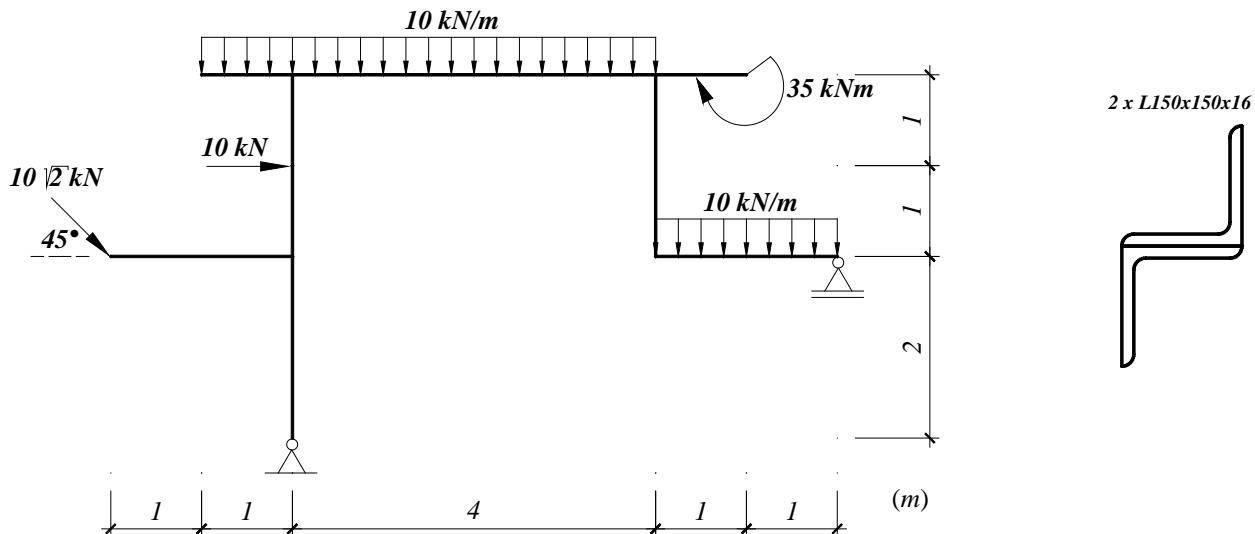
$$\sigma_z^A = 5 \cdot 10^2 \left( \frac{\cos 315^\circ}{1510} \cdot 0 + \frac{\sin 315^\circ}{391} \cdot 6.067 \right) = -5.48 \frac{kN}{cm^2}$$

$$\sigma_z^B = 5 \cdot 10^2 \left( \frac{\cos 315^\circ}{1510} \cdot 10.607 + \frac{\sin 315^\circ}{391} \cdot (-4.54) \right) = 6.59 \frac{kN}{cm^2}$$

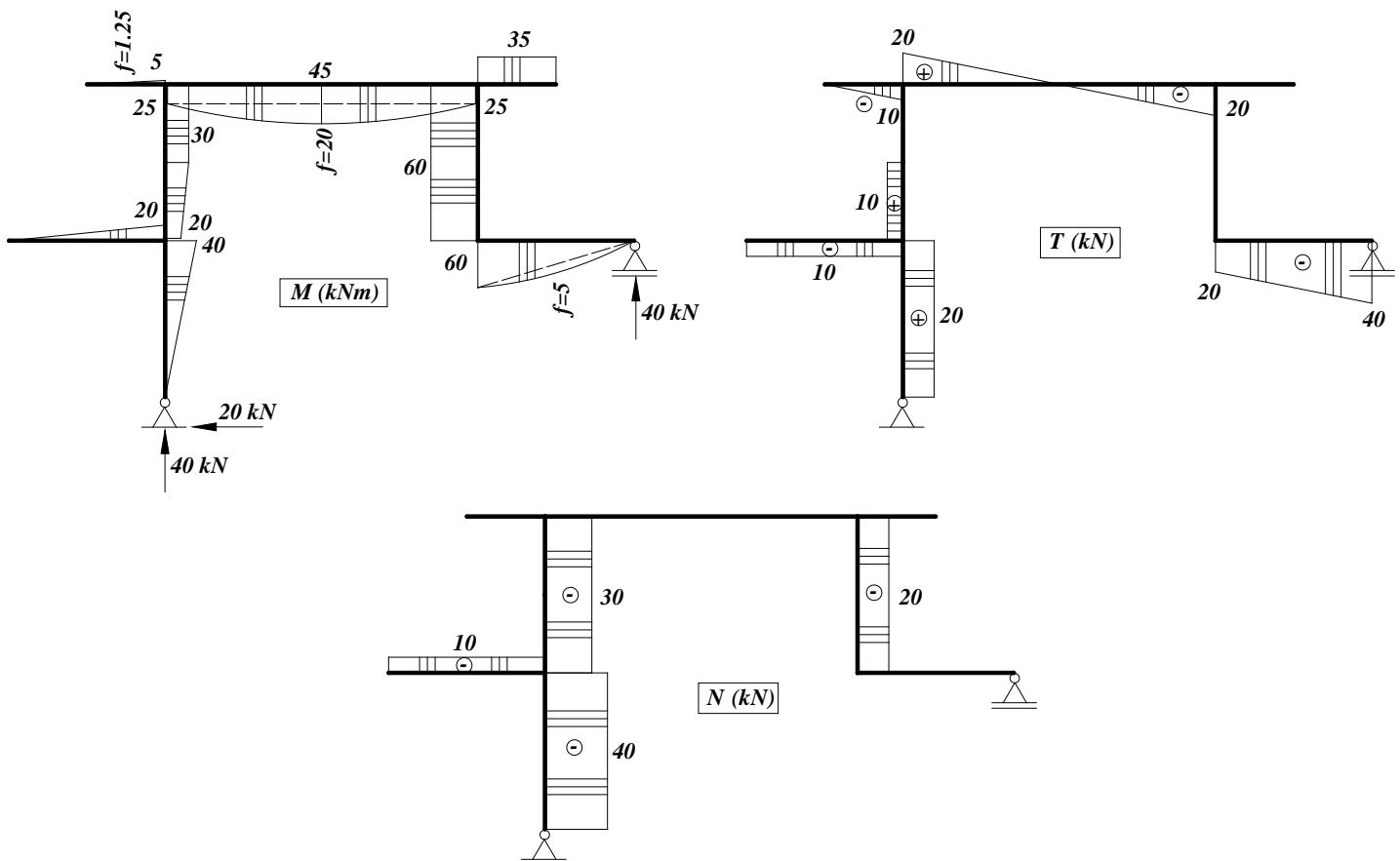
2. Za nosač na slici:

a. Nacrtati dijagrame presječnih sila ( $M$ ,  $T$ ,  $N$ );

b. Na gornjoj rigli, na mjestu maksimalnog momenta, nacrtati dijagram ekstremnih vrijednosti normalnog napona.



**RJEŠENJE**



Geometrijske karakteristike poprečnog presjeka:

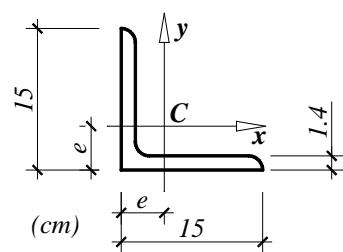
**L150x150x14**

$$e = 4.21 \text{ cm}$$

$$A = 40.3 \text{ cm}^2$$

$$I_x = I_y = 845 \text{ cm}^4$$

$$I_{xy} = -495 \text{ cm}^4$$



$$I_{\xi} = 2[845 + 40.3 \cdot 4.21^2] = 3118.6 \text{ cm}^4$$

$$I_{\eta} = 2[845 + 40.3 \cdot (7.5 - 4.21)^2] = 2562.4 \text{ cm}^4$$

$$I_{\xi\eta} = 2[495 + 40.3 \cdot 4.21 \cdot 3.29] = 2106.4 \text{ cm}^4$$

$$I_{1/2} = \dots = 2840.5 \pm 2124.7$$

$$I_1 = 4965.2 \text{ cm}^4$$

$$I_2 = 715.8 \text{ cm}^4$$

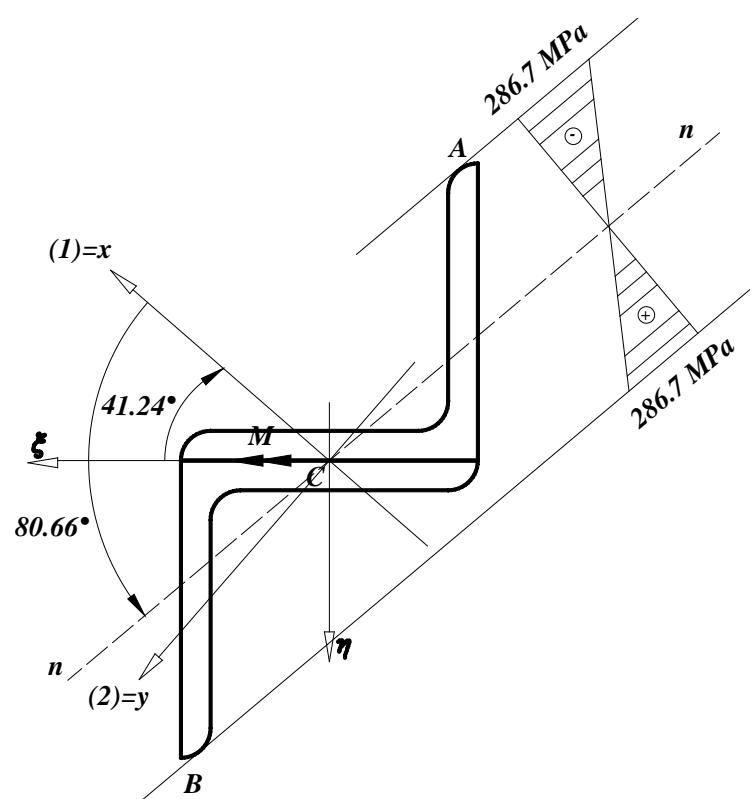
$$\tan 2\alpha = \frac{-2 \cdot 2106.4}{3118.6 - 2562.4} = -7.575 \rightarrow \alpha = -41.24^\circ$$

Ekstremni normalni naponi:

$$\beta = -41.24^\circ$$

$$\tan \varphi = -\frac{I_1}{I_2} \tan \beta = -\frac{4968.2}{715.8} \tan(-41.24^\circ) = 6.081$$

$$\varphi = \arctan 6.081 = 80.66^\circ$$



Tačke u kojima se javljaju ekstremne vrijednosti normalnih napona u sistemu osa  $\xi\eta$ :

$$A(\xi_A; \eta_A) = (-7.5; -15) \text{ cm}$$

$$B(\xi_B; \eta_B) = (7.5; 15) \text{ cm}$$

Tačke u kojima se javljaju ekstremne vrijednosti normalnih napona u sistemu glavnih osa  $xy$ :

$$x_A = \xi_A \cos \alpha + \eta_A \sin \alpha = (-7.5) \cos(-41.24^\circ) + (-15) \sin(-41.24^\circ) = 4.25 \text{ cm}$$

$$y_A = -\xi_A \sin \alpha + \eta_A \cos \alpha = 7.5 \sin(-41.24^\circ) + (-15) \cos(-41.24^\circ) = -16.22 \text{ cm}$$

$$x_B = \xi_B \cos \alpha + \eta_B \sin \alpha = 7.5 \cos(-41.24^\circ) + 15 \sin(-41.24^\circ) = -4.25 \text{ cm}$$

$$y_B = -\xi_B \sin \alpha + \eta_B \cos \alpha = -7.5 \sin(-41.24^\circ) + 15 \cos(-41.24^\circ) = 16.22 \text{ cm}$$

$$A(x_A; y_A) = (4.25; -16.22) \text{ cm}$$

$$B(x_B; y_B) = (-4.25; 16.22) \text{ cm}$$

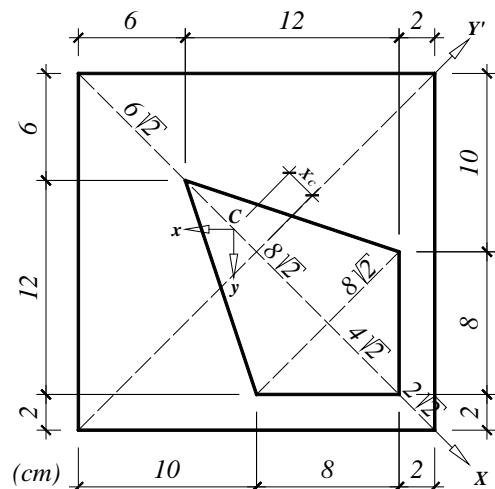
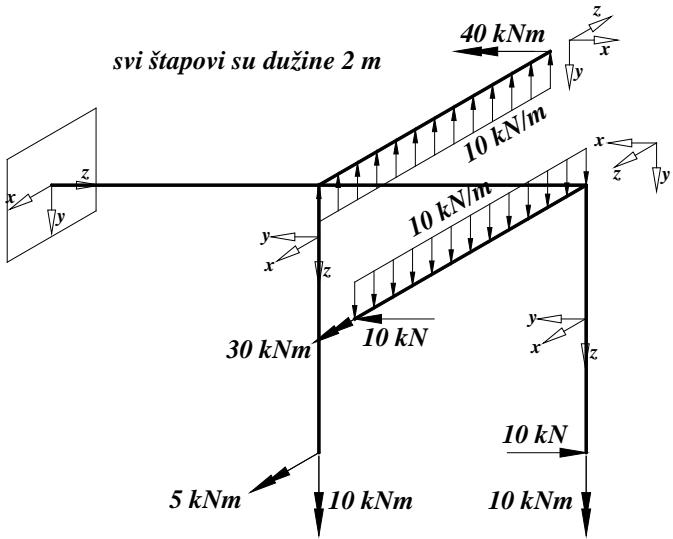
$$\sigma_z^{A,B} = M \left( \frac{\cos \beta}{I_1} \cdot y_{A,B} + \frac{\sin \beta}{I_2} \cdot x_{A,B} \right)$$

$$\sigma_z^A = 45 \cdot 10^2 \left( \frac{\cos(-41.24^\circ)}{4965.2} \cdot (-16.22) + \frac{\sin(-41.24^\circ)}{715.8} \cdot 4.25 \right) = -28.666 \frac{kN}{cm^2}$$

$$\sigma_z^B = 45 \cdot 10^2 \left( \frac{\cos(-41.24^\circ)}{4965.2} \cdot 16.22 + \frac{\sin(-41.24^\circ)}{715.8} \cdot (-4.25) \right) = 28.666 \frac{kN}{cm^2}$$

3. Za nosač na slici:

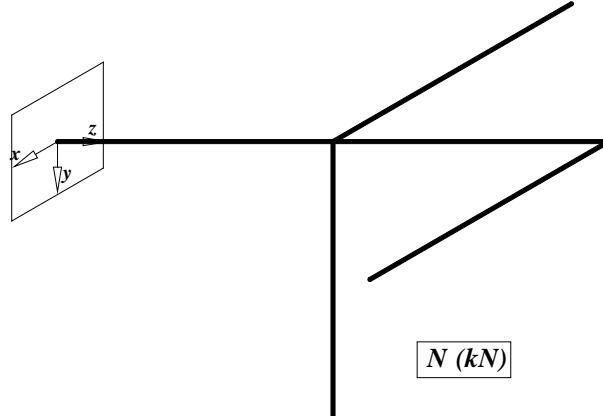
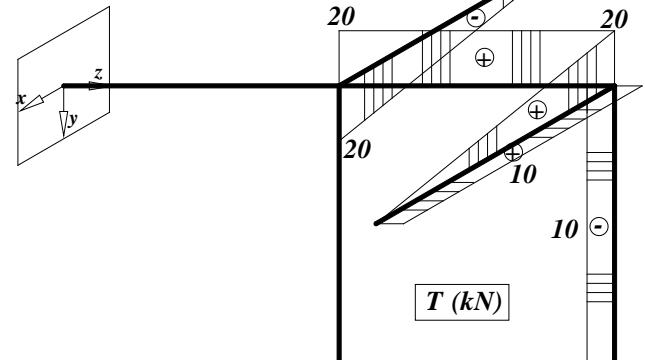
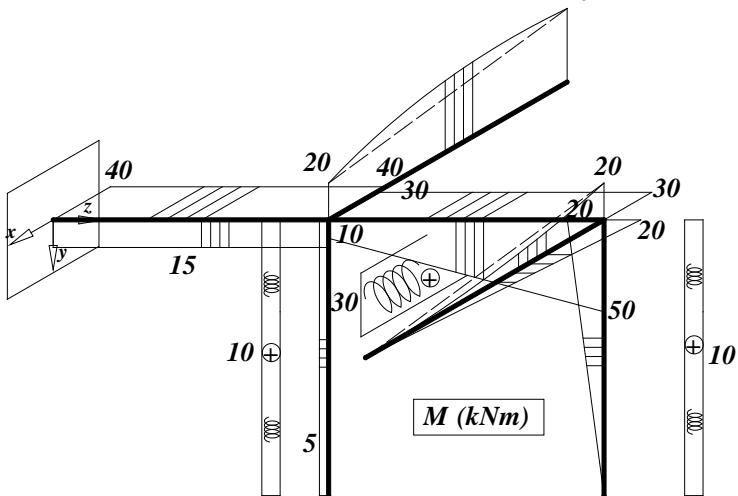
- Nacrtati dijagrame presječnih sila ( $M$ ,  $T$ ,  $N$ );
- Za presjek u uklještenju nacrtati dijagram ekstremnih vrijednosti normalnog napona.



**RJEŠENJE**

20 +

40



Geometrijske karakteristike poprečnog presjeka:

$$X_c = \frac{-1.333\sqrt{2} \cdot 64 - 5.333\sqrt{2} \cdot 32}{304} = -1.191 \text{ cm}$$

$$I_x = \frac{20^4}{12} - \frac{1}{48} \left[ \left(8\sqrt{2}\right)^4 + 4\sqrt{2} \cdot \left(8\sqrt{2}\right)^3 \right] = 12821 \text{ cm}^4 \equiv I_1$$

$$I_y = \frac{20^4}{12} + 400 \cdot 1.191^2 - \frac{1}{36} \left(8\sqrt{2}\right)^4 - 64 \cdot 3.077^2 - \frac{1}{36} 8\sqrt{2} \cdot \left(4\sqrt{2}\right)^3 - 32 \cdot 8.733^2 = 10342 \text{ cm}^4 \equiv I_2$$

$$I_{xy} = 0$$

Ekstremni normalni naponi:

$$M = \sqrt{15^2 + 40^2} = 42.72 \text{ kNm}$$

$$\beta = 45 + \arctg \frac{15}{40} = 65.556^\circ$$

$$\tg \varphi = -\frac{I_1}{I_2} \tg \beta = -\frac{12821}{10342} \tg 65.556^\circ = -2.7279$$

$$\varphi = \arctg (-2.7279) = -69.864^\circ$$

Tačke u kojima se javljaju ekstremne vrijednosti normalnih napona u sistemu glavnih osa **XY**:

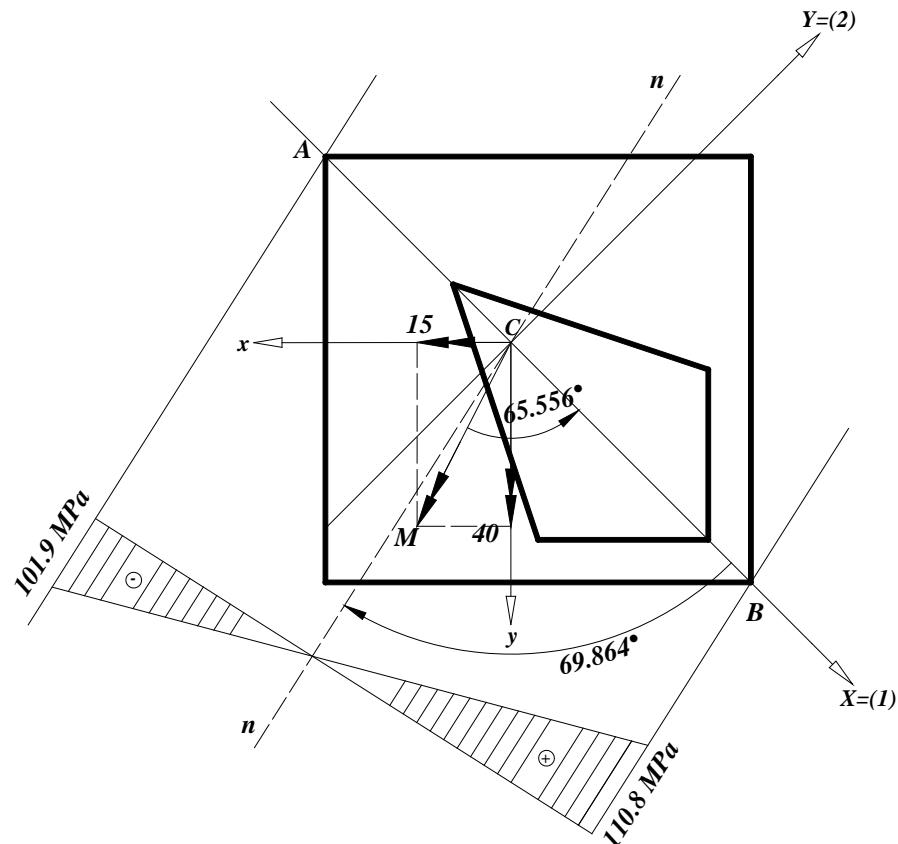
$$A(X_A; Y_A) = (-12.951; 0) \text{ cm}$$

$$B(X_B; Y_B) = (15.333; 0) \text{ cm}$$

$$\sigma_z^{A,B} = M \left( \frac{\cos \beta \cdot Y_{A,B}}{I_1} + \frac{\sin \beta \cdot X_{A,B}}{I_2} \right)$$

$$\sigma_z^A = 42.72 \cdot 10^2 \left( \frac{\cos 65.556^\circ}{12821} \cdot 0 + \frac{\sin 65.556^\circ}{10342} \cdot (-12.951) \right) = -10.19 \frac{\text{kN}}{\text{cm}^2}$$

$$\sigma_z^B = 42.72 \cdot 10^2 \left( \frac{\cos 65.556^\circ}{12821} \cdot 0 + \frac{\sin 65.556^\circ}{10342} \cdot 15.333 \right) = 11.08 \frac{\text{kN}}{\text{cm}^2}$$



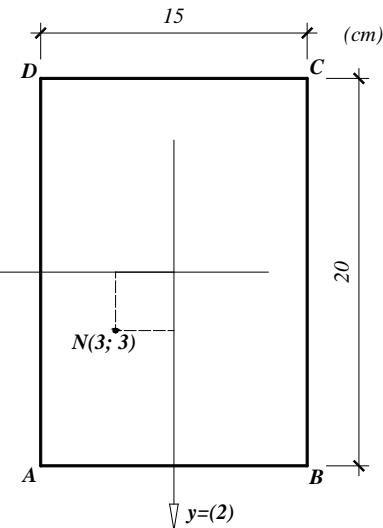
# VJEŽBA BR. 13

## EKSCENTRIČNO NAPREZANJE

- I. Drveni stub pravougaonog poprečnog presjeka  $15 \times 20\text{cm}$  opterećen je ekscentričnom silom pritiska od  $90\text{kN}$  koja djeluje u tački  $N(3;3)$ . Odrediti položaj neutralne linije, dijagram ekstremnih normalnih naponi, kao i dijagram normalnih naponi duž stranice  $BC$  pravougaonika.

**RJEŠENJE**

$$N(3;3) \rightarrow a=3\text{cm}, b=3\text{cm}$$



Geometrijske karakteristike poprečnog presjeka:

$$A = 15 \cdot 20 = 300 \text{ cm}^2$$

$$I_x = \frac{15 \cdot 20^3}{12} = 10000 \text{ cm}^4$$

$$I_y = \frac{15^3 \cdot 20}{12} = 5625 \text{ cm}^4$$

$$i_x^2 = \frac{I_x}{A} = \frac{10000}{300} = 33.333 \text{ cm}^2$$

$$i_y^2 = \frac{I_y}{A} = \frac{5625}{300} = 18.75 \text{ cm}^2$$

Neutralna linija i normalni naponi:

$$p = -\frac{i_y^2}{a} = -\frac{18.75}{3} = -6.25 \text{ cm}$$

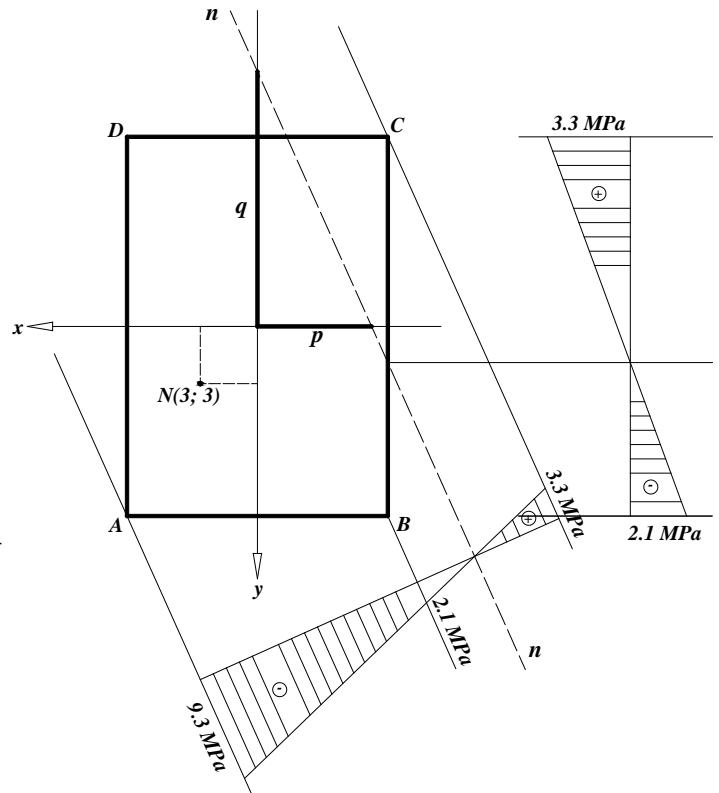
$$q = -\frac{i_x^2}{b} = -\frac{33.333}{3} = -11.111 \text{ cm}$$

$$\sigma_z = \pm \frac{P}{A} \left( 1 + \frac{b}{i_x^2} \cdot y + \frac{a}{i_y^2} \cdot x \right)$$

$$\sigma_z^A = -\frac{90}{300} \left( 1 + \frac{3}{33.333} \cdot 10 + \frac{3}{18.75} \cdot 7.5 \right) = -0.93 \frac{\text{kN}}{\text{cm}^2}$$

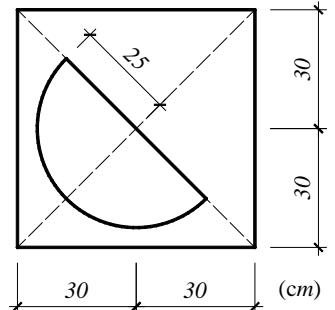
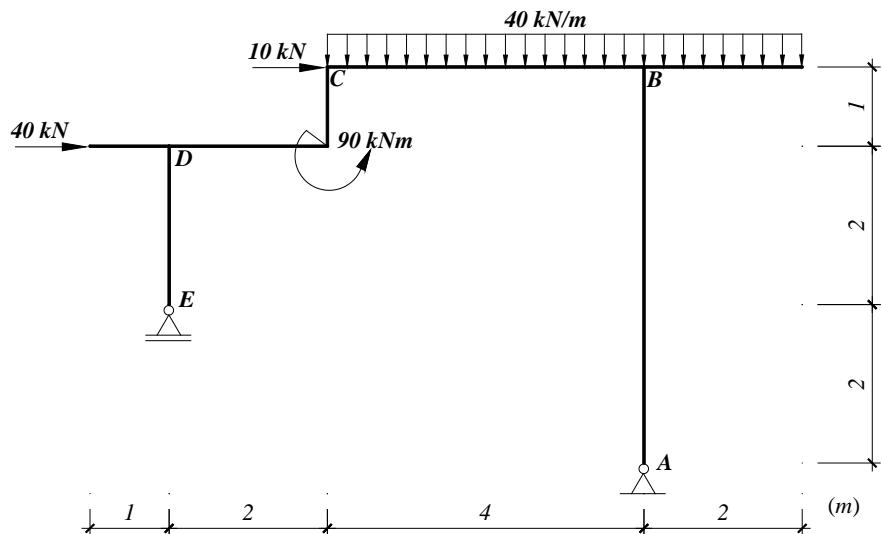
$$\sigma_z^C = -\frac{90}{300} \left( 1 + \frac{3}{33.333} \cdot (-10) + \frac{3}{18.75} \cdot (-7.5) \right) = 0.33 \frac{\text{kN}}{\text{cm}^2}$$

$$\sigma_z^B = -\frac{90}{300} \left( 1 + \frac{3}{33.333} \cdot (-7.5) + \frac{3}{18.75} \cdot 10 \right) = -0.21 \frac{\text{kN}}{\text{cm}^2}$$

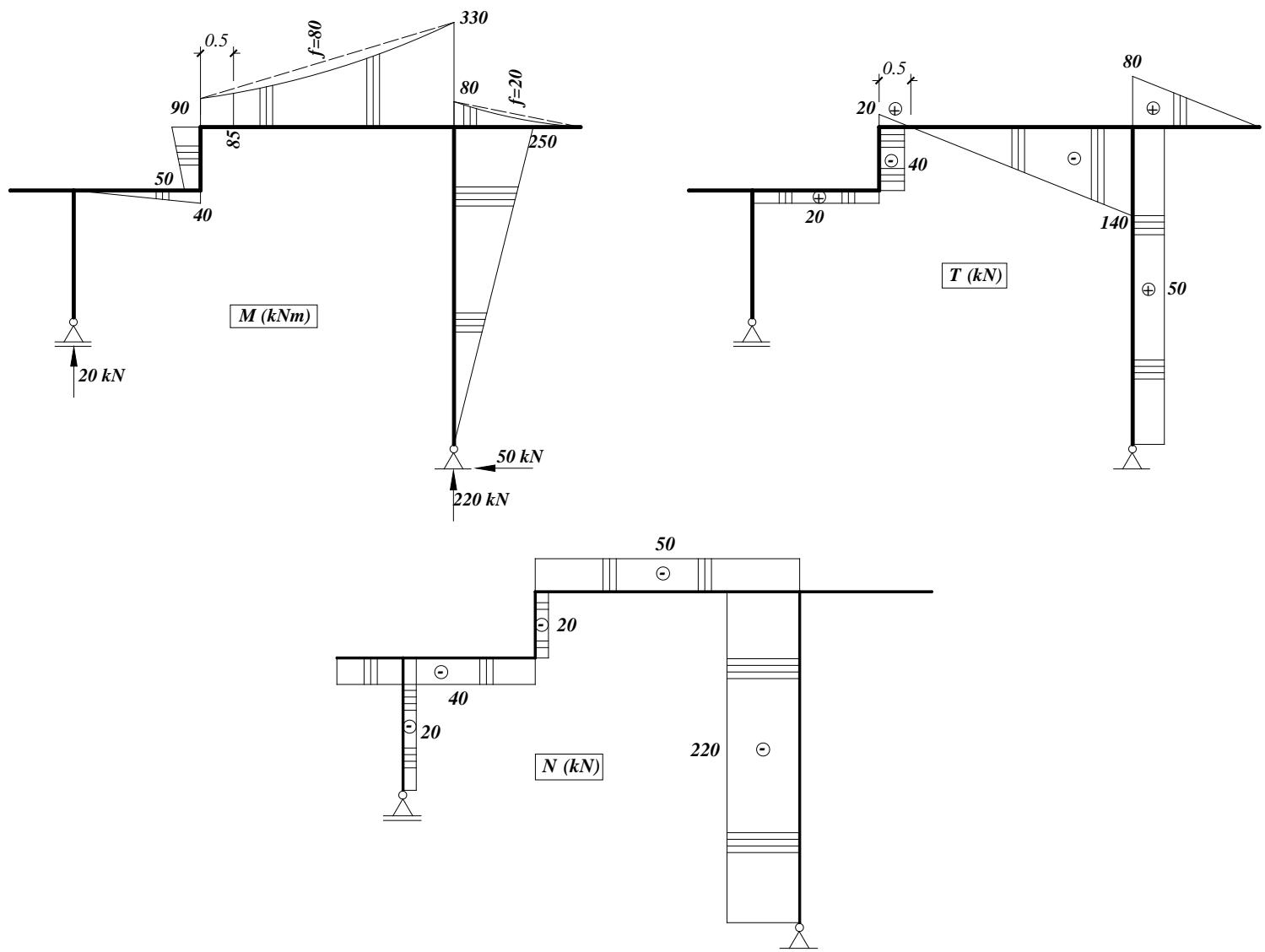


2. Za nosač na slici:

- a. Nacrtati dijagrame presječnih sila ( $M$ ,  $T$ ,  $N$ );
  - b. Na gornjoj rigli (dio  $BC$ ), na mjestu gdje je transverzalna sila jednaka nuli, nacrtati dijagram ekstremnih vrijednosti normalnog napona.



RJEŠENJE



Geometrijske karakteristike poprečnog presjeka:

$$C_1(0;0) \quad A_1 = 3600 \text{ cm}^2$$

$$C_2\left(-\frac{4 \cdot 25}{3\pi}; 0\right) = (-10.61; 0) \quad A_1 = 981.742 \text{ cm}^2$$

$$x_C = \frac{-(-10.61) \cdot 981.748}{2618.252} = 3.978 \text{ cm}$$

$$I_x = \frac{1}{12} 60^4 - \frac{25^4 \pi}{8} = 926\,602 \text{ cm}^4 \equiv I_1$$

$$I_y = \frac{1}{12} 60^4 + 60^2 \cdot 3.978^2 - 0.10976 \cdot 25^4 - 981.748 \cdot 14.588^2 = 885\,168 \text{ cm}^4 \equiv I_2$$

$$i_1^2 = i_x^2 = \frac{926\,602}{2618.252} = 353.9 \text{ cm}^2$$

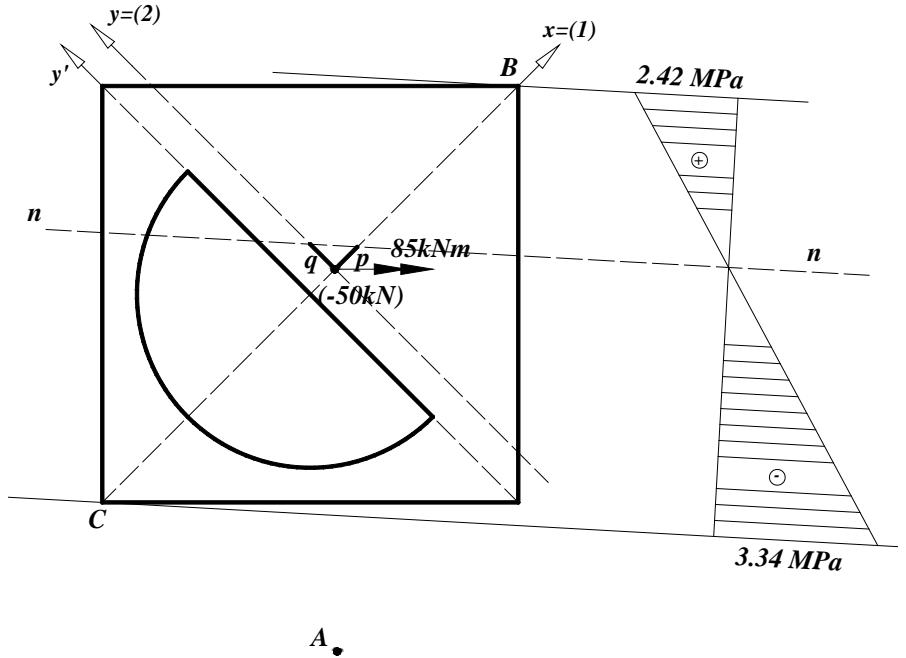
$$i_2^2 = i_y^2 = \frac{885\,168}{2618.252} = 338.1 \text{ cm}^2$$

Ekvivalentno opterećenje:

$$A(a; b) = A\left(\frac{\Sigma M_y}{P}; \frac{\Sigma M_x}{P}\right)$$

$$A = \left( \frac{8500 \frac{\sqrt{2}}{2}}{-50}; \frac{8500 \frac{\sqrt{2}}{2}}{-50} \right)$$

$$A = (-120.21; -120.21) \text{ cm}$$



Neutralna linija:

$$p = -\frac{i_y^2}{a} = -\frac{338.1}{-121.21} = 2.812 \text{ cm}$$

$$q = -\frac{i_x^2}{b} = -\frac{353.9}{-121.21} = 2.944 \text{ cm}$$

Tačke u kojima se javljaju ekstremne vrijednosti normalnih napona u sistemu **glavnih osa xy**:

$$B(x_B; y_B) = \left( \left( 30\sqrt{2} - 3.978 \right); 0 \right) = (38.448; 0) \text{ cm}$$

$$C(x_C; y_C) = \left( -\left( 30\sqrt{2} + 3.978 \right); 0 \right) = (-46.404; 0) \text{ cm}$$

Ekstremni normalni naponi:

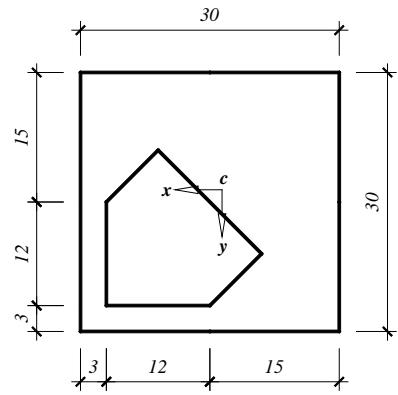
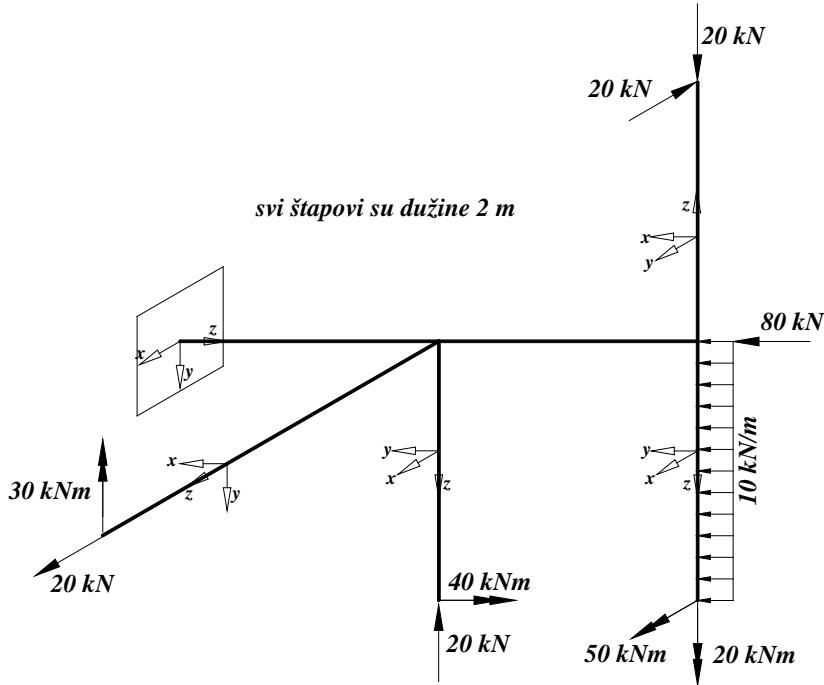
$$\sigma_z = \pm \frac{P}{A} \left( 1 + \frac{b}{i_x^2} \cdot y + \frac{a}{i_y^2} \cdot x \right)$$

$$\sigma_z^B = -\frac{50}{2618.252} \left( 1 + \frac{-120.21}{353.9} \cdot 0 + \frac{-120.21}{338.1} \cdot 38.448 \right) = 0.242 \frac{kN}{cm^2}$$

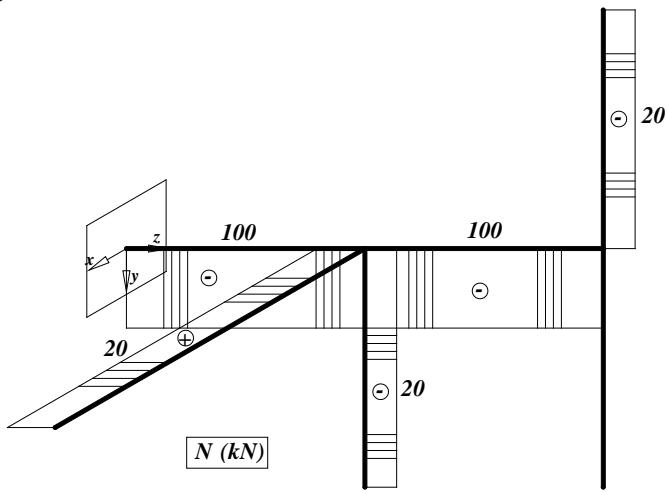
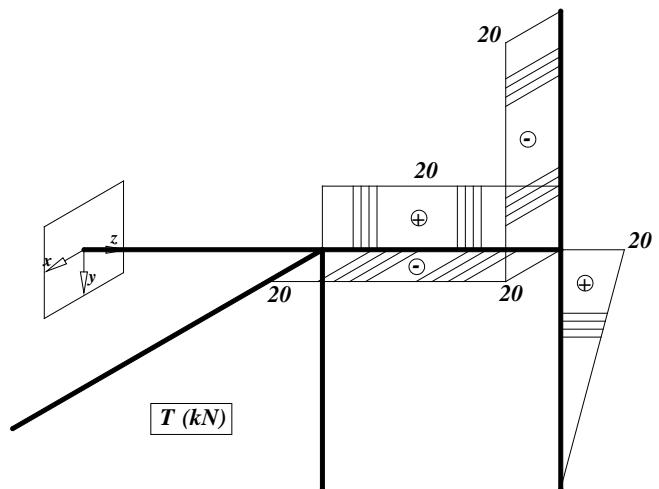
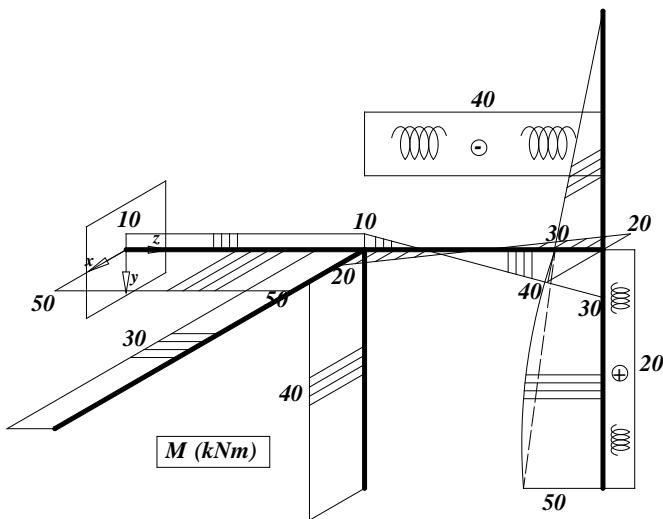
$$\sigma_z^C = -\frac{50}{2618.252} \left( 1 + \frac{-120.21}{353.9} \cdot 0 + \frac{-120.21}{338.1} \cdot (-46.404) \right) = -0.334 \frac{kN}{cm^2}$$

3. Za nosač na slici:

- Nacrtati dijagrame presječnih sila ( $M$ ,  $T$ ,  $N$ );
- Za presjek u uklještenju nacrtati dijagram ekstremnih vrijednosti normalnog napona.



**RJEŠENJE**



Geometrijske karakteristike poprečnog presjeka:

$$C_1(0;0) \ A_l = 900 \text{ cm}^2$$

$$C_2(3\sqrt{2};0) \ A_l = 144 \text{ cm}^2$$

$$C_3(8\sqrt{2};0) \ A_l = 72 \text{ cm}^2$$

$$\Sigma A = 684 \text{ cm}^2$$

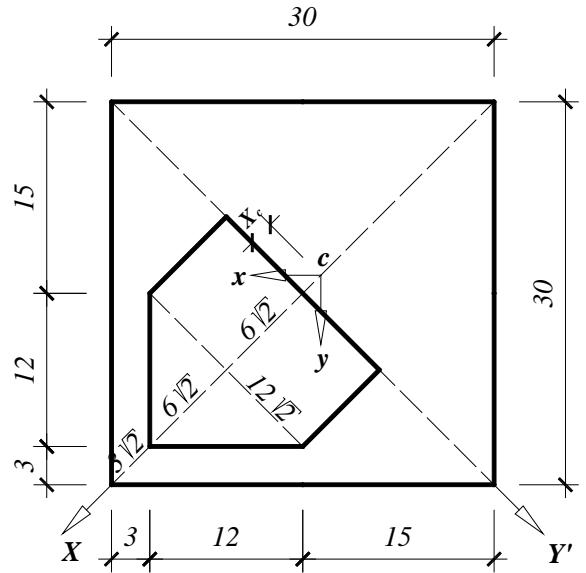
$$X_C = \frac{-3\sqrt{2} \cdot 144 - 8\sqrt{2} \cdot 72}{684} = -2.084 \text{ cm}$$

$$I_x = \frac{1}{12} 30^4 - \frac{1}{48} 6\sqrt{2} (12\sqrt{2})^3 - \frac{1}{12} 6\sqrt{2} (12\sqrt{2})^3 = 63180 \text{ cm}^4 \equiv I_1$$

$$I_y = \frac{1}{12} 30^4 + 30^2 \cdot 2.084^2 - \left[ \frac{1}{36} 12\sqrt{2} (6\sqrt{2})^3 + 72 \cdot 13.40^2 \right] - \left[ \frac{1}{12} 12\sqrt{2} (6\sqrt{2})^3 + 144 \cdot 6.327^2 \right] = 51564 \text{ cm}^4 \equiv I_2$$

$$i_l^2 = i_x^2 = \frac{63180}{684} = 92.37 \text{ cm}^2$$

$$i_2^2 = i_y^2 = \frac{51564}{684} = 75.386 \text{ cm}^2$$



Ekvivalentno opterećenje:

**I način** (određivanje koordinata napadne tačke **A** u sistemu osa **xy**, a zatim rotacija koordinata da bi se dobile koordinate napadne tačke u sistemu **glavnih osa 12 (XY)**, tj **a** i **b**)

$$A(x_A; y_A) = A\left(\frac{\Sigma M_y}{P}; \frac{\Sigma M_x}{P}\right) = \left(\frac{50 \cdot 10^2}{-100}; \frac{-10 \cdot 10^2}{-100}\right) = (-50; 10) \text{ cm}$$

$$X_A = x_A \cos \alpha + y_A \sin \alpha = (-50) \cos 45^\circ + 10 \sin 45^\circ = -28.28 \text{ cm} \equiv a$$

$$Y_A = -x_A \sin \alpha + y_A \cos \alpha = 50 \sin 45^\circ + 10 \cos 45^\circ = 42.43 \text{ cm} \equiv b$$

$$A(X_A; Y_A) = (a; b) = (-28.28; 42.43) \text{ cm}$$

**II način** (projektovanje vektora napadnih momenata na **glavne ose** i proračun koordinata napadne tačke **A** u sistemu **glavnih osa 12 (XY)**, tj **a** i **b**)

$$\Sigma M_x = -\left(50 \frac{\sqrt{2}}{2} + 10 \frac{\sqrt{2}}{2}\right) = -42.43 \text{ kNm}$$

$$\Sigma M_y = 50 \frac{\sqrt{2}}{2} - 10 \frac{\sqrt{2}}{2} = 28.28 \text{ kNm}$$

$$A(X_A; Y_A) = (a; b) = \left(\frac{\Sigma M_y}{P}; \frac{\Sigma M_x}{P}\right) = \left(\frac{28.28 \cdot 10^2}{-100}; \frac{-42.43 \cdot 10^2}{-100}\right) = (-28.28; 42.43) \text{ cm}$$

Neutralna linija:

$$p = -\frac{i_y^2}{a} = -\frac{75.386}{-28.28} = 2.666 \text{ cm}$$

$$q = -\frac{i_x^2}{b} = -\frac{92.37}{42.43} = -2.17 \text{ cm}$$

Tačke u kojima se javljaju ekstremne vrijednosti normalnih napona u sistemu glavnih osa **XY**:

$$B(X_B; Y_B) = (2.084; 15\sqrt{2}) \text{ cm}$$

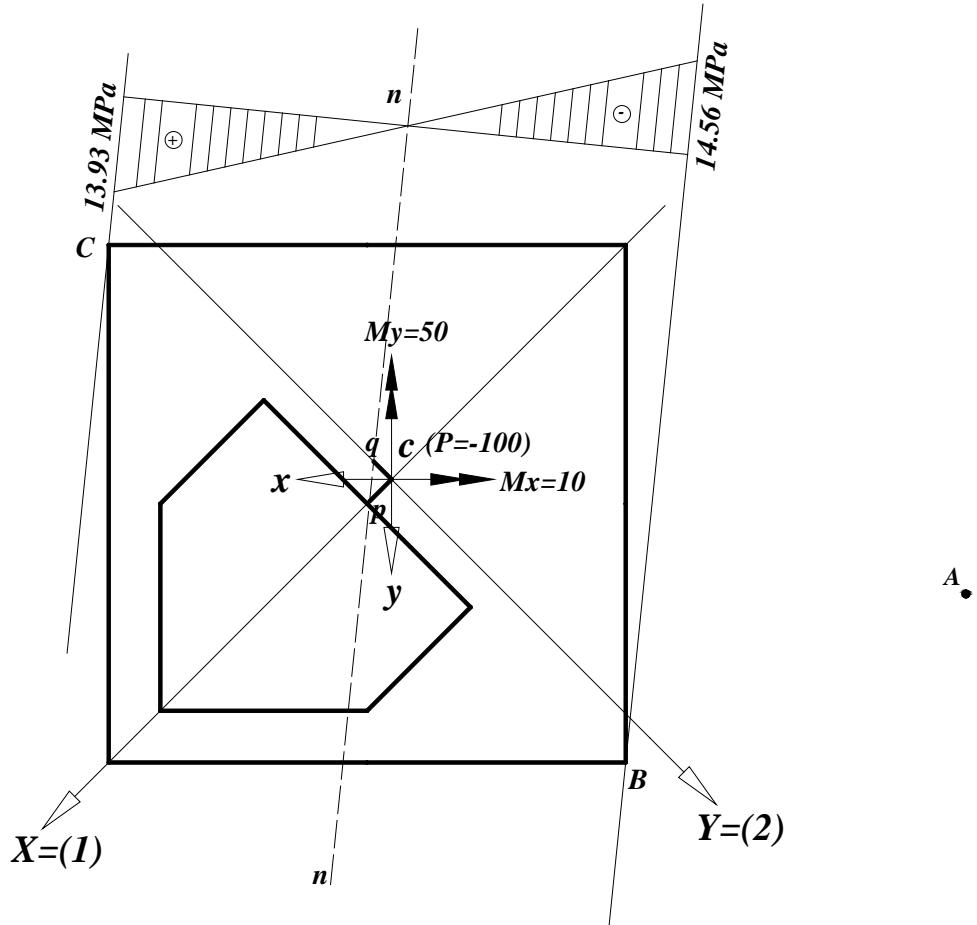
$$C(X_C; Y_C) = (2.084; -15\sqrt{2}) \text{ cm}$$

Ekstremni normalni naponi:

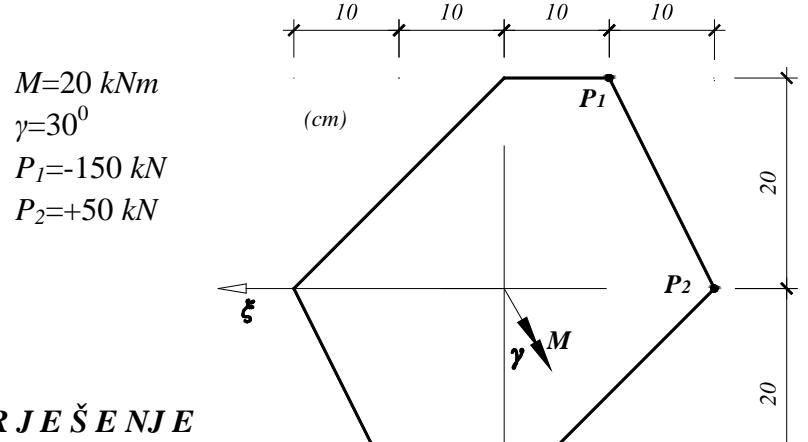
$$\sigma_z = \pm \frac{P}{A} \left( 1 + \frac{b}{i_X^2} \cdot Y + \frac{a}{i_Y^2} \cdot X \right)$$

$$\sigma_z^B = -\frac{100}{684} \left( 1 + \frac{42.43}{92.37} \cdot 15\sqrt{2} + \frac{-28.28}{75.386} \cdot 2.084 \right) = -1.456 \frac{kN}{cm^2}$$

$$\sigma_z^C = -\frac{100}{684} \left( 1 + \frac{42.43}{92.37} \cdot (-15\sqrt{2}) + \frac{-28.28}{75.386} \cdot 2.084 \right) = 1.393 \frac{kN}{cm^2}$$



4. Betonski stub je opterećen kao na slici. Sračunati ekstremne normalne napone u poprečnom presjeku stuba i nacrtati dijagram ekstremnih normalnih napona.



Geometrijske karakteristike poprečnog presjeka:

$$A = 40^2 - 2 \frac{10 \cdot 20}{2} - 2 \frac{20 \cdot 20}{2} = 1000 \text{ cm}^2$$

$$I_{\xi} = \frac{1}{12} 40^4 - 2 \left[ \frac{1}{36} 20^4 + 200 \cdot 13.333^2 \right] - 2 \left[ \frac{1}{36} 10 \cdot 20^3 + 100 \cdot 13.333^2 \right] = 93.333 \text{ cm}^4$$

$$I_{\eta} = \frac{1}{12} 40^4 - 2 \left[ \frac{1}{36} 20^4 + 200 \cdot 13.333^2 \right] - 2 \left[ \frac{1}{36} 10^3 \cdot 20 + 100 \cdot 16.667^2 \right] = 76.667 \text{ cm}^4$$

$$I_{\xi\eta} = -2 \left[ \frac{20^4}{72} + 200 \cdot 13.333 \cdot (-13.333) \right] - 2 \left[ -\frac{10^2 \cdot 20^2}{72} + 100 \cdot 13.333 \cdot 16.667 \right] = 23.333 \text{ cm}^4$$

$$I_{1/2} = \dots = 85.000 \pm 24.777$$

$$I_1 = 109.777 \text{ cm}^4 \equiv I_x$$

$$I_2 = 60.223 \text{ cm}^4 \equiv I_y$$

$$\tan 2\alpha = \frac{-2 \cdot 23.333}{76.667 - 23.333} = -2.8 \quad \rightarrow \quad \alpha = -35.173^\circ$$

$$i_1^2 = i_x^2 = \frac{109.777}{1000} = 109.777 \text{ cm}^2$$

$$i_2^2 = i_y^2 = \frac{60.223}{1000} = 60.223 \text{ cm}^2$$

Ekvivalentno opterećenje:

$$\Sigma M_{\xi} = -20 \sin 30^\circ + 150 \cdot 0.2 = 20 \text{ kNm}$$

$$\Sigma M_{\eta} = -20 \cos 30^\circ + 150 \cdot 0.1 - 50 \cdot 0.2 = -12.32 \text{ kNm}$$

$$\Sigma P = P_1 + P_2 = -150 + 50 = -100 \text{ kN}$$

$$A(\xi_A; \eta_A) = \left( \frac{\Sigma M_{\eta}}{\Sigma P}; \frac{\Sigma M_{\xi}}{\Sigma P} \right) = \left( \frac{-12.32 \cdot 10^2}{-100}; \frac{20 \cdot 10^2}{-100} \right) = (12.32; -20) \text{ cm}$$

$$x_A = \xi_A \cos \alpha + \eta_A \sin \alpha = 12.32 \cos(-35.173^\circ) - 20 \sin(-35.173^\circ) = 21.59 \text{ cm} \equiv a$$

$$y_A = -\xi_A \sin \alpha + \eta_A \cos \alpha = -12.32 \sin(-35.173^\circ) - 20 \cos(-35.173^\circ) = -9.25 \text{ cm} \equiv b$$

$$A(x_A; y_A) = (a; b) = (21.59; -9.25) \text{ cm}$$

Neutralna linija:

$$p = -\frac{i_y^2}{a} = -\frac{60.223}{21.59} = -2.79 \text{ cm}$$

$$q = -\frac{i_x^2}{b} = -\frac{109.777}{-9.25} = 11.88 \text{ cm}$$

Tačke u kojima se javljaju ekstremne vrijednosti normalnih napona u sistemu osa  $\xi\eta$ :

$$B(\xi_B; \eta_B) = (0; -20) \text{ cm}$$

$$C(\xi_C; \eta_C) = (0; 20) \text{ cm}$$

Tačke u kojima se javljaju ekstremne vrijednosti normalnih napona u sistemu glavnih osa  $xy$ :

$$x_B = \xi_B \cos \alpha + \eta_B \sin \alpha = 0 \cos(-35.173^\circ) - 20 \sin(-35.173^\circ) = 11.521 \text{ cm}$$

$$y_B = -\xi_B \sin \alpha + \eta_B \cos \alpha = 0 \sin(-35.173^\circ) - 20 \cos(-35.173^\circ) = -16.348 \text{ cm}$$

$$x_C = \xi_C \cos \alpha + \eta_C \sin \alpha = 0 \cos(-35.173^\circ) + 20 \sin(-35.173^\circ) = -11.521 \text{ cm}$$

$$y_C = -\xi_C \sin \alpha + \eta_C \cos \alpha = 0 \sin(-35.173^\circ) + 20 \cos(-35.173^\circ) = 16.348 \text{ cm}$$

$$B(x_B; y_B) = (11.521; -16.348) \text{ cm}$$

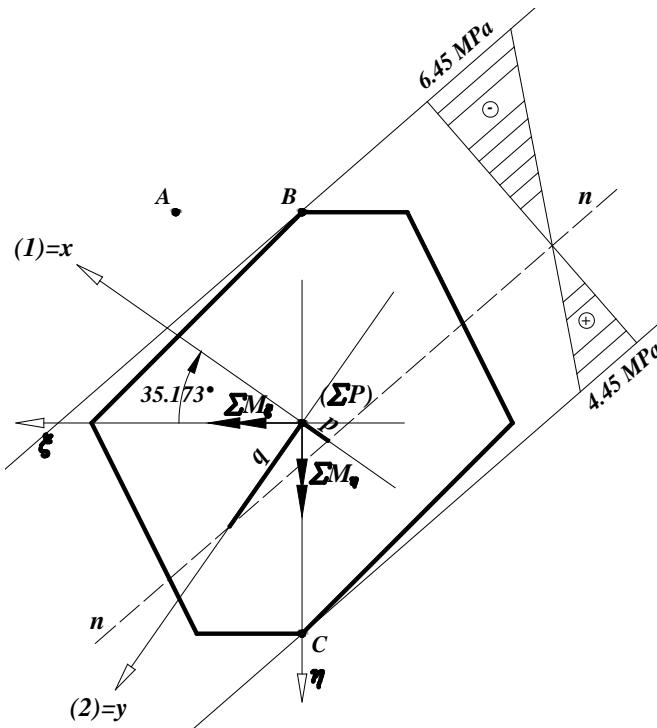
$$C(x_C; y_C) = (-11.521; 16.348) \text{ cm}$$

Ekstremni normalni naponi:

$$\sigma_z = \pm \frac{\Sigma P}{A} \left( 1 + \frac{b}{i_x^2} \cdot y + \frac{a}{i_y^2} \cdot x \right)$$

$$\sigma_z^B = -\frac{100}{684} \left( 1 + \frac{-9.25}{109.777} \cdot (-16.348) + \frac{21.59}{60.223} \cdot 11.521 \right) = -0.645 \frac{kN}{cm^2}$$

$$\sigma_z^C = -\frac{100}{684} \left( 1 + \frac{-9.25}{109.777} \cdot 16.348 + \frac{21.59}{60.223} \cdot (-11.521) \right) = 0.445 \frac{kN}{cm^2}$$

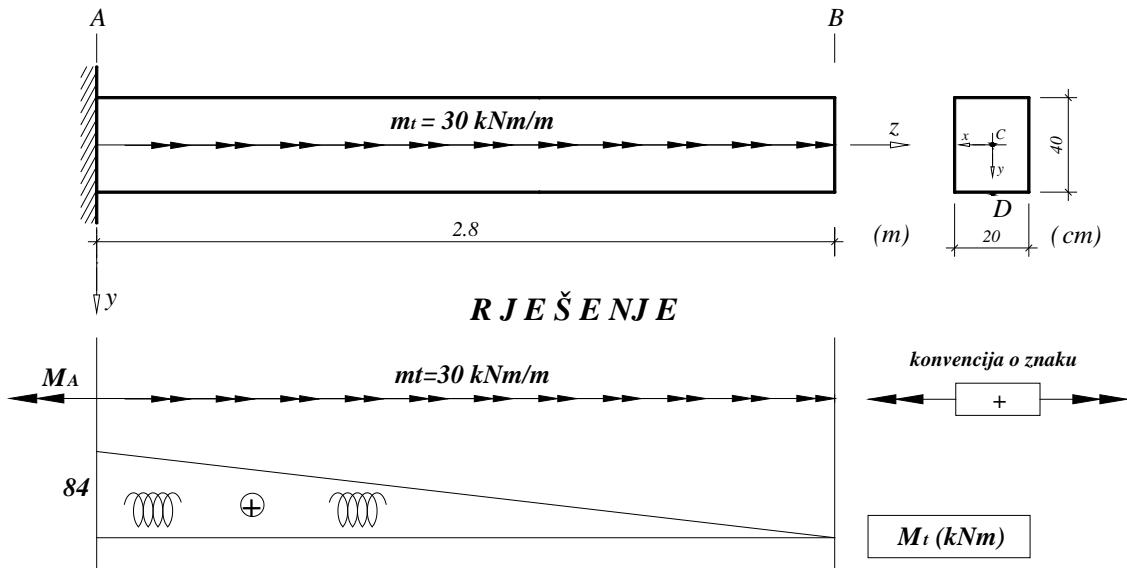


# VJEŽBA BR. 14

## TORZIJA

I. Za konzolni nosač na slici:

- Nacrtati dijagram promjene momenta torzije duž nosača ( $M_t$ );
- Sračunati maksimalni smičući napon ( $\tau_{max}$ ) i definisati presjek u kome on djeluje;
- Nacrtati dijagram smičućih napona u presjeku u uklještenju;
- Analizirati stanje napona u tački D;
- Sračunati obrtanje poprečnog presjeka na slobodnom kraju nosača ( $\varphi_{BA}$ ), ako je  $G=80 \text{ GPa}$ .



a.  $M_A = m_t \cdot l = 30 \text{ kNm} / \text{m} \cdot 2.8 \text{ m} = 84 \text{ kNm}$

b.  $\tau_{max} = \frac{M_{t,max}}{W_t}; W_t = const \text{ (const poprečni presjek)} \rightarrow \tau_{max} \text{ u presjeku u uklještenju}$

$$\frac{h}{b} = \frac{40}{20} = 2 \rightarrow \begin{cases} \alpha = 0.457 \\ \beta = 0.493 \\ \gamma = 0.795 \end{cases} \rightarrow I_t = \alpha \cdot b^4 = 0.457 \cdot 20^4 = 7.312 \cdot 10^4 \text{ cm}^4$$

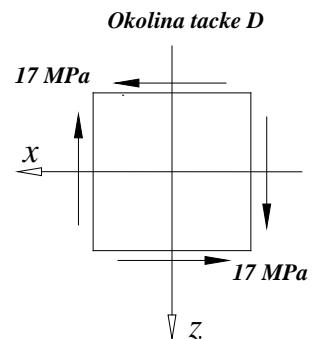
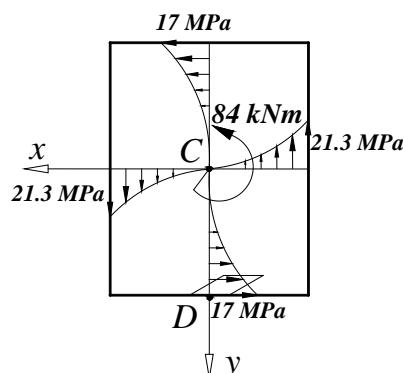
$$W_t = \beta \cdot b^3 = 0.493 \cdot 20^3 = 3.944 \cdot 10^3 \text{ cm}^3$$

$$\tau_{max} = \frac{M_{t,max}}{W_t} = \frac{84 \cdot 10^2}{3.944 \cdot 10^3} = 2.13 \text{ kN/cm}^2$$

c.  $\tau_{max} = 2.13 \text{ kN/cm}^2$

$$\tau' = \gamma \cdot \tau_{max} = 0.795 \cdot 2.13 = 1.7 \text{ kN/cm}^2$$

d.  $\tau_{zx}^D = -1.7 \text{ kN/cm}^2$

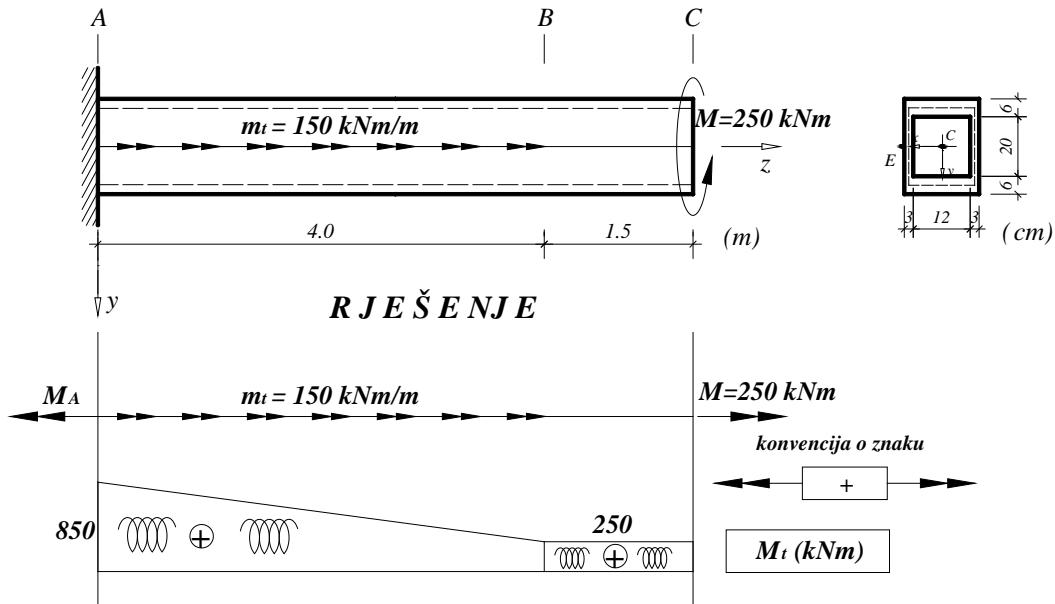


e.  $\varphi_{BA} = \int_0^{2.8} \frac{84 - 30 \cdot z}{G \cdot I_t} dz = \frac{1}{G \cdot I_t} \left( 84 \cdot z - 30 \frac{z^2}{2} \right) \Big|_0^{2.8} =$

$$= \frac{1}{G \cdot I_t} \left( 84 \cdot 2.8 - 30 \frac{2.8^2}{2} \right) = \frac{117.6 \text{ kNm}^2}{G \cdot I_t} = \frac{117.6 \cdot 10^4}{80 \cdot 10^2 \cdot 7.312 \cdot 10^4} = 2.01 \cdot 10^{-3} \text{ rad} \cdot \frac{180}{\pi} = 0.115^\circ$$

2. Za konzolni nosač na slici:

- Nacrtati dijagram promjene momenta torzije duž nosača ( $M_t$ );
- Sračunati maksimalni smičući napon ( $\tau_{max}$ ) i definisati presjek u kome on djeluje;
- Nacrtati dijagram smičućih napona u presjeku na slobodnom kraju konzole;
- Analizirati stanje napona u tački E;
- Sračunati obrtanje poprečnog presjeka na slobodnom kraju konzole ( $\varphi_{CA}$ ), ako je  $G=80 \text{ GPa}$ .



$$a. M_A = m_t \cdot l + M = 150 \cdot 4 + 250 = 850 \text{ kNm}$$

$$b. \tau_{max} = \frac{M_{t,max}}{W_t} \quad (\tau_{max} \text{ djeluje u presjeku u kome djeluje maksimalni } M_t, \text{ a to je presjek u uklještenju})$$

Geometrijske karakteristike poprečnog presjeka:

$$A_s = 26 \cdot 15 = 390 \text{ cm}^2$$

$$W_{t,i} = 2 \cdot A_s \cdot t_i$$

$$W_{t,1} = 2 \cdot 390 \cdot 6 = 4680 \text{ cm}^3$$

$$W_{t,2} = 2 \cdot 390 \cdot 3 = 2340 \text{ cm}^3$$

$$I_t = \frac{4 \cdot A_s^2}{\sum_i \frac{s_i}{t_i}} = \frac{4 \cdot 390^2}{\left( \frac{26}{3} + \frac{15}{6} \right) \cdot 2} = 27241 \text{ cm}^4$$

Maksimalni smičući napon:

$$\tau_{max} = \frac{M_{t,max}}{W_t^{\min}} = \frac{M_{t,max}}{W_{t,2}} = \frac{850 \cdot 10^2}{2340} = 36.325 \text{ kN/cm}^2$$

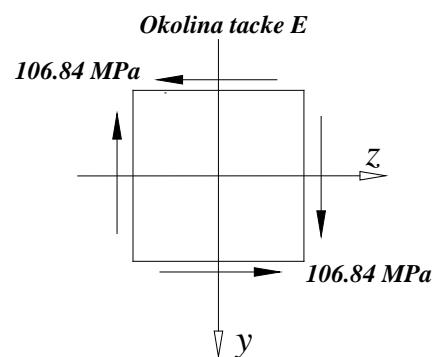
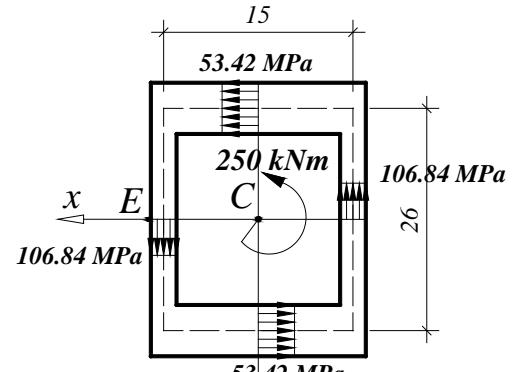
- Presjek na slobodnom kraju konzole (presjek C):

$$M_t^C = 250 \text{ kNm}$$

$$\tau_C^{t=6} = \frac{M_t^C}{W_{t,1}} = \frac{250 \cdot 10^2}{4680} = 5.342 \text{ kN/cm}^2$$

$$\tau_C^{t=3} = \frac{M_t^C}{W_{t,2}} = \frac{250 \cdot 10^2}{2340} = 10.684 \text{ kN/cm}^2$$

- $\tau_{zy}^E = +10.684 \text{ kN/cm}^2$



$$e. \quad \varphi_{CA} = \varphi_{BA} + \varphi_{CB}$$

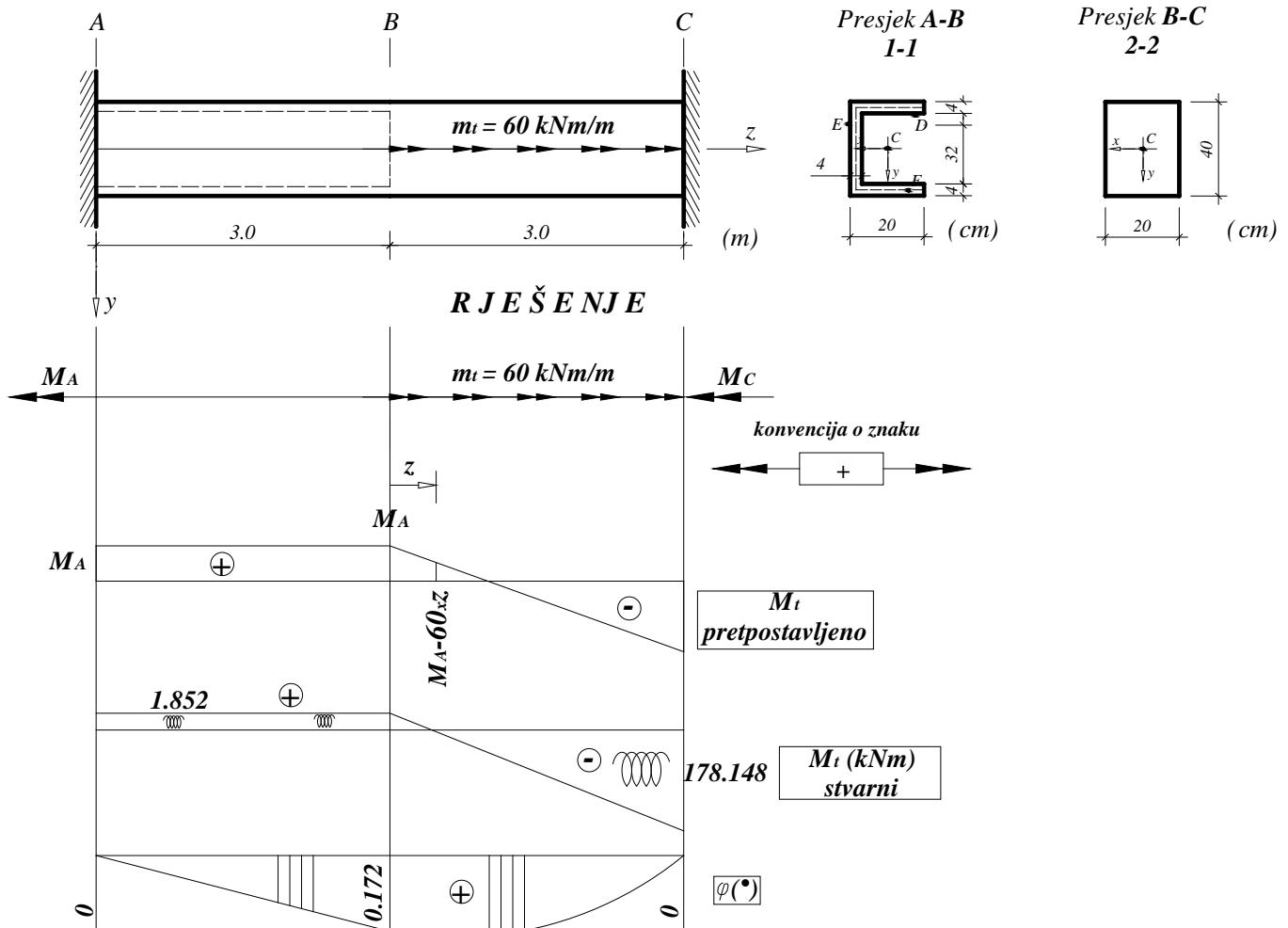
$$\begin{aligned}\varphi_{BA} &= \int_0^4 \frac{850 - 150 \cdot z}{G \cdot I_t} dz = \frac{1}{G \cdot I_t} \left( 850 \cdot z - 150 \frac{z^2}{2} \right) \Big|_0^4 = \\ &= \frac{1}{G \cdot I_t} \left( 850 \cdot 4 - 150 \frac{4^2}{2} \right) = \frac{2200 \cdot 10^4}{80 \cdot 10^2 \cdot 27241} = 0.1 \text{ rad} \cdot \frac{180}{\pi} = 5.73^\circ\end{aligned}$$

$$\varphi_{CB} = \frac{M_{CB} \cdot l_{CB}}{G \cdot I_t} = \frac{250 \cdot 10^2 \cdot 1.5 \cdot 10^2}{80 \cdot 10^2 \cdot 27241} = 0.017 \text{ rad} \cdot \frac{180}{\pi} = 0.974^\circ$$

$$\varphi_{CA} = 0.1 + 0.017 = 0.117 \text{ rad} \cdot \frac{180}{\pi} = 6.7^\circ$$

3. Za obostrano uklješteni nosač na slici:

- Nacrtati dijagram promjene momenta torzije duž nosača ( $M_t$ );
- Sračunati maksimalni smičući napon ( $\tau_{max}$ ) i definisati presjek u kome on djeluje;
- Nacrtati dijagram smičućih napona u presjeku A;
- Analizirati stanje napona u tačkama D, E i F;
- Nacrtati dijagram obratanja poprečnih presjeka duž nosača, ako su oba dijela (poprečna presjeka) izrađenana od istog materijala čiji je modul klizanja  $G=120 \text{ GPa}$ .



- 2 statički nepoznate veličine (reaktivni torzionalni momenti  $M_A$  i  $M_C$ )  
1 raspoloživi uslov ravnoteže ( $\sum M_t = 0$ )  
Znači, 1x statički neodređen problem.
  - Uslov ravnoteže  $\sum M_t = 0 \rightarrow M_A + M_C = m_t \cdot l / 2$
  - Deformacijski uslov  $\varphi_{CA} = 0$

Geometrijske karakteristike poprečnih presjeka:

Presjek 1-1:

$$I_t^{1-1} = \frac{1}{3} \sum_i h_i \cdot t_i^3 = \frac{1}{3} (18 \cdot 4^3 \cdot 2 + 36 \cdot 4^3) = 1536 \text{ cm}^4$$

$$W_{t,i} = \frac{I_t}{t_i} \quad (t_i = 4 \text{ cm} = \text{const} \rightarrow \text{imamo sam 1 otporni moment})$$

$$W_t^{1-1} = \frac{1536}{4} = 384 \text{ cm}^3$$

Presjek 2-2:

$$\frac{h}{b} = \frac{40}{20} = 2 \rightarrow \begin{cases} \alpha = 0.457 \\ \beta = 0.493 \\ \gamma = 0.795 \end{cases} \rightarrow I_t^{2-2} = \alpha \cdot b^4 = 0.457 \cdot 20^4 = 7.312 \cdot 10^4 \text{ cm}^4$$

$$W_t^{2-2} = \beta \cdot b^3 = 0.493 \cdot 20^3 = 3.944 \cdot 10^3 \text{ cm}^3$$

Proračun reakcija oslonaca:

$$\varphi_{CA} = \varphi_{BA} + \varphi_{CB} = 0$$

$$\frac{M_A \cdot l / 2}{GI_t^{1-1}} + \int_0^{l/2} \frac{M_A - m_t \cdot z}{GI_t^{2-2}} dz = 0 \quad | \cdot GI_t^{2-2}$$

$$\frac{I_t^{2-2}}{I_t^{1-1}} M_A \cdot l / 2 + \left( M_A \cdot z - m_t \frac{z^2}{2} \right) \Big|_0^{l/2} = 0$$

$$\frac{I_t^{2-2}}{I_t^{1-1}} M_A \cdot l / 2 + M_A \cdot l / 2 - m_t \frac{(l/2)^2}{2} = 0 \quad | / l / 2$$

$$M_A \left( \frac{I_t^{2-2}}{I_t^{1-1}} + 1 \right) = m_t \frac{l}{4}$$

$$M_A = \frac{m_t \cdot l}{4 \left( \frac{I_t^{2-2}}{I_t^{1-1}} + 1 \right)} = \frac{60 \cdot 6}{4 \left( \frac{73120}{1536} + 1 \right)} = 1.852 \text{ kNm}$$

$$\text{Iz uslova ravnoteže } \rightarrow M_C = m_t \cdot l / 2 - M_A = 60 \cdot 3 - 1.852 = 178.148 \text{ kNm}$$

Vidimo da je presjek 2-2 „navukao“ odnosno prihvatio skoro kompletno spoljašnje torziono opterećenje iz razloga što ima višestruko veću torzionu krutost koja se ogleda kroz geometrijsku komponentu krutosti odnosno torzioni moment inercije ( $I_t^{2-2} = 73120 \text{ cm}^4$  vs  $I_t^{1-1} = 1536 \text{ cm}^4$ ).

*Presjek A*

b. Maksimalni smičući naponi:

Presjek 1-1, dio AB ( $M_{t,\max}^{AB} = 1.852 \text{ kNm}$ ):

$$\tau_{\max}^{AB} = \frac{M_{t,\max}^{AB}}{W_t^{1-1}} = \frac{1.852 \cdot 10^2}{384} = 0.482 \text{ kN/cm}^2$$

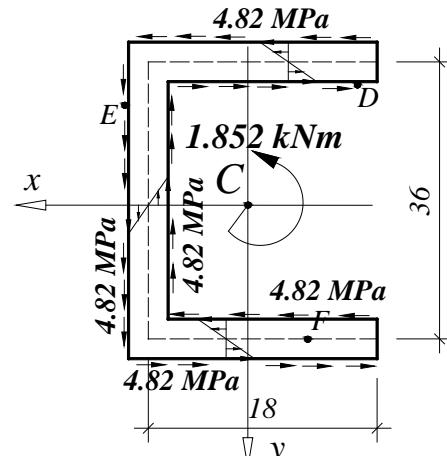
Presjek 2-2, dio BC ( $M_{t,\max}^C = 178.148 \text{ kNm}$ ):

$$\tau_{\max}^{BC} = \frac{M_{t,\max}^C}{W_t^{2-2}} = \frac{178.148 \cdot 10^2}{3944} = 4.517 \text{ kN/cm}^2$$

$\tau_{\max}^C \equiv \tau_{\max}$  (djeluje u presjeku C)

c. Smičući naponi u presjeku A:

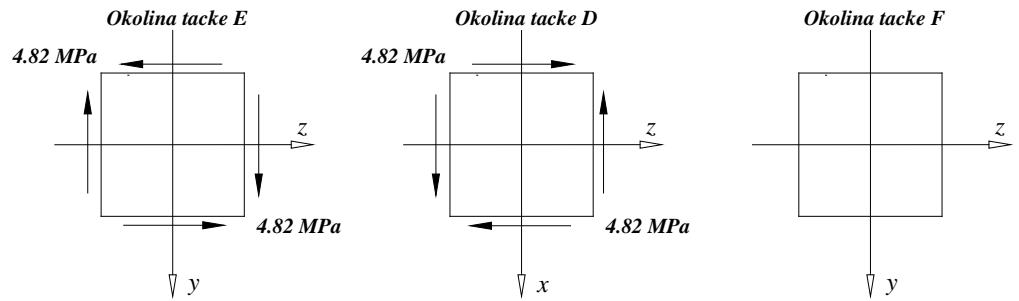
$$M_t^A = 1.852 \text{ kNm} \quad \tau_A = 0.482 \text{ kN/cm}^2$$



$$d. \quad \tau_{zx}^D = -0.482 \text{ kN/cm}^2$$

$$\tau_{zy}^E = +0.482 \text{ kN/cm}^2$$

$$\tau_{zx}^F = 0$$



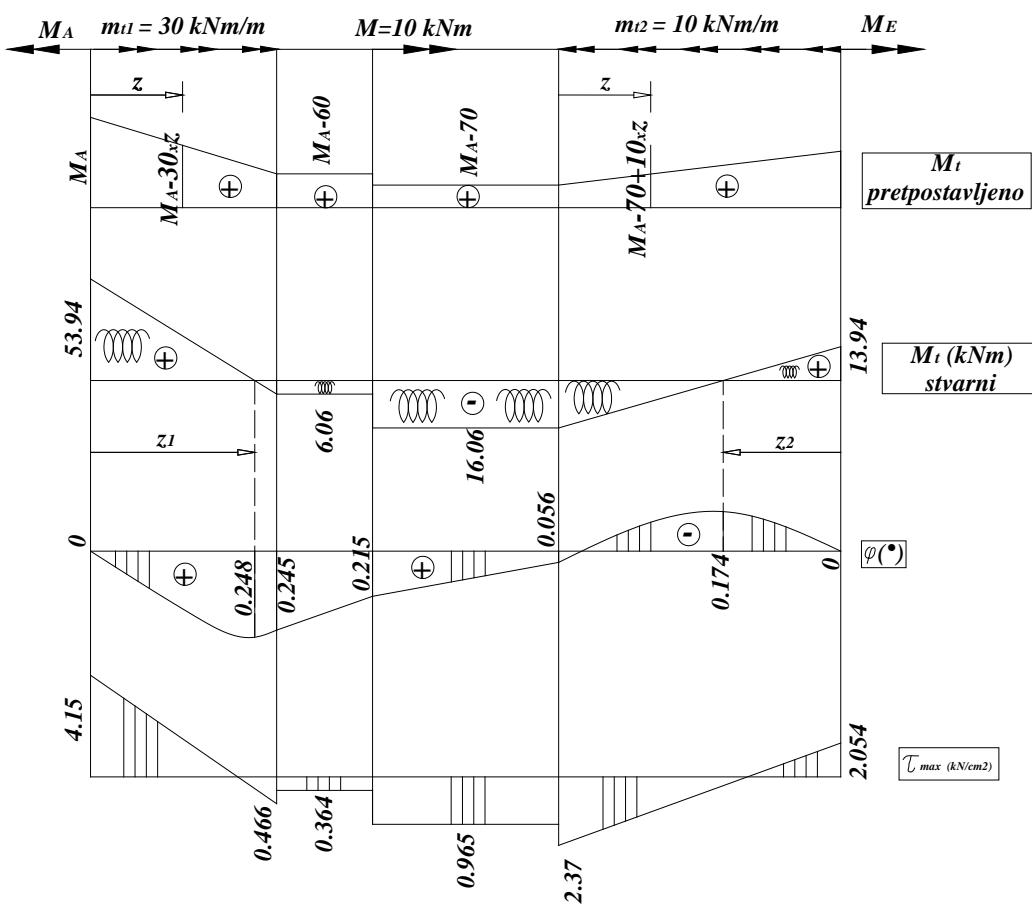
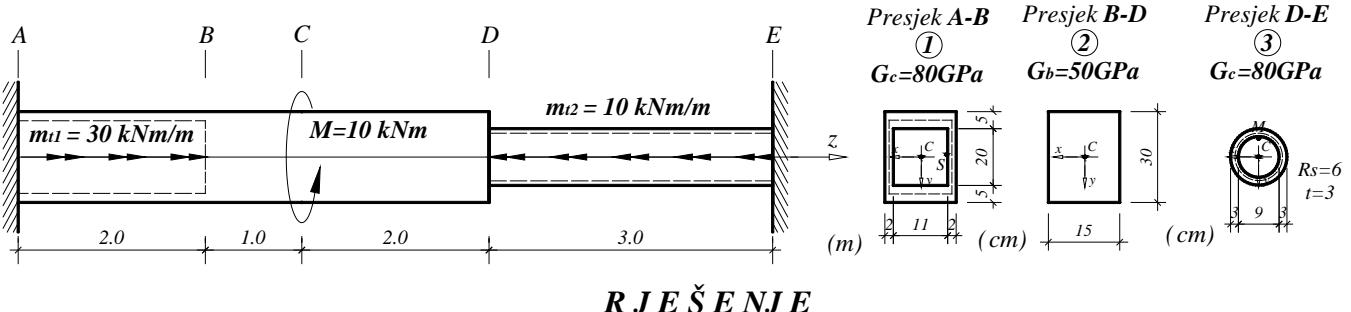
$$e. \quad \varphi_{BA} = \frac{1.852 \cdot 10^2 \cdot 3 \cdot 10^2}{120 \cdot 10^2 \cdot 1536} = 0.003 \text{ rad} \cdot \frac{180}{\pi} = 0.172^\circ$$

Kontrola

$$\begin{aligned} \varphi_{CB} &= \int_0^3 \frac{1.852 - 60 \cdot z}{G \cdot I_t^{2-2}} dz = \frac{1}{G \cdot I_t^{2-2}} \left( 1.852 \cdot z - 60 \frac{z^2}{2} \right) \Big|_0^3 = \\ &= \frac{1}{G \cdot I_t^{2-2}} \left( 1.852 \cdot 3 - 60 \frac{3^2}{2} \right) = \frac{-264.444 \cdot 10^4}{120 \cdot 10^2 \cdot 73120} = -0.003 \text{ rad} \cdot \frac{180}{\pi} = -0.172^\circ \end{aligned}$$

$$\varphi_{CA} = 0.003 - 0.003 = 0$$

4. Na slici je prikazan obostrano uklješteni nosač koji je izrađen od čelika (djelovi AB i DE -  $G_c=80 \text{ GPa}$ ) odnosno od bakra (dio BD -  $G_b=50 \text{ GPa}$ ).
- Nacrtati dijagram promjene momenta torzije duž nosača ( $M_t$ );
  - Nacrtati dijagram promjene maksimalnog napona ( $\tau_{max}$ ) duž nosača;
  - Nacrtati dijagram smičućih napona u presjecima A i D<sub>desno</sub>;
  - Analizirati stanje napona u tačkama S i M;
  - Nacrtati dijagram obratanja poprečnih presjeka duž nosača. Naznačiti obrtanja u presjecima A, B, C, D i E;
  - Sračunati maksimalno obrtanje po absolutnoj vrijednosti i definisati presjek u kome se javlja.



- 2 statički nepoznate veličine (reaktivni torzionalni momenti  $M_A$  i  $M_E$ )  
1 raspoloživi uslov ravnoteže ( $\sum M_t = 0$ )  
Znači, 1x statički neodređen problem.
  - Uslov ravnoteže  $\sum M_t = 0 \rightarrow M_C + m_t^{[1]} \cdot 2 + M_E = M_A + m_t^{[2]} \cdot 3$
  - Deformacijski uslov  $\varphi_{EA} = 0$

Geometrijske karakteristike poprečnih presjeka:

Presjek 1:

$$A_s = 13 \cdot 25 = 325 \text{ cm}^2$$

$$W_{t,1}^{[1]} = 2 \cdot 325 \cdot 2 = 1300 \text{ cm}^3$$

$$W_{t,2}^{[1]} = 2 \cdot 325 \cdot 5 = 3250 \text{ cm}^3$$

$$I_t^{[1]} = \frac{4 \cdot A_s^2}{\sum_i \frac{s_i}{t_i}} = \frac{4 \cdot 325^2}{\left(\frac{25}{2} + \frac{13}{5}\right) \cdot 2} = 13990 \text{ cm}^4$$

Presjek 2:

$$\frac{h}{b} = \frac{30}{15} = 2 \rightarrow \begin{cases} \alpha = 0.457 \\ \beta = 0.493 \\ \gamma = 0.795 \end{cases} \rightarrow I_t^{[2]} = \alpha \cdot b^4 = 0.457 \cdot 15^4 = 23136 \text{ cm}^4$$

$$W_t^{[2]} = \beta \cdot b^3 = 0.493 \cdot 15^3 = 1664 \text{ cm}^3$$

Presjek 3:

$$A_s = \pi \cdot 6^2 = 113.1 \text{ cm}^2$$

$$W_t^{[3]} = 2 \cdot 113.1 \cdot 3 = 678.6 \text{ cm}^3$$

$$I_t^{[1]} = \frac{4 \cdot A_s^2}{\sum_i \frac{s_i}{t_i}} = \frac{4 \cdot 113.1^2}{\frac{2\pi \cdot 6}{3}} = 4071.5 \text{ cm}^4$$

$$\frac{G_c}{G_b} = \frac{80}{50} = 1.6 \quad \frac{I_t^{[1]}}{I_t^{[2]}} = \frac{13990}{23135.6} = 0.6047 \quad \frac{I_t^{[1]}}{I_t^{[3]}} = \frac{13990}{4071.5} = 3.436$$

Proračun reakcija oslonaca:

$$\varphi_{EA} = \varphi_{BA} + \varphi_{CB} + \varphi_{DC} + \varphi_{ED} = 0$$

$$\int_0^2 \frac{M_A - 30 \cdot z}{G_c I_t^{[1]}} dz + \frac{(M_A - 60) \cdot 1}{G_b I_t^{[2]}} + \frac{(M_A - 70) \cdot 2}{G_b I_t^{[2]}} + \int_0^3 \frac{M_A - 70 + 10 \cdot z}{G_c I_t^{[3]}} dz = 0 \quad | \cdot G_c I_t^{[1]}$$

$$\left( M_A \cdot z - 30 \frac{z^2}{2} \right) \Big|_0^2 + \frac{G_c}{G_b} \frac{I_t^{[1]}}{I_t^{[2]}} (M_A - 60) \cdot 1 + \frac{G_c}{G_b} \frac{I_t^{[1]}}{I_t^{[2]}} (M_A - 70) \cdot 2 + \frac{I_t^{[1]}}{I_t^{[3]}} \left( M_A \cdot z - 70 \cdot z + 10 \frac{z^2}{2} \right) \Big|_0^3 = 0$$

$$\left( M_A \cdot 2 - 30 \frac{2^2}{2} \right) + 1.6 \cdot 0.6047 (M_A - 60) \cdot 1 + 1.6 \cdot 0.6047 (M_A - 70) \cdot 2 + 3.436 \left( M_A \cdot 3 - 70 \cdot 3 + 10 \frac{3^2}{2} \right) = 0$$

$$M_A = 53.94 \text{ kNm}$$

$$\text{Iz uslova ravnoteže } \rightarrow M_E = M_A + m_t^{[2]} \cdot 3 - M_C - m_t^{[1]} \cdot 2 = 53.94 + 10 \cdot 3 - 10 - 30 \cdot 2 = 13.94 \text{ kNm}$$

b.

Maksimalni smičući naponi:

$$\text{Presjek } A \quad (M_t^A = 53.94 \text{ kNm}): \tau_{\max}^A = \frac{M_t^A}{W_{t,1}^{[1]}} = \frac{53.94 \cdot 10^2}{1300} = 4.15 \text{ kN/cm}^2$$

$$\text{Presjek } B_{\text{lijevo}} \quad (M_t^B = -6.06 \text{ kNm}): \tau_{\max}^{B_{\text{lijevo}}} = \frac{M_t^B}{W_{t,1}^{[1]}} = \frac{-6.06 \cdot 10^2}{1300} = -0.466 \text{ kN/cm}^2$$

$$\text{Presjeci } B_{\text{desno}} \text{ i } C_{\text{lijevo}} \quad (M_t^{B_d, C_l} = -6.06 \text{ kNm}): \tau_{\max}^{B_d, C_l} = \frac{M_t^{B_d, C_l}}{W_t^{[2]}} = \frac{-6.06 \cdot 10^2}{1664} = -0.364 \text{ kN/cm}^2$$

$$\text{Presjeci } C_{desno} \text{ i } D_{lijevo} \quad (M_t^{C_d, D_l} = -16.06 \text{ kNm}) : \tau_{\max}^{C_d, D_l} = \frac{M_t^{C_d, D_l}}{W_t^{[2]}} = \frac{-16.06 \cdot 10^2}{1664} = -0.965 \text{ kN/cm}^2$$

$$\text{Presjek } D_{desno} \quad (M_t^{D_{desno}} = -16.06 \text{ kNm}) : \tau_{\max}^{D_{desno}} = \frac{M_t^{D_{desno}}}{W_t^{[3]}} = \frac{-16.06 \cdot 10^2}{678.6} = -2.37 \text{ kN/cm}^2$$

$$\text{Presjek } E \quad (M_t^E = 13.94 \text{ kNm}) : \tau_{\max}^E = \frac{M_t^E}{W_t^{[3]}} = \frac{13.94 \cdot 10^2}{678.6} = 2.054 \text{ kN/cm}^2$$

c. Smičući naponi u presjecima A i D<sub>desno</sub>:

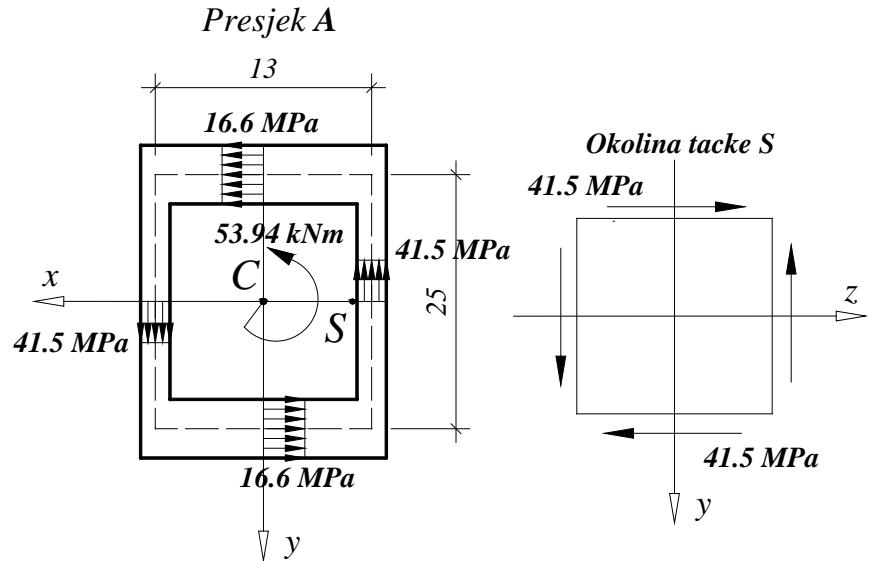
$$M_t^A = 53.94 \text{ kNm}$$

$$\tau_{A}^{t=2} = \frac{M_t^A}{W_{t,1}^{[1]}} = \frac{53.94 \cdot 10^2}{1300} = 4.15 \text{ kN/cm}^2$$

$$\tau_{A}^{t=5} = \frac{M_t^A}{W_{t,2}^{[1]}} = \frac{53.94 \cdot 10^2}{3250} = 1.66 \text{ kN/cm}^2$$

$$M_t^{D_{desno}} = -16.06 \text{ kNm}$$

$$\begin{aligned} \tau_{\max}^{D_{desno}} &= \frac{M_t^{D_{desno}}}{W_t^{[3]}} = \frac{-16.06 \cdot 10^2}{678.6} = \\ &= -2.37 \text{ kN/cm}^2 \end{aligned}$$

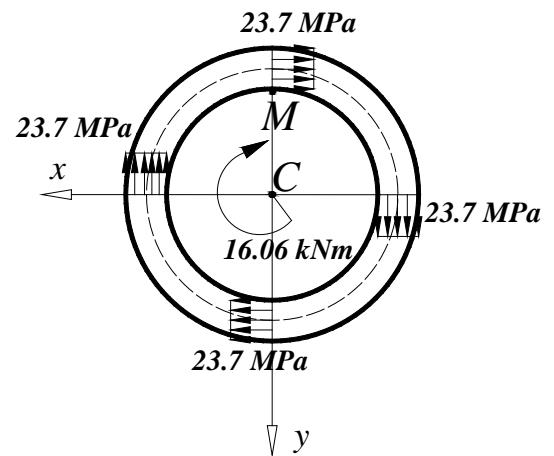


d.

$$\tau_{zy}^S = -4.15 \text{ kN/cm}^2$$

$$\tau_{zx}^M = -2.37 \text{ kN/cm}^2$$

Presjek D<sub>desno</sub>



e.

$$\begin{aligned} \varphi_{BA} &= \int_0^{\frac{z}{2}} \frac{53.94 - 30 \cdot z}{G_c \cdot I_t^{[1]}} dz = \frac{1}{G_c \cdot I_t^{[1]}} \left( 53.94 \cdot z - 30 \frac{z^2}{2} \right) \Big|_0^{\frac{z}{2}} = \\ &= \frac{1}{G_c \cdot I_t^{[1]}} \left( 53.94 \cdot 2 - 30 \frac{2^2}{2} \right) = \frac{47.88 \cdot 10^4}{80 \cdot 10^2 \cdot 13990} = \\ &= 0.004278 \text{ rad} \cdot \frac{180}{\pi} = 0.245^\circ \end{aligned}$$

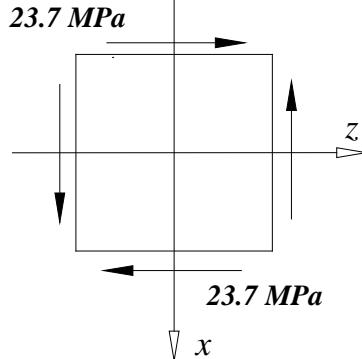
$$\varphi_{CB} = \frac{M_{CB} \cdot l_{CB}}{G_b I_t^{[2]}} = \frac{-6.06 \cdot 10^2 \cdot 1 \cdot 10^2}{50 \cdot 10^2 \cdot 23135.6} = -0.0005239 \text{ rad}$$

$$\varphi_{CA} = \varphi_{BA} + \varphi_{CB} = 0.004278 - 0.0005239 =$$

$$= 0.003754 \text{ rad} \cdot \frac{180}{\pi} = 0.215^\circ$$

$$\varphi_{DC} = \frac{M_{DC} \cdot l_{DC}}{G_b I_t^{[2]}} = \frac{-16.06 \cdot 10^2 \cdot 2 \cdot 10^2}{50 \cdot 10^2 \cdot 23135.6} = -0.002777 \text{ rad}$$

Okolina tачke M



$$\varphi_{DA} = \varphi_{CA} + \varphi_{DC} = 0.003754 - 0.002777 =$$

$$= 0.000977 \text{ rad} \cdot \frac{180}{\pi} = 0.056^0$$

$$\begin{aligned}\varphi_{ED} &= \int_0^3 \frac{-16.06 + 10 \cdot z}{G_c \cdot I_t^{[3]}} dz = \frac{1}{G_c \cdot I_t^{[3]}} \left( -16.06 \cdot z + 10 \frac{z^2}{2} \right) \Big|_0^3 = \\ &= \frac{1}{G_c \cdot I_t^{[3]}} \left( -16.06 \cdot 3 + 10 \frac{3^2}{2} \right) = \frac{-3.18 \cdot 10^4}{80 \cdot 10^2 \cdot 4071.5} = -0.000977 \text{ rad}\end{aligned}$$

Kontrola:

$$\varphi_{EA} = \varphi_{DA} + \varphi_{ED} = 0.000977 - 0.000977 = 0$$

$$f. \quad \varphi(z) = \int_0^l \frac{M(z)}{GI_t} dz$$

$$\max \varphi = \varphi(z_{\varphi_{\max}}) = ? \quad \frac{d\varphi(z)}{dz} = 0 \quad \frac{d\varphi(z)}{dz} = \frac{d}{dz} \left( \int_0^l \frac{M(z)}{GI_t} dz \right) = \frac{M(z)}{GI_t} = 0 \rightarrow M(z) = 0 \rightarrow \boxed{z_{\varphi_{\max}}}$$

Sa dijagrama momenata torzije ( $M_t$ ) se mogu uočiti dva presjeka u kojima je moment torzije jednak 0.

Položaj predmetnih presjeka ćemo odrediti iz sličnosti trouglova:

$$\frac{53.94 + 6.06}{2} = \frac{53.94}{z_1} \rightarrow z_1 = 1.798m$$

$$\frac{13.94 + 16.06}{3} = \frac{13.94}{z_2} \rightarrow z_2 = 1.394m$$

Obrtanje presjeka definisanih sa  $z_1$  i  $z_2$ :

$$\begin{aligned}\varphi_{\max} = \varphi(z_1) &= \int_0^{1.798} \frac{53.94 - 30 \cdot z}{G_c \cdot I_t^{[1]}} dz = \frac{1}{G_c \cdot I_t^{[1]}} \left( 53.94 \cdot z - 30 \frac{z^2}{2} \right) \Big|_0^{1.798} = \\ &= \frac{1}{G_c \cdot I_t^{[1]}} \left( 53.94 \cdot 1.798 - 30 \frac{1.798^2}{2} \right) = \frac{48.492 \cdot 10^4}{80 \cdot 10^2 \cdot 13990} = 0.00433 \text{ rad} \cdot \frac{180}{\pi} = 0.248^0 \equiv \varphi_{\max} - \text{apsolutni max}\end{aligned}$$

$$\begin{aligned}\varphi_{\max} = \varphi(z_2) &= \int_0^{1.394} \frac{13.94 - 10 \cdot z}{G_c \cdot I_t^{[3]}} dz = \frac{1}{G_c \cdot I_t^{[3]}} \left( 13.94 \cdot z - 10 \frac{z^2}{2} \right) \Big|_0^{1.394} = \\ &= \frac{1}{G_c \cdot I_t^{[3]}} \left( 13.94 \cdot 1.394 - 10 \frac{1.394^2}{2} \right) = \frac{9.716 \cdot 10^4}{80 \cdot 10^2 \cdot 4071.5} = 0.00298 \text{ rad} \cdot \frac{180}{\pi} = 0.171^0 \equiv \varphi_{\max} - \text{lokalni max}\end{aligned}$$